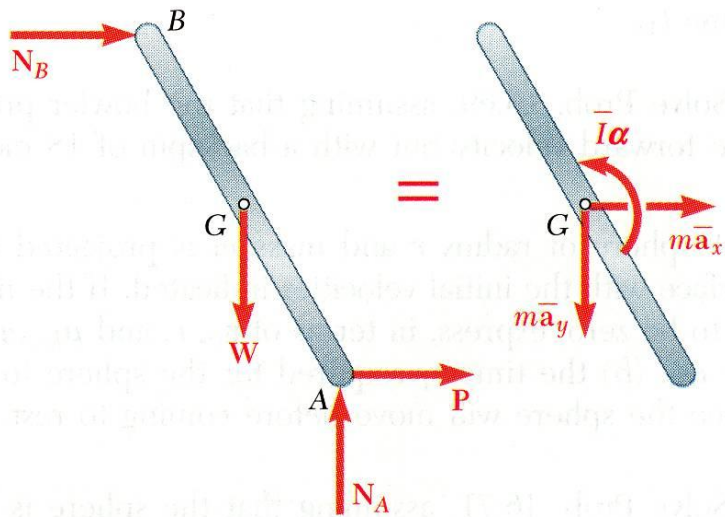
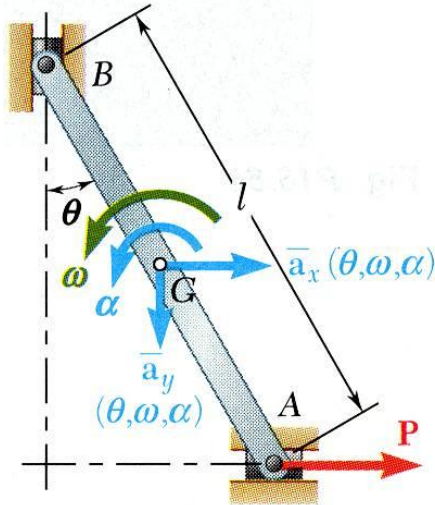


Kinetics of Rigid Bodies

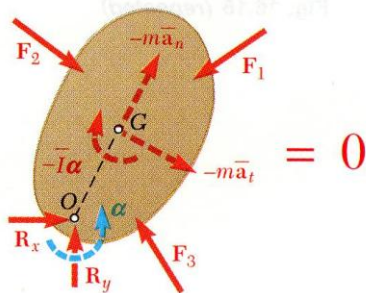
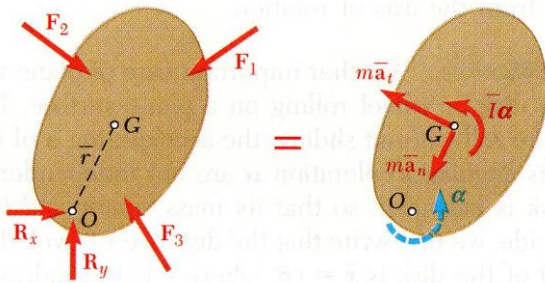
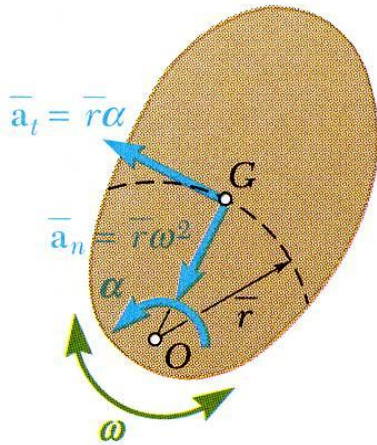
Preview of 16.7- 16.9

16.7 Constrained Plane Motion



- Most engineering applications involve rigid bodies which are moving under given constraints, e.g., cranks, connecting rods, and non-slipping wheels.
- *Constrained plane motion*: motions with definite relations between the components of acceleration of the mass center and the angular acceleration of the body.
- Solution of a problem involving constrained plane motion begins with a kinematic analysis.
- e.g., given θ , ω , and α , find P , N_A , and N_B .
 - kinematic analysis yields \bar{a}_x and \bar{a}_y .
 - application of d'Alembert's principle yields P , N_A , and N_B .

