

## **2. Stress and infinitesimal strain**

## 2.1 Problem definition

- Shallow depth – Block movement in jointed rock mass under low-stress environment
- Deep mining – Stress is main concern for the mine stability.  
Rock mass behaves as continuum.
- Force
  - Body force: Gravity, magnetic, inertia ( $F/V$ )
  - Surface force: Conveyed by physical contact ( $F/A$ )

## 2.2 Force and stress

- Force is a vector with magnitude and direction.  
Ex.)  $F_x, F_y, F_z, F_1, F_2, \dots$
- Stress is a tensor with magnitude, direction and surface orientation.

Ex.)  $\sigma_{xx} (= \sigma_x), \sigma_{xy} (= \sigma_{xy}), \sigma_{yz}, \dots$

surface orientation (normal to the surface)

stress direction

## 2.2 Force and stress

- Traction is a force per unit area acting on a surface (Fig.2.1(b)).
- “Tensor is a further extension of ideas we already use when defining quantities like scalars and vectors.”
  - Ex.) Scalar → rank zero
  - Vector → rank one
  - Stress → rank two
  - :

## 2.2 Force and stress

- Tensor notation
  - Introduced in deriving the theory of relativity by Einstein.
  - Basic idea is to make summation for the repeatedly used subscripts.

$$\text{Ex.) } a_i b_i = a_x b_x + a_y b_y + a_z b_z$$

$a_{ii} b_k = a_{xx} b_k + a_{yy} b_k + a_{zz} b_k$  ( $i$ : dummy index,  $k$ : free index)

$$a_{ik} b_i c_k = a_{xk} b_x c_k + a_{yk} b_y c_k + a_{zk} b_z c_k$$

$$= a_{xx} b_x c_x + a_{xy} b_x c_y + a_{xz} b_x c_z$$

$$+ a_{yx} b_y c_x + a_{yy} b_y c_y + a_{yz} b_y c_z$$

$$+ a_{zx} b_z c_x + a_{zy} b_z c_y + a_{zz} b_z c_z$$

## 2.2 Force and stress

- Important symbols in tensor notation

- Kronecker delta

$$\begin{aligned}\delta_{ij} &= 1 \text{ if } i=j. \\ &= 0 \text{ if } i \neq j.\end{aligned}$$

Ex.)  $\delta_{xx} = 1, \quad \delta_{zz} = 1, \quad \delta_{xy} = 0, \quad \delta_{yz} = 0$

$$\sum_{ij} a_i b_j = a_x b_x + a_y b_y + a_z b_z = a_i b_i$$

- Permutation symbol

$$\begin{aligned}\epsilon_{ijk} &= 0 \text{ if any pair of subscripts are identical.} \\ &= (-1)^p, \text{ where } p \text{ is the number of subscript transposition.}\end{aligned}$$

Ex.)  $\epsilon_{xyz} = 1, \quad \epsilon_{xyx} = 0, \quad \epsilon_{yyz} = 0, \quad \epsilon_{xzz} = 0$   
 $\epsilon_{xzy} = -1, \quad \epsilon_{zxy} = 1, \quad \epsilon_{yxz} = -1, \quad \epsilon_{zyx} = -1$

## 2.2 Force and stress

- Linear algebra in tensor notation

- Matrix operation

$$Ax = b \rightarrow a_{ij}x_j = b_i$$

$$AB = Y \rightarrow a_{ij}b_{jk} = Y_{ik}$$

$$A^T B = Z \rightarrow a_{ji}b_{jk} = Z_{ik}$$

$$x^T B x = c \rightarrow x_i b_{ij} x_j = c$$

- Derivatives

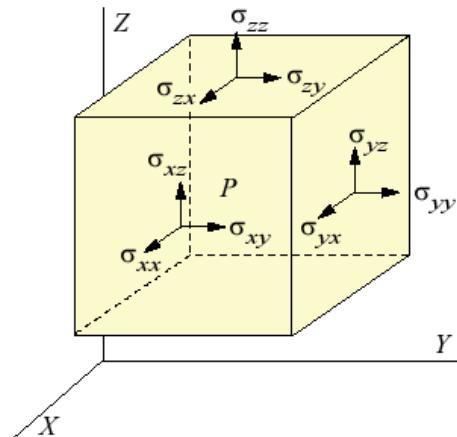
$$Grad \mathbf{f} = \nabla \mathbf{f} = \frac{\partial \mathbf{f}}{\partial x} \hat{e}_x + \frac{\partial \mathbf{f}}{\partial y} \hat{e}_y + \frac{\partial \mathbf{f}}{\partial z} \hat{e}_z = \mathbf{f}_{,i}$$

$$Div \vec{V} = \nabla \cdot \vec{V} = \frac{\partial V_x}{\partial x} + \frac{\partial V_y}{\partial y} + \frac{\partial V_z}{\partial z} = V_{i,i}$$

