

# System Control

## 3. Transfer Function and State Equation

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# Transfer Functions

- Transfer Functions (Linear Time Invariant Systems)

-The ratio of the Laplace Transform of the output (response function) to the Laplace Transform of the input (driving function) under the assumption that all initial conditions are zero.

$$\frac{Y(s)}{U(s)} = G(s)$$

$U(s) \longrightarrow \boxed{G(s)} \longrightarrow Y(s)$

- Differential Equation

$$a_0 y^{(n)} + a_1 y^{(n-1)} + \dots + a_{n-1} y' + a_n y = b_0 x^m + b_1 x^{m-1} + \dots + b_{m-1} x' + b_m x$$

$$\text{T. F. } G(s) = \frac{Y(s)}{X(s)} = \frac{b_0 s^m + b_1 s^{m-1} + \dots + b_{m-1} s + b_m}{a_0 s^n + a_1 s^{n-1} + \dots + a_{n-1} s + a_n} \quad n \geq m$$

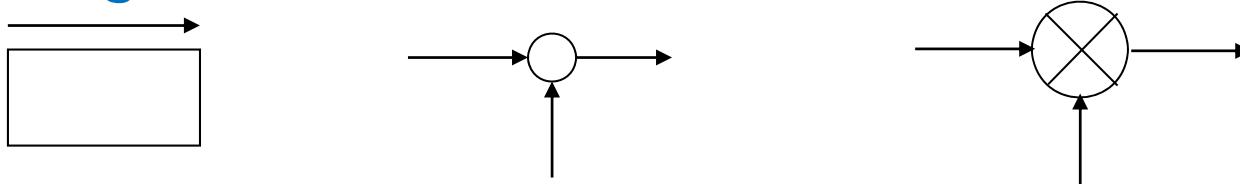
- T.F.

- ① A mathematical model
- ② A property of a system itself independent of the magnitude and nature of the input
- ③ T. F. includes the units( input-output relations) however, does not provide any information concerning the physical structure of the system, many different systems can have identical T. F.

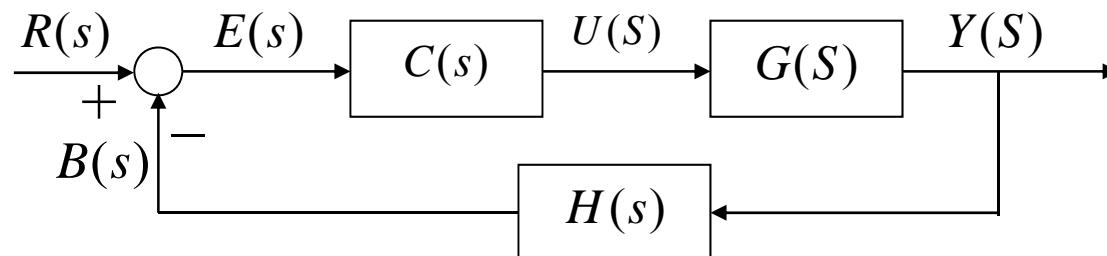


# Feedback Control System

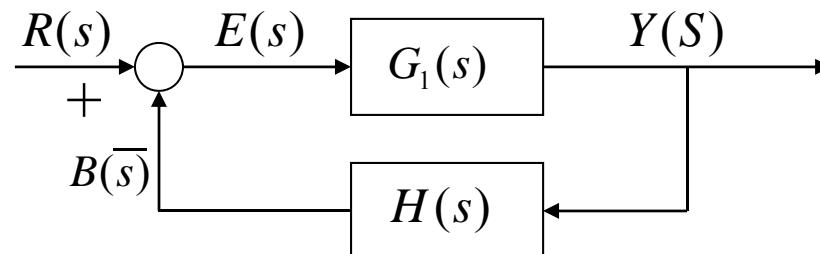
- **Block Diagram**



A block diagram (of a system) : a pictorial representation of the function performed by each component and of the flow of signals.



or



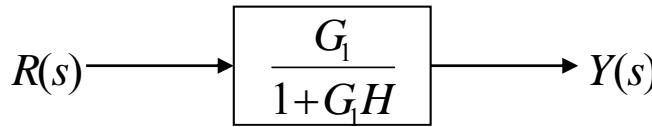
$$G_1(S) = C(s) \cdot G(s)$$

# Transfer Functions

- **Open loop T.F.**  $= \frac{B(s)}{E(s)} = \frac{H(s) \cdot Y(S)}{E(s)} = \frac{H(s)G_1(s)E(s)}{E(S)}$   
 $= H(s) \cdot G_1(s)$

- **Feed forward T.F.**  $= \frac{Y(s)}{E(s)} = G_1(s)$

- **Closed loop T.F.**  $= \frac{Y(s)}{R(s)}$



$$\begin{aligned} Y(s) &= G_1(s)E(s) = G_1(s)[R(s) - B(s)] \\ &= G_1(s)[R(s) - H(s) \cdot Y(s)] \end{aligned}$$

$$[1 + G_1(s)H(s)]Y(s) = G_1(s)R(s)$$

$$\frac{Y(s)}{R(s)} = \frac{G_1(s)}{1 + G_1(s)H(s)}$$

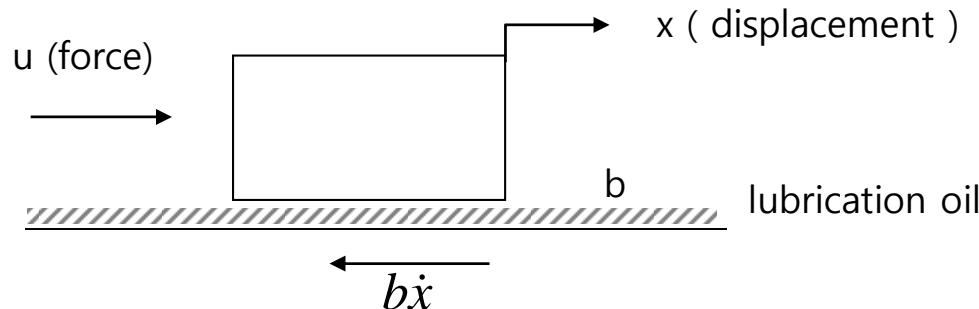


$$\frac{G_{\text{feed forward}}}{1 + G_{\text{open}}(s)}$$



# Transfer Functions

Ex)



$$m\ddot{x} = u - b\dot{x}$$

$$m\ddot{x} + b\dot{x} = u$$

$$x(0) = 0$$

$$\dot{x}(0) = 0$$

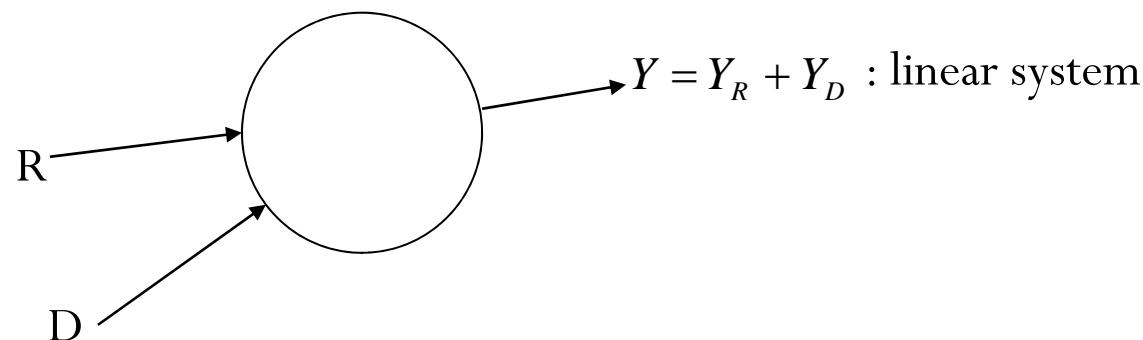
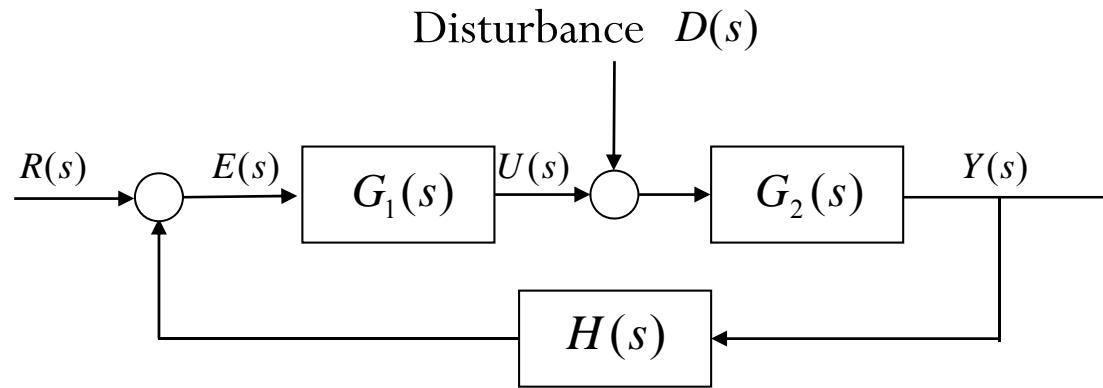
$$ms^2 X(s) + bsX(s) = U(s)$$

$$(ms^2 + bs)X(s) = U(s)$$

$$\frac{X(s)}{U(s)} = \frac{1}{s(ms + b)} \quad : \text{Transfer Function}$$



# Closed-loop system subjected to a disturbance



$$\begin{aligned} Y_D(s) &= G_2(s)[D(s) + U(s)] \\ &= G_2(s)D(s) + G_2(s)G_1(s)E(s) \\ &= G_2(s)D(s) + G_2(s)G_1(s)[-H(s)Y_D(s)] \end{aligned}$$