16.8 Constrained Motion: Noncentroidal Rotation



- *Noncentroidal rotation*: motion of a body is constrained to rotate about a fixed axis that does not pass through its mass center.
- Kinematic relation between the motion of the mass center G and the motion of the body about G ,

$$\bar{a}_t = r\alpha \quad \bar{a}_n = r\omega^2$$

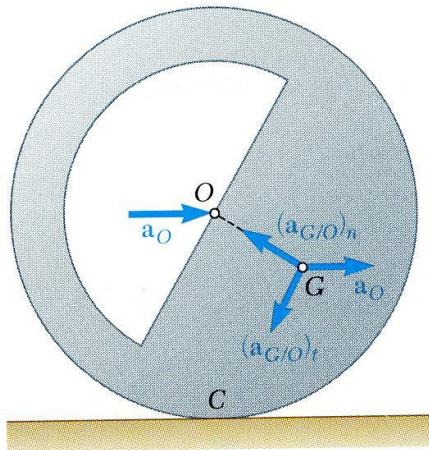
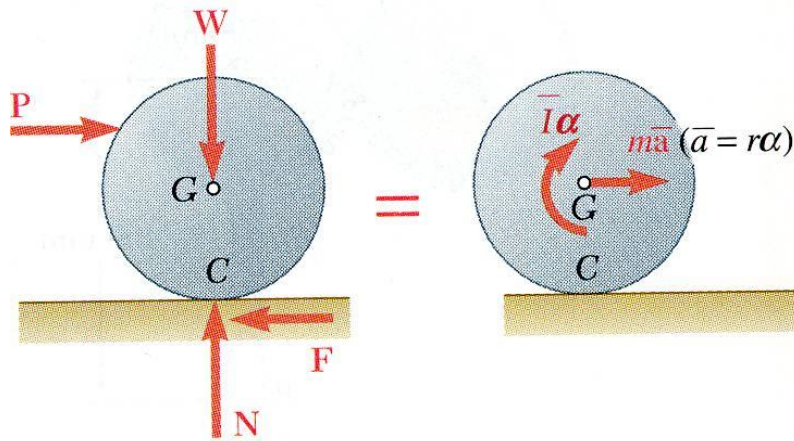
- The kinematic relations are used to eliminate \bar{a}_t and \bar{a}_n from equations derived from d'Alembert's principle or from the method of dynamic equilibrium.

- Equate the moment about O :

- Motion with constant angular speed?

Kinetics of Rigid Bodies

16.8 Constrained Plane Motion: Rolling Motion



- For a balanced disk constrained to roll without sliding,
 $\bar{x} = r\theta \rightarrow \bar{a} = r\alpha$

- Rolling, no sliding:

$$F \leq \mu_s N \quad \bar{a} = r\alpha$$

Rolling, sliding impending:

$$F = \mu_s N \quad \bar{a} = r\alpha$$

Rotating and sliding:

$$F = \mu_k N \quad \bar{a}, r\alpha \text{ independent}$$

- For the geometric center of an unbalanced disk,

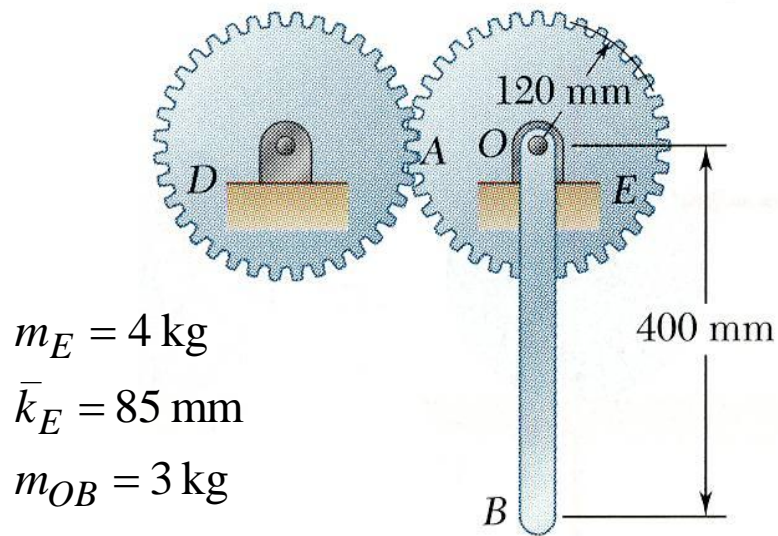
$$a_O = r\alpha$$

The acceleration of the mass center,

$$\begin{aligned} \vec{a}_G &= \vec{a}_O + \boxed{\phantom{\vec{a}_G}} \\ &= \vec{a}_O + \boxed{\phantom{\vec{a}_G}} \end{aligned}$$

Kinetics of Rigid Bodies

Sample Problem 16.6



The portion AOB of the mechanism is actuated by gear D and at the instant shown has a clockwise angular velocity of 8 rad/s and a counterclockwise angular acceleration of 40 rad/s^2 .