## 2.2 Force and stress

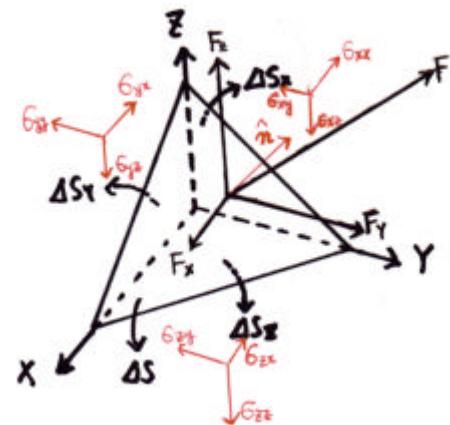
- Stress in rectangular coordinates

$$[\boldsymbol{S}] = \begin{bmatrix} S_{xx} & S_{xy} & S_{xz} \\ S_{yx} & S_{yy} & S_{yz} \\ S_{zx} & S_{zy} & S_{zz} \end{bmatrix} = \begin{bmatrix} S_{xx} & S_{xy} & S_{xz} \\ S_{xy} & S_{yy} & S_{yz} \\ S_{xz} & S_{yz} & S_{zz} \end{bmatrix}$$



- Stress in an arbitrary plane

$$\hat{n} = \left( \frac{\Delta S_x}{\Delta S}, \frac{\Delta S_y}{\Delta S}, \frac{\Delta S_z}{\Delta S} \right)$$



## 2.2 Force and stress

$$\vec{F} = \vec{F}_x + \vec{F}_y + \vec{F}_z$$

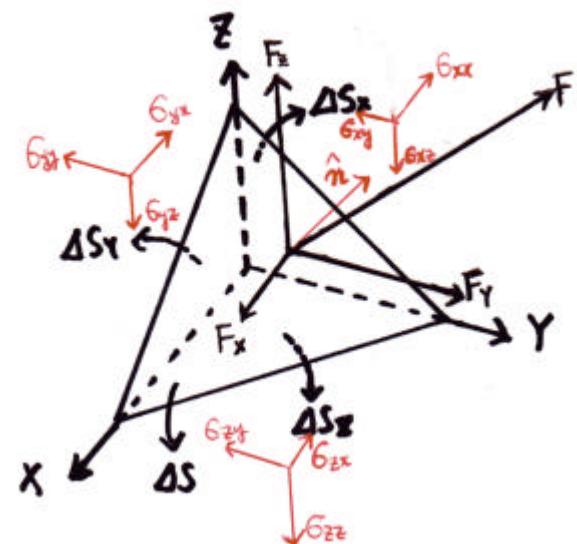
$$\Delta S t^{\hat{n}} = \Delta S t_x^{\hat{n}} + \Delta S t_y^{\hat{n}} + \Delta S t_z^{\hat{n}}$$

$$\Delta S t_x^{\hat{n}} = \Delta S_x \mathbf{S}_{xx} + \Delta S_y \mathbf{S}_{yx} + \Delta S_z \mathbf{S}_{zx}$$

$$\Delta S t_y^{\hat{n}} = \Delta S_x \mathbf{S}_{xy} + \Delta S_y \mathbf{S}_{yy} + \Delta S_z \mathbf{S}_{zy}$$

$$\Delta S t_z^{\hat{n}} = \Delta S_x \mathbf{S}_{xz} + \Delta S_y \mathbf{S}_{yz} + \Delta S_z \mathbf{S}_{zz}$$

$$t_i^{\hat{n}} = n_j \mathbf{S}_{ji} \left( = \mathbf{S}_{ij} n_j \right)$$



## 2.2 Force and stress

- Equilibrium conditions
  - Force equilibrium & moment equilibrium

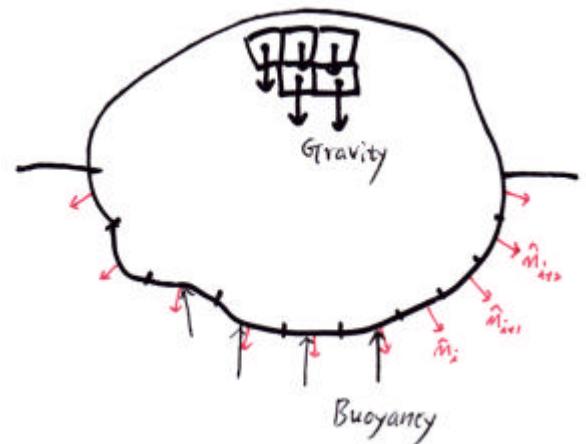
$$\sum \vec{F} = 0, \quad \sum \vec{M} = 0$$

