

## **2. Stress and infinitesimal strain**

# 2.1 Problem definition

- Shallow depth – Block movement in jointed rock mass under low-stress environment

Deep mining – Stress is main concern for the mine stability.  
Rock mass behaves as continuum.

- Force
  - Body force: Gravity, magnetic, inertia ( $F/V$ )
  - Surface force: Conveyed by physical contact ( $F/A$ )

## 2.2 Force and stress

- Force is a vector with magnitude and direction.

Ex.)  $F_x, F_y, F_z, F_1, F_2, \dots$

- Stress is a tensor with magnitude, direction and surface orientation.

Ex.)  $\sigma_{xx} (= \sigma_x), \sigma_{xy} (= \tau_{xy}), \sigma_{yz}, \dots$

↙  
surface orientation (normal to the surface)

↘  
stress direction

## 2.2 Force and stress

- Traction is a force per unit area acting on a surface (Fig.2.1(b)).
- “Tensor is a further extension of ideas we already use when defining quantities like scalars and vectors.”

Ex.) Scalar  $\rightarrow$  rank zero

Vector  $\rightarrow$  rank one

Stress  $\rightarrow$  rank two

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## 2.2 Force and stress

- Tensor notation
  - Introduced in deriving the theory of relativity by Einstein.
  - Basic idea is to make summation for the repeatedly used subscripts.

Ex.)  $a_i b_i = a_x b_x + a_y b_y + a_z b_z$

$$a_{ii} b_k = a_{xx} b_k + a_{yy} b_k + a_{zz} b_k \quad (i: \text{dummy index}, k: \text{free index})$$
$$a_{ik} b_i c_k = a_{xk} b_x c_k + a_{yk} b_y c_k + a_{zk} b_z c_k$$
$$= a_{xx} b_x c_x + a_{xy} b_x c_y + a_{xz} b_x c_z$$
$$+ a_{yx} b_y c_x + a_{yy} b_y c_y + a_{yz} b_y c_z$$
$$+ a_{zx} b_z c_x + a_{zy} b_z c_y + a_{zz} b_z c_z$$

## 2.2 Force and stress

- Important symbols in tensor notation

- Kronecker delta

$$\delta_{ij} = 1 \text{ if } i=j.$$

$$= 0 \text{ if } i \neq j.$$

EX.)  $\delta_{xx} = 1, \delta_{zz} = 1, \delta_{xy} = 0, \delta_{yz} = 0$

$$\delta_{ij} a_i b_j = a_x b_x + a_y b_y + a_z b_z = a_i b_i$$

- Permutation symbol

$$\varepsilon_{ijk} = 0 \text{ if any pair of subscripts are identical.}$$

$$= (-1)^p, \text{ where } p \text{ is the number of subscript transposition.}$$

EX.)  $\varepsilon_{xyz} = 1, \varepsilon_{xyx} = 0, \varepsilon_{yyz} = 0, \varepsilon_{xzz} = 0$

$$\varepsilon_{xzy} = -1, \varepsilon_{zxy} = 1, \varepsilon_{yxz} = -1, \varepsilon_{zyx} = -1$$

## 2.2 Force and stress

- Linear algebra in tensor notation

- Matrix operation

$$A\mathbf{x} = \mathbf{b} \rightarrow a_{ij}x_j = b_i$$

$$AB = Y \rightarrow a_{ij}b_{jk} = Y_{ik}$$

$$A^T B = Z \rightarrow a_{ji}b_{jk} = Z_{ik}$$

$$\mathbf{x}^T B \mathbf{x} = c \rightarrow x_i b_{ij} x_j = c$$

- Derivatives

$$\text{Grad } \phi = \nabla \phi = \frac{\partial \phi}{\partial x} \hat{e}_x + \frac{\partial \phi}{\partial y} \hat{e}_y + \frac{\partial \phi}{\partial z} \hat{e}_z = \phi_{,i}$$

$$\text{Div } \vec{V} = \nabla \cdot \vec{V} = \frac{\partial V_x}{\partial x} + \frac{\partial V_y}{\partial y} + \frac{\partial V_z}{\partial z} = V_{i,i}$$

