

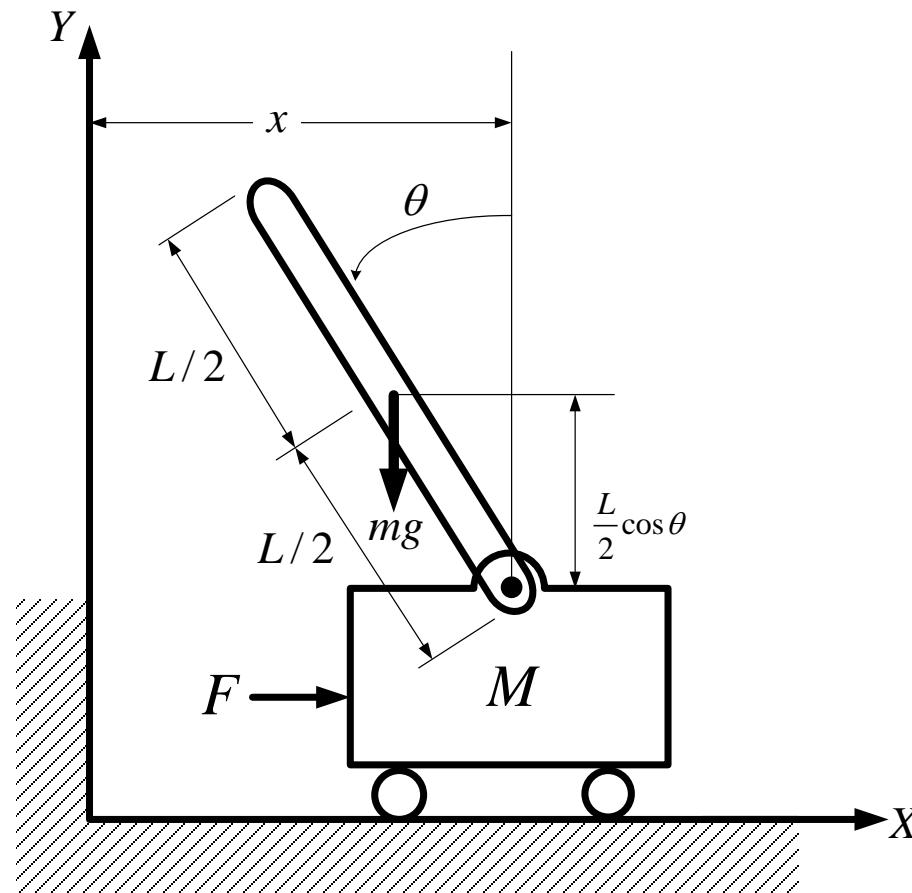
# Inverted Pendulum System

## Fall 2014

Output Feedback vs. State  
Space Approach for Tracking

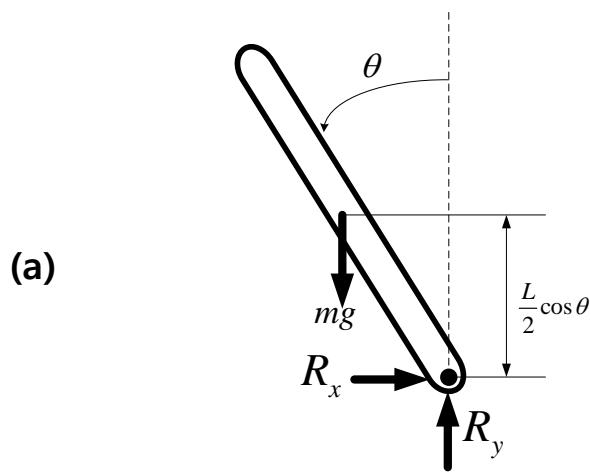
2014 Fall Inverted Pendulum Tracking

# Inverted Pendulum System



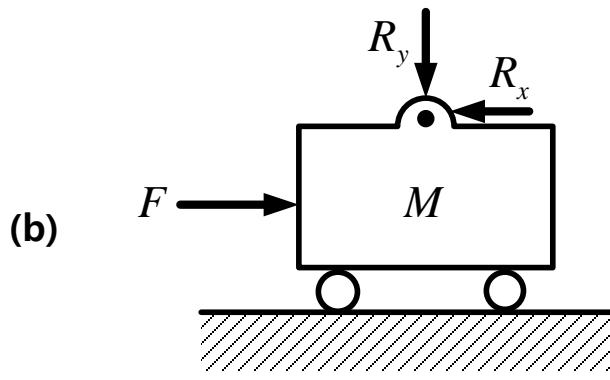
**Fig 1. Inverted Pendulum System**

# Free Body Diagram : Rod & Cart



## *About bar having mass 'm'*

$$\sum M = R_x \frac{L}{2} \cos \theta + R_y \frac{L}{2} \sin \theta = I \ddot{\theta} \quad \dots\dots(3)$$



### *About cart having mass 'M'*

$$\sum F_x = F - R_x = M \ddot{x} \quad \dots\dots\dots(4)$$

**Fig 2.**

- (a) free-body diagram of rod
  - (b) free-body diagram of cart

# Variable Relation

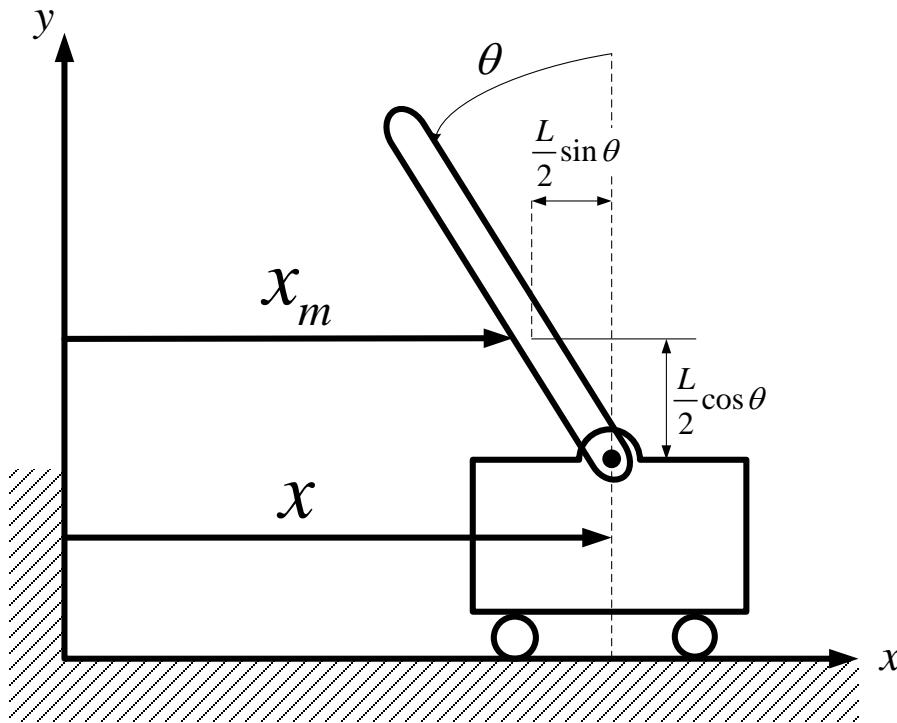


Fig 3. Relation between two center of gravities

$$x_m = x - \frac{L}{2} \sin \theta$$

$$y_m = \frac{L}{2} \cos \theta$$



$$\dot{x}_m = \dot{x} - \frac{L}{2} \dot{\theta} \cos \theta$$

$$\dot{y}_m = -\frac{L}{2} \dot{\theta} \sin \theta$$



$$\ddot{x}_m = \ddot{x} + \frac{L}{2} \dot{\theta}^2 \sin \theta - \frac{L}{2} \ddot{\theta} \cos \theta$$

$$\ddot{y}_m = -\frac{L}{2} \dot{\theta}^2 \cos \theta - \frac{L}{2} \ddot{\theta} \sin \theta$$

# Equations & Desired State

From above relations, equations (1) ~ (4) can be rewritten as follow

$$(1) \quad R_x = m \left( \ddot{x} + \frac{L}{2} \dot{\theta}^2 \sin \theta - \frac{L}{2} \ddot{\theta} \cos \theta \right)$$

$$(2) \quad R_y = mg + m \left( -\frac{L}{2} \dot{\theta}^2 \cos \theta - \frac{L}{2} \ddot{\theta} \sin \theta \right)$$

$$(3) \quad I \ddot{\theta} = m \ddot{x} \frac{L}{2} \cos \theta - m \left( \frac{L}{2} \right)^2 \ddot{\theta} + mg \frac{L}{2} \sin \theta \quad \dots\dots (5)$$

$$(4) \quad (M+m) \ddot{x} + m \frac{L}{2} (\dot{\theta}^2 \sin \theta - \ddot{\theta} \cos \theta) = F \quad \dots\dots (6)$$

Initial condition  
 $\theta(0) = \theta_0$   
 $x(0) = 0$

# Equations & Desired State

From equation (5)

$$\ddot{x} = \frac{1}{m \frac{L}{2} \cos \theta} \left\{ \left[ I + m \left( \frac{L}{2} \right)^2 \right] \ddot{\theta} - mg \frac{L}{2} \sin \theta \right\}$$

put this into (6) and arrange

$$\ddot{\theta} = \frac{1}{(M+m) \left( I + m(L/2)^2 \right) - m^2 (L/2)^2 \cos^2(\theta)} \left[ -m^2 (L/2)^2 \sin(\theta) \cos(\theta) \dot{\theta}^2 + (M+m)mg \frac{L}{2} \sin(\theta) + m(L/2) \cos(\theta) F \right]$$

and from this,  $\ddot{x}$  also can be

$$\ddot{x} = \frac{1}{(M+m) \left( I + m(L/2)^2 \right) - m^2 (L/2)^2 \cos^2(\theta)} \left[ m^2 (L/2)^2 \sin(\theta) \cos(\theta) g - m(I + m(L/2)^2)(L/2) \sin(\theta) \dot{\theta}^2 + (I + m(L/2)^2) F \right]$$

# 1. Nonlinear System Control

Nonlinear System

+

Angle( $\theta$ ) and Angle Rate( $\dot{\theta}$ ) Controller

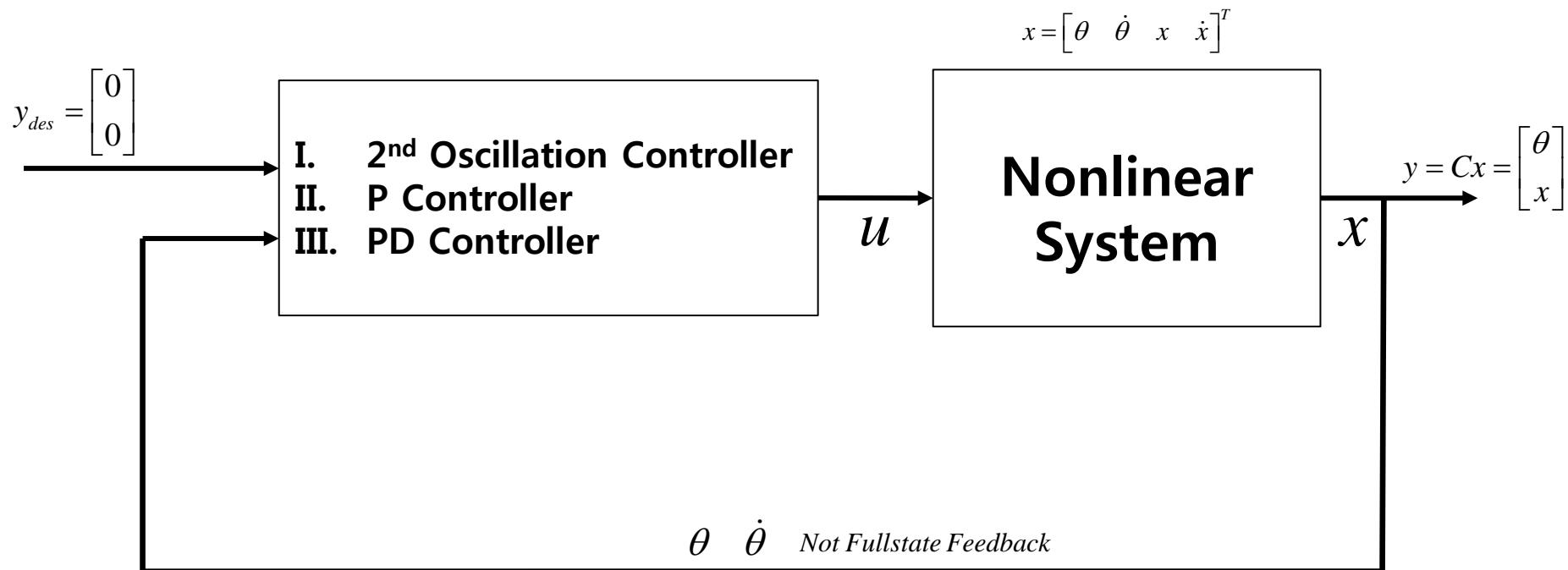
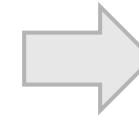


Fig 4. Block diagram for nonlinear system control

## $\theta$ Control : by Force control

Desired state is that  $\theta$  goes to zero at  $t \rightarrow \infty$   
(We do not care about  $x(t)$ )



$$\lim_{t \rightarrow \infty} \theta(t) = 0$$

$$\ddot{\theta} = \frac{1}{(M+m)\left(I + m(L/2)^2\right) - m^2(L/2)^2 \cos^2(\theta)} \left[ -m^2(L/2)^2 \sin(\theta) \cos(\theta) \dot{\theta}^2 + (M+m)mg \frac{L}{2} \sin(\theta) + m(L/2) \cos(\theta) F \right]$$

From above equation, we want to make  $\theta \rightarrow 0$  by controlling appropriate  $F(t)$

$F$  is a control input  $u(t)$ ,  $F(t) = u(t)$

Three ways of control using force are introduced

I Make above equation about  $\ddot{\theta}$  like  $\ddot{\theta} = -2\zeta\omega_n\dot{\theta} - \omega_n^2\theta$

II  $u(t) = F = -K\theta(t)$

III  $u(t) = F = -K\theta(t) - C\dot{\theta}$

# **$\theta$ Control : by Force control**

I

$$\ddot{\theta} = \frac{1}{(M+m)\left(I + m\left(\frac{L}{2}\right)^2\right) - m^2\left(\frac{L}{2}\right)^2 \cos^2(\theta)} \left[ -m^2\left(\frac{L}{2}\right)^2 \sin(\theta) \cos(\theta) \dot{\theta}^2 + (M+m)mg \frac{L}{2} \sin(\theta) + m\left(\frac{L}{2}\right) \cos(\theta) F \right]$$

$$\text{Let } A = \frac{(M+m)}{m\frac{L}{2}\cos\theta} \left[ I + m\left(\frac{L}{2}\right)^2 \right] - m\frac{L}{2}\cos\theta \quad \text{and} \quad B = \frac{(M+m)}{m\frac{L}{2}\cos\theta} \frac{L}{2} \sin\theta$$

Then above equation can be expressed as follows

$$\ddot{\theta} = -\frac{1}{A} m \frac{L}{2} \dot{\theta}^2 \sin\theta + \frac{B}{A} \cdot mg + \frac{1}{A} F$$

If  $\theta$  satisfies the following equation,

$$\ddot{\theta} = -2\zeta\omega_n\dot{\theta} - \omega_n^2\theta$$

$$\therefore u(t) = F = -2A \cdot \zeta\omega_n \cdot \dot{\theta} - A \cdot \omega_n^2 \cdot \theta - B \cdot mg + m \frac{L}{2} \sin\theta \cdot \dot{\theta}^2$$

With appropriate values of  $\zeta$  and  $\omega_n$ ,  $\theta$  can be zero at  $t \rightarrow \infty$

## **$\theta$ Control : by Force control**

From the equation  $\ddot{\theta} = -\frac{1}{A}m\frac{L}{2}\dot{\theta}^2 \sin \theta + \frac{B}{A} \cdot mg + \frac{1}{A} F$

We want to control by

II

$$u(t) = F = -K\theta(t) \quad (K > 0)$$

$$\ddot{\theta} = -\frac{1}{A}m\frac{L}{2}\dot{\theta}^2 \sin \theta + \frac{B}{A} \cdot mg - \frac{K}{A}\theta$$

In this case,  $\theta$  goes to zero when  $t \rightarrow \infty$ ?