- Force equilibrium

$$\int_s \hat{t}_i^n ds + \int_v \mathbf{r} b_i dv = 0$$

$$\int_s n_j \mathbf{s}_{ji} ds = \int_v \mathbf{s}_{ji,j} dv = 0 \quad (\text{Divergence theorem})$$

$$\rightarrow \mathbf{s}_{ji,j} + \mathbf{r} b_i = 0$$



## 2.2 Force and stress

- Moment equilibrium

$$\vec{M} = \vec{r} \times \vec{F} \left( = \epsilon_{ijk} r_j F_k \right)$$

$$\int_s \vec{r} \times t \hat{n} ds + \int_v \vec{r} \times \vec{r} \vec{b} dv = 0$$

$$\rightarrow \epsilon_{ijk} S_{jk} = 0$$

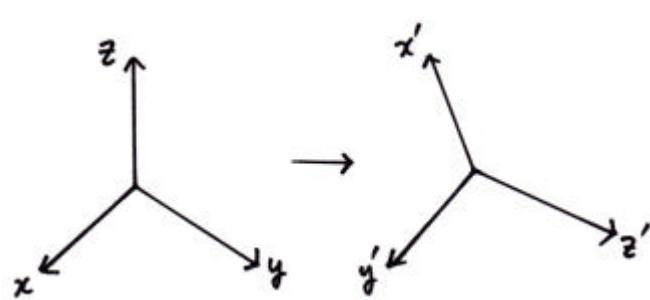
$$i = x : \quad S_{yz} - S_{zy} = 0$$

$$i = y : \quad S_{zx} - S_{xz} = 0 \quad \rightarrow \quad S_{ij} = S_{ji}$$

$$i = z : \quad S_{xy} - S_{yx} = 0$$

## 2.3 Stress transformation

- Vector transformation



The diagram illustrates a coordinate transformation. On the left, a fixed coordinate system is shown with axes labeled  $x$ ,  $y$ , and  $z$ . An arrow points to the right, indicating a rotation to a new coordinate system. The new system has axes labeled  $x'$ ,  $y'$ , and  $z'$ , where  $x'$  is parallel to the original  $x$ -axis.

$x'$	$x$	$y$	$z$
$y'$	$\cos q_{x'x}$	$\cos q_{x'y}$	$\cos q_{x'z}$
$z'$	$\cos q_{y'x}$	$\cos q_{y'y}$	$\cos q_{y'z}$
	$\cos q_{z'x}$	$\cos q_{z'y}$	$\cos q_{z'z}$

→  $\begin{bmatrix} a_{11} & a_{12} & a_{13} \\ a_{21} & a_{22} & a_{23} \\ a_{31} & a_{32} & a_{33} \end{bmatrix}$

$$\begin{bmatrix} x' \\ y' \\ z' \end{bmatrix} = \begin{bmatrix} a_{11} & a_{12} & a_{13} \\ a_{21} & a_{22} & a_{23} \\ a_{31} & a_{32} & a_{33} \end{bmatrix} \begin{bmatrix} x \\ y \\ z \end{bmatrix}$$

## 2.3 Stress transformation

- Stress transformation

$$u_i' = a_{ip} u_p, \quad v_j' = a_{jq} v_q \quad (\text{vector transformation})$$

$$u_i' v_j' = \mathbf{S}_{ij}' \quad (\text{outer product})$$

$$u_i' v_j' = a_{ip} u_p a_{jq} v_q = a_{ip} a_{jq} u_p v_q = a_{ip} a_{jq} \mathbf{S}_{pq} = a_{ip} \mathbf{S}_{pq} a_{jq}$$

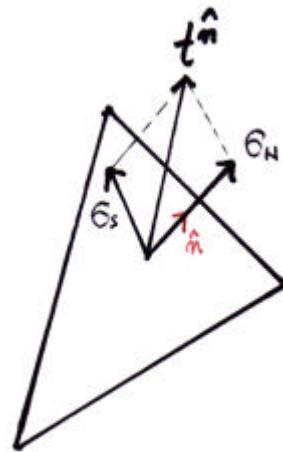
$$\rightarrow \mathbf{S}_{ij}' = a_{ip} \mathbf{S}_{pq} a_{jq} \quad (\Sigma' = A \cdot \Sigma \cdot A^T)$$

## 2.4 Principal stress and stress invariants

- Principal stress
  - Traction in a surface can be decomposed into normal and shear stresses which are in the same plane with the traction.

$$\mathbf{S}_N = \hat{n} \cdot t^{\hat{n}}$$

$$\mathbf{S}_S = t^{\hat{n}} - \mathbf{S}_N$$



## 2.4 Principal stress and stress invariants

- The surface on which shear component of the traction is zero is a **principal plane** and its normal component is called **principal stress**.

$$\begin{aligned} t^{\hat{n}} &= \Sigma \cdot \hat{n} = \mathbf{s} \cdot \hat{n} \\ \mathbf{s}_{ij} n_j - \mathbf{s} n_i &= 0 \\ (\mathbf{s}_{ij} - \mathbf{s} \delta_{ij}) n_j &= 0 \end{aligned} \quad \left[ \begin{array}{ccc} \mathbf{s}_{xx} - \mathbf{s} & \mathbf{s}_{xy} & \mathbf{s}_{xz} \\ \mathbf{s}_{xy} & \mathbf{s}_{yy} - \mathbf{s} & \mathbf{s}_{yz} \\ \mathbf{s}_{xz} & \mathbf{s}_{yz} & \mathbf{s}_{zz} - \mathbf{s} \end{array} \right] \begin{bmatrix} x \\ y \\ z \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}$$