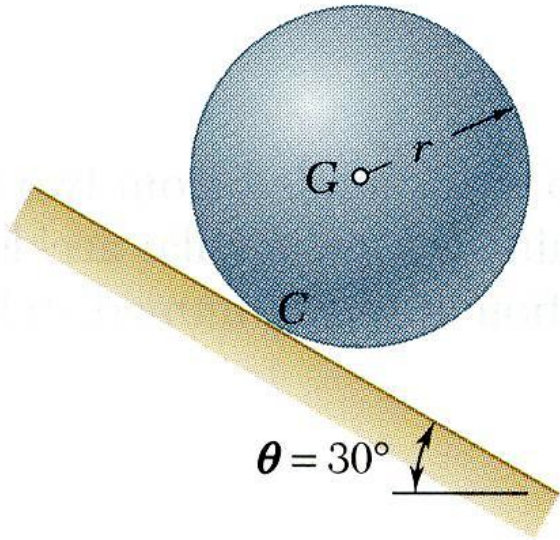
Determine: a) tangential force exerted by gear D , and b) components of the reaction at shaft O .

Kinetics of Rigid Bodies

Sample Problem 16.6

Kinetics of Rigid Bodies

Sample Problem 16.8



A sphere of weight W is released with no initial velocity and rolls without slipping on the incline.

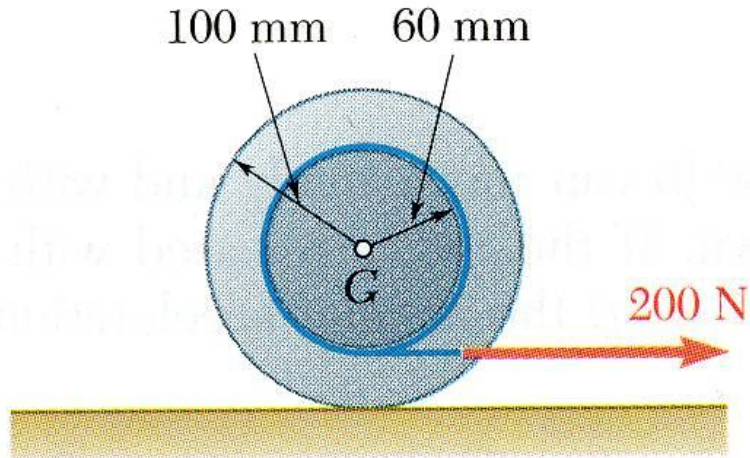
Determine: *a*) the minimum value of the coefficient of friction, *b*) the velocity of G after the sphere has rolled 10 m and *c*) the velocity of G if the sphere were to move 10 m down a frictionless incline.

Kinetics of Rigid Bodies

Sample Problem 16.8

Kinetics of Rigid Bodies

Sample Problem 16.9



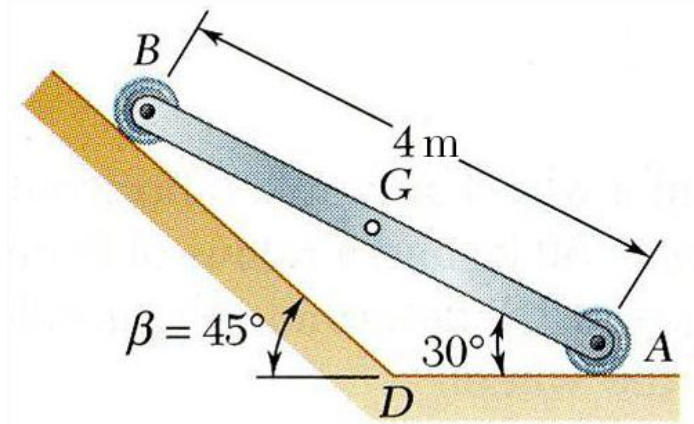
A cord is wrapped around the inner hub of a wheel and pulled horizontally with a force of 200 N. The wheel has a mass of 50 kg and a radius of gyration of 70 mm. Knowing $\mu_s = 0.20$ and $\mu_k = 0.15$, determine the acceleration of G and the angular acceleration of the wheel.

Kinetics of Rigid Bodies

Sample Problem 16.9

Kinetics of Rigid Bodies

Sample Problem 16.10



The extremities of a 4-m rod weighing 50 N can move freely and with no friction along two straight tracks. The rod is released with no velocity from the position shown.

Determine: *a*) the angular acceleration of the rod, and *b*) the reactions at A and B.

Kinetics of Rigid Bodies

Sample Problem 16.10