# Closed-loop system subjected to a disturbance

$$R=0 : \quad \frac{Y_D(s)}{D(s)} = \frac{G_2}{1+G_1G_2H} = G_D(s)$$

$$D=0 : \quad \frac{Y_R(s)}{R(s)} = \frac{G_1G_2}{1+G_1G_2H} = G_R(s)$$

$$\begin{aligned} Y(s) &= \frac{G_1G_2}{1+G_1G_2H} R + \frac{G_2}{1+G_1G_2H} D \\ &= \frac{G_2}{1+G_1G_2H} [G_1R + D] \end{aligned}$$

- $G_1G_2H \gg 1$

$$G_D(s) \approx \frac{G_2}{G_1G_2H} = \frac{1}{G_1H}$$

$$G_1H \gg 1 \quad G_D = \varepsilon \ll 1$$

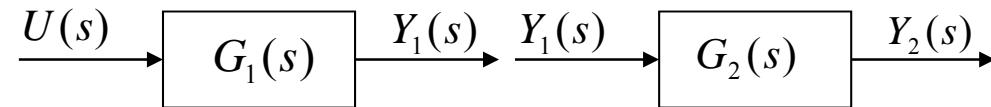
The effect of the disturbance is reduced → Advantage of the closed-loop system

$$G_R(s) \approx \frac{G_1G_2}{G_1G_2H} = \frac{1}{H}$$



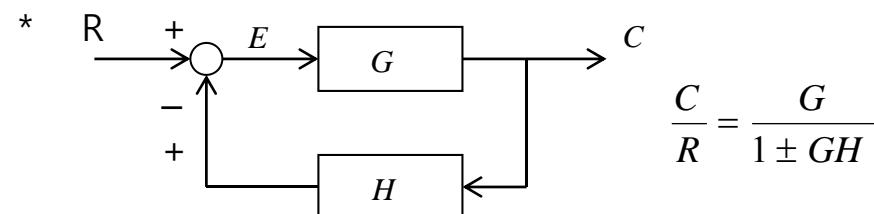
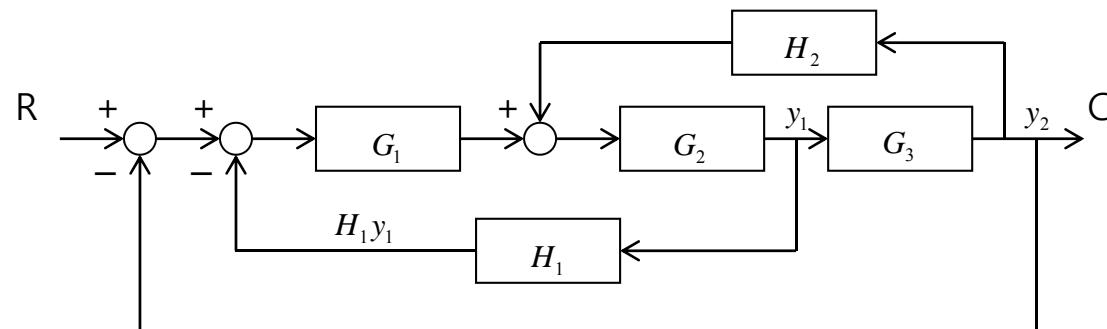
# Block Diagram Reduction

ex1)



$$Y_2(s) = G_2(s)Y_1(s) = G_2(s)G_1(s)U(s)$$

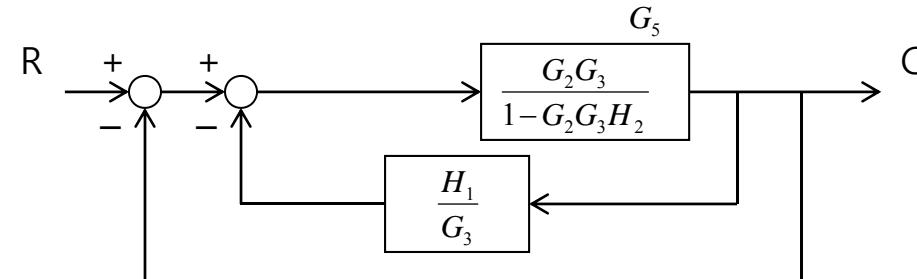
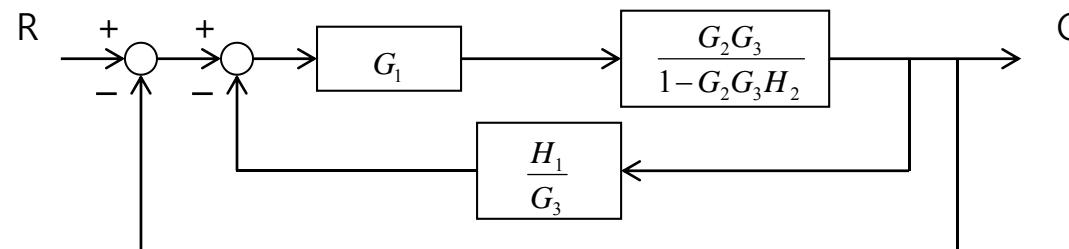
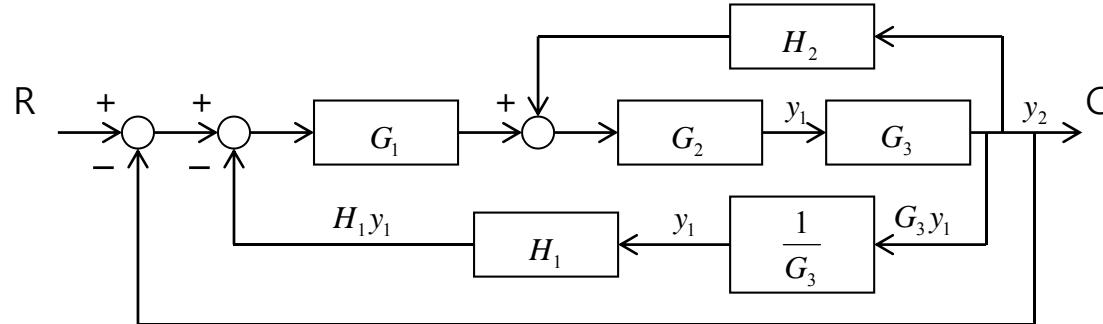
ex2)



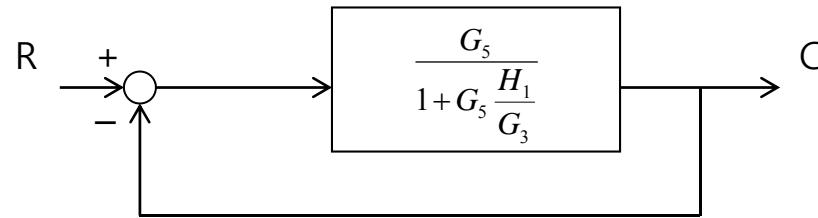
$$\frac{C}{R} = \frac{G}{1 \pm GH}$$



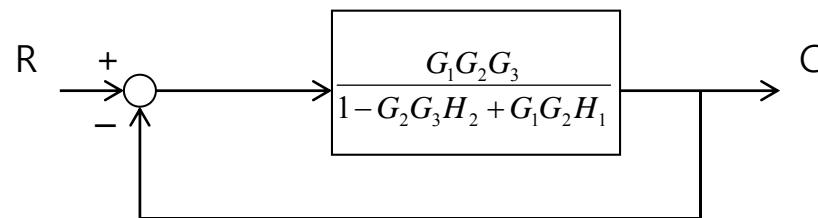
# Block Diagram Reduction



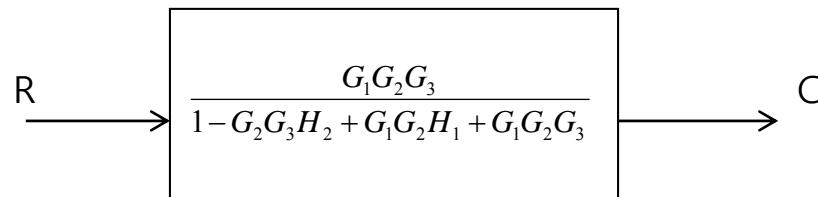
# Block Diagram Reduction



$$\frac{\frac{G_1 G_2 G_3}{1 - G_2 G_3 H_2}}{1 + \frac{G_1 G_2 G_3}{1 - G_2 G_3 H_2} \frac{H_1}{G_3}} = \frac{G_1 G_2 G_3}{1 - G_2 G_3 H_2 + G_1 G_2 H_1}$$



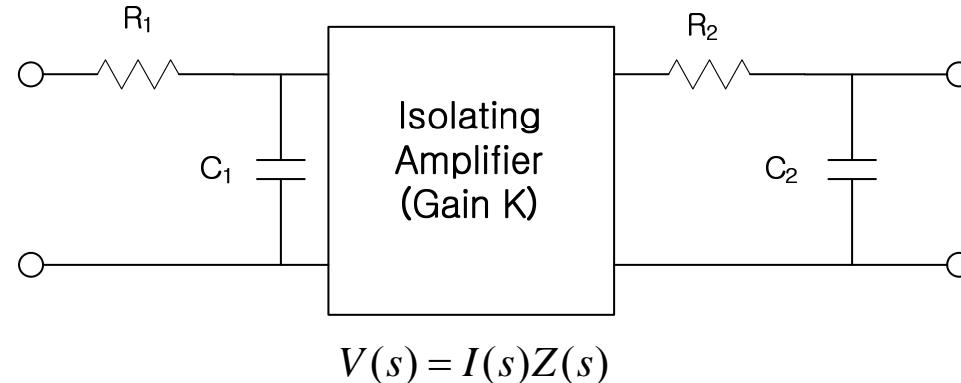
$$\frac{\frac{G_1 G_2 G_3}{1 - G_2 G_3 H_2 + G_1 G_2 H_1}}{1 + \frac{G_1 G_2 G_3}{1 - G_2 G_3 H_2 + G_1 G_2 H_1}} = \frac{G_1 G_2 G_3}{1 - G_2 G_3 H_2 + G_1 G_2 H_1 + G_1 G_2 G_3}$$



# No Loading Effect

- No Loading Effect

Block can be connected in series only if the output of one block is not affected by the next following block.