When  $\mathbf{s}_1 \geq \mathbf{s}_2 \geq \mathbf{s}_3$

$\mathbf{s}_1$  : major principal stress

$\mathbf{s}_2$  : intermediate principal stress

$\mathbf{s}_3$  : minor principal stress

## 2.4 Principal stress and stress invariants

- Stress invariants

$$\begin{vmatrix} \mathbf{s}_{xx} - \mathbf{s} & \mathbf{s}_{xy} & \mathbf{s}_{xz} \\ \mathbf{s}_{xy} & \mathbf{s}_{yy} - \mathbf{s} & \mathbf{s}_{yz} \\ \mathbf{s}_{xz} & \mathbf{s}_{yz} & \mathbf{s}_{zz} - \mathbf{s} \end{vmatrix} = 0 \quad \text{for } \hat{\mathbf{n}} \neq (0,0,0)$$

$$\rightarrow \mathbf{s}^3 - I_1 \mathbf{s}^2 + I_2 \mathbf{s} - I_3 = 0$$

$$I_1 = \mathbf{s}_{ii} \left(= \mathbf{s}_{xx} + \mathbf{s}_{yy} + \mathbf{s}_{zz}\right)$$

$$I_2 = \frac{1}{2} \left[ \mathbf{s}_{ii} \mathbf{s}_{jj} - \mathbf{s}_{ij} \mathbf{s}_{ij} \right] \left(= \mathbf{s}_{xx} \mathbf{s}_{yy} + \mathbf{s}_{xx} \mathbf{s}_{zz} + \mathbf{s}_{yy} \mathbf{s}_{zz} - \mathbf{s}_{xy} \mathbf{s}_{xy} - \mathbf{s}_{xz} \mathbf{s}_{xz} - \mathbf{s}_{yz} \mathbf{s}_{yz} \right)$$

$$I_3 = |\mathbf{s}_{ij}|$$

**CUBIC EQUATION:**  $x^3 + a_1x^2 + a_2x + a_3 = 0$

Let

$$Q = \frac{3a_2 - a_1^2}{9}, \quad R = \frac{9a_1a_2 - 27a_3 - 2a_1^3}{54},$$

$$S = \sqrt[3]{R + \sqrt{Q^3 + R^2}}, \quad T = \sqrt[3]{R - \sqrt{Q^3 + R^2}}$$

**Solutions:**

$$\begin{cases} x_1 = S + T - \frac{1}{3}a_1 \\ x_2 = -\frac{1}{2}(S + T) - \frac{1}{3}a_1 + \frac{1}{2}i\sqrt{3}(S - T) \\ x_3 = -\frac{1}{2}(S + T) - \frac{1}{3}a_1 - \frac{1}{2}i\sqrt{3}(S - T) \end{cases}$$

If  $a_1, a_2, a_3$  are real and if  $D = Q^3 + R^2$  is the *discriminant*, then

- (i) one root is real and two complex conjugate if  $D > 0$
- (ii) all roots are real and at least two are equal if  $D = 0$
- (iii) all roots are real and unequal if  $D < 0$ .

If  $D < 0$ , computation is simplified by use of trigonometry.

**Solutions if  $D < 0$ :**

$$\begin{cases} x_1 = 2\sqrt{-Q} \cos\left(\frac{1}{3}\theta\right) - \frac{1}{3}a_1 \\ x_2 = 2\sqrt{-Q} \cos\left(\frac{1}{3}\theta + 120^\circ\right) - \frac{1}{3}a_1 \\ x_3 = 2\sqrt{-Q} \cos\left(\frac{1}{3}\theta + 240^\circ\right) - \frac{1}{3}a_1 \end{cases} \quad \text{where } \cos \theta = R/\sqrt{-Q^3}$$

$$x_1 + x_2 + x_3 = -a_1, \quad x_1x_2 + x_2x_3 + x_3x_1 = a_2, \quad x_1x_2x_3 = -a_3$$

where  $x_1, x_2, x_3$  are the three roots.

## 2.5 Differential equations of static equilibrium

Refer to section 2.2

$$\mathbf{s}_{ji,j} + \mathbf{r}b_i = 0$$

$$\frac{\partial \mathbf{s}_{xx}}{\partial x} + \frac{\partial \mathbf{s}_{xy}}{\partial y} + \frac{\partial \mathbf{s}_{xz}}{\partial z} + \mathbf{r}b_x = 0$$

$$\frac{\partial \mathbf{s}_{yx}}{\partial x} + \frac{\partial \mathbf{s}_{yy}}{\partial y} + \frac{\partial \mathbf{s}_{yz}}{\partial z} + \mathbf{r}b_y = 0$$

$$\frac{\partial \mathbf{s}_{zx}}{\partial x} + \frac{\partial \mathbf{s}_{zy}}{\partial y} + \frac{\partial \mathbf{s}_{zz}}{\partial z} + \mathbf{r}b_z = 0$$