III

$$u(t) = F = -K\theta(t) - C\dot{\theta} \quad (K > 0, C > 0)$$

$$\ddot{\theta} = -\frac{1}{A}m\frac{L}{2}\dot{\theta}^2 \sin \theta + \frac{B}{A} \cdot mg - \frac{1}{A}(K\theta + C\dot{\theta})$$

In this case,  $\theta$  goes to zero when  $t \rightarrow \infty$ ?

# Simulation : Force Control

Simulation Condition

$$\theta(0) = 0.1 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$\zeta = 0.5$$

$$K = 30$$

$$\dot{\theta}(0) = 0 \text{ rad / s}$$

$$\dot{x}(0) = 0 \text{ m / s}^2$$

$$\omega_n = 0.7 \text{ rad / s} \quad C = 15$$

I

Simulation Results Plots

— :  $\zeta, \omega_n$  control   --- : P control   - - - : PD control

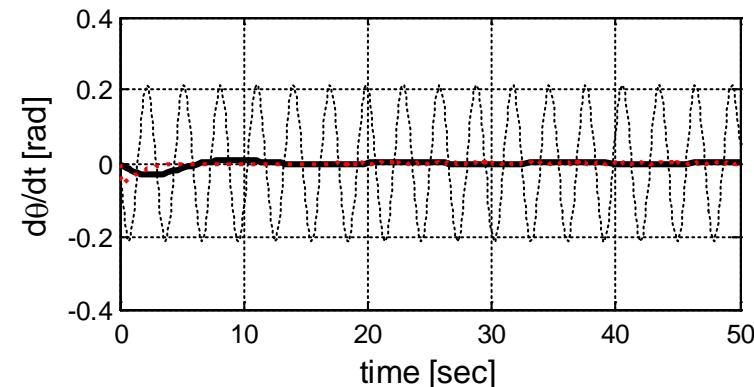
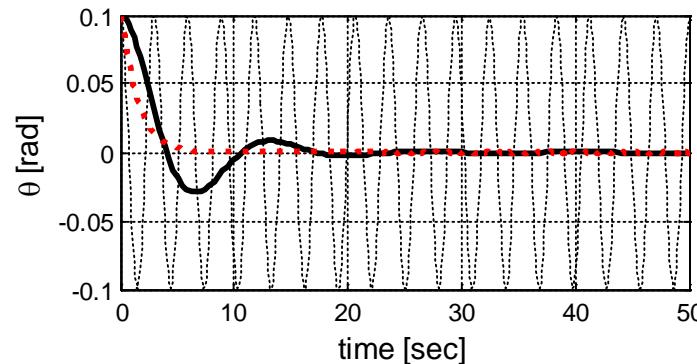
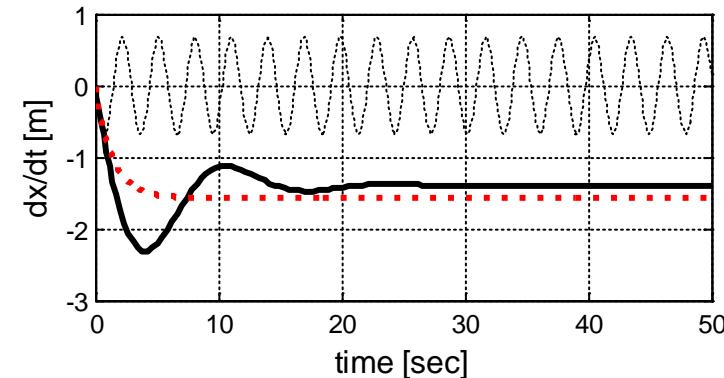
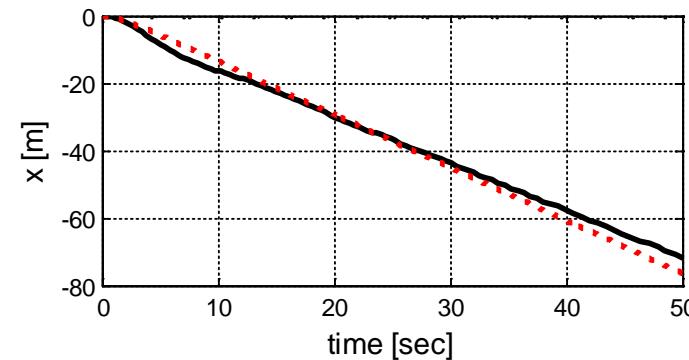


Fig 5. Simulation results of nonlinear system for three kinds of control

# Simulation : Animation

Simulation Condition

$$\theta(0) = 0.1 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$\zeta = 0.5$$

$$K = 30$$

$$\dot{\theta}(0) = 0 \text{ rad / s}$$

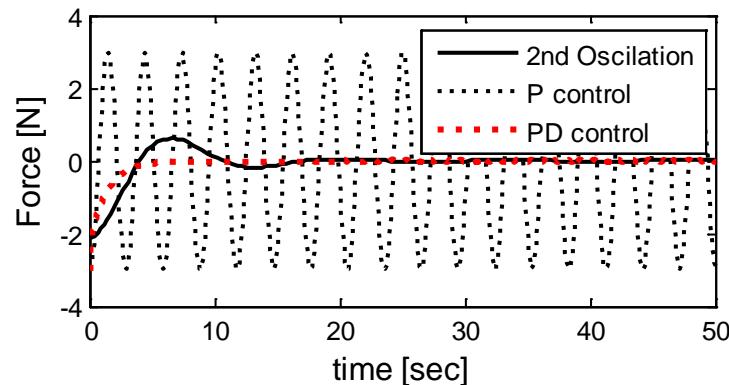
$$\dot{x}(0) = 0 \text{ m / s}^2$$

$$\omega_n = 0.7 \text{ rad / s}$$

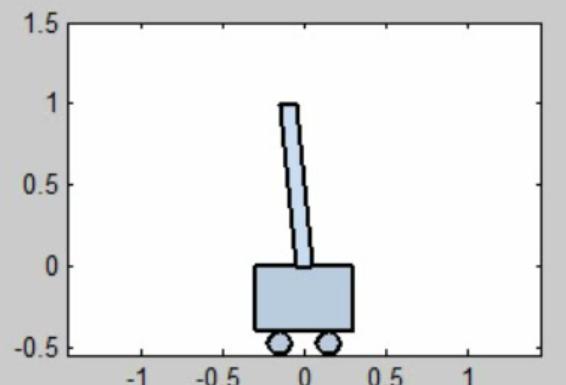
$$C = 15$$

II

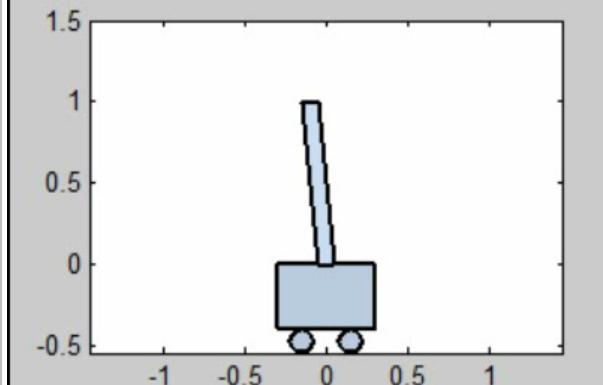
Force Plot and Simulation Results Animation



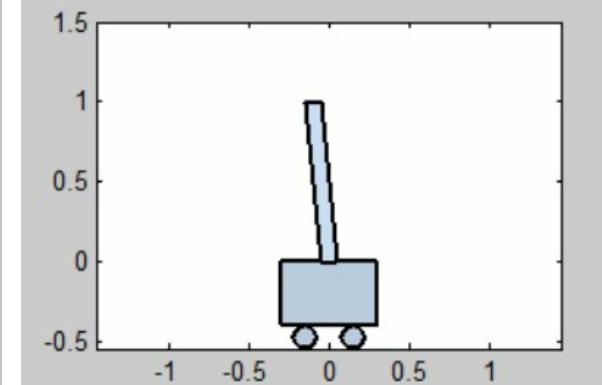
— :  $\zeta, \omega_n$  control  
- - - : P control  
- - - : PD control



(a) 2<sup>nd</sup> Oscillation Control



(b) P Control



(c) PD Control

Fig 6. Simulation results Animation

# After run simulation model

Open 'RUN\_file.m' and execute  
⇒ Four graphs ( $\theta$  vs  $t$ ,  $\dot{\theta}$  vs  $t$ ,  $x$  vs  $t$ ,  $\dot{x}$  vs  $t$ )  
&  
*Animation*

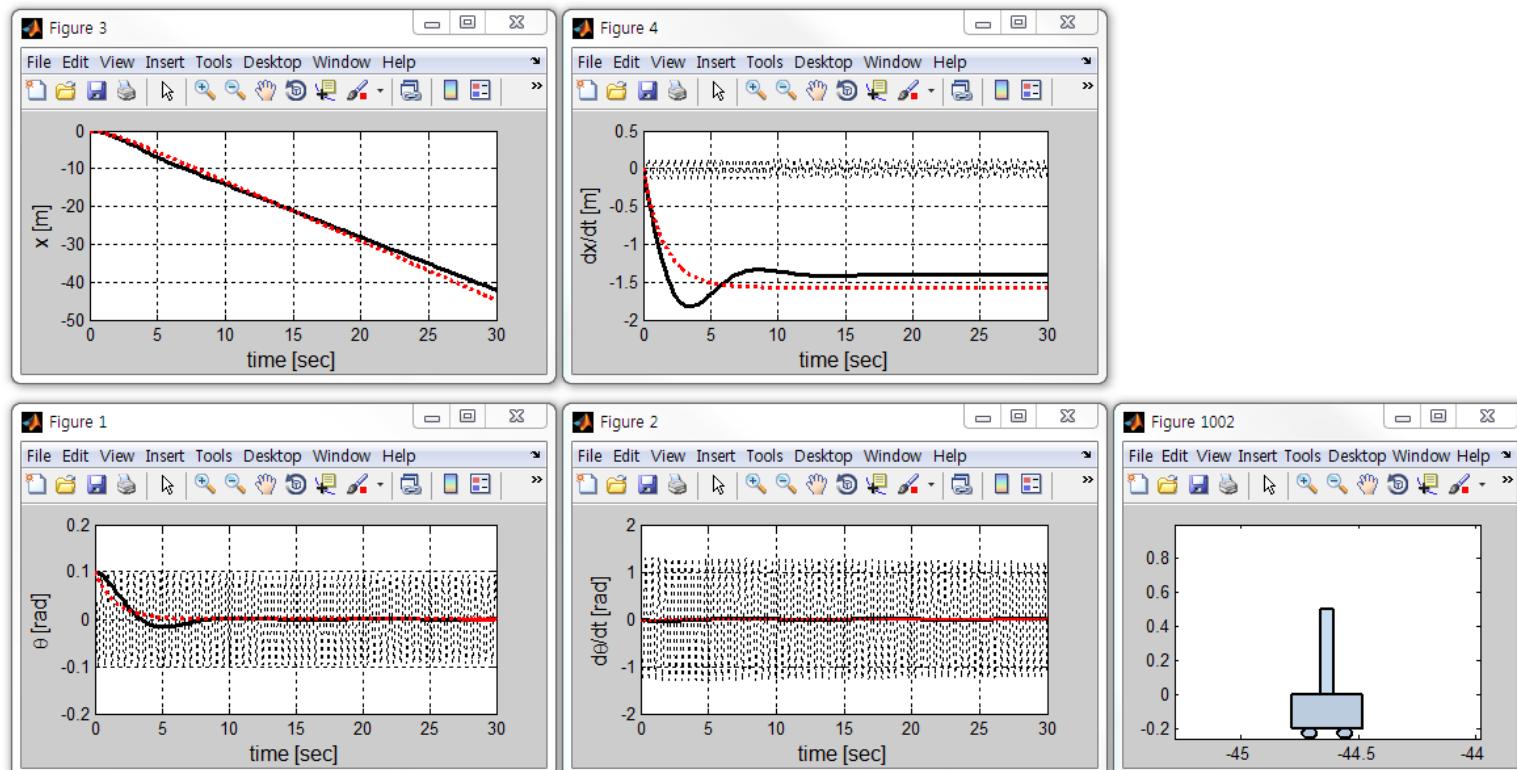
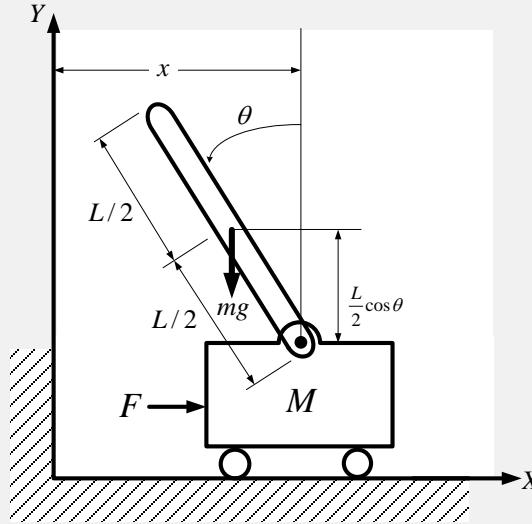


Fig 7. Screenshot after run simulation

# Nonlinear System



*Equation of motions*

$$I\ddot{\theta} = m\ddot{x}\frac{L}{2}\cos\theta - m\left(\frac{L}{2}\right)^2\dot{\theta} + mg\frac{L}{2}\sin\theta$$

$$(M+m)\ddot{x} + m\frac{L}{2}(\dot{\theta}^2 \sin\theta - \ddot{\theta}\cos\theta) = F$$

*Control input :  $u(t) = F$*

*Initial Condition :  $\theta(0) = \theta_0, \dot{\theta}(0) = 0, x(0) = 0, \dot{x}(0) = 0$*

*From combining above two equations*

$$\ddot{\theta} = \frac{B}{A}mg - \frac{1}{A}m\frac{L}{2}\dot{\theta}^2 \sin\theta + \frac{1}{A}F$$

$$\ddot{x} = C\left[-\frac{1}{A}m\frac{L}{2}\sin\theta\dot{\theta}^2 + \frac{B}{A}mg + \frac{1}{A}F\right] - g\tan\theta$$

$$\dot{\mathbf{x}} = \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} x_2 \\ \frac{B}{A}mg - \frac{1}{A}m\frac{L}{2}\dot{\theta}^2 \sin\theta \\ x_4 \\ C\left[-\frac{1}{A}m\frac{L}{2}\sin\theta\dot{\theta}^2 + \frac{B}{A}mg\right] - g\tan\theta \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{A} \\ 0 \\ \frac{C}{A} \end{bmatrix}F(t)$$