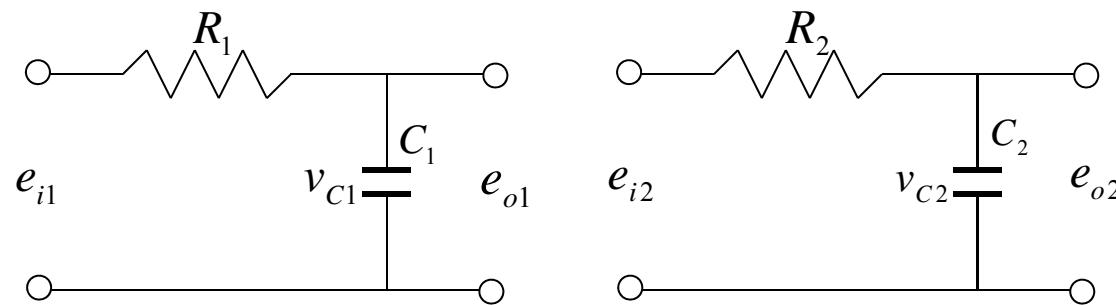
$Z(s)$  : complex impedance

If the input impedance of the second element is infinite, the output of the first element is not affected by connecting it to the second element.



# No Loading Effect

ex1)



$$e_{i1} = R_1 i_1 + e_{o1}$$

$$\frac{dv_{C1}}{dt} = \frac{1}{C_1} i_1, \quad v_{C1} = e_{o1}$$

$$\Rightarrow \frac{de_{o1}}{dt} = \frac{1}{C_1} \left[ \frac{1}{R_1} (e_{i1} - e_{o1}) \right] = -\frac{1}{C_1 R_1} e_{o1} + \frac{1}{C_1 R_1} e_{i1}$$

$$\frac{E_{o1}(s)}{E_{i1}(s)} = \frac{\frac{1}{C_1 R_1}}{S + \frac{1}{C_1 R_1}} \rightarrow a_1$$

$$\frac{E_{o2}(s)}{E_{i2}(s)} = \frac{\frac{1}{C_2 R_2}}{S + \frac{1}{C_2 R_2}} \rightarrow a_2$$

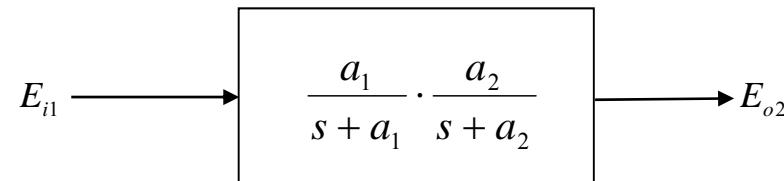
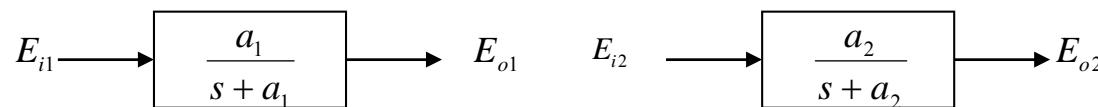


# No Loading Effect

ex1)

$$\frac{E_{o1}(s)}{E_{i1}(s)} = \frac{\frac{1}{C_1 R_1}}{S + \frac{1}{C_1 R_1}}$$

$$\frac{E_{o2}(s)}{E_{i2}(s)} = \frac{\frac{1}{C_2 R_2}}{S + \frac{1}{C_2 R_2}}$$



No !!  
Incorrect

P.90.Ogata

$$\frac{E_o(s)}{E_i(s)} = \frac{1}{R_1 C_1 R_2 C_2 s^2 + (R_1 C_1 + R_2 C_2 + R_1 C_2) s + 1}$$



# Mason's Gain Formula

The overall gain

$$P = \frac{1}{\Delta} \sum_k P_k \Delta_k$$

$P_k$  = path gain of k-th forward path

$\Delta$  = determinant

$$= 1 - \sum_a L_a - \sum_{b,c} L_b L_c - \sum_{d,e,f} L_d L_e L_f + \dots$$

$\sum_a L_a$  = sum of all individual loop gains

$\sum_{b,c} L_b L_c$  = sum of gain products of all possible combination of two non touching loops

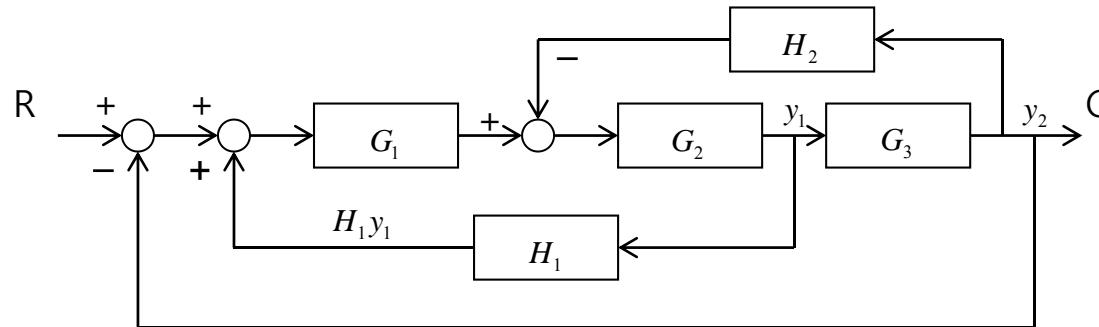
$\sum_{d,e,f} L_d L_e L_f$  = sum of gain products of all possible combination of three non touching loops

$\Delta_k$  = cofactors of the k-th forward path determinant of the graph with the loops touching the k-th forward path removed, that is, the cofactor  $\Delta_k$  is obtained from  $\Delta$  by removing the loops that touch path  $P_k$



# Mason's Gain Formula

Ex 3-13) Ogata



① **One Forward path :**  $P_1 = G_1G_2G_3$

② **Three Individual Loops :**  $L_1 = G_1G_2H_1$

$$L_2 = -G_2G_3H_2$$

$$L_3 = -G_1G_2G_3$$

③ **No Non-touching Loops :**  $\Delta = 1 - (L_1 + L_2 + L_3)$

$$= 1 - G_1G_2H_1 + G_2G_3H_2 + G_1G_2G_3$$

④  $\Delta_1$  :  $P_1$  touches all loops

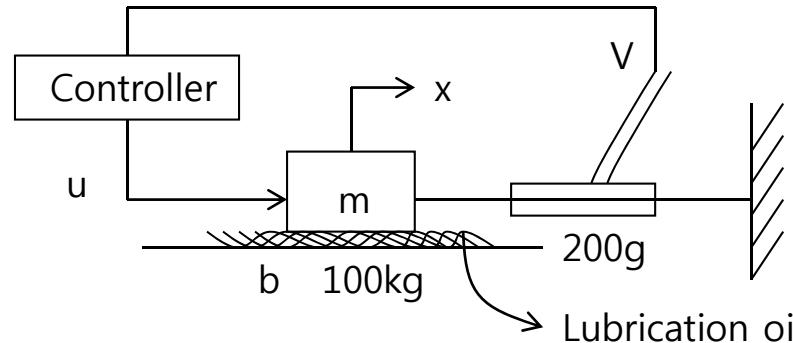
$$\Delta_1 = 1$$

⑤  $\frac{C(s)}{R(s)} = \frac{1}{\Delta} (P_1 \Delta_1)$

$$= \frac{G_1G_2G_3}{1 - G_1G_2H_1 + G_2G_3H_2 + G_1G_2G_3}$$

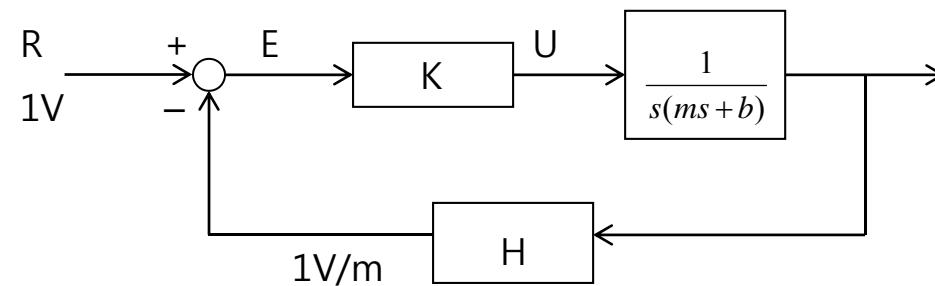


# State Equation



$$m\ddot{x} = u - b\dot{x}$$

$$(ms^2 + bs)X(s) = U(s)$$



# State Equation

- $m\ddot{x} + b\dot{x} = u$

$$\begin{aligned}x_1 &= x & \dot{x}_1 &= \dot{x} = x_2 \\x_2 &= \dot{x} & \dot{x}_2 &= \ddot{x} = -\frac{b}{m}\dot{x} + \frac{u}{m} = -\frac{b}{m}x_2 + \frac{1}{m}u\end{aligned}$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ 0 & -\frac{b}{m} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{m} \end{bmatrix} u$$

$$y = x_1 = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

$$x = \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} \quad \begin{cases} \dot{x} = Ax + Bu \\ y = Cx + Du \end{cases}$$

First order matrix differential Eq.  
→ State Equation



# State Equation

- **State**  $x$

The smallest set of variables such that knowledge of these variables at  $t = t_0$ , together with the knowledge of the input for  $t \geq t_0$ , completely determines the behavior of the system at any time  $t \geq t_0$