## **2.6 Plane problems and biaxial stress**

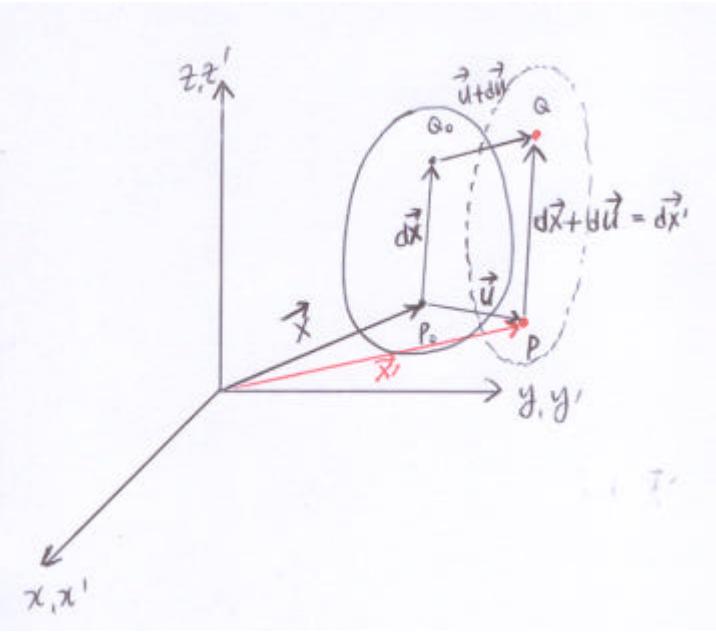
Apply the principles in 3D to 2D problems

# Problem

- Calculate traction, normal stress and shear stress in a surface whose normal vector is as follow ( $\Sigma$  is stress on the object).

$$\hat{n} = \left( \frac{2}{3}, -\frac{2}{3}, \frac{1}{3} \right) \quad \Sigma = \begin{bmatrix} 7 & 0 & -2 \\ 0 & 5 & 0 \\ -2 & 0 & 4 \end{bmatrix}$$

# 2.7 Displacement and strain



$$\vec{X} = (x, y, z) \quad \vec{X}' = (x', y', z')$$

$$\vec{u} = \vec{X}' - \vec{X} \quad d\vec{X}' = d\vec{X} + d\vec{u}$$

$$(d\vec{X}')^2 = d\vec{X}' \cdot d\vec{X}' \rightarrow d\vec{X}'_k d\vec{X}'_k$$

$$d\vec{X}'_k = \frac{\partial \vec{X}'_k}{\partial \vec{X}_j} d\vec{X}_j$$

$$(d\vec{X}')^2 - (d\vec{X})^2 = \frac{\partial \vec{X}'_k}{\partial \vec{X}_i} \frac{\partial \vec{X}'_k}{\partial \vec{X}_j} d\vec{X}_i d\vec{X}_j - \mathbf{d}_{ij} d\vec{X}_i d\vec{X}_j$$

$$= \left( \frac{\partial \vec{X}'_k}{\partial \vec{X}_i} \frac{\partial \vec{X}'_k}{\partial \vec{X}_j} - \mathbf{d}_{ij} \right) d\vec{X}_i d\vec{X}_j$$

## 2.7 Displacement and strain

$$L_{ij} = \frac{1}{2} \left( \frac{\partial X_k^{'}}{\partial X_i} \frac{\partial X_k^{'}}{\partial X_j} - d_{ij} \right)$$

$$X_k^{'} = u_k + X_k$$

$$\begin{aligned} L_{ij} &= \frac{1}{2} \left( \left( \frac{\partial u_k}{\partial X_i} + d_{ki} \right) \left( \frac{\partial u_k}{\partial X_j} + d_{kj} \right) - d_{ij} \right) \\ &= \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} + \frac{\partial u_k}{\partial X_i} \frac{\partial u_k}{\partial X_j} + d_{ij} - d_{ij} \right) \\ &\approx \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right) \text{ for infinitesimal displacement} \end{aligned}$$

$$\therefore (dX^{'})^2 - (dX)^2 = 2L_{ij} dX_i dX_j$$

## 2.7 Displacement and strain

$$L_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right)$$

$$\begin{bmatrix} L_{ij} \end{bmatrix} = \begin{bmatrix} \frac{\partial u_x}{\partial X_x} & \frac{1}{2} \left( \frac{\partial u_x}{\partial X_y} + \frac{\partial u_y}{\partial X_x} \right) & \frac{1}{2} \left( \frac{\partial u_x}{\partial X_z} + \frac{\partial u_z}{\partial X_x} \right) \\ \frac{1}{2} \left( \frac{\partial u_y}{\partial X_x} + \frac{\partial u_x}{\partial X_y} \right) & \frac{\partial u_y}{\partial X_y} & \frac{1}{2} \left( \frac{\partial u_y}{\partial X_z} + \frac{\partial u_z}{\partial X_y} \right) \\ \frac{1}{2} \left( \frac{\partial u_z}{\partial X_x} + \frac{\partial u_x}{\partial X_z} \right) & \frac{1}{2} \left( \frac{\partial u_z}{\partial X_y} + \frac{\partial u_y}{\partial X_z} \right) & \frac{\partial u_z}{\partial X_z} \end{bmatrix}$$