Where  $A = \frac{(M+m)}{m\frac{L}{2}\cos\theta}\left[I + m\left(\frac{L}{2}\right)^2\right] - m\frac{L}{2}\cos\theta$ ,  $B = \frac{(M+m)}{m\frac{L}{2}\cos\theta}\frac{L}{2}\sin\theta$ ,  $C = \frac{I + m(L/2)^2}{m(L/2)\cos\theta}$

# Angle and Angle Rate Controller

## Control Strategy

- How to control  $u(t) = F(t)$ ?
- Desired state when  $t \rightarrow \infty$  is  $\theta \rightarrow 0$

i) Applying a assumption that  $\theta$  satisfies 2nd oscillation, bellow equation comes

$$\ddot{\theta} = \frac{B}{A}mg - \frac{1}{A}m\frac{L}{2}\dot{\theta}^2 \sin \theta + \frac{1}{A}F = -2\zeta\omega_n\dot{\theta} - \omega_n^2\theta$$

In this case, control input becomes

$$u(t) = F(t) = A(-2\zeta\omega_n\dot{\theta} - \omega_n^2\theta) - Bmg + m\frac{L}{2}\sin \theta \cdot \dot{\theta}^2$$

ii) P control

$$u(t) = F(t) = -K\theta$$

iii) PD control

$$u(t) = F(t) = -K\theta - C\dot{\theta}$$

## 2. Linear System Control

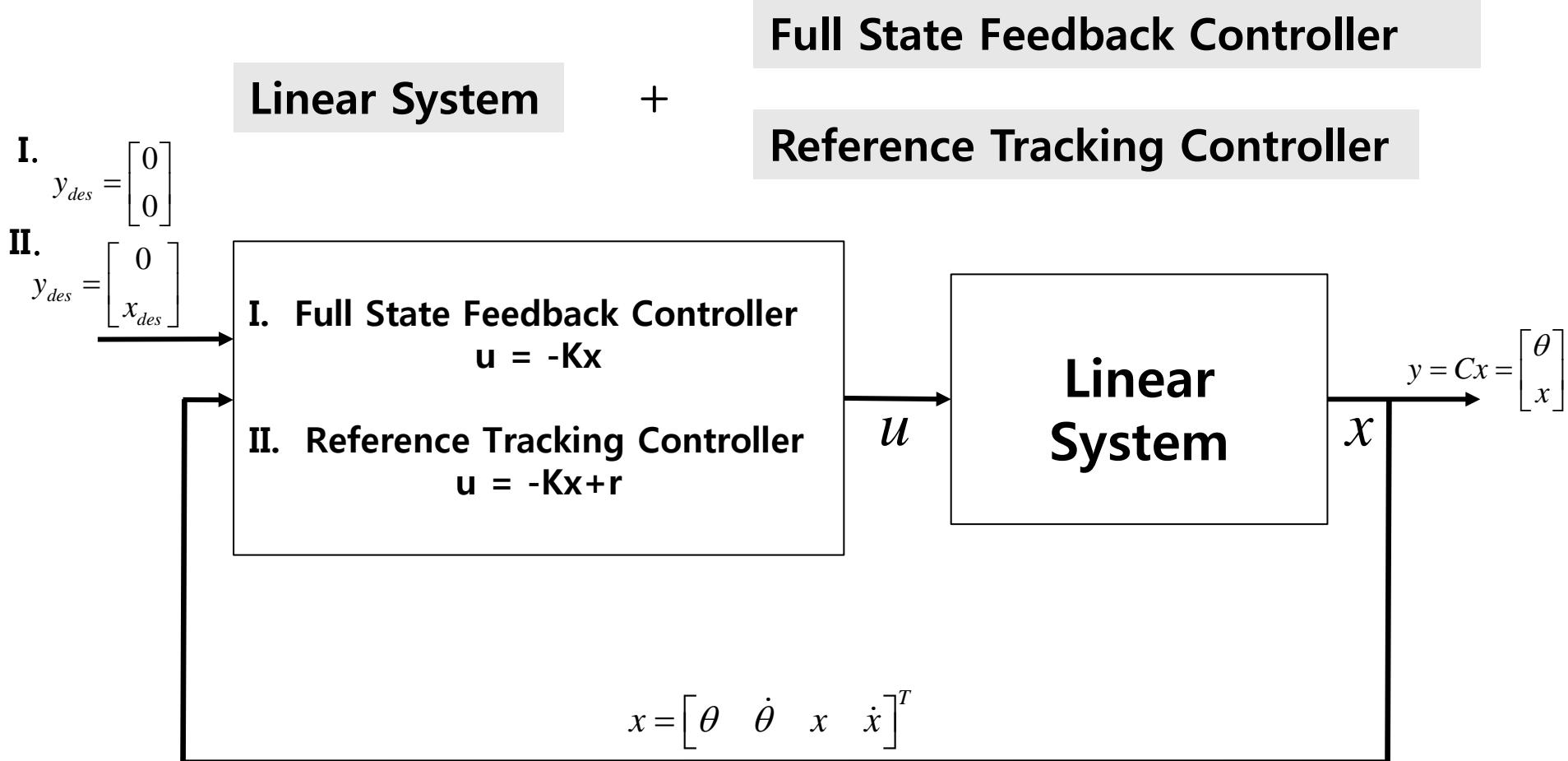


Fig 8. Block diagram for linear system control

# Linear Model

By small angle assumption,  $\sin(\theta) \approx \theta$ ,  $\cos(\theta) \approx 1$ ,  $\dot{\theta}^2 \approx 0$

then,  $\dot{\mathbf{x}} = \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} x_2 \\ \frac{B}{A}mg - \frac{1}{A}m\frac{L}{2}\dot{\theta}^2 \sin\theta \\ x_4 \\ C\left[-\frac{1}{A}m\frac{L}{2}\sin\theta\dot{\theta}^2 + \frac{B}{A}mg\right] - g\tan\theta \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{A} \\ 0 \\ \frac{C}{A} \end{bmatrix} F(t)$  becomes as follows

$$\dot{\mathbf{x}} = \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} 0 \\ \frac{mg(L/2)}{I+m(L/2)^2 - m^2(L/2)^2/(M+m)} \\ 0 \\ \left(\frac{(I+m(L/2)^2)}{I+m(L/2)^2 - m^2(L/2)^2/(M+m)} - 1\right)g \end{bmatrix} + \begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 1 \\ 0 & 0 & 0 \end{bmatrix} \mathbf{x} + \begin{bmatrix} 0 \\ \frac{m(L/2)/(M+m)}{I+m(L/2)^2 - m^2(L/2)^2/(M+m)} \\ 0 \\ \frac{(I+m(L/2)^2)/(M+m)}{I+m(L/2)^2 - m^2(L/2)^2/(M+m)} \end{bmatrix} F(t)$$

where  $\mathbf{x} = [x_1 \ x_2 \ x_3 \ x_4]^T = [\theta \ \dot{\theta} \ x \ \dot{x}]^T$

Where  $A = \frac{(M+m)}{m\frac{L}{2}} \left[ I + m\left(\frac{L}{2}\right)^2 \right] - m\frac{L}{2}$ ,  $B = \frac{(M+m)}{m\frac{L}{2}} \frac{L}{2} \theta$ ,  $C = \frac{I + m(L/2)^2}{m(L/2)}$

# Linear Model : Feedback Control

When a system is defined as  $\dot{\mathbf{x}} = A\mathbf{x} + Bu$ , we want to control  $\mathbf{x}$  to make  $\mathbf{x}(\infty) = 0$

Let  $u = -K\mathbf{x}$

Then,  $\dot{\mathbf{x}} = A\mathbf{x} - Bu = (A - BK)\mathbf{x} \Rightarrow \mathbf{x} = e^{(A-BK)t}$

( $\mathbf{x}$  :  $n \times 1$  matrix,  $A$  :  $n \times n$  matrix,  $B$  :  $n \times 1$  matrix,  $K$  :  $1 \times n$  matrix)

Appropriate gain  $K$  makes the system stable

where  $\mathbf{x} = [x_1 \ x_2 \ x_3 \ x_4]^T = [\theta \ \dot{\theta} \ x \ \dot{x}]^T$ ,  $K = [k_1 \ k_2 \ k_3 \ k_4]$

→ If eigenvalues(poles) of matrix  $A - BK$  have negative-real part, then  $\mathbf{x}$  goes to zero at  $t \rightarrow \infty$



The control methods, which were introduced at the 'Nonlinear system' part, were failed to control state 'x' to be zero at  $t \rightarrow \infty$ . On the otherhand, linear feedback control make the system to be stable about all state

Command 'acker' can be used to get desired gain  $K$

# Simulation : Linear Feedback Control

Simulation Condition

$$\theta(0) = 0.1 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$\dot{\theta}(0) = 0 \text{ rad/s}$$

$$\dot{x}(0) = 0 \text{ m/s}^2$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

## Simulation Results Plots

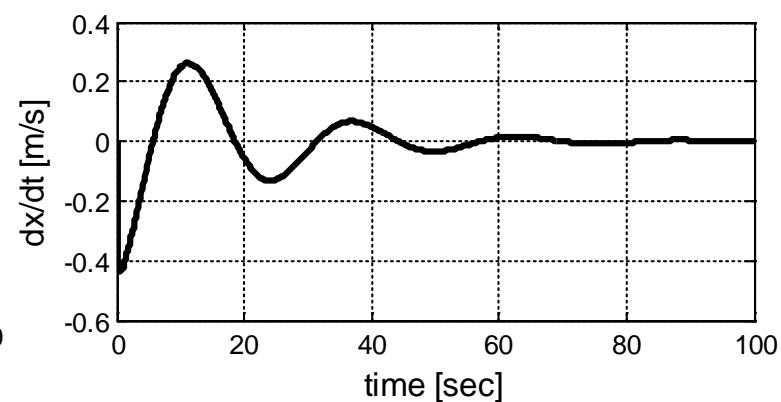
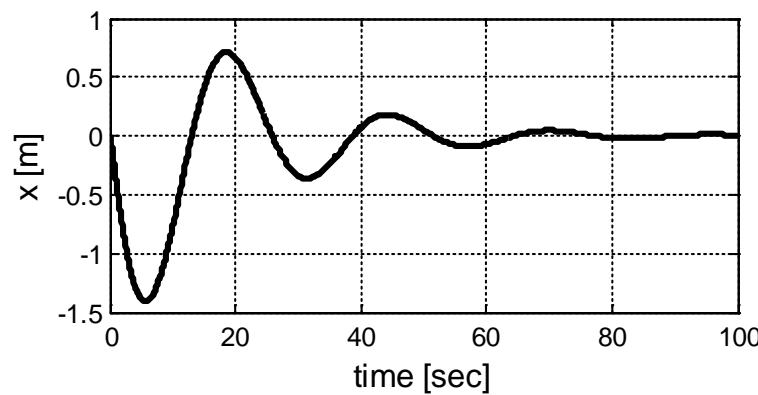
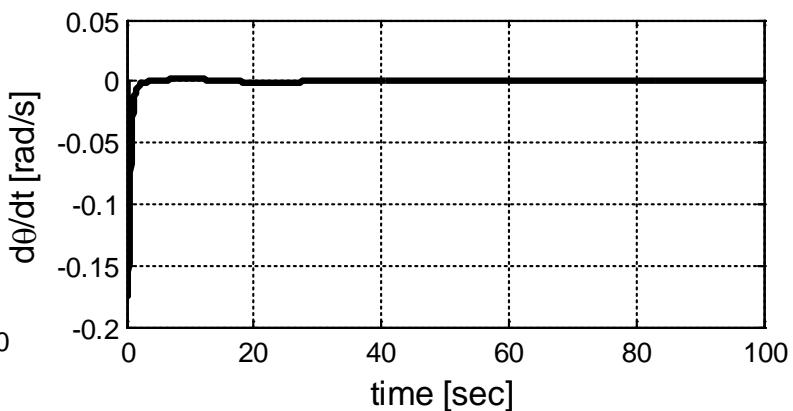
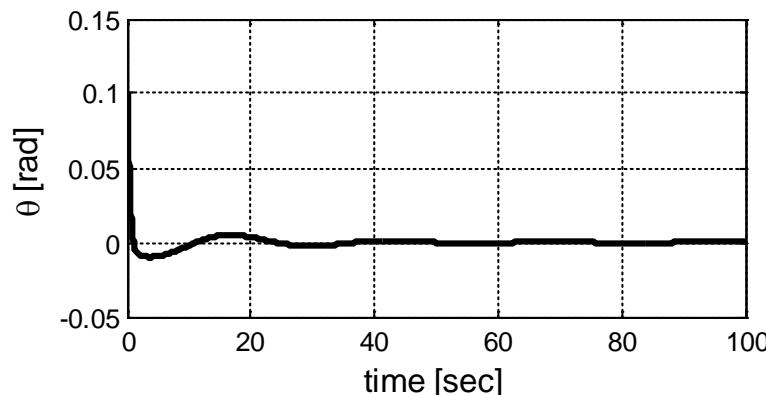


Fig 9. Simulation results of linear system for tracking control

# Simulation : Linear Feedback Control - Animation

Simulation Condition

$$\theta(0) = 0.1 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$\dot{\theta}(0) = 0 \text{ rad / s}$$

$$\dot{x}(0) = 0 \text{ m / s}^2$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

Force Plot and Simulation Results Animation

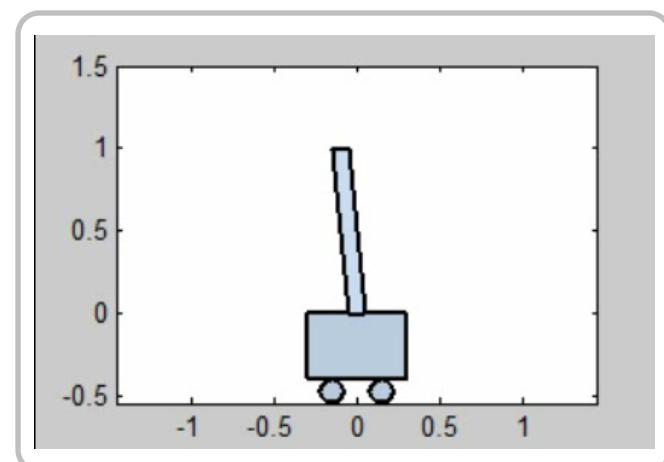
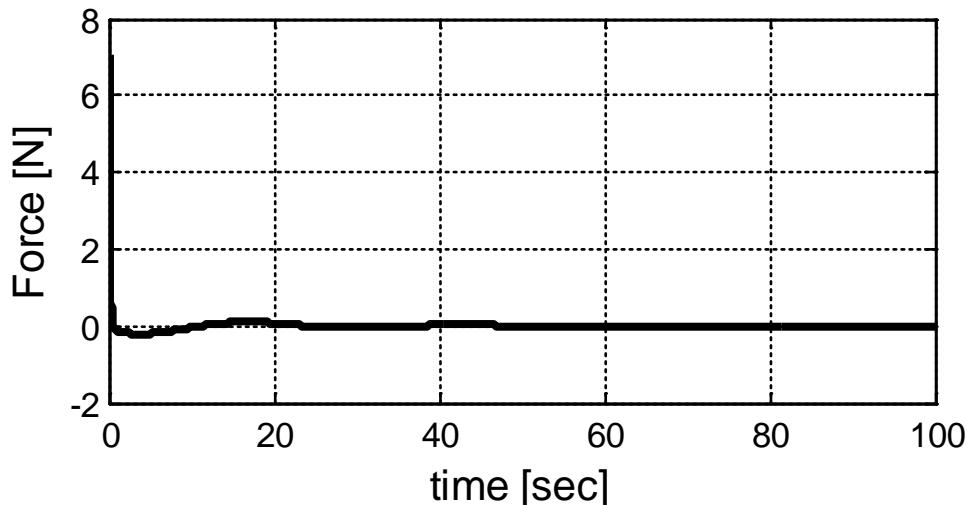


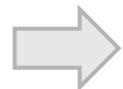
Fig 10. Feedback Control Input and Simulation results Animation

# Linear Model : Tracking Problem

We discussed linear feedback control.