- **State Variables**

The variables making up the smallest set of variables that determine the state of the dynamic system

ex)  $x_1$  : displacement       $x_2$  : velocity

- **State Vector**

$x_1, x_2, x_3 \dots$  state variables

$$x = \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{bmatrix}$$

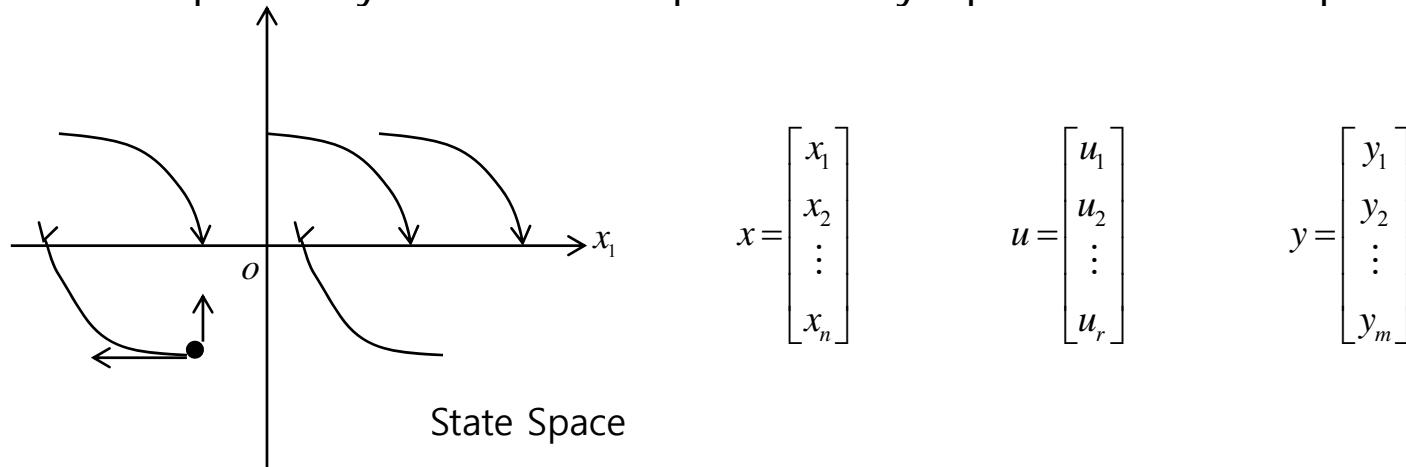
A vector that determines uniquely the system state  $x(t)$  for any time once the state at  $t = t_0$  is given and the input  $u(t)$  for  $t \geq t_0$  is specified



# State Equation

- **State Space**

The n-dimensional space, whose coordinate axes consist of the  $x_1$  axis,  $x_2$  axis, ...,  $x_n$  axis is called a state space. Any state can be represented by a point in the state space



$$x = \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{bmatrix} \quad u = \begin{bmatrix} u_1 \\ u_2 \\ \vdots \\ u_r \end{bmatrix} \quad y = \begin{bmatrix} y_1 \\ y_2 \\ \vdots \\ y_m \end{bmatrix}$$

$$\begin{cases} \dot{x}_1 = f_1(x_1, x_2, \dots, x_n; u_1, u_2, \dots, u_r; t) \\ \dot{x}_2 = f_2(x_1, x_2, \dots, x_n; u_1, u_2, \dots, u_r; t) \\ \vdots \\ \dot{x}_n = f_n(x_1, x_2, \dots, x_n; u_1, u_2, \dots, u_r; t) \end{cases}$$

$$\begin{cases} y_1 = g_1(x_1, x_2, \dots, x_n; u_1, u_2, \dots, u_r; t) \\ \vdots \\ y_m = g_m(x_1, x_2, \dots, x_n; u_1, u_2, \dots, u_r; t) \end{cases}$$

$$\begin{aligned} \dot{x} &= f(x, u, t) \\ y &= g(x, u, t) \end{aligned}$$

$$f = \begin{bmatrix} f_1 \\ f_2 \\ \vdots \\ f_n \end{bmatrix} \quad g = \begin{bmatrix} g_1 \\ g_2 \\ \vdots \\ g_m \end{bmatrix}$$



# Linear Systems

- **Linear Systems**

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx + Du\end{aligned}$$

- **Linearization (of nonlinear system)**

$$\dot{x} = f(x, u, t) = f(x, u)$$

$$x = x_0, u = u_0, \dot{x} = \dot{x}_0 = f_0 = 0$$

$$x = x_0 + \Delta x, u = u_0 + \Delta u$$

$$\dot{x} = f(x_0, u_0) + \left. \frac{\partial f}{\partial x} \right|_{x_0 u_0} \Delta x + \left. \frac{\partial f}{\partial u} \right|_{x_0 u_0} \Delta u + \left. \frac{\partial^2 f}{\partial x^2} \right|_{x_0 u_0} \Delta x^2 + \left. \frac{\partial^2 f}{\partial u^2} \right|_{x_0 u_0} \Delta u^2 + \dots$$

$$\cong f_0 + K_1 \Delta x + K_2 \Delta u$$

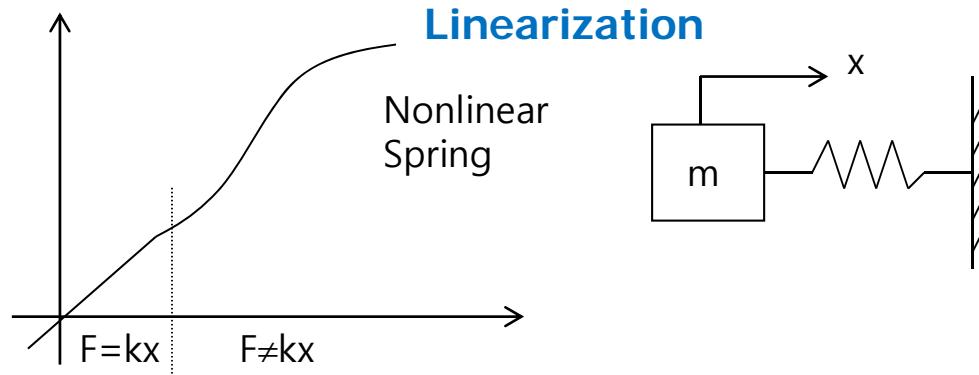
$$\dot{x} - \dot{x}_0 = K_1 \Delta x + K_2 \Delta u \quad \Delta x, \Delta u : small$$

$$\Delta x = K_1 \Delta x + K_2 \Delta u \quad Approximation$$



# Linear Systems

- Nonlinear Systems  $\longrightarrow$  Linear systems



- State : mathematical concept, not physical meaning

$$\begin{cases} x_1 = x \\ x_2 = \dot{x} \end{cases}$$

$$\dot{x} = \begin{bmatrix} 0 & 1 \\ 0 & -\frac{b}{m} \end{bmatrix} x + \begin{bmatrix} 0 \\ \frac{1}{m} \end{bmatrix} u$$

$$y = [1 \quad 0]x$$

$$\begin{aligned}\hat{x} &= \begin{bmatrix} 1 & 1 \\ 0 & 1 \end{bmatrix} x = \begin{bmatrix} x_1 + x_2 \\ x_2 \end{bmatrix} = \begin{bmatrix} x + \dot{x} \\ \dot{x} \end{bmatrix} = Tx \\ x &= T^{-1}\hat{x}\end{aligned}$$

$$T^{-1}\dot{\hat{x}} = AT^{-1}\hat{x} + BU$$

$$\begin{cases} \dot{\hat{x}} = TAT^{-1}\hat{x} + TBU \\ y = CT^{-1}\hat{x} \end{cases}$$

$$\begin{cases} \dot{\hat{x}} = \hat{A}\hat{x} + \hat{B}U \\ y = \hat{C}\hat{x} \end{cases}$$



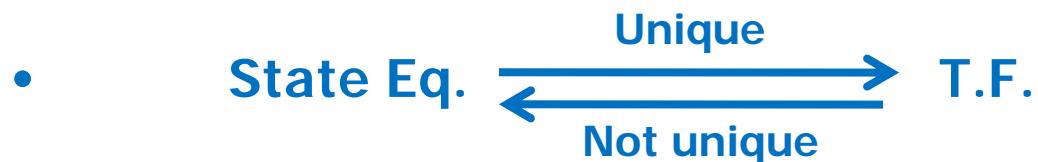
# Linear Systems

- Model :

Differential eq.

Transfer Functions

State eq.



$$\dot{x} = Ax + Bu$$

$$y = Cx + Du$$

$$sX(s) = AX(s) + BU(s)$$

$$X(s) = (sI - A)^{-1} BU(s)$$

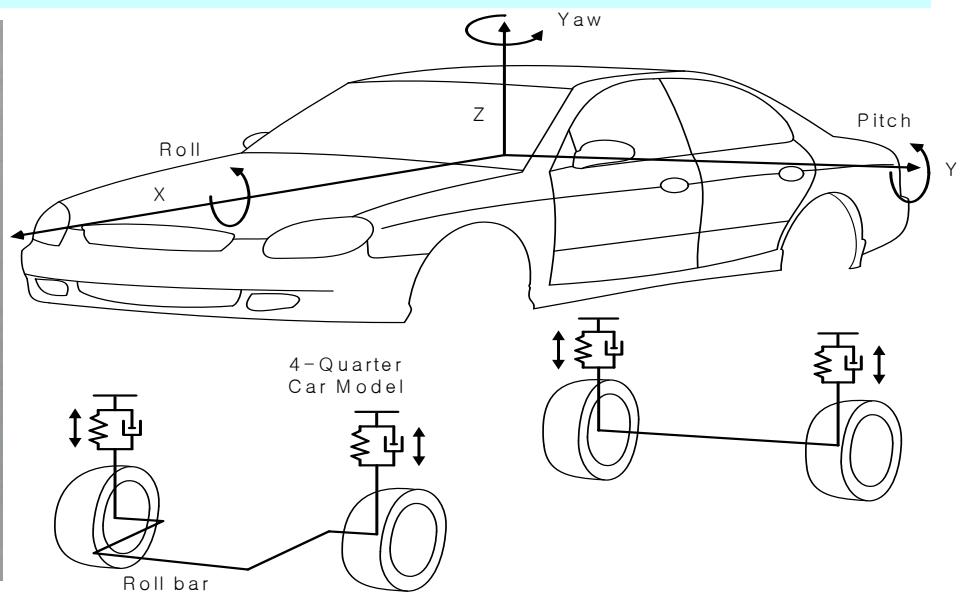
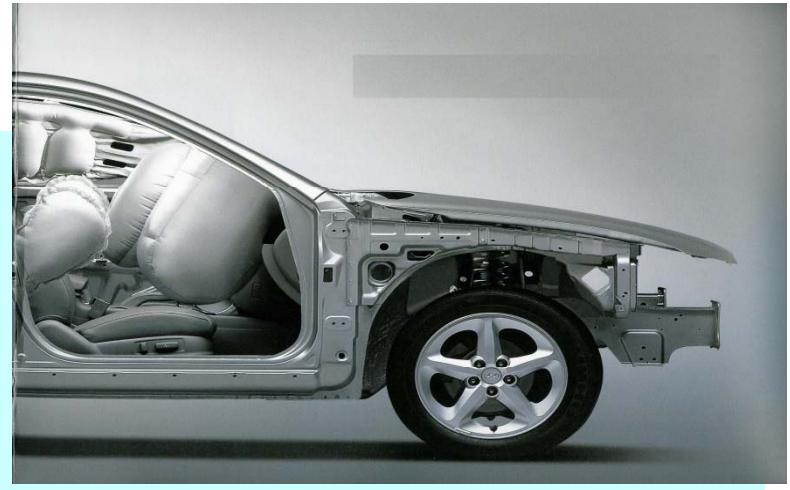
$$Y(s) = CX(s) + DU(s)$$