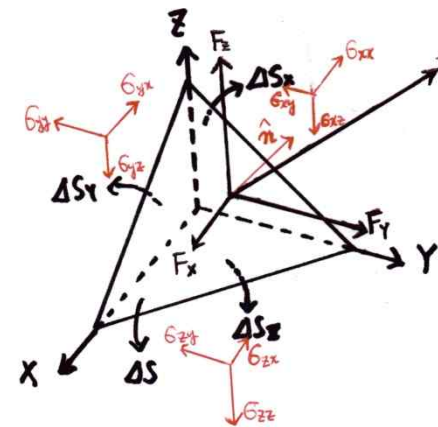
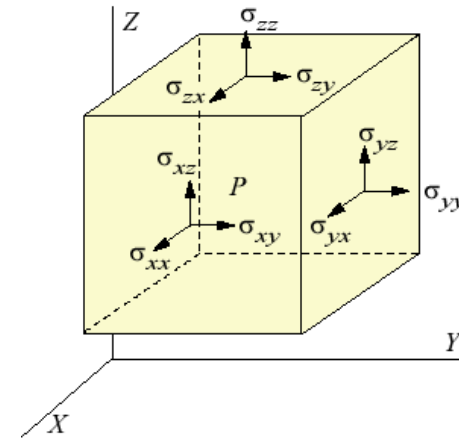
## 2.2 Force and stress

- Stress in rectangular coordinates

$$[\sigma] = \begin{bmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{yx} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{zx} & \sigma_{zy} & \sigma_{zz} \end{bmatrix} = \begin{bmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{xy} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{xz} & \sigma_{yz} & \sigma_{zz} \end{bmatrix}$$

- Stress in an arbitrary plane

$$\hat{n} = \left( \frac{\Delta S_x}{\Delta S}, \frac{\Delta S_y}{\Delta S}, \frac{\Delta S_z}{\Delta S} \right)$$





## 2.2 Force and stress

$$\vec{F} = \vec{F}_x + \vec{F}_y + \vec{F}_z$$

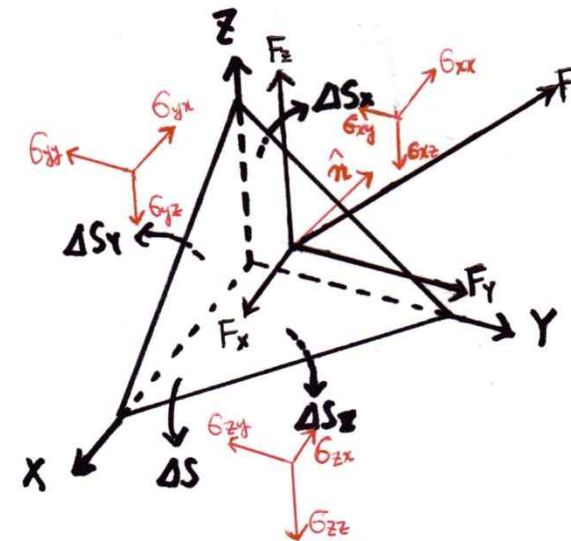
$$\Delta S t^{\vec{n}} = \Delta S t_x^{\vec{n}} + \Delta S t_y^{\vec{n}} + \Delta S t_z^{\vec{n}}$$

$$\Delta S t_x^{\hat{n}} = \Delta S_x \sigma_{xx} + \Delta S_y \sigma_{yx} + \Delta S_z \sigma_{zx}$$

$$\Delta S t_y^{\hat{n}} = \Delta S_x \sigma_{xy} + \Delta S_y \sigma_{yy} + \Delta S_z \sigma_{zy}$$

$$\Delta S t_z^{\hat{n}} = \Delta S_x \sigma_{xz} + \Delta S_y \sigma_{yz} + \Delta S_z \sigma_{zz}$$

$$t_i^{\hat{n}} = n_j \sigma_{ji} \quad (= \sigma_{ij} n_j)$$



## 2.2 Force and stress

- Equilibrium conditions

- Force equilibrium & moment equilibrium

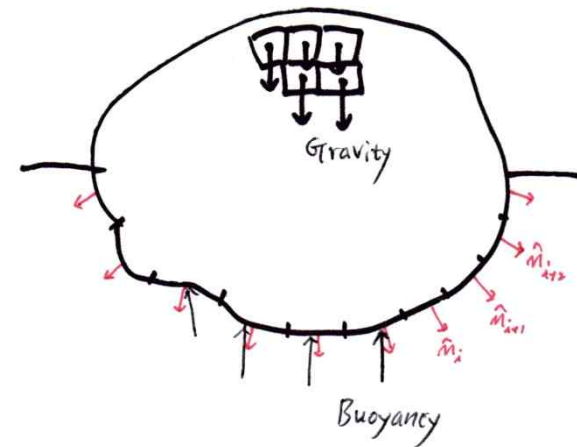
$$\sum \vec{F} = 0, \quad \sum \vec{M} = 0$$