This make it possible that all the states go to zero at  $t \rightarrow \infty$ .

But what if we want to make the states converge to specific values?



Q) How to make the states converge to nonzero values

## Reference Input Tracking

Introduce reference input  $u = -K\mathbf{x} + r$

Let  $x_{ss}$  and  $u_{ss}$  as state  $x$  and input  $u$  respectively at steady state

Then,  $u = u_{ss} - K(x - x_{ss})$

if the system is like  $\dot{\mathbf{x}} = A\mathbf{x} + Bu$  where  $\mathbf{x} = [x_1 \ x_2 \ x_3 \ x_4]^T = [\theta \ \dot{\theta} \ x \ \dot{x}]^T$ ,  
 $y = C\mathbf{x} + Du$

At the steady state, this system becomes

$$0 = A\mathbf{x}_{ss} + Bu_{ss} \quad \dots\dots (*) \quad (\because \text{At steady state, } \dot{\mathbf{x}} = 0)$$
$$y_{ss} = C\mathbf{x}_{ss} + Du_{ss}$$

# Linear Model : Tracking Problem

We want to make  $y_{ss} = r_{ss}$  for any value of  $r_{ss}$

To do this, assume that  $\begin{aligned}\mathbf{x}_{ss} &= N_x r_{ss} \\ u_{ss} &= N_u r_{ss}\end{aligned}$  and put these equations to (\*)

Then  $0 = AN_x r_{ss} + BN_u r_{ss}$

$$r_{ss} = CN_x r_{ss} + DN_u r_{ss}$$

It can be also written as a matrix form like  $\begin{bmatrix} A & B \\ C & D \end{bmatrix} \begin{bmatrix} N_x \\ N_u \end{bmatrix} = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$

Assume that the inverse of  $\begin{bmatrix} A & B \\ C & D \end{bmatrix}$  exists, then this equation can be solved for  $N_x$  and  $N_u$

$$\begin{bmatrix} N_x \\ N_u \end{bmatrix} = \begin{bmatrix} A & B \\ C & D \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

From above relation

$$\begin{aligned}u &= N_u r - K(x - N_x r) = -K\mathbf{x} + (N_u + KN_x)r \\ &= -K\mathbf{x} + \bar{N}r\end{aligned} \quad \text{where } \bar{N} = N_u + KN_x$$

# Simulation : Reference Tracking Control

Simulation Condition

$$\theta(0) = 0.1 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$\dot{\theta}(0) = 0 \text{ rad/s}$$

$$\dot{x}(0) = 0 \text{ m/s}^2$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

$$x_{ss} = r_{ss} = 5$$

## Simulation Results Plots

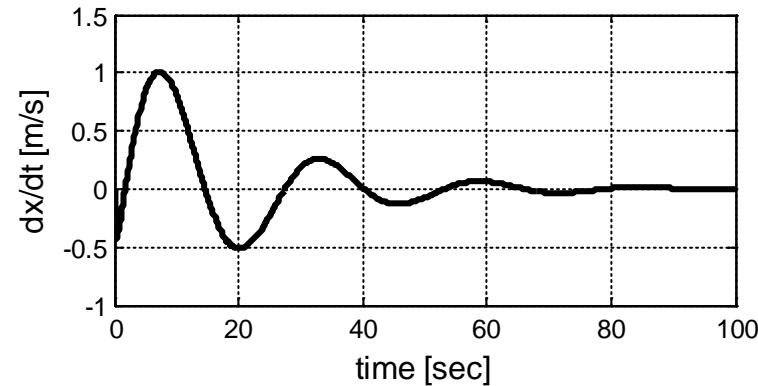
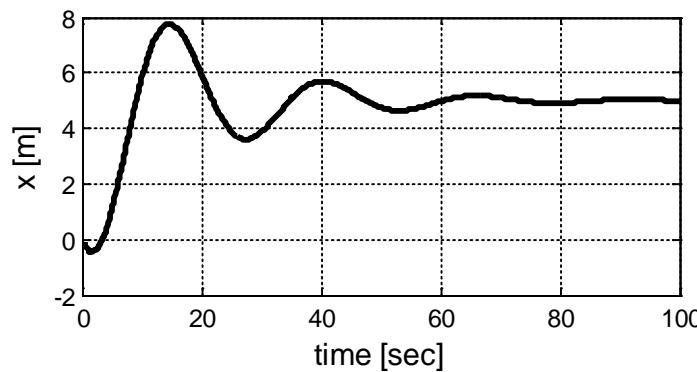
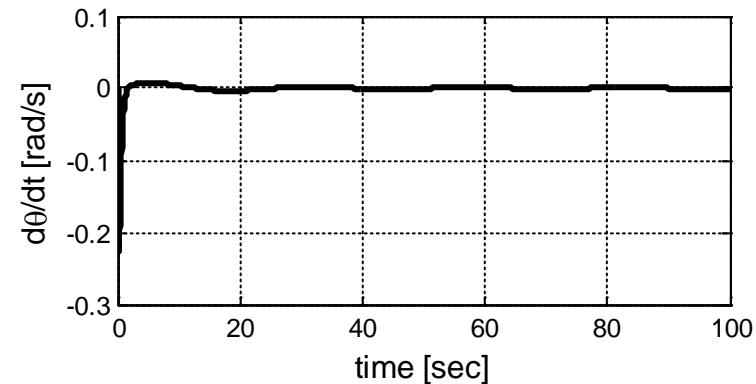
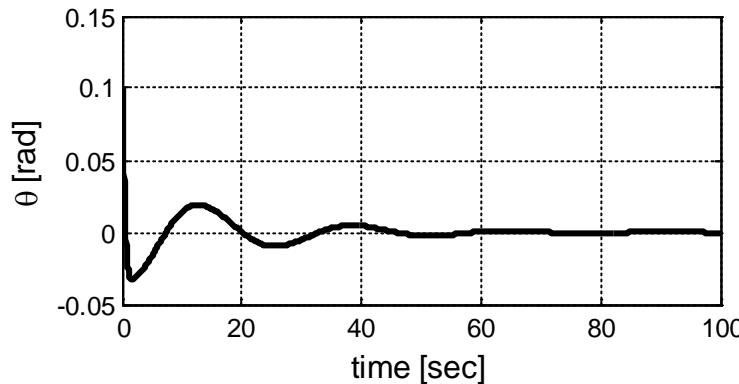


Fig 11. Simulation results of linear system for tracking control

# Simulation : Reference Tracking Control - Animation

Simulation Condition

$$\theta(0) = 0.1 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

$$\dot{\theta}(0) = 0 \text{ rad / s}$$

$$\dot{x}(0) = 0 \text{ m / s}^2$$

$$x_{ss} = r_{ss} = 5$$

Force Plot and Simulation Results Animation

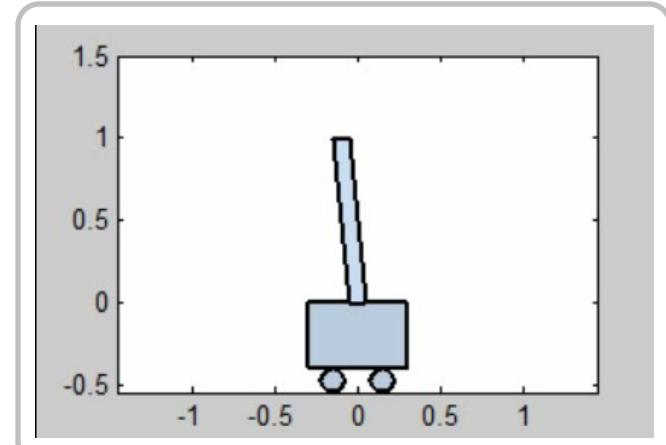
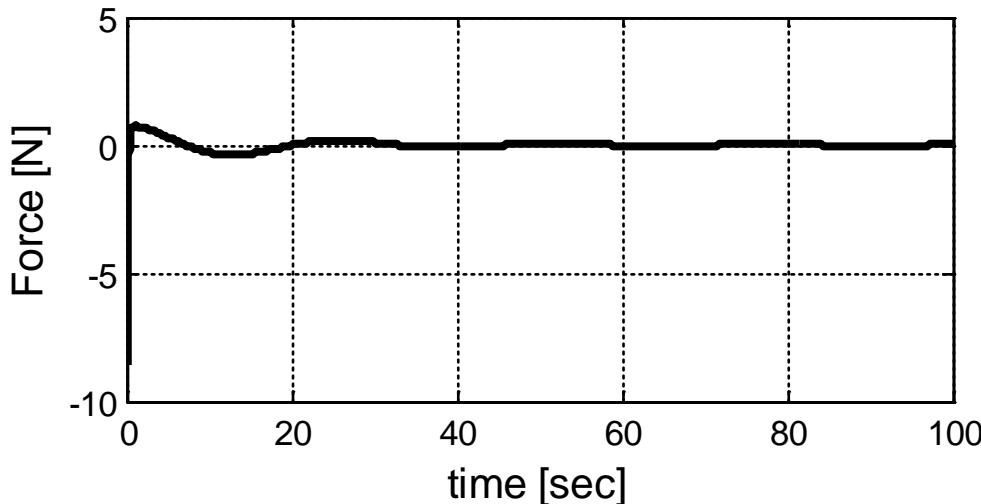


Fig 12. Reference Tracking Input and Simulation results Animation

### 3. Nonlinear System with Linear Controller

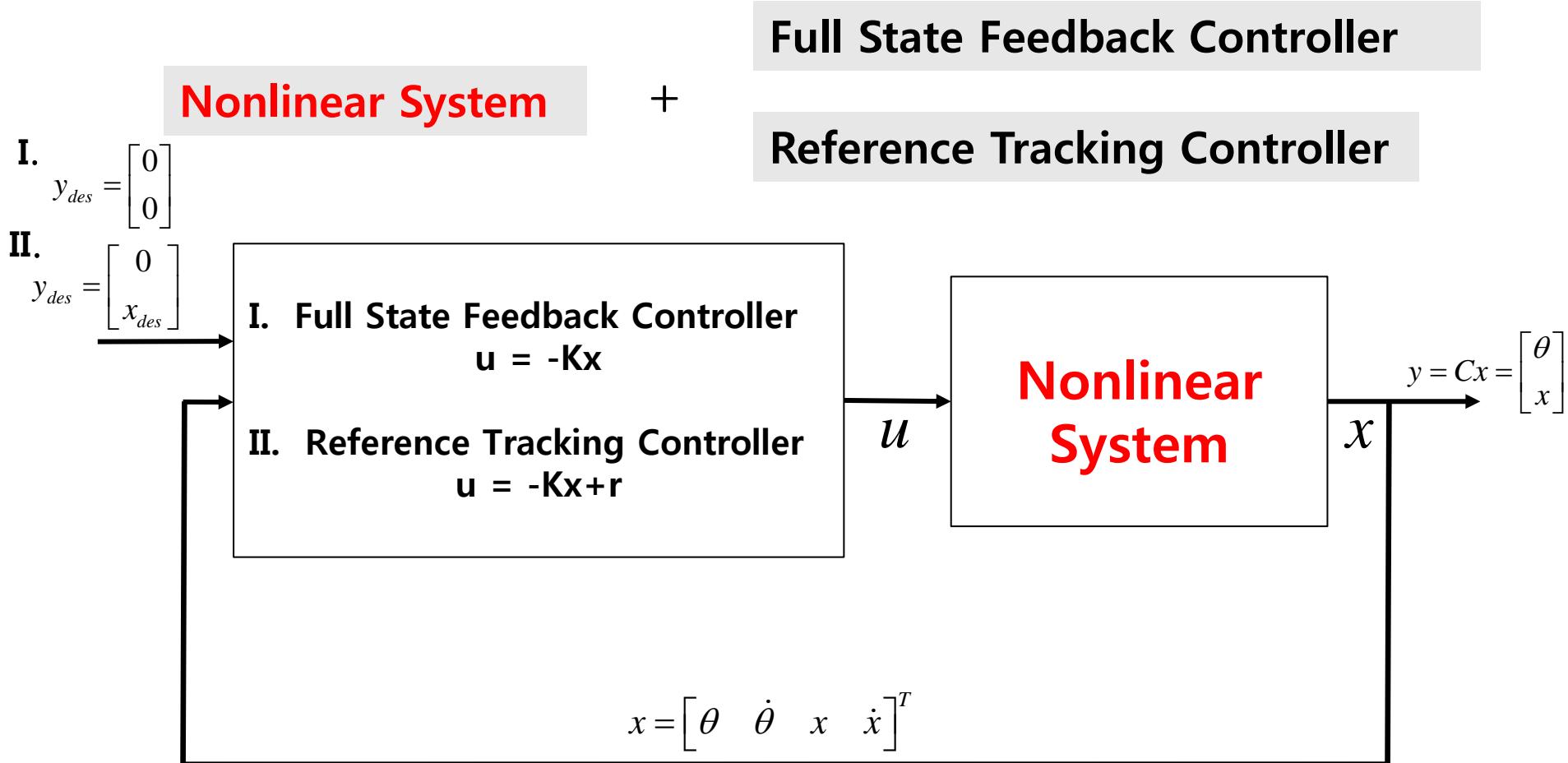
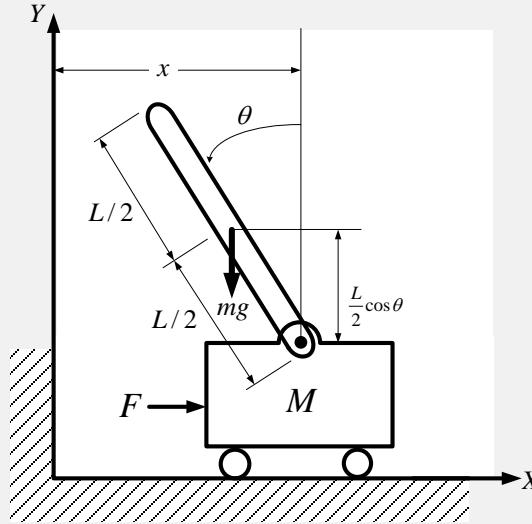


Fig 8. Block diagram for nonlinear system with linear controller

# Nonlinear System



*Equation of motions*

$$I\ddot{\theta} = m\ddot{x}\frac{L}{2}\cos\theta - m\left(\frac{L}{2}\right)^2\dot{\theta} + mg\frac{L}{2}\sin\theta$$

$$(M+m)\ddot{x} + m\frac{L}{2}(\dot{\theta}^2 \sin\theta - \ddot{\theta}\cos\theta) = F$$

*Control input :  $u(t) = F$*

*Initial Condition :  $\theta(0) = \theta_0, \dot{\theta}(0) = 0, x(0) = 0, \dot{x}(0) = 0$*