$$= \left[ C(sI - A)^{-1} B + D \right] U(s)$$

$$= G(s)U(s)$$

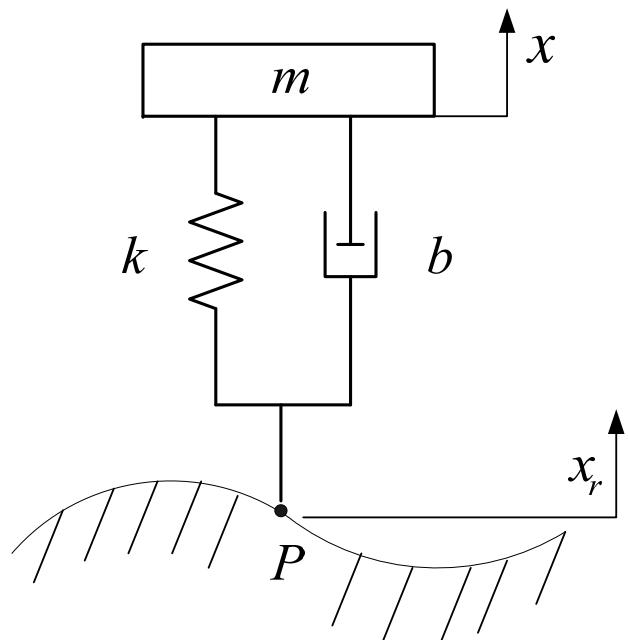


# Vehicle Suspension



# Vehicle Suspension

## Ex 1) Simplified Quarter Car Model



$$\dot{\mathbf{x}} = \begin{bmatrix} 0 & 1 & 0 \\ -\frac{k}{m} & -\frac{b}{m} & \frac{k}{m} \\ 0 & 0 & 0 \end{bmatrix} \mathbf{x} + \begin{bmatrix} 0 \\ \frac{b}{m} \\ 1 \end{bmatrix} u$$

$$m\ddot{x} = -b(\dot{x} - \dot{x}_r) - k(x - x_r)$$

$$m\ddot{x} + b(\dot{x} - \dot{x}_r) + k(x - x_r) = 0$$

$$m\ddot{x} + b\dot{x} + kx = b\dot{x}_r + kx_r$$

Laplace Transform

$$(ms^2 + bs + k)X(s) = (bs + k)X_r(s)$$

The transfer function

$$\frac{X(s)}{X_r(s)} = \frac{bs + k}{ms^2 + bs + k}$$

$$\text{State Eq : } \ddot{x} = -\frac{b}{m}\dot{x} - \frac{k}{m}x + \frac{b}{m}\dot{x}_r + \frac{k}{m}x_r$$

$$\text{let } x_1 = x \quad u = \dot{x}_r$$

$$x_2 = \dot{x}$$

$$x_3 = x_r$$

$$\text{then } \dot{x}_1 = \dot{x} = x_2$$

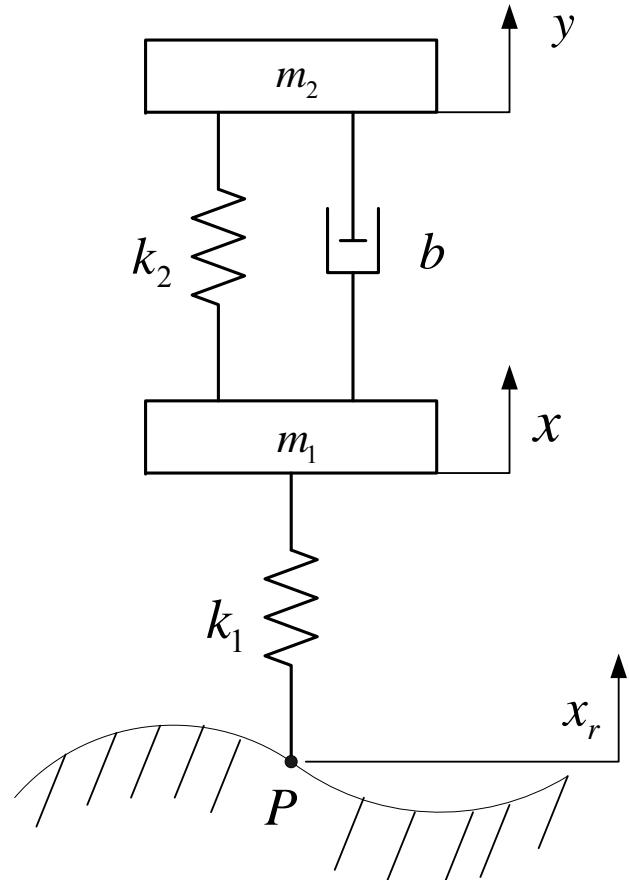
$$\dot{x}_2 = \ddot{x} = -\frac{b}{m}x_2 - \frac{k}{m}x_1 + \frac{k}{m}x_3 + \frac{b}{m}u$$

$$\dot{x}_3 = \dot{x}_r = u$$



# Vehicle Suspension

## Ex 2) Another Quarter Car Model ( 2 DOF ¼ Car model)



Applying the Newton's second law to the system, we obtain

$$m_1 \ddot{x} = -k_2(x - y) - b(\dot{x} - \dot{y}) - k_1(x - x_r)$$

$$m_2 \ddot{y} = k_2(x - y) + b(\dot{x} - \dot{y})$$

Hence we have

$$m_1 \ddot{x} + b\dot{x} + (k_1 + k_2)x = b\dot{y} + k_2y + k_1x_r,$$

$$m_2 \ddot{y} + b\dot{y} + k_2y = b\dot{x} + k_2x$$

Taking Laplace Transform

$$(m_1 s^2 + bs + k_1 + k_2)X(s) = (bs + k_2)Y(s) + k_1 X_r(s)$$

$$(m_2 s^2 + bs + k_2)Y(s) = (bs + k_2)X(s)$$

Eliminating  $X(s)$  from the last two equations, we have

$$\frac{Y(s)}{X_r(s)} = \frac{k_1(bs + k_2)}{m_1 m_2 s^4 + (m_1 + m_2)bs^3 + [k_1 m_2 + (m_1 + m_2)k_2]s^2 + k_1 bs + k_1 k_2}$$



# Vehicle Suspension

## Ex 2) Another Quarter Car Model ( 2 DOF ¼ Car model)

State Equation :

$$\begin{array}{ll} \text{let } x_1 = y - x & \dot{x}_1 = \dot{y} - \dot{x} = x_2 - x_4 \\ x_2 = \dot{y} & x_2 = \ddot{y} = -\frac{k_2}{m_2}x_1 - \frac{b}{m_2}(x_2 - x_4) \\ x_3 = x - x_r & \dot{x}_3 = \dot{x} - \dot{x}_r = x_4 - \dot{x}_r \\ x_4 = \dot{x} & \dot{x}_4 = \ddot{x} = \frac{k_2}{m_1}x_1 + \frac{b}{m_1}(x_2 - x_4) - \frac{k_1}{m_1}x_3 \end{array}$$

$$\dot{\mathbf{x}} = \begin{bmatrix} 0 & 1 & 0 & -1 \\ -\frac{k_2}{m_2} & -\frac{b}{m_2} & 0 & -\frac{b}{m_2} \\ 0 & 0 & 0 & 1 \\ \frac{k_2}{m_1} & \frac{b}{m_1} & -\frac{k_1}{m_1} & -\frac{b}{m_1} \end{bmatrix} \mathbf{x} + \begin{bmatrix} 0 \\ 0 \\ -1 \\ 0 \end{bmatrix} u \quad u = \dot{x}_r$$