- Force equilibrium

$$\int_s t_i^{\hat{n}} ds + \int_v \rho b_i dv = 0$$

$$\int_s n_j \sigma_{ji} ds = \int_v \sigma_{ji,j} dv = 0 \quad (\text{Divergence theorem})$$

$$\rightarrow \sigma_{ji,j} + \rho b_i = 0$$



## 2.2 Force and stress

- Moment equilibrium

$$\vec{M} = \vec{r} \times \vec{F} \quad (= \varepsilon_{ijk} r_j F_k)$$

$$\int_s \vec{r} \times t^{\hat{n}} ds + \int_v \vec{r} \times \rho \vec{b} dv = 0$$

$$\rightarrow \varepsilon_{ijk} \sigma_{jk} = 0$$

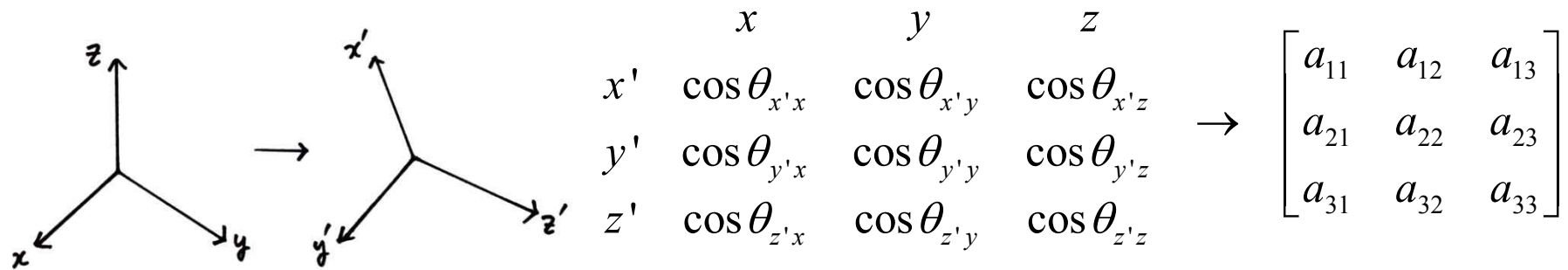
$$i = x: \quad \sigma_{yz} - \sigma_{zy} = 0$$

$$i = y: \quad \sigma_{zx} - \sigma_{xz} = 0 \quad \rightarrow \quad \sigma_{ij} = \sigma_{ji}$$

$$i = z: \quad \sigma_{xy} - \sigma_{yx} = 0$$

## 2.3 Stress transformation

- Vector transformation



$$\begin{bmatrix} x' \\ y' \\ z' \end{bmatrix} = \begin{bmatrix} a_{11} & a_{12} & a_{13} \\ a_{21} & a_{22} & a_{23} \\ a_{31} & a_{32} & a_{33} \end{bmatrix} \begin{bmatrix} x \\ y \\ z \end{bmatrix}$$

## 2.3 Stress transformation

- Stress transformation

$$u_i' = a_{ip} u_p, \quad v_j' = a_{jq} v_q \quad (\text{vector transformation})$$

$$u_i' v_j' = \sigma_{ij}' \quad (\text{outer product})$$

$$u_i' v_j' = a_{ip} u_p a_{jq} v_q = a_{ip} a_{jq} u_p v_q = a_{ip} a_{jq} \sigma_{pq} = a_{ip} \sigma_{pq} a_{jq}$$

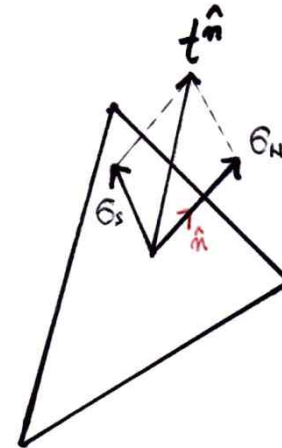
$$\rightarrow \sigma_{ij}' = a_{ip} \sigma_{pq} a_{jq} \quad (\Sigma' = A \cdot \Sigma \cdot A^T)$$

## 2.4 Principal stress and stress invariants

- Principal stress
  - Traction in a surface can be decomposed into normal and shear stresses which are in the same plane with the traction.

$$\sigma_N = \hat{n} \cdot \hat{t}$$

$$\sigma_S = \hat{t} - \sigma_N \hat{n}$$



## 2.4 Principal stress and stress invariants

- The surface on which shear component of the traction is zero is a **principal plane** and its normal component is called **principal stress**.