.....Linear (Infinitesimal) strain tensor

# 2.7 Displacement and strain

- Normal strain

$$\begin{aligned}(dX')^2 - (dX)^2 &= (dX' - dX)(dX' + dX) \\ &\approx 2(dX' - dX)dX = 2L_{ij}dX_i dX_j\end{aligned}$$

$$dX' - dX = \frac{L_{ij}dX_i dX_j}{dX}$$

$$\frac{dX' - dX}{dX} = \frac{dX_i}{dX} L_{ij} \frac{dX_j}{dX} = \hat{n} \cdot L \cdot \hat{n} \quad (\text{normal strain})$$

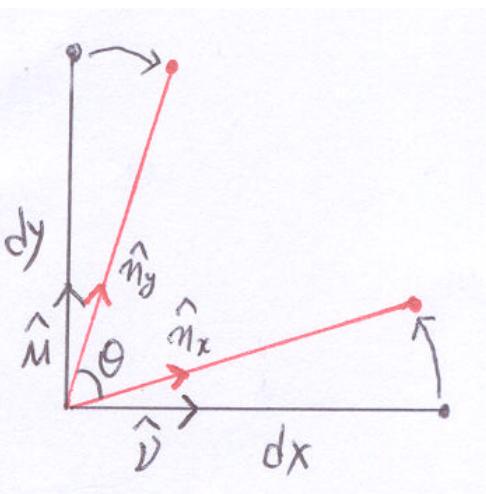
$$\hat{e}_x \cdot L \cdot \hat{e}_x = L_{xx}$$

$$\hat{e}_y \cdot L \cdot \hat{e}_y = L_{yy}$$

$$\hat{e}_z \cdot L \cdot \hat{e}_z = L_{zz}$$

# 2.7 Displacement and strain

- Shear strain



$$\hat{n}_x \cdot \hat{n}_y = \cos q = \sin\left(\frac{p}{2} - q\right) = \sin g \approx g \quad (\text{shear strain})$$

$$L_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right) = \frac{1}{2} (J + J_c)$$

$$d\vec{u} = \frac{\partial \vec{u}_i}{\partial X_j} dX_j = J \cdot d\vec{X}$$

$$\hat{n}_x \approx \hat{u} + J \cdot \hat{u}, \quad \hat{n}_y \approx \hat{m} + J \cdot \hat{m}$$

$$\begin{aligned} \mathbf{g} &= \hat{n}_x \cdot \hat{n}_y = (\hat{u} + J \cdot \hat{u}) \cdot (\hat{m} + J \cdot \hat{m}) \\ &= \hat{u} \cdot \hat{m} + \hat{u} \cdot (J + J_c) \cdot \hat{m} + \hat{u} \cdot J_c \cdot J \cdot \hat{m} \\ &= \hat{u} \cdot 2L \cdot \hat{m} \end{aligned}$$

$$\hat{e}_x \cdot L \cdot \hat{e}_y = L_{xy} = \frac{1}{2} \mathbf{g}_{xy} \quad \hat{e}_y \cdot L \cdot \hat{e}_z = L_{yz} = \frac{1}{2} \mathbf{g}_{yz}$$

$$\hat{e}_z \cdot L \cdot \hat{e}_x = L_{zx} = \frac{1}{2} \mathbf{g}_{zx}$$

## 2.7 Displacement and strain

$$L_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right)$$

$$\begin{bmatrix} L_{ij} \end{bmatrix} = \begin{bmatrix} \mathbf{e}_x & \frac{1}{2}\mathbf{g}_{xy} & \frac{1}{2}\mathbf{g}_{xz} \\ \frac{1}{2}\mathbf{g}_{xy} & \mathbf{e}_y & \frac{1}{2}\mathbf{g}_{yz} \\ \frac{1}{2}\mathbf{g}_{xz} & \frac{1}{2}\mathbf{g}_{yz} & \mathbf{e}_z \end{bmatrix}$$

# 2.7 Displacement and strain

- Decomposition of relative displacement ( $du$ )

$$\begin{aligned} du_i &= \frac{\partial u_i}{\partial X_j} dX_j \\ &= \left[ \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right) + \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} - \frac{\partial u_j}{\partial X_i} \right) \right] dX_j \\ &= [L_{ij} + \Omega_{ij}] dX_j \end{aligned}$$

$$[\Omega_{ij}] = \begin{bmatrix} 0 & \mathbf{w}_{xy} & \mathbf{w}_{xz} \\ -\mathbf{w}_{xy} & 0 & \mathbf{w}_{yz} \\ -\mathbf{w}_{xz} & -\mathbf{w}_{yz} & 0 \end{bmatrix} \quad \dots \text{ rotation tensor}$$

$$= \begin{bmatrix} 0 & -\Omega_z & \Omega_y \\ \Omega_z & 0 & -\Omega_x \\ -\Omega_y & \Omega_x & 0 \end{bmatrix} \left( \vec{\Omega} = \frac{1}{2} \vec{\nabla} \times \vec{u} = \frac{1}{2} \begin{bmatrix} \hat{e}_x & \hat{e}_y & \hat{e}_z \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ u_x & u_y & u_z \end{bmatrix} \right)$$