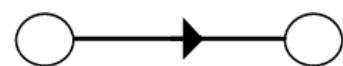
# Signal Flow

$$\dot{x}(t) = Ax + Bu$$

$$y = Cx + Du$$



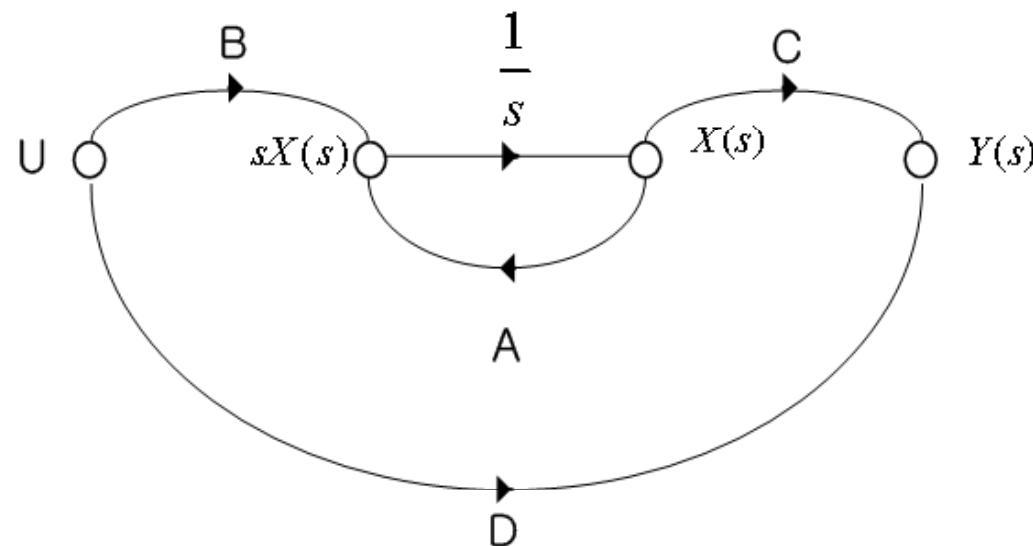
: signal



: relation dynamic  $\left(\frac{1}{s}, z^{-1}\right)$



: static relation



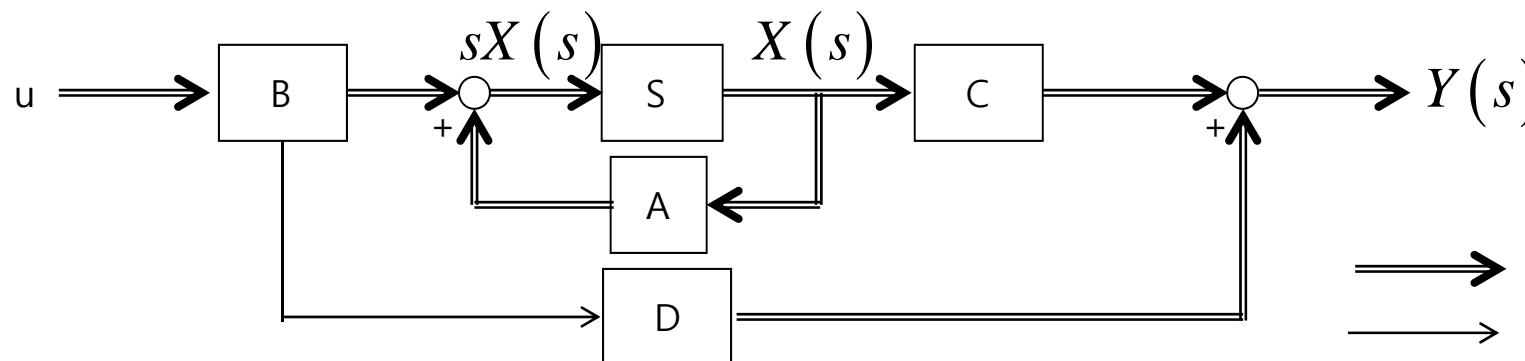
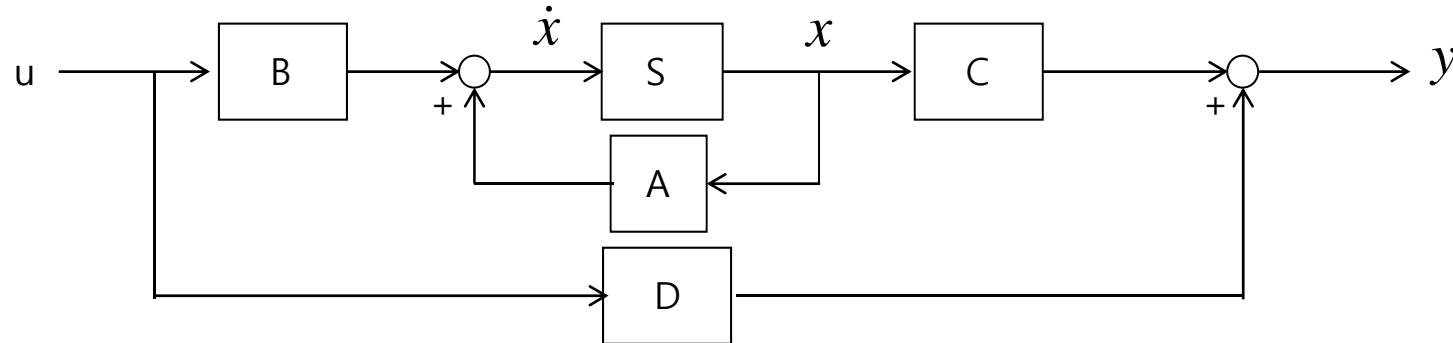
# Block Diagram



: Signals on Line



: Transfer function inside the box (block)



→ : vector  
→ : scalar

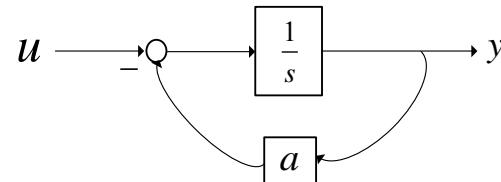


# Block Diagram

ex1. First order

$$\dot{y} + ay = u$$

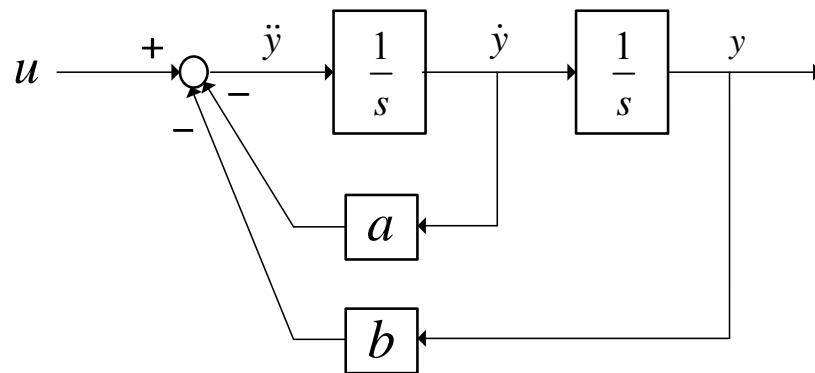
$$\frac{Y}{U} = \frac{1}{s + a}$$



ex2. Second order

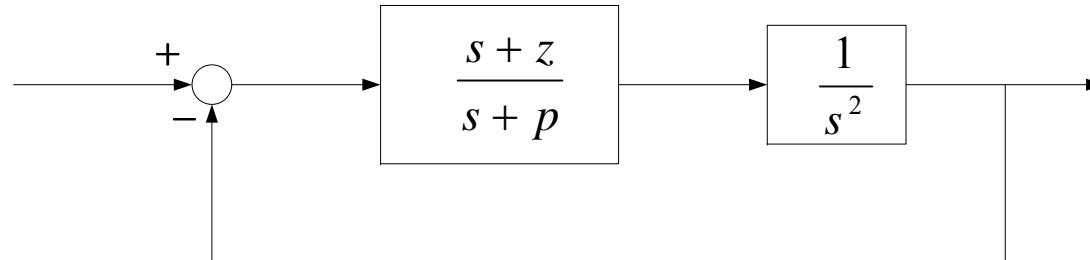
$$\ddot{y} + a\dot{y} + by = u$$

$$\frac{Y}{U} = \frac{1}{s^2 + as + b} = \frac{1}{s(s + a) + b}$$



# Block Diagram

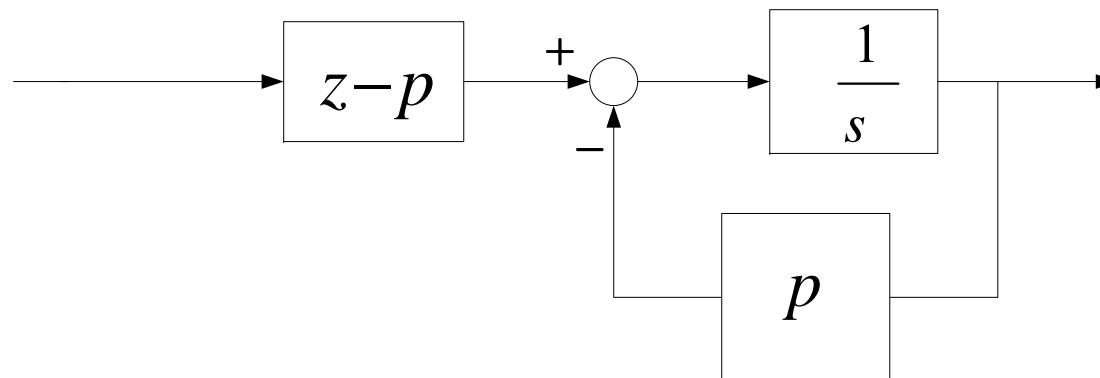
ex3.



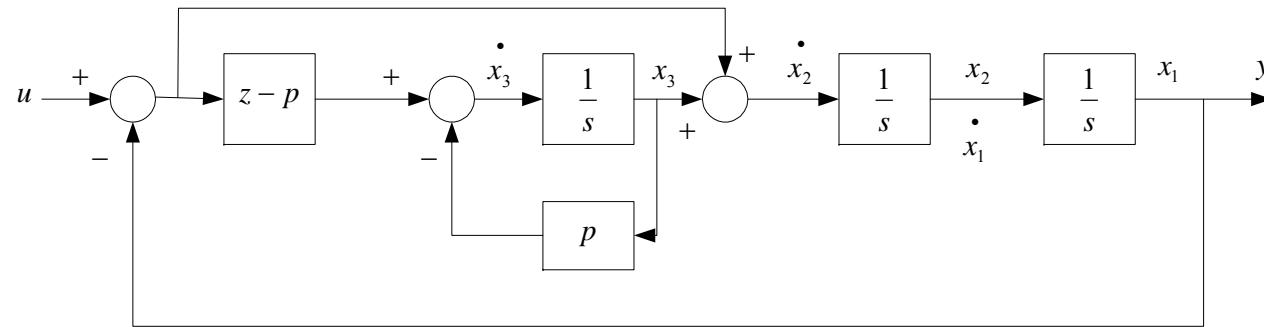
$$\frac{s+z}{s+p} = \frac{s+p+z-p}{s+p} = 1 + \frac{z-p}{s+p}$$

let  $\frac{y}{u} = \frac{z-p}{s+p}$

then  $sy + py = (z-p)u$



# Block Diagram



$$\text{Thus, } \dot{x}_1 = x_2$$

$$\dot{x}_2 = x_3 + (u - x_1)$$

$$\dot{x}_3 = (z - p) \cdot (u - x_1) - px_3$$

The state representation is as follows

$$\dot{\mathbf{x}} = \begin{bmatrix} 0 & 1 & 0 \\ -1 & 0 & 1 \\ p-z & 0 & -p \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \\ z-p \end{bmatrix} u$$

$$y = [1 \ 0 \ 0] x_1$$



# Canonical Forms

- Canonical Forms (T.F. → State Eq.)

