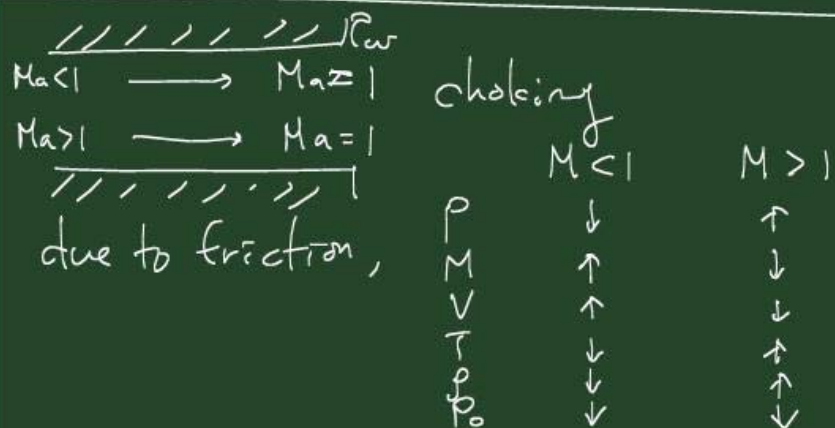
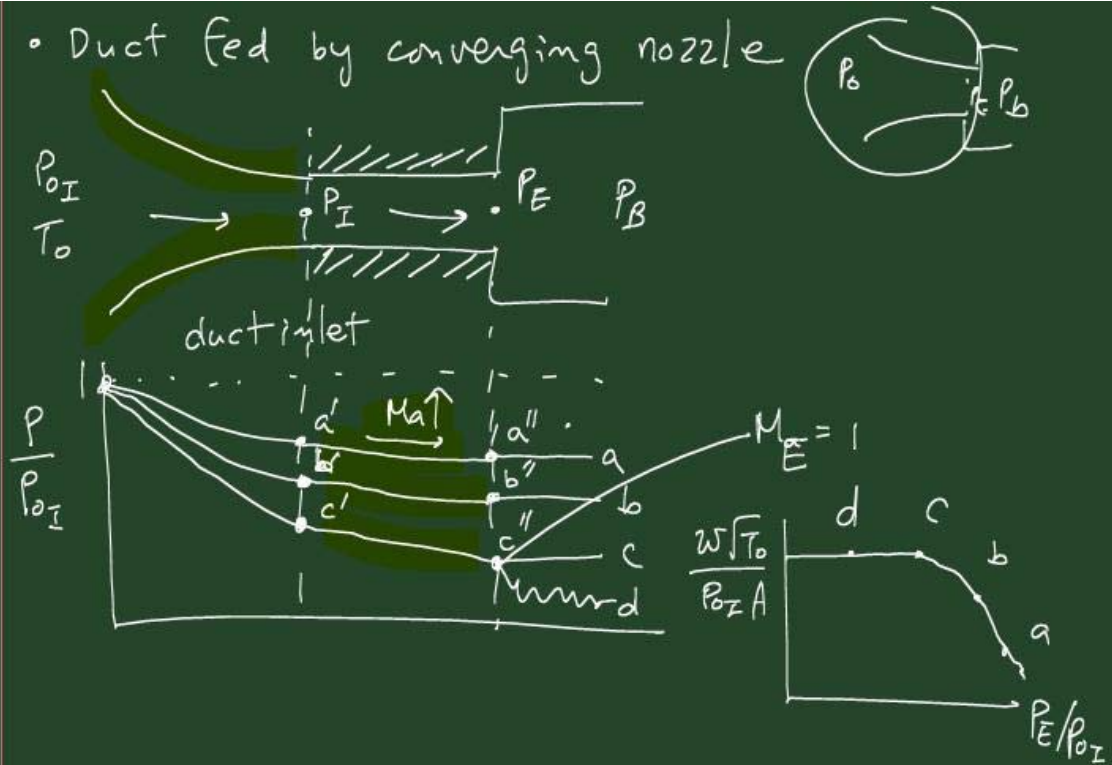
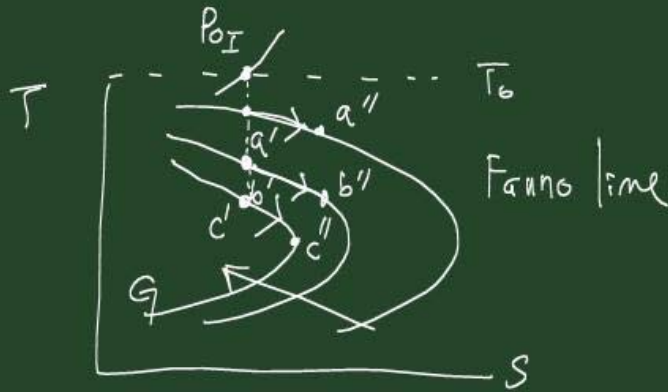