## 2.8 Principal strains, strain transformation, volumetric strain and deviator strain

Principal strains: Eigen values/vectors of strain tensor

Strain transformation: analogous to the stress transformation

$$\begin{aligned}\text{Volumetric strain} &= \frac{V - V_0}{V_0} = \frac{(1 + \mathbf{e}_1)(1 + \mathbf{e}_2)(1 + \mathbf{e}_3)dX_1 dX_2 dX_3}{dX_1 dX_2 dX_3} - 1 \\ &= \mathbf{e}_1 + \mathbf{e}_2 + \mathbf{e}_3 + \mathbf{e}_1 \mathbf{e}_2 + \mathbf{e}_2 \mathbf{e}_3 + \mathbf{e}_3 \mathbf{e}_1 + \mathbf{e}_1 \mathbf{e}_2 \mathbf{e}_3 \\ &\approx \mathbf{e}_1 + \mathbf{e}_2 + \mathbf{e}_3\end{aligned}$$

$$\text{Deviator strain} = \begin{bmatrix} \mathbf{e}_x - \Delta/3 & \mathbf{g}_{xy} & \mathbf{g}_{xz} \\ \mathbf{g}_{xy} & \mathbf{e}_y - \Delta/3 & \mathbf{g}_{yz} \\ \mathbf{g}_{xz} & \mathbf{g}_{yz} & \mathbf{e}_z - \Delta/3 \end{bmatrix} \quad \Delta : \text{volumetric strain}$$

# 2.9 Strain compatibility equations

Independent strain components: 6 ( $\epsilon_x, \epsilon_y, \epsilon_z, g_{xy}, g_{yz}, g_{zx}$ )

Independent displacement components : 3 ( $u_x, u_y, u_z$ )

Random  $u_x, u_y, u_z \rightarrow \epsilon_x, \epsilon_y, \epsilon_z, g_{xy}, g_{yz}, g_{zx}$  (o)

Random  $\epsilon_x, \epsilon_y, \epsilon_z, g_{xy}, g_{yz}, g_{zx} \rightarrow u_x, u_y, u_z$  (x)

Finding restricting conditions:

$$\epsilon_{xx} = \frac{\partial u_x}{\partial X_x}, \quad \epsilon_{yy} = \frac{\partial u_y}{\partial X_y}, \quad \epsilon_{xy} = \frac{1}{2} \left( \frac{\partial u_x}{\partial X_y} + \frac{\partial u_y}{\partial X_x} \right)$$

$$\frac{\partial^2 \epsilon_{xx}}{\partial X_y^2} = \frac{\partial^3 u_x}{\partial X_x \partial X_y^2}, \quad \frac{\partial^2 \epsilon_{yy}}{\partial X_x^2} = \frac{\partial^3 u_y}{\partial X_x^2 \partial X_y}, \quad \frac{\partial^2 \epsilon_{xy}}{\partial X_x \partial X_y} = \frac{1}{2} \left( \frac{\partial^3 u_x}{\partial X_x \partial X_y^2} + \frac{\partial^3 u_y}{\partial X_x^2 \partial X_y} \right)$$

$$\therefore \frac{\partial^2 \epsilon_{xx}}{\partial X_y^2} + \frac{\partial^2 \epsilon_{yy}}{\partial X_x^2} = 2 \frac{\partial^2 \epsilon_{xy}}{\partial X_x \partial X_y}$$

## 2.9 Strain compatibility equations

$$\frac{\partial^2 \mathbf{e}_{xx}}{\partial X_y^2} + \frac{\partial^2 \mathbf{e}_{yy}}{\partial X_x^2} = 2 \frac{\partial^2 \mathbf{e}_{xy}}{\partial X_x \partial X_y}$$

$$\frac{\partial^2 \mathbf{e}_{yy}}{\partial X_z^2} + \frac{\partial^2 \mathbf{e}_{zz}}{\partial X_x^2} = 2 \frac{\partial^2 \mathbf{e}_{yz}}{\partial X_y \partial X_z}$$

$$\frac{\partial^2 \mathbf{e}_{zz}}{\partial X_x^2} + \frac{\partial^2 \mathbf{e}_{xx}}{\partial X_z^2} = 2 \frac{\partial^2 \mathbf{e}_{zx}}{\partial X_x \partial X_z}$$