Controllable Canonical Form	← Direct Programming Method
Observable Canonical Form	← Nested Programming Method
Diagonal (Jordan) Canonical Form	← Partial Fraction Expansion

- Controllable Canonical Form

$$G(s) = \frac{b_0 s^n + b_1 s^{n-1} + \cdots + b_n}{s^n + a_1 s^{n-1} + \cdots + a_n}$$

Ex) n=3

$$\begin{aligned} G(s) &= \frac{b_0 s^3 + b_1 s^2 + b_2 s + b_3}{s^3 + a_1 s^2 + a_2 s + a_3} \\ &= b_0 + \frac{(b_1 - b_0 a_1)s^2 + (b_2 - b_0 a_2)s + (b_3 - b_0 a_3)}{s^3 + a_1 s^2 + a_2 s + a_3} \end{aligned}$$

$$\left\{ \begin{array}{l} b_1' = b_1 - b_0 a_1 \\ b_2' = b_2 - b_0 a_2 \\ b_3' = b_3 - b_0 a_3 \end{array} \right.$$



# Canonical Forms

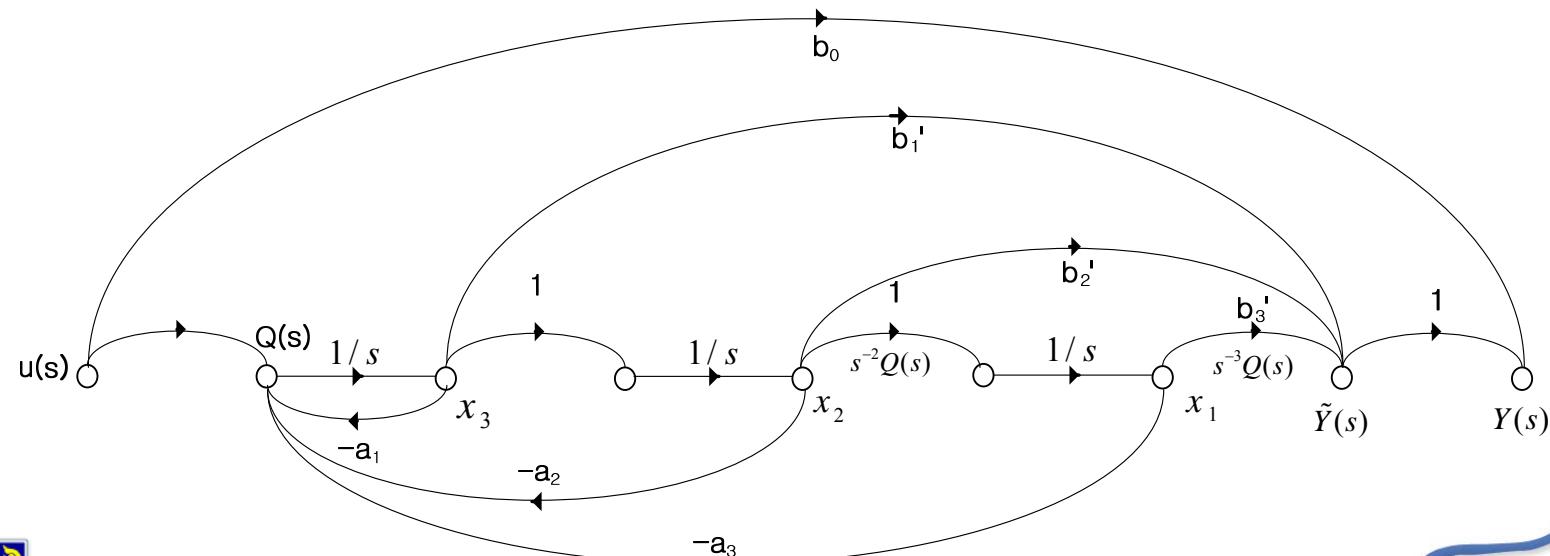
$$\tilde{Y}(s) = \left( \frac{b_1' s^2 + b_2' s + b_3'}{s^3 + a_1 s^2 + a_2 s + a_3} \right) u(s) \times \frac{s^{-3}}{s^{-3}} \quad Y(s) = b_0 u(s) + \tilde{Y}(s)$$

$$Y(s) = \left[ C(sI - A)^{-1} B + D \right] u(s)$$

$$\Rightarrow \frac{\tilde{Y}(s)}{b_1' s^{-1} + b_2' s^{-2} + b_3' s^{-3}} = \frac{u(s)}{1 + a_1 s^{-1} + a_2 s^{-2} + a_3 s^{-3}} = Q(s)$$

$$Q(s) = u(s) - a_1 s^{-1} Q(s) - a_2 s^{-2} Q(s) - a_3 s^{-3} Q(s)$$

$$\tilde{Y}(s) = (b_1' s^{-1} + b_2' s^{-2} + b_3' s^{-3}) Q(s)$$



# Canonical Forms

- **Controllable Canonical Form**

$$\frac{dx_1}{dt} = x_2$$

$$\frac{dx_2}{dt} = x_3$$

$$\frac{dx_3}{dt} = -a_3x_1 - a_2x_2 - a_1x_3 + u$$

$$y = b_3' x_1 + b_2' x_2 + b_1' x_3 + b_0 u$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -a_3 & -a_2 & -a_1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} u$$

$$y = [b_3' \quad b_2' \quad b_1'] x + [b_0] u$$



# Canonical Forms

- Observable Canonical Form (Nested Programming)

$$\frac{dx_1}{dt} = x_2$$

$$\frac{dx_2}{dt} = x_3$$

$$\frac{dx_3}{dt} = -a_3x_1 - a_2x_2 - a_1x_3 + u$$

$$y = b_3' x_1 + b_2' x_2 + b_1' x_3 + b_0 u$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -a_1 & 1 & 0 \\ -a_2 & 0 & 1 \\ -a_3 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} b_1' \\ b_2' \\ b_3' \end{bmatrix} u$$

$$y = [1 \ 0 \ 0] x + [b_0] u$$

(Note :  $x$  in the controllable canonical form  $\neq$   $x$  in the observable canonical form)



# Canonical Forms

- Diagonal (or Jordan) Canonical Form (Partial Fraction Expansion)

Case 1. Distinct Roots ( $\lambda_1 \neq \lambda_2 \neq \lambda_3$ )

$$G(s) = \frac{B(s)}{A(s)} = \frac{K_1}{s - \lambda_1} + \frac{K_2}{s - \lambda_2} + \frac{K_3}{s - \lambda_3}$$

$$Y(s) = \sum_{i=1}^3 \frac{K_i}{s - \lambda_i} u(s) = y_1 + y_2 + y_3$$

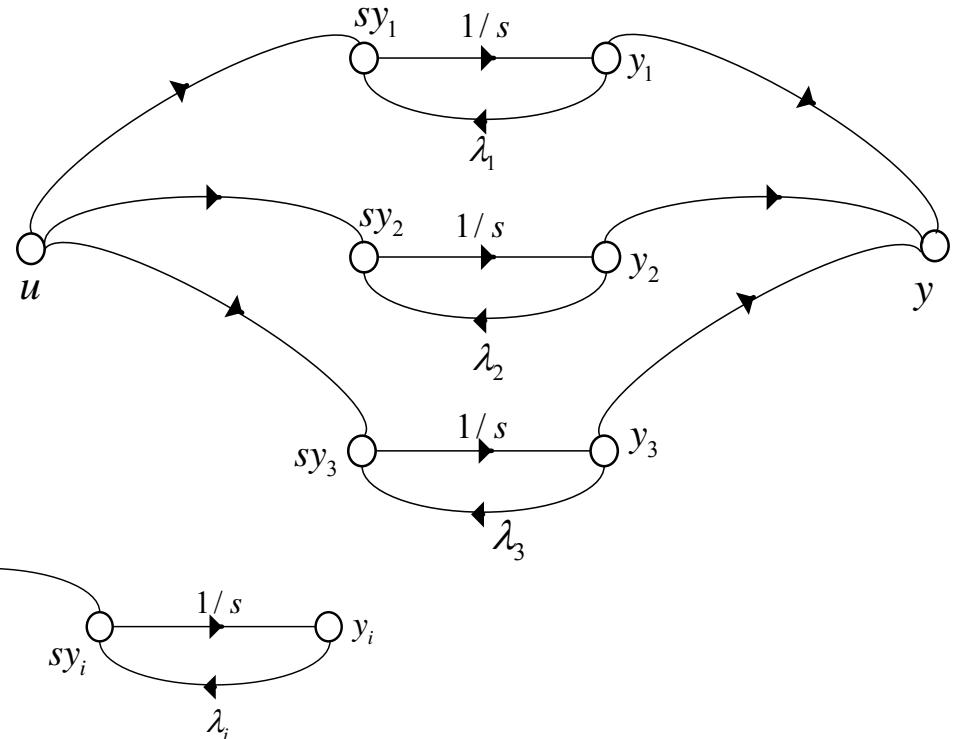
$$y_i = \frac{K_i}{s - \lambda_i} u$$

$$sy_i = \lambda_i y_i + K_i u$$

let  $x_1 = y_1, x_2 = y_2, x_3 = y_3$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} \lambda_1 & 0 & 0 \\ 0 & \lambda_2 & 0 \\ 0 & 0 & \lambda_3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} K_1 \\ K_2 \\ K_3 \end{bmatrix} u$$

$$y = [1 \ 1 \ 1] x$$



# Canonical Forms

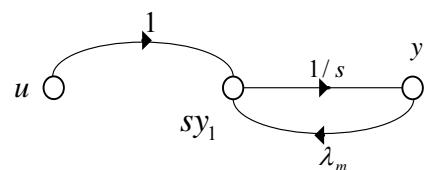
- Diagonal (or Jordan) Canonical Form (Partial Fraction Expansion)