*From combining above two equations*

$$\ddot{\theta} = \frac{B}{A}mg - \frac{1}{A}m\frac{L}{2}\dot{\theta}^2 \sin\theta + \frac{1}{A}F$$

$$\ddot{x} = C\left[-\frac{1}{A}m\frac{L}{2}\sin\theta\dot{\theta}^2 + \frac{B}{A}mg + \frac{1}{A}F\right] - g\tan\theta$$

$$\dot{\mathbf{x}} = \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} x_2 \\ \frac{B}{A}mg - \frac{1}{A}m\frac{L}{2}\dot{\theta}^2 \sin\theta \\ x_4 \\ C\left[-\frac{1}{A}m\frac{L}{2}\sin\theta\dot{\theta}^2 + \frac{B}{A}mg\right] - g\tan\theta \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{A} \\ 0 \\ \frac{C}{A} \end{bmatrix}F(t)$$

Where  $A = \frac{(M+m)}{m\frac{L}{2}\cos\theta}\left[I + m\left(\frac{L}{2}\right)^2\right] - m\frac{L}{2}\cos\theta$ ,  $B = \frac{(M+m)}{m\frac{L}{2}\cos\theta}\frac{L}{2}\sin\theta$ ,  $C = \frac{I + m(L/2)^2}{m(L/2)\cos\theta}$

# Linear Feedback Control

When a system is defined as  $\dot{\mathbf{x}} = A\mathbf{x} + Bu$ , we want to control  $\mathbf{x}$  to make  $\mathbf{x}(\infty) = 0$

Let  $u = -K\mathbf{x}$

Then,  $\dot{\mathbf{x}} = A\mathbf{x} - Bu = (A - BK)\mathbf{x} \Rightarrow \mathbf{x} = e^{(A-BK)t}$

( $\mathbf{x}$  :  $n \times 1$  matrix,  $A$  :  $n \times n$  matrix,  $B$  :  $n \times 1$  matrix,  $K$  :  $1 \times n$  matrix)

Appropriate gain  $K$  makes the system stable

where  $\mathbf{x} = [x_1 \ x_2 \ x_3 \ x_4]^T = [\theta \ \dot{\theta} \ x \ \dot{x}]^T$ ,  $K = [k_1 \ k_2 \ k_3 \ k_4]$

→ If eigenvalues(poles) of matrix  $A - BK$  have negative-real part, then  $\mathbf{x}$  goes to zero at  $t \rightarrow \infty$



The control methods, which were introduced at the 'Nonlinear system' part, were failed to control state 'x' to be zero at  $t \rightarrow \infty$ . On the otherhand, linear feedback control make the system to be stable about all state

Command 'acker' can be used to get desired gain  $K$

# Nonlinear System vs Linear System with Linear Feedback Control

Simulation Condition

$$\theta(0) = 0.1 \text{ rad}$$

$$\dot{\theta}(0) = 0 \text{ rad/s}$$

$$x(0) = 0 \text{ m}$$

$$\dot{x}(0) = 0 \text{ m/s}^2$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

## Simulation Results Plots

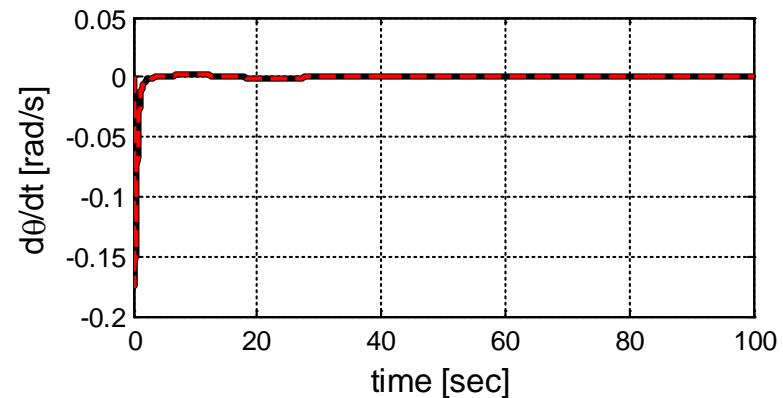
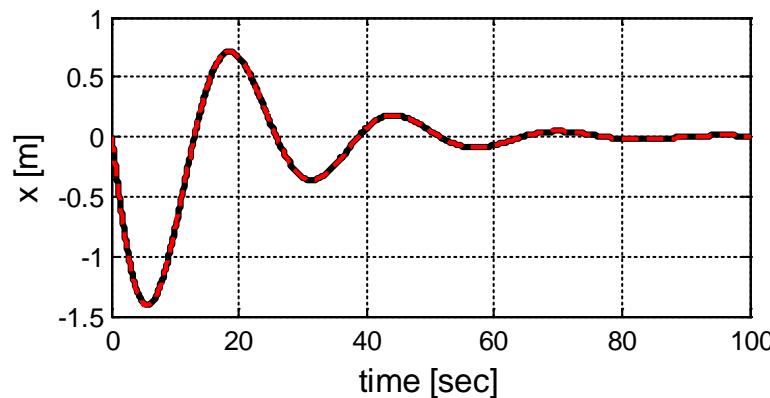
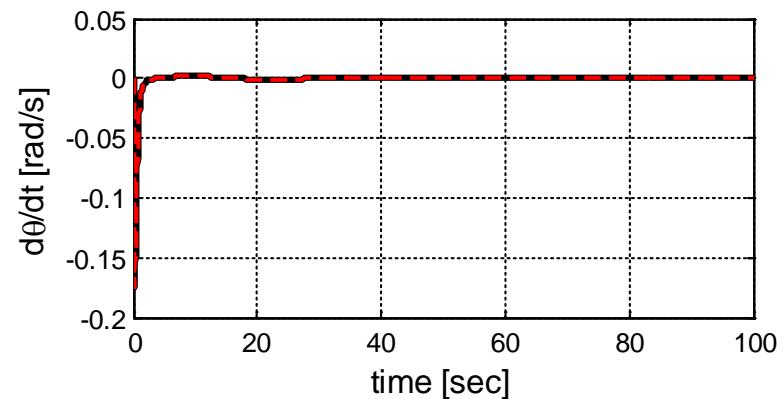
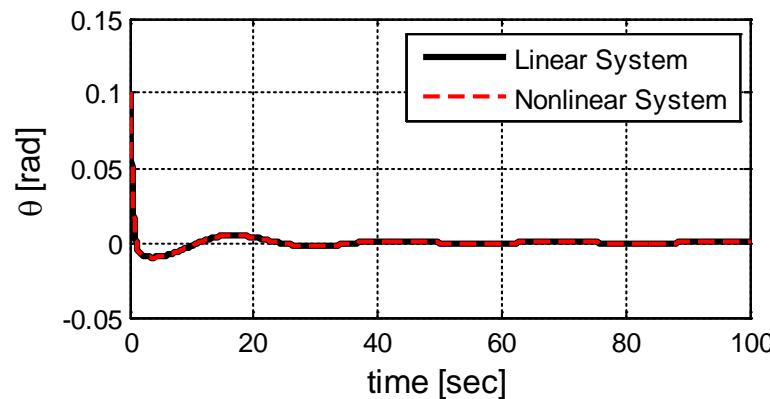


Fig 9. Simulation results for linear Feedback control

# Nonlinear System vs Linear System with Linear Feedback Control

Simulation Condition

$$\theta(0) = 0.1 \text{ rad}$$

$$\dot{\theta}(0) = 0 \text{ rad/s}$$

$$x(0) = 0 \text{ m}$$

$$\dot{x}(0) = 0 \text{ m/s}^2$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

Force Plot and Simulation Results Animation

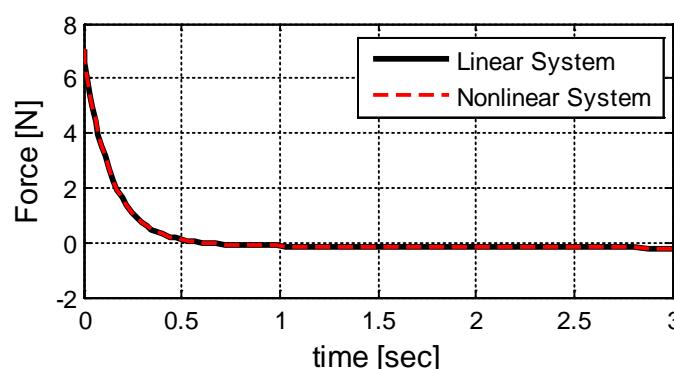
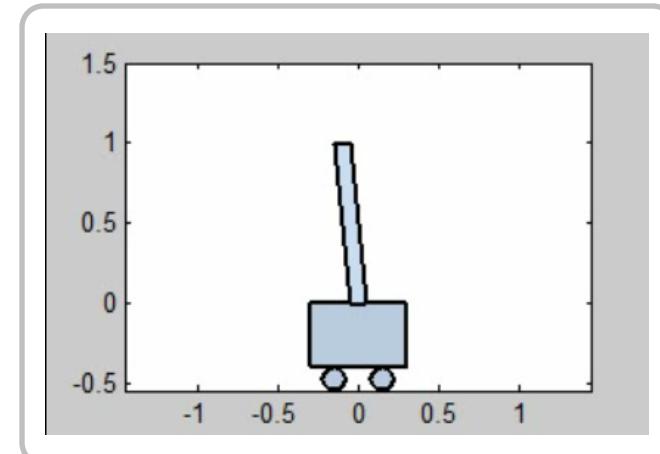
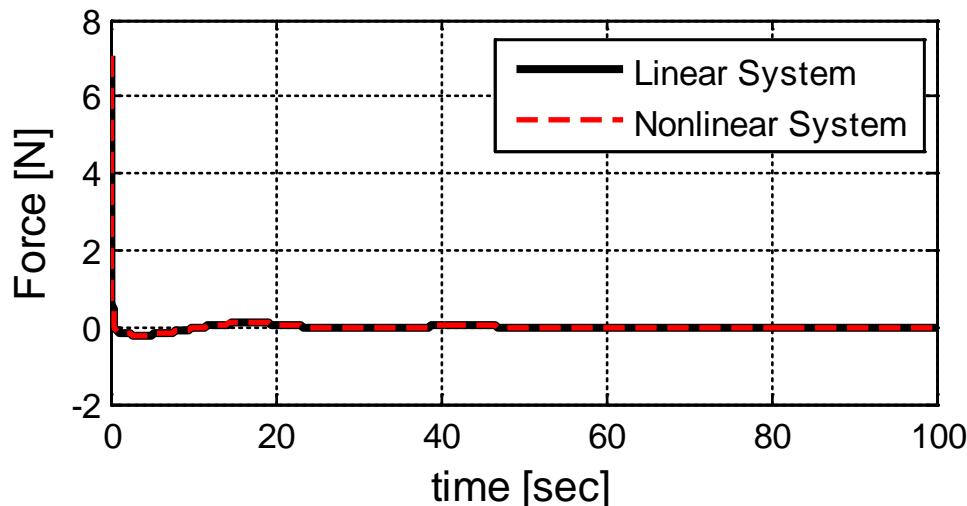


Fig 10. Feedback Control Input (Nonlinear system) and Simulation results Animation

# Nonlinear System vs Linear System with Linear Feedback Control

Simulation Condition

$$\theta(0) = 1.57 \text{ rad}$$

$$\dot{\theta}(0) = 0 \text{ rad/s}$$

$$x(0) = 0 \text{ m}$$

$$\dot{x}(0) = 0 \text{ m/s}^2$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

## Simulation Results Plots

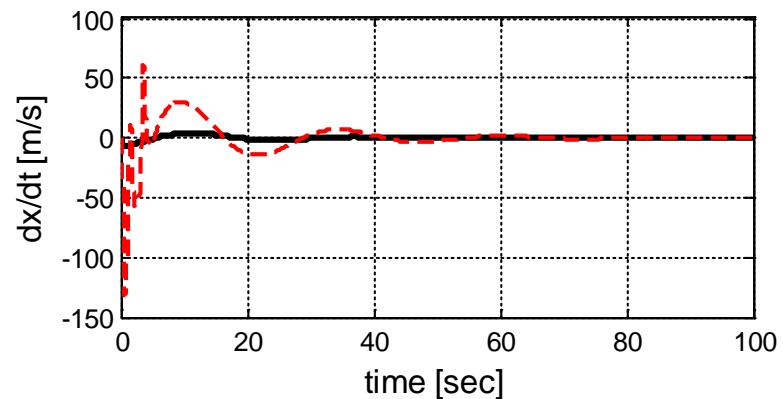
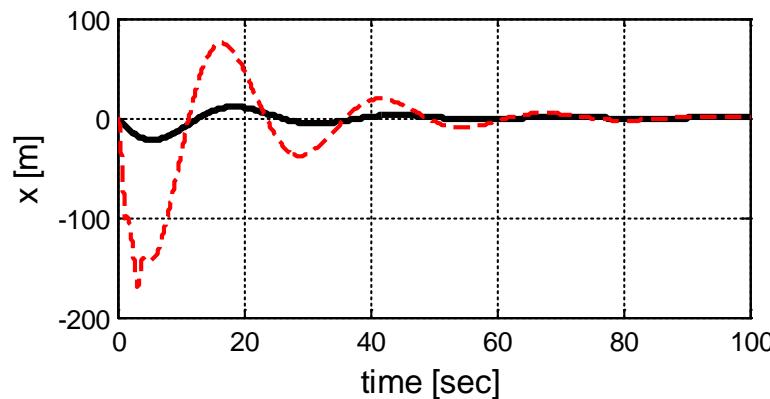
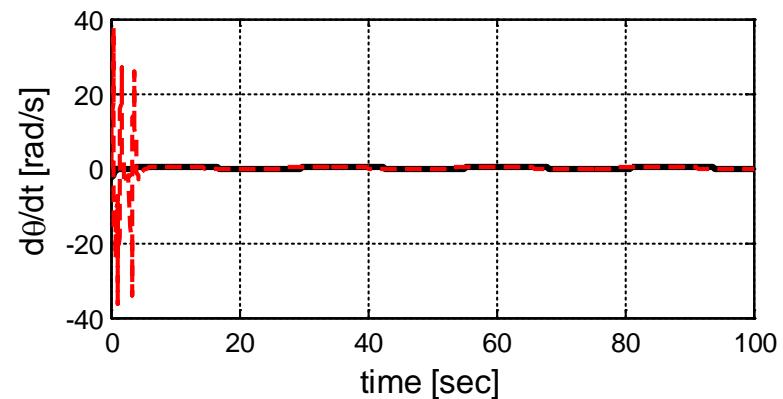
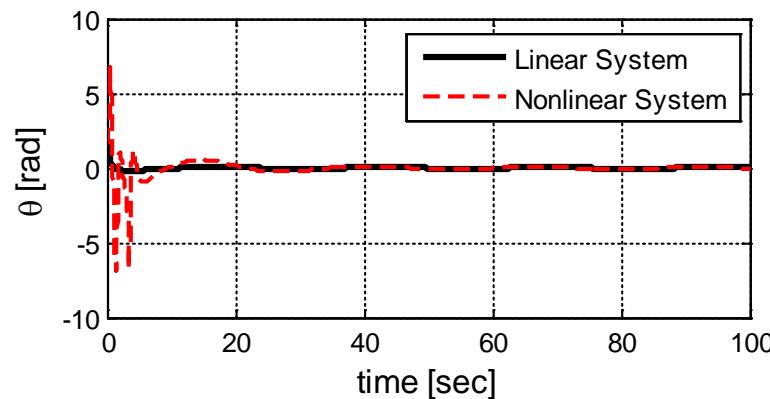