- There was an explanation on the panel method to get the potential-flow solution.

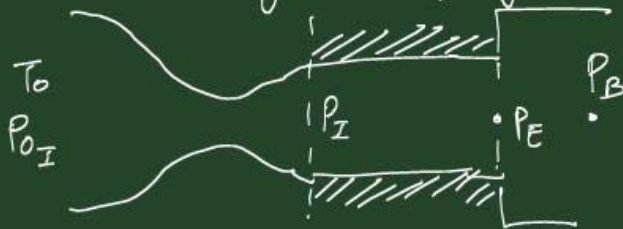


- Duct fed by converging nozzle





duct fed by converging - diverging nozzle



more complex than the converging - nozzle case.

2. Isothermal flow in long ducts

energy eq. $dQ = c_p dT + d\left(\frac{V^2}{2}\right) = c_p dT_0$

isothermal $x T, T_0$ $T_0 = T \left(1 + \frac{k-1}{2} M^2\right)$
 Take diff'l & $dT = 0$

$$\rightarrow \frac{dT_0}{T_0} = \frac{(k-1)M^2}{2\left(1 + \frac{k-1}{2}M^2\right)} \frac{dM^2}{M^2}$$

Eq. of state $p = \rho R T \rightarrow \frac{dp}{p} = \frac{d\rho}{\rho}$ for $dT = 0$

Mach number $M^2 = \frac{V^2}{kRT} \rightarrow \frac{dM^2}{M^2} = 2 \frac{dV}{V}$

$$\rightarrow \frac{dp}{p} = \frac{d\rho}{\rho} = -\frac{dV}{V} = -\frac{1}{2} \frac{dM^2}{M^2} = -\frac{kM^2}{2(1-kM^2)} \cdot 4f \frac{dx}{D}$$

$f = \frac{\tau_w}{\frac{1}{2}\rho V^2}$ and D : hydraulic diameter

$$\frac{dP_0}{P_0} = \frac{kM^2(1-\frac{k+1}{2}M^2)}{2(kM^2-1)(1+\frac{k-1}{2}M^2)} \cdot 4f \frac{dx}{D} \Rightarrow M = \sqrt{\frac{1}{k}}$$

$$\frac{dT_0}{T_0} = \frac{k(k-1)M^4}{2(1-kM^2)(1+\frac{k-1}{2}M^2)} \cdot 4f \frac{dx}{D}$$

$M < 1/\sqrt{k}$ (subsonic) $M > 1/\sqrt{k}$ (subsonic / supersonic)

P	↓	↑
ρ	↓	↑
V	↑	↓
M	↑	↓
T_0	↑	↓
P_0	↓	↑

$\Rightarrow M \rightarrow 1/\sqrt{k}$
 heat added to fluid (for $M < 1/\sqrt{k}$)
 heat rejected from the flow (for $M > 1/\sqrt{k}$)
 for $M < \sqrt{2/(k+1)}$
 $M > "$

① Flow in ducts with heating or cooling