Case 2. Multiple Roots

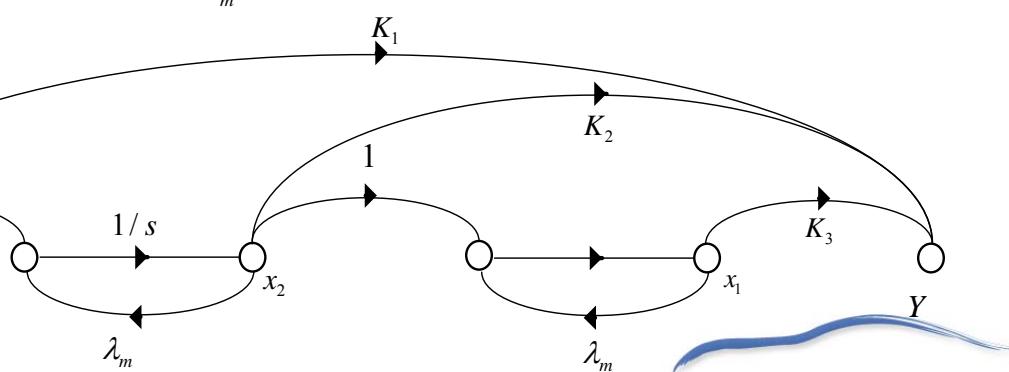
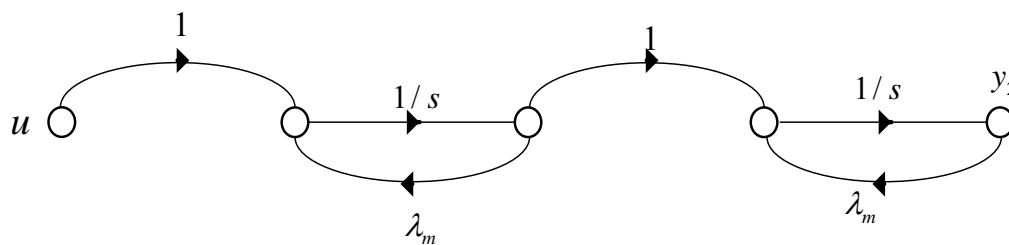
$$G(s) = \frac{B(s)}{(s - \lambda_m)^3} = \frac{K_1}{s - \lambda_m} + \frac{K_2}{(s - \lambda_m)^2} + \frac{K_3}{(s - \lambda_m)^3} \quad \lambda_1 = \lambda_2 = \lambda_3 = \lambda_m$$

$$y_1 = \frac{1}{s - \lambda_m} u$$

$$sy_1 = \lambda_m y_1 + u$$



$$y_2 = \frac{1}{(s - \lambda_m)^2} u$$



# Canonical Forms

- Diagonal (or Jordan) Canonical Form (Partial Fraction Expansion)

Case 2. Multiple Roots

$$\dot{x}_1 = \lambda_m x_1 + x_2$$

$$\dot{x}_2 = \lambda_m x_2 + x_3$$

$$\dot{x}_3 = \lambda_m x_3 + u$$

$$y = K_3 x_1 + K_2 x_2 + K_1 x_3$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} \lambda_1 & 1 & 0 \\ 0 & \lambda_2 & 1 \\ 0 & 0 & \lambda_3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} u$$

$$y = [K_3 \quad K_2 \quad K_1] \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$



# Canonical Forms

- Diagonal (or Jordan) Canonical Form (Partial Fraction Expansion)

Case 3. Complex Roots

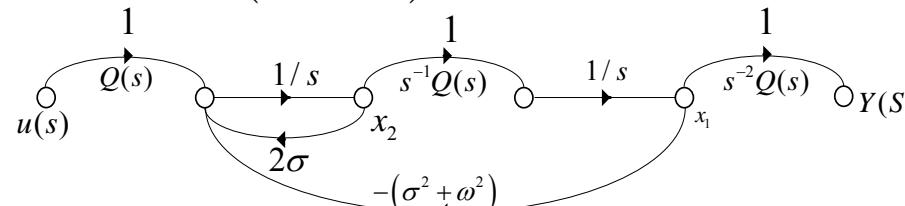
$$G(s) = \frac{1}{s^2 - 2\sigma s + \sigma^2 + \omega^2} = \frac{1}{[s - (\sigma + j\omega)][s - (\sigma - j\omega)]}$$

$$\begin{aligned}\frac{Y}{u} &= G(s) = \frac{1}{s^2 - 2\sigma s + \sigma^2 + \omega^2} \\ &= \frac{s^{-2}}{1 - 2\sigma s^{-1} + (\sigma^2 + \omega^2)s^{-2}}\end{aligned}$$

$$\frac{Y}{s^{-2}} = \frac{u}{1 - 2\sigma s^{-1} + (\sigma^2 + \omega^2)s^{-2}} = Q(s)$$

$$\left\{ \begin{array}{l} Y = s^{-2}Q(s) \\ Q(s)(1 - 2\sigma s^{-1} + (\sigma^2 + \omega^2)s^{-2}) = u \end{array} \right.$$

$$Q(s) = u + 2\sigma s^{-1}Q(s) - (\sigma^2 + \omega^2)s^{-2}Q(s)$$



# Canonical Forms

- Diagonal (or Jordan) Canonical Form (Partial Fraction Expansion)

Case 3. Complex Roots

$$AP = P\Lambda$$

Note : Complex Roots, Complex State x

$$\dot{x} = \Lambda x + bu$$

$$y = Cx$$

→ Complex case의 diagonalization 방법 이용

$$\Lambda K = KJ$$

$$\Lambda = \begin{bmatrix} \sigma + j\omega & 0 \\ 0 & \sigma - j\omega \end{bmatrix}$$

$$K = \begin{bmatrix} \frac{1}{2} & -\frac{j}{2} \\ \frac{1}{2} & \frac{j}{2} \end{bmatrix}$$

$$K^{-1} = \frac{2}{j} \begin{bmatrix} \frac{j}{2} & \frac{j}{2} \\ -\frac{1}{2} & \frac{1}{2} \end{bmatrix}$$

$$J = K^{-1}\Lambda K$$

$$= K^{-1}P^{-1}APK$$

$$J = \begin{bmatrix} \sigma & \omega \\ -\omega & \sigma \end{bmatrix}$$

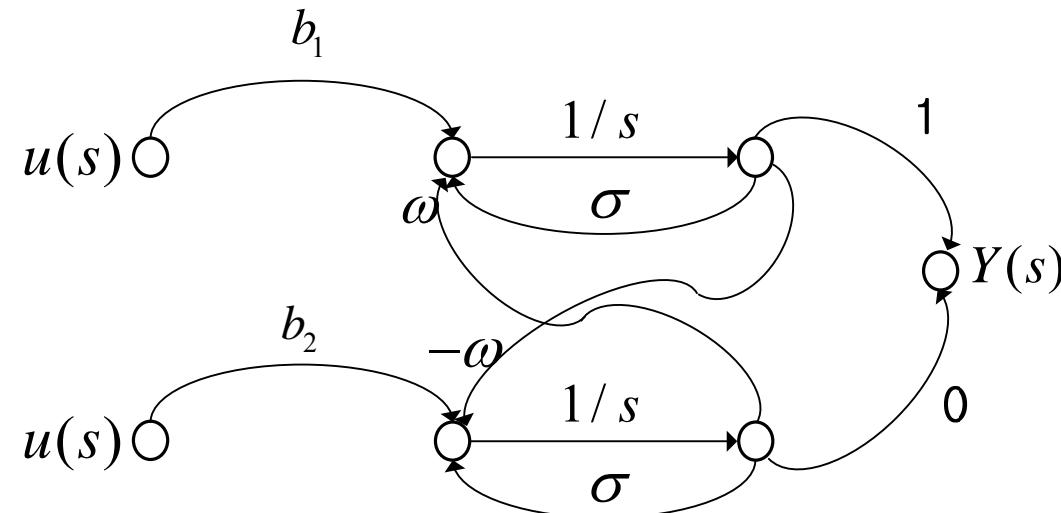


# Canonical Forms

- Diagonal (or Jordan) Canonical Form (Partial Fraction Expansion)

Case 3. Complex Roots

Ex)  $\dot{z} = \begin{bmatrix} \sigma & \omega \\ -\omega & \sigma \end{bmatrix} z + \begin{bmatrix} b_1 \\ b_2 \end{bmatrix} u$   
 $y = [1 \ 0] z$



# Canonical Forms

- Diagonal (or Jordan) Canonical Form (Partial Fraction Expansion)

Case 3. Complex Roots

Step 1     $\dot{x} = Ax + Bu$

Step 2    let     $x = P\xi$   
 $\dot{\xi} = \underbrace{P^{-1}AP}_{\Lambda} \xi + P^{-1}Bu$     : diagonal

Step 3    let     $\xi = Kz$   
 $\dot{z} = \underbrace{K^{-1}\Lambda K}_J z + K^{-1}P^{-1}Bu$   
 $= \begin{bmatrix} \sigma & \omega \\ -\omega & \sigma \end{bmatrix} z + \begin{bmatrix} b_1 \\ b_2 \end{bmatrix} u$

$$\begin{cases} x = P\xi = PKz \\ \dot{z} = \underbrace{K^{-1}\Lambda K}_J z + K^{-1}P^{-1}bu \\ y = CPKz \end{cases}$$