Fig 11. Simulation results for linear feedback control

# Nonlinear System vs Linear System with Linear Feedback Control

Simulation Condition

$$\theta(0) = 1.57 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$\dot{\theta}(0) = 0 \text{ rad/s}$$

$$\dot{x}(0) = 0 \text{ m/s}^2$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

## Force Plot and Simulation Results Animation

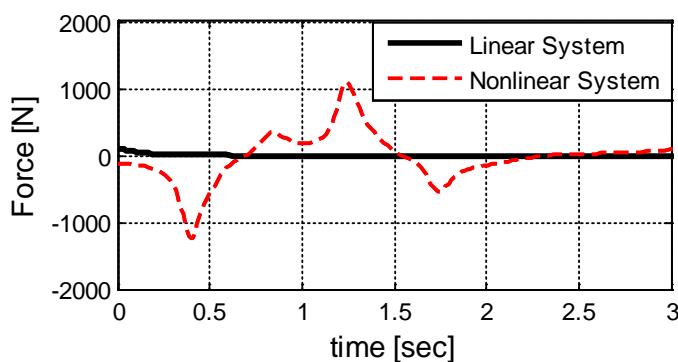
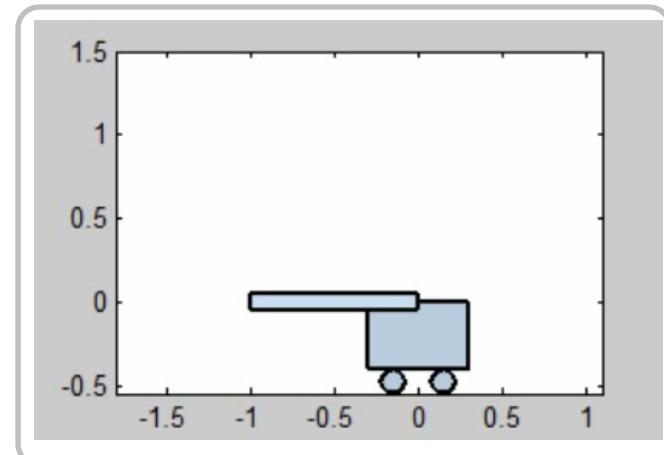
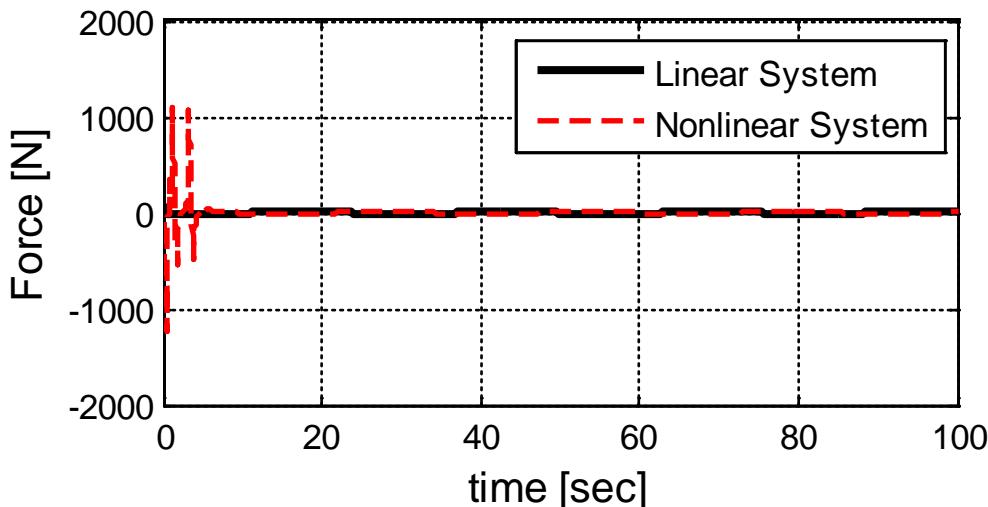


Fig 12. Feedback Control Input (Nonlinear system) and Simulation results Animation

# Nonlinear System with Linear Feedback Control

Simulation Condition

$$\theta(0) = 2.44 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$\dot{\theta}(0) = 0 \text{ rad/s}$$

$$\dot{x}(0) = 0 \text{ m/s}^2$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

## Simulation Results Plots

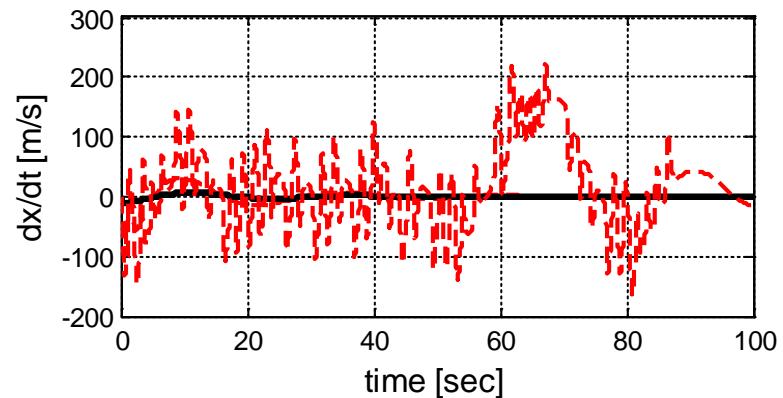
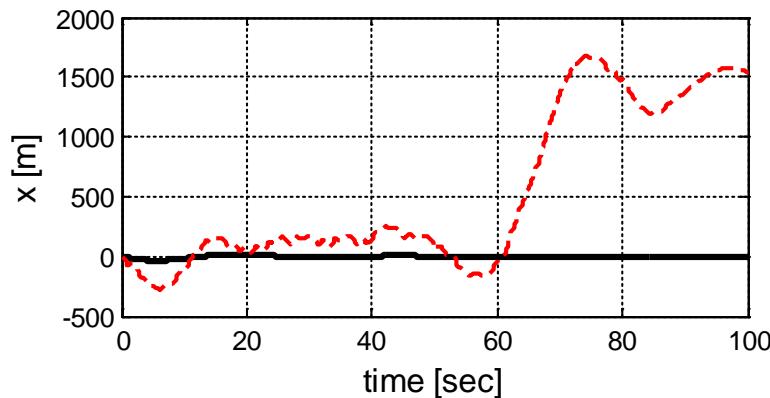
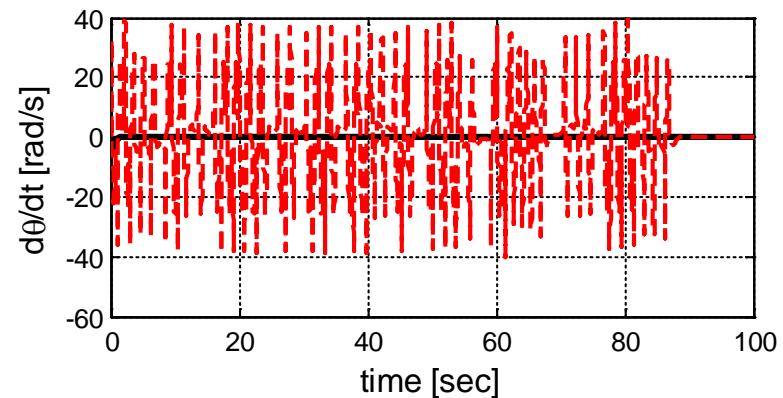
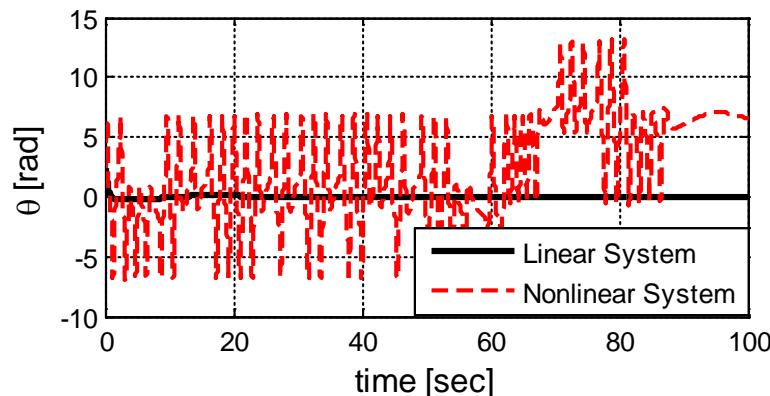


Fig 13. Simulation results for linear feedback control

# Nonlinear System with Linear Feedback Control - Animation

Simulation Condition

$$\theta(0) = 2.44 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$\dot{\theta}(0) = 0 \text{ rad / s}$$

$$\dot{x}(0) = 0 \text{ m / s}^2$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

Force Plot and Simulation Results Animation

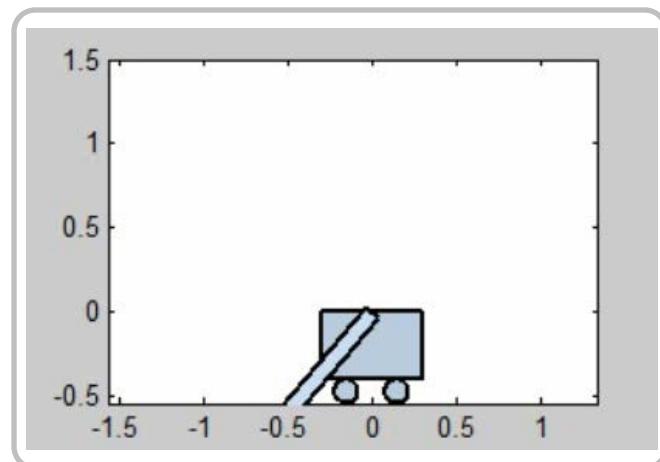
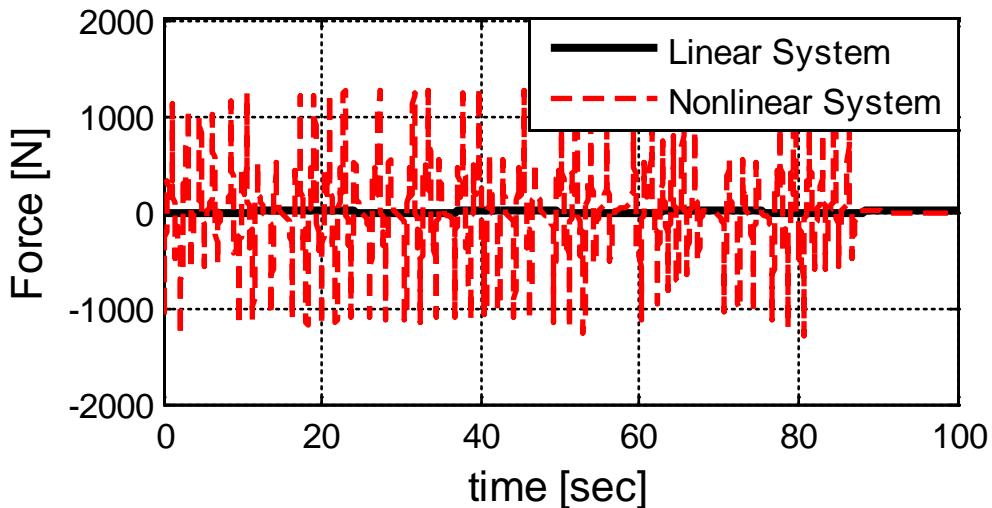


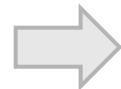
Fig 14. Feedback Control Input (Nonlinear system) and Simulation results Animation

# Tracking Problem

We discussed linear feedback control.

This make it possible that all the states go to zero at  $t \rightarrow \infty$ .

But what if we want to make the states converge to specific values?



Q) How to make the states converge to nonzero values

## Reference Input Tracking

Introduce reference input  $u = -K\mathbf{x} + r$

Let  $x_{ss}$  and  $u_{ss}$  as state  $x$  and input  $u$  respectively at steady state

Then,  $u = u_{ss} - K(x - x_{ss})$

if the system is like  $\dot{\mathbf{x}} = A\mathbf{x} + Bu$  where  $\mathbf{x} = [x_1 \ x_2 \ x_3 \ x_4]^T = [\theta \ \dot{\theta} \ x \ \dot{x}]^T$ ,  
 $y = C\mathbf{x} + Du$

At the steady state, this system becomes

$$0 = A\mathbf{x}_{ss} + Bu_{ss} \quad \dots\dots (*) \quad (\because \text{At steady state, } \dot{\mathbf{x}} = 0)$$
$$y_{ss} = C\mathbf{x}_{ss} + Du_{ss}$$

# Tracking Problem

We want to make  $y_{ss} = r_{ss}$  for any value of  $r_{ss}$

To do this, assume that  $\begin{aligned}\mathbf{x}_{ss} &= N_x r_{ss} \\ u_{ss} &= N_u r_{ss}\end{aligned}$  and put these equations to (\*)

Then  $0 = AN_x r_{ss} + BN_u r_{ss}$

$$r_{ss} = CN_x r_{ss} + DN_u r_{ss}$$

It can be also written as a matrix form like  $\begin{bmatrix} A & B \\ C & D \end{bmatrix} \begin{bmatrix} N_x \\ N_u \end{bmatrix} = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$

Assume that the inverse of  $\begin{bmatrix} A & B \\ C & D \end{bmatrix}$  exists, then this equation can be solved for  $N_x$  and  $N_u$

$$\begin{bmatrix} N_x \\ N_u \end{bmatrix} = \begin{bmatrix} A & B \\ C & D \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

From above relation

$$\begin{aligned}u &= N_u r - K(x - N_x r) = -K\mathbf{x} + (N_u + KN_x)r \\ &= -K\mathbf{x} + \bar{N}r\end{aligned} \quad \text{where } \bar{N} = N_u + KN_x$$