$$\begin{aligned} t^{\hat{n}} &= \Sigma \cdot \hat{n} = \sigma \hat{n} \\ \sigma_{ij} n_j - \sigma n_i &= 0 \\ (\sigma_{ij} - \sigma \delta_{ij}) n_j &= 0 \end{aligned} \quad \begin{bmatrix} \sigma_{xx} - \sigma & \sigma_{xy} & \sigma_{xz} \\ \sigma_{xy} & \sigma_{yy} - \sigma & \sigma_{yz} \\ \sigma_{xz} & \sigma_{yz} & \sigma_{zz} - \sigma \end{bmatrix} \begin{bmatrix} x \\ y \\ z \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}$$

When  $\sigma_1 \geq \sigma_2 \geq \sigma_3$

$\sigma_1$  : major principal stress

$\sigma_2$  : intermediate principal stress

$\sigma_3$  : minor principal stress

## 2.4 Principal stress and stress invariants

- Stress invariants

$$\begin{vmatrix} \sigma_{xx} - \sigma & \sigma_{xy} & \sigma_{xz} \\ \sigma_{xy} & \sigma_{yy} - \sigma & \sigma_{yz} \\ \sigma_{xz} & \sigma_{yz} & \sigma_{zz} - \sigma \end{vmatrix} = 0 \quad \text{for } \hat{n} \neq (0,0,0)$$

$$\rightarrow \sigma^3 - I_1 \sigma^2 + I_2 \sigma - I_3 = 0$$

$$I_1 = \sigma_{ii} \quad (= \sigma_{xx} + \sigma_{yy} + \sigma_{zz})$$

$$I_2 = \frac{1}{2} [\sigma_{ii} \sigma_{jj} - \sigma_{ij} \sigma_{ij}] \quad (= \sigma_{xx} \sigma_{yy} + \sigma_{xx} \sigma_{zz} + \sigma_{yy} \sigma_{zz} - \sigma_{xy} \sigma_{xy} - \sigma_{xz} \sigma_{xz} - \sigma_{yz} \sigma_{yz})$$

$$I_3 = |\sigma_{ij}|$$



## CUBIC EQUATION: $x^3 + a_1x^2 + a_2x + a_3 = 0$

Let

$$Q = \frac{3a_2 - a_1^2}{9}, \quad R = \frac{9a_1a_2 - 27a_3 - 2a_1^3}{54},$$

$$S = \sqrt[3]{R + \sqrt{Q^3 + R^2}}, \quad T = \sqrt[3]{R - \sqrt{Q^3 + R^2}}$$

Solutions:

$$\begin{cases} x_1 = S + T - \frac{1}{3}a_1 \\ x_2 = -\frac{1}{2}(S + T) - \frac{1}{3}a_1 + \frac{1}{2}i\sqrt{3}(S - T) \\ x_3 = -\frac{1}{2}(S + T) - \frac{1}{3}a_1 - \frac{1}{2}i\sqrt{3}(S - T) \end{cases}$$

If  $a_1, a_2, a_3$  are real and if  $D = Q^3 + R^2$  is the *discriminant*, then

- (i) one root is real and two complex conjugate if  $D > 0$
- (ii) all roots are real and at least two are equal if  $D = 0$
- (iii) all roots are real and unequal if  $D < 0$ .

If  $D < 0$ , computation is simplified by use of trigonometry.

Solutions if  $D < 0$ :

$$\begin{cases} x_1 = 2\sqrt{-Q} \cos(\frac{1}{3}\theta) - \frac{1}{3}a_1 \\ x_2 = 2\sqrt{-Q} \cos(\frac{1}{3}\theta + 120^\circ) - \frac{1}{3}a_1 \\ x_3 = 2\sqrt{-Q} \cos(\frac{1}{3}\theta + 240^\circ) - \frac{1}{3}a_1 \end{cases} \quad \text{where } \cos \theta = R/\sqrt{-Q^3}$$

$$x_1 + x_2 + x_3 = -a_1, \quad x_1x_2 + x_2x_3 + x_3x_1 = a_2, \quad x_1x_2x_3 = -a_3$$

where  $x_1, x_2, x_3$  are the three roots.

## 2.5 Differential equations of static equilibrium

Refer to section 2.2

$$\sigma_{ji,j} + \rho b_i = 0$$

$$\frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{xy}}{\partial y} + \frac{\partial \sigma_{xz}}