$$\frac{\partial}{\partial X_x} \left( -\frac{\partial \mathbf{e}_{yz}}{\partial X_x} + \frac{\partial \mathbf{e}_{zx}}{\partial X_y} + \frac{\partial \mathbf{e}_{xy}}{\partial X_z} \right) = \frac{\partial^2 \mathbf{e}_{xx}}{\partial X_y \partial X_z}$$

$$\frac{\partial}{\partial X_y} \left( \frac{\partial \mathbf{e}_{yz}}{\partial X_x} - \frac{\partial \mathbf{e}_{zx}}{\partial X_y} + \frac{\partial \mathbf{e}_{xy}}{\partial X_z} \right) = \frac{\partial^2 \mathbf{e}_{yy}}{\partial X_z \partial X_x}$$

$$\frac{\partial}{\partial X_z} \left( \frac{\partial \mathbf{e}_{yz}}{\partial X_x} + \frac{\partial \mathbf{e}_{zx}}{\partial X_y} - \frac{\partial \mathbf{e}_{xy}}{\partial X_z} \right) = \frac{\partial^2 \mathbf{e}_{zz}}{\partial X_x \partial X_y}$$

# 2.10 Stress-strain relations

$$\boldsymbol{e}_{xx} = \frac{1}{E} [\boldsymbol{s}_{xx} - \mathbf{n} (\boldsymbol{s}_{yy} + \boldsymbol{s}_{zz})], \quad \boldsymbol{e}_{yy} = \frac{1}{E} [\boldsymbol{s}_{yy} - \mathbf{n} (\boldsymbol{s}_{xx} + \boldsymbol{s}_{zz})], \quad \boldsymbol{e}_{zz} = \frac{1}{E} [\boldsymbol{s}_{zz} - \mathbf{n} (\boldsymbol{s}_{xx} + \boldsymbol{s}_{yy})]$$

$$\boldsymbol{g}_{xy} = \frac{1}{G} \boldsymbol{t}_{xy}, \quad \boldsymbol{g}_{yz} = \frac{1}{G} \boldsymbol{t}_{yz}, \quad \boldsymbol{g}_{zx} = \frac{1}{G} \boldsymbol{t}_{zx} \quad \left( G = \frac{E}{2(1+\mathbf{n})} \right)$$

$$\begin{bmatrix} \boldsymbol{e}_{xx} \\ \boldsymbol{e}_{yy} \\ \boldsymbol{e}_{zz} \\ \boldsymbol{g}_{xy} \\ \boldsymbol{g}_{yz} \\ \boldsymbol{g}_{zx} \end{bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\mathbf{n} & -\mathbf{n} & 0 & 0 & 0 \\ -\mathbf{n} & 1 & -\mathbf{n} & 0 & 0 & 0 \\ -\mathbf{n} & -\mathbf{n} & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 2(1+\mathbf{n}) & 0 & 0 \\ 0 & 0 & 0 & 0 & 2(1+\mathbf{n}) & 0 \\ 0 & 0 & 0 & 0 & 0 & 2(1+\mathbf{n}) \end{bmatrix} \begin{bmatrix} \boldsymbol{s}_{xx} \\ \boldsymbol{s}_{yy} \\ \boldsymbol{s}_{zz} \\ \boldsymbol{t}_{xy} \\ \boldsymbol{t}_{yz} \\ \boldsymbol{t}_{zx} \end{bmatrix}$$

$$\begin{bmatrix} \boldsymbol{s}_{xx} \\ \boldsymbol{s}_{yy} \\ \boldsymbol{s}_{zz} \\ \boldsymbol{t}_{xy} \\ \boldsymbol{t}_{yz} \\ \boldsymbol{t}_{zx} \end{bmatrix} = \frac{E(1-\mathbf{n})}{(1+\mathbf{n})(1-2\mathbf{n})} \begin{bmatrix} 1 & \mathbf{n}/(1-\mathbf{n}) & \mathbf{n}/(1-\mathbf{n}) & 0 & 0 & 0 \\ \mathbf{n}/(1-\mathbf{n}) & 1 & \mathbf{n}/(1-\mathbf{n}) & 0 & 0 & 0 \\ \mathbf{n}/(1-\mathbf{n}) & \mathbf{n}/(1-\mathbf{n}) & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{(1-2\mathbf{n})}{2(1-\mathbf{n})} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{(1-2\mathbf{n})}{2(1-\mathbf{n})} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{(1-2\mathbf{n})}{2(1-\mathbf{n})} \end{bmatrix} \begin{bmatrix} \boldsymbol{e}_{xx} \\ \boldsymbol{e}_{yy} \\ \boldsymbol{e}_{zz} \\ \boldsymbol{g}_{xy} \\ \boldsymbol{g}_{yz} \\ \boldsymbol{g}_{zx} \end{bmatrix}$$

## **2.11 Cylindrical polar co-ordinates**

Refer to p.37, textbook

## **2.12 Geomechanics conventions for displacement, strain and stress**

Compressive/contractile force/displacement are positive.  
Refer to Figure 2.12.