# Nonlinear System vs Linear System with Reference Tracking Control

Simulation Condition

$$\theta(0) = 0.1 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

$$\dot{\theta}(0) = 0 \text{ rad/s}$$

$$\dot{x}(0) = 0 \text{ m/s}^2$$

$$x_{ss} = r_{ss} = 5$$

## Simulation Results Plots

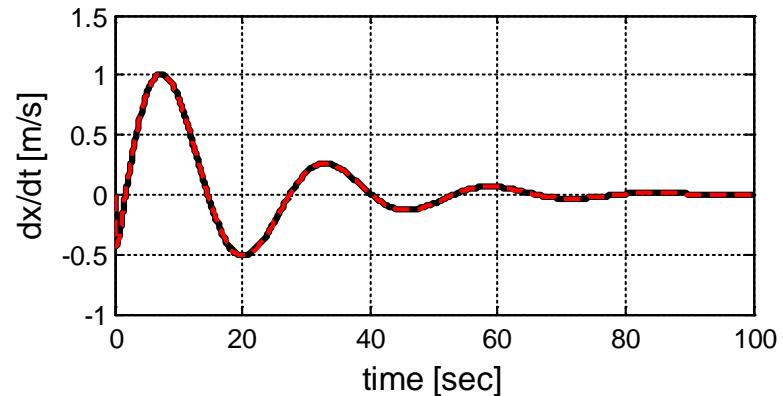
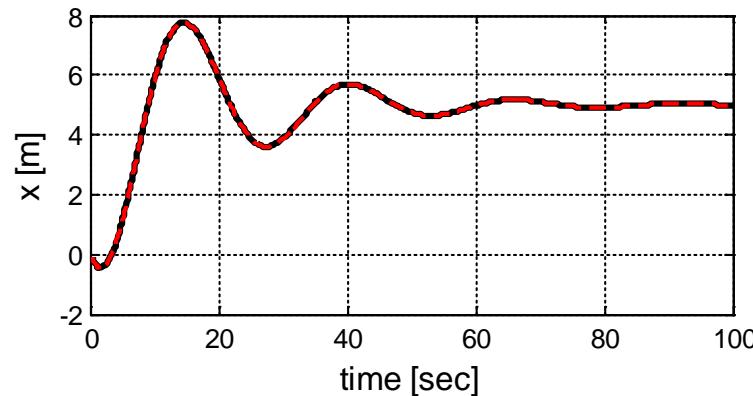
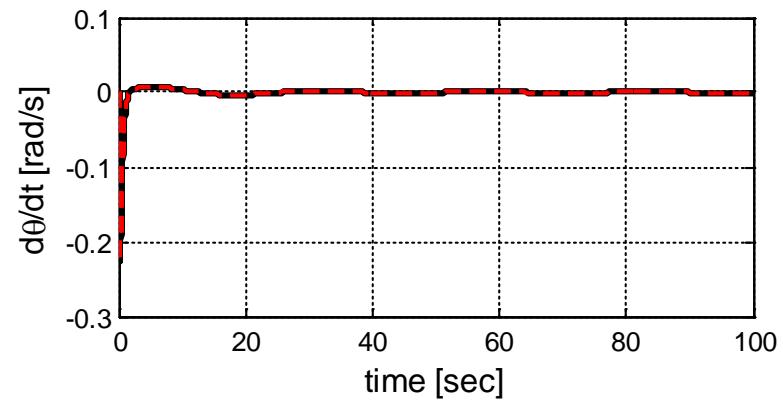
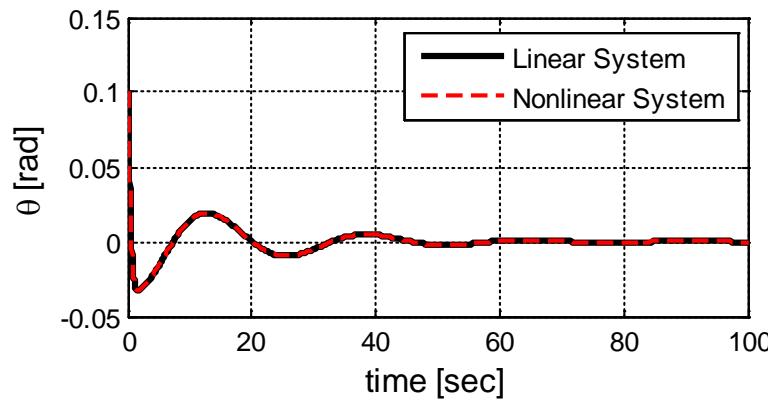


Fig 15. Simulation results for tracking control

# Nonlinear System vs Linear System with Reference Tracking Control

Simulation Condition

$$\theta(0) = 0.1 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

$$\dot{\theta}(0) = 0 \text{ rad / s}$$

$$\dot{x}(0) = 0 \text{ m / s}^2$$

$$x_{ss} = r_{ss} = 5$$

## Force Plot and Simulation Results Animation

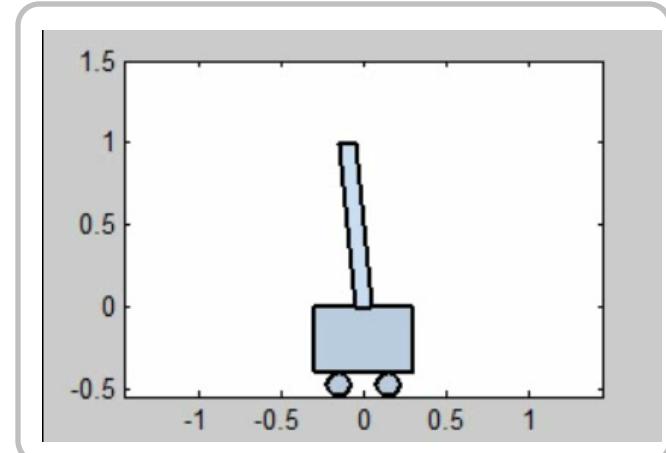
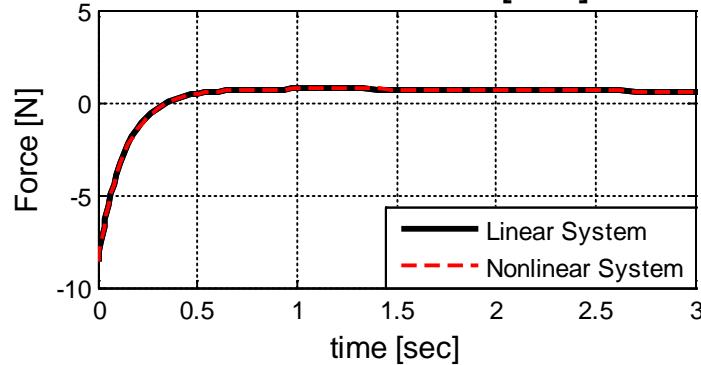
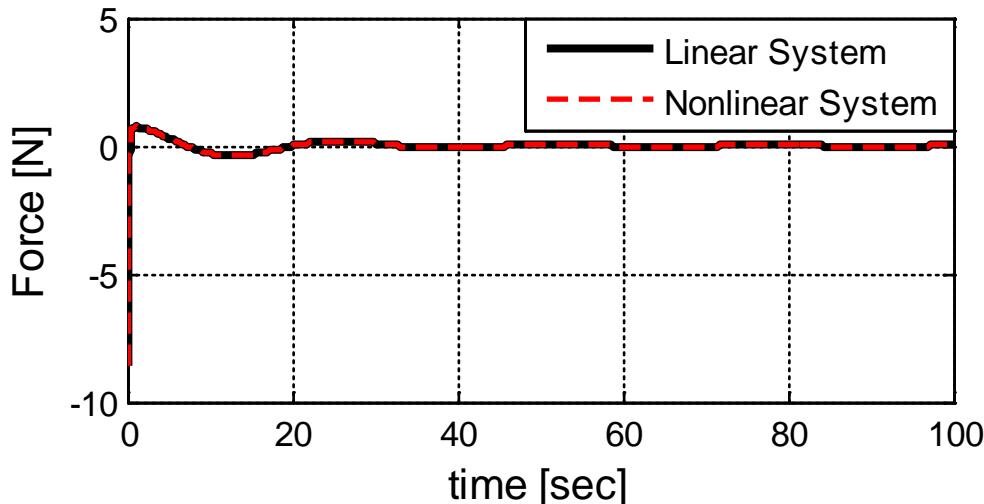


Fig 16. Reference Tracking Input (Nonlinear system) and Simulation results Animation

# Nonlinear System vs Linear System with Reference Tracking Control

Simulation Condition

$$\theta(0) = 1.57 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

$$\dot{\theta}(0) = 0 \text{ rad/s}$$

$$\dot{x}(0) = 0 \text{ m/s}^2$$

$$x_{ss} = r_{ss} = 5$$

## Simulation Results Plots

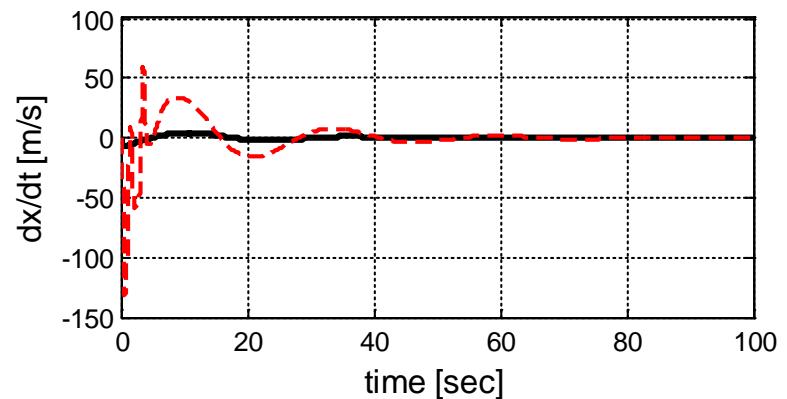
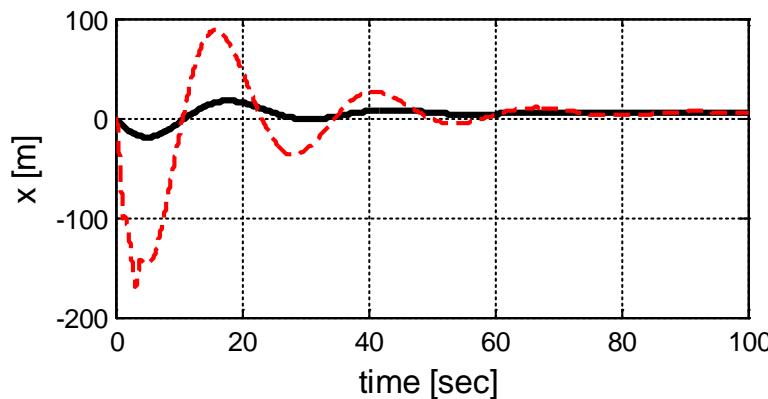
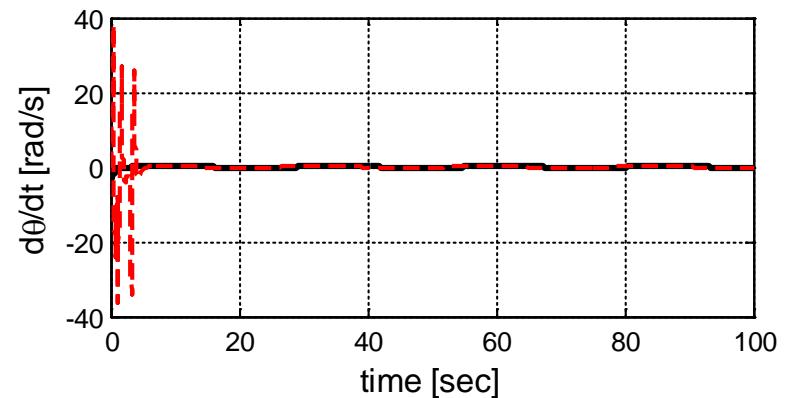
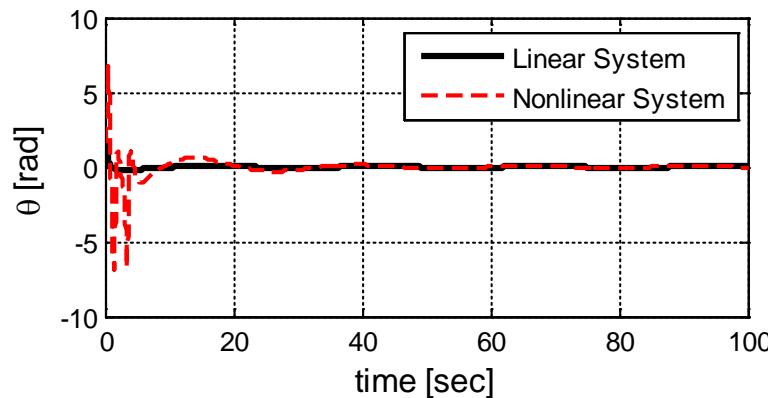


Fig 17. Simulation results for tracking control

# Nonlinear System vs Linear System with Reference Tracking Control

Simulation Condition

$$\theta(0) = 1.57 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

$$\dot{\theta}(0) = 0 \text{ rad / s}$$

$$\dot{x}(0) = 0 \text{ m / s}^2$$

$$x_{ss} = r_{ss} = 5$$

Force Plot and Simulation Results Animation

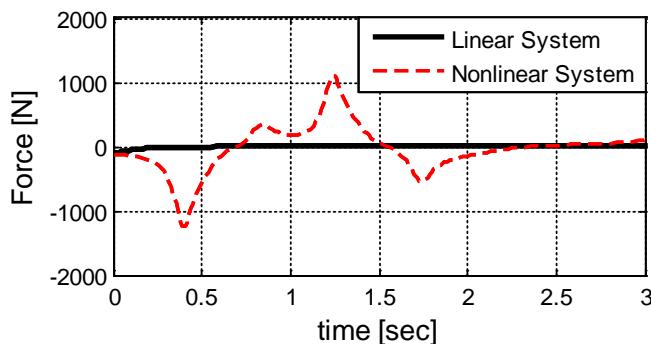
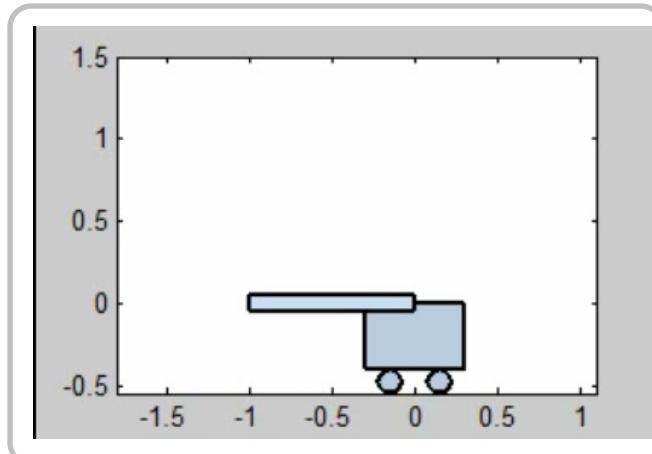
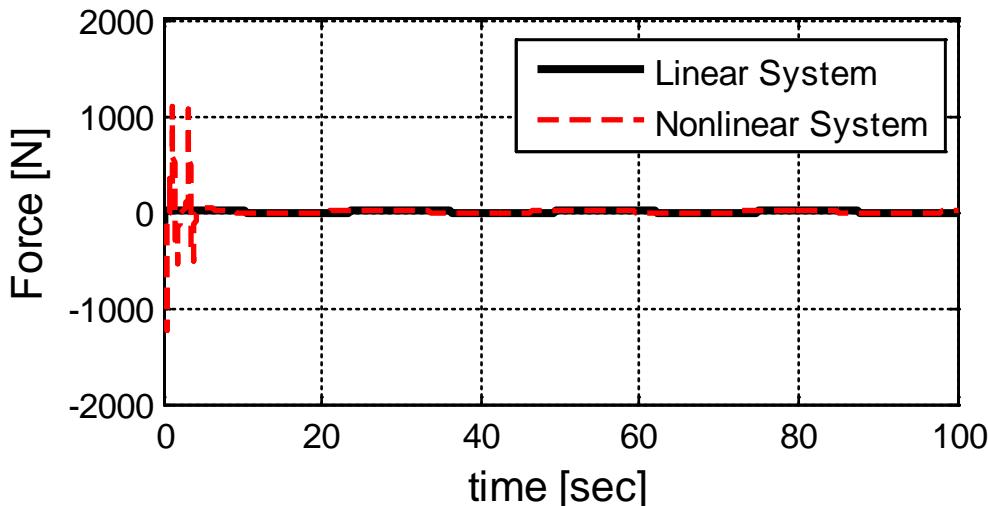


Fig 18. Reference Tracking Input (Nonlinear system) and Simulation results Animation

# Nonlinear System vs Linear System with Reference Tracking Control

Simulation Condition

$$\theta(0) = 3.05 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

$$\dot{\theta}(0) = 0 \text{ rad/s}$$

$$\dot{x}(0) = 0 \text{ m/s}^2$$

$$x_{ss} = r_{ss} = 5$$

## Simulation Results Plots

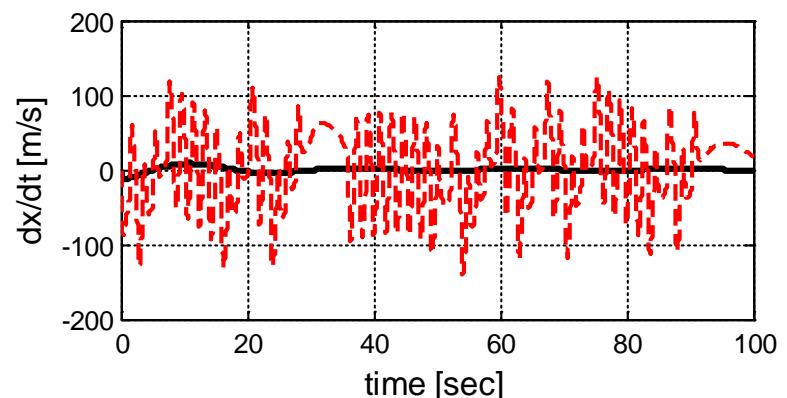
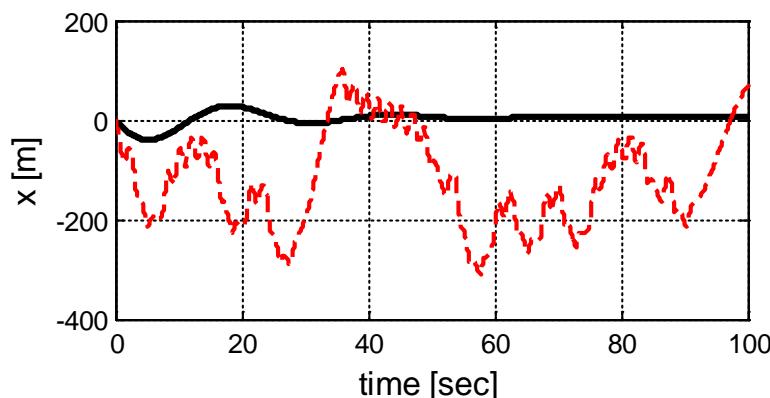
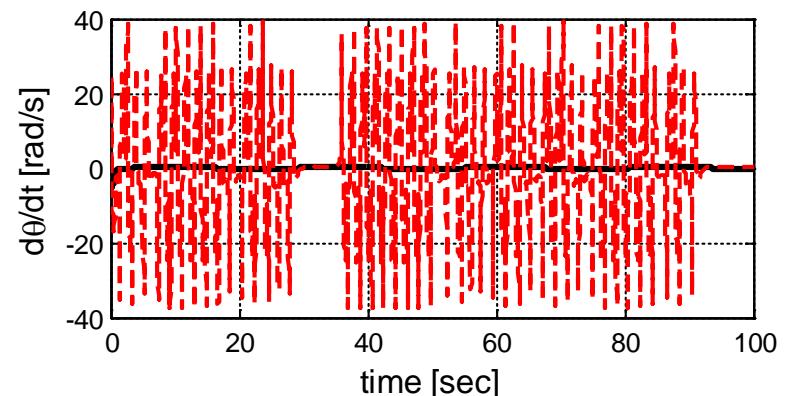
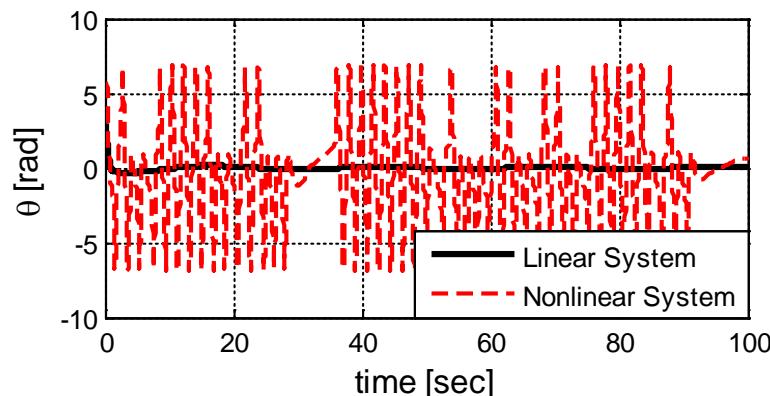


Fig 19. Simulation results for tracking control

# Nonlinear System vs Linear System with Reference Tracking Control

Simulation Condition

$$\theta(0) = 3.05 \text{ rad}$$

$$x(0) = 0 \text{ m}$$

$$K = [70.5, 22.7, -0.3, -0.6483]$$

$$\dot{\theta}(0) = 0 \text{ rad / s}$$

$$\dot{x}(0) = 0 \text{ m / s}^2$$

$$x_{ss} = r_{ss} = 5$$

Force Plot and Simulation Results Animation

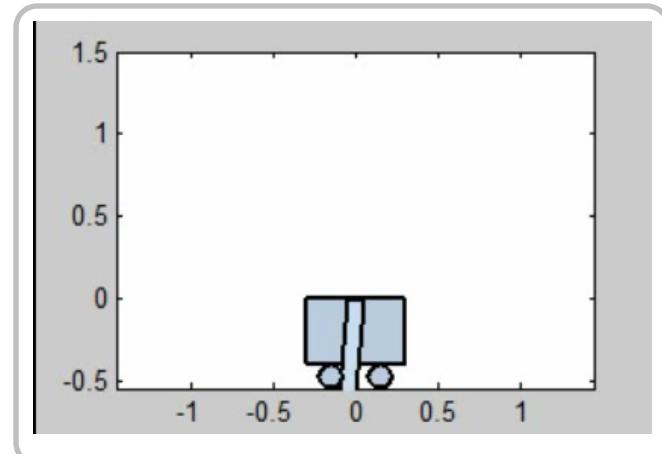
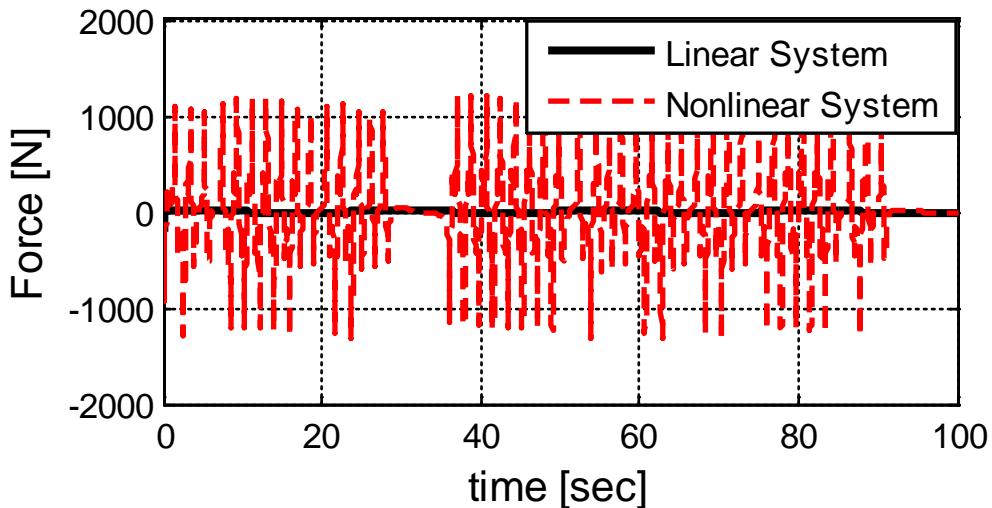


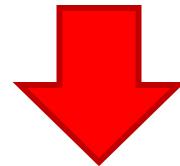
Fig 20. Reference Tracking Input (Nonlinear system) and Simulation results Animation

# **Appendix. A**

# How to use

## Open 'RUN\_file.m'

1. Simulation(시간, 초기조건 등) 환경 설정
2. Parameter(질량, 길이 등) 값 설정
3. Gain 값 설정
4. 실행
5. 'Simulaton\_file.m'이 실행된다.



## 'Simulation\_file.m'

5. Nonlinear Equation을 실시간으로 계산(By function '*ODE4*')
6. State의 실시간 값들이 얻어짐(x, theta...)
7. State의 시간에 대한 그래프 출력

# How to use – RUN\_file.m



Open 'RUN\_file.m'

Simulation Setting

```
global m M L I g Kp Kd zeta wn mode Kf A B u_ss x_ss;
```

%% Simulation Configuration

```
t0=0; % Initial Time [sec]  
tf=100; % Final Time that simulation ends [sec]  
del_t = 0.05; % Sampling time [sec]
```

%% Initial Condition

```
x0 = 0; % Initial Position of Cart [m]  
d_x0 = 0; % Initial velocity of Cart [m/s]  
theta_0 = 0.1; % initial Condition of Angle 1 [rad]  
d_theta_0 = 0; % initial Condition of Angular Velocity [rad/s]
```

%% Tracing Control

```
rss = 0; % Desried 'x'
```

%% System Type Selection

```
mode = 3; % < Choose the mode >  
% mode 1 : Nonlinear System +  
% Angle and Angle rate Controller  
% mode 2 : Linear System  
% mode 3 : Nonlinear System +  
% Linear Feedback Controller
```

%% Dynamic Parameters

```
m=0.1; % Mass of rod [kg]  
M=2; % Mass of cart [kg]  
L=1; % Length of rod [m]  
I = m*(L/2)^2; % Moment of inertia of rod [kg*m^2]  
g = 9.81; % Gravity [m/s^2]
```

Global 변수 선언

시뮬레이션에 필요한 함수인 ode45에 쓰일 변수를 선언.  
부록B에 ode45에 대해서 설명해두었다.

Simulation time 설정

t0 : 초기 시간(=0)  
tf : simulation 종료 시간  
del\_t : sampling time 설정

Initial Condition 설정

x0 : Cart의 초기 위치  
d\_x0 : Cart의 초기 속도  
theta\_0 : Rod의 초기 각도  
d\_theta\_0 : Rod의 초기 각속도

Desired X 좌표 설정

rss : 목표 도달 X 좌표

Dynamic Parameter 설정

m : Rod의 질량  
M : Cart의 질량  
L : Rod의 길이  
I : Rod의 moment of inertia (자동으로 계산)  
g : 중력가속도 (고정값)

Control Gain 설정

K : P gain 값 설정 (양수)  
C : D gain 값 설정 (양수, 0일 시에는 P Control)  
Zeta : 댐핑 값 설정  
wn : 공진주파수 설정

# How to use – Simulation\_file.m

- ▶ 'RUN\_file'을 실행시키면 자동으로 'Simulation\_file.m'이 실행된다.

## % Simulation File

```
% - Run ode function to calculate 'theta' in real time  
% - If mode selection is invalid, display the message  
% - Display result plot  
% - Display animation
```

## % Run Simulation

```
[t,X]=ode45(@OSC_control,[0, tf],[theta_0, d_theta_0, x0, d_x0]);  
figure(1)  
set.figure(1,'Position',[200,400,400,200]);  
plot(t,X(:,1),'k','LineWidth',2.3);  
grid on;  
xlabel('time [sec]', 'FontSize',12);  
ylabel('theta [rad]', 'FontSize',12);  
figure(2);  
set.figure(2,'Position',[620,400,400,200]);  
plot(t,X(:,2),'k','LineWidth',2.3);  
grid on;  
xlabel('time [sec]', 'FontSize',12);  
ylabel('dtheta/dt [rad]', 'FontSize',12);  
figure(3)  
set.figure(3,'Position',[200,700,400,200]);
```

### ode45

Ode45라는 함수를 통해 nonlinear equation을 실시간으로 풀어낼 수 있다.(부록B에 ode45에 대해 설명)

### State의 실시간 값들이 얻어짐

앞서 설정한 simulation time동안의 매 순간의 data 값(카트 위치, 바의 각도 등)들이 ode45 함수의 결과로 얻어진다.

### 결과의 시간에 대한 그래프 출력

이렇게 구해진 실시간 데이터 결과들을 시간에 대한 그래프로 출력한다.

# **Appendix. B**

# 부록 : ode45

## ▶ ODE45 ?

MATLAB에 내장돼있는 함수 & 미분방정식을 실시간으로 풀어주는 함수  
이 때 exact solution을 구하는 것이 아닌, 수치해석 방법으로 실시간 해를 구한다.  
Runge-Kutta method를 사용하는 numerical solver  
내가 풀고자 하는 미분방정식을 풀어주는 함수를 만든다.

## ▶ Why ode45?

선형 뿐 아니라 비선형 미분방정식도 풀 수 있으며

사용방법 및 수식작성이 직관적이다.

## ▶ How to use?

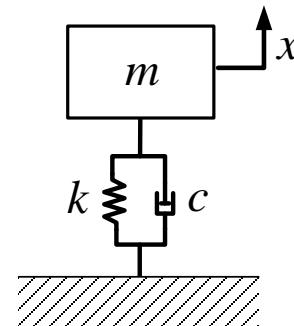
함수 이름 작성, 변수 선언  
(이 때 함수 이름은 저장 시에도 동일한 이름으로 저장해야 한다.)

$\dot{x} = f(x)$  꼴 그대로 수식으로 옮긴다.

## 부록 : ode45

## Example

$$m\ddot{x} + c\dot{x} + kx = 0, \quad x(0) = x_0, \dot{x}(0) = \dot{x}_0$$



1. 함수의 이름을 정하고 변수를 차례로 적어준다.(이때는 시간과 state x)

`function dx=function_name(t,x)`

-----> 미분 term을 나타내는 변수, state를 한번 미분한 것을 의미한다. 즉,  $\dot{x}$ 를 의미  
-----> 함수이름(저장 시 동일한 이름으로 저장해야 한다.)

2. 미분방정식에 쓰이는 global 변수(state 제외)를 선언한다.

`global m c k` 이 함수에서 이 변수들이 쓰이기 때문에 이 함수가 돌아가기 전, 이 변수들이 workspace 상에 선언되어 있어야 한다.

3.  $\dot{x} = f(x)$  꼴로 식을 만들어주고 code에 그대로 옮겨 적는다.

$$\mathbf{x} = \begin{bmatrix} x \\ \dot{x} \end{bmatrix} = \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}, \quad \begin{aligned} \dot{x}_1 &= x_2 \\ \dot{x}_2 &= -\frac{c}{m}x_2 - \frac{k}{m}x_1 \end{aligned}$$

`dx = zeros(2,1);`

# 부록 : ode45

- ▶ 문제의 미분방정식을 푸는 ode45를 MATLAB에 작성하면 다음과 같다.

## Example

$$m\ddot{x} + c\dot{x} + kx = 0, \quad x(0) = x_0, \dot{x}(0) = \dot{x}_0$$

```
function dx=function_name(t,x)
global m c k
dx = zeros(2,1);
dx(1) = x(2);
dx(2) = -(c/m)*x(2)-(k/m)*x(1);
```

- ▶ 그럼 이 때 다음과 같이, 앞에서 만든 ode45를 이용한 m-file을 작성하여 실행하면 다음과 같다.

```
global m c k

simtime = 10;
samtime = 0.1;

m = 10;
c = 20;
k = 100;

x0 = 2;
d_x0 = 5;

[t,X] = ode45(@function_name,[0,simtime],[x0,d_x0]);
```

사용할 함수 이름

t의 초기, 말기 값

State의 초기 값(x1, x2 차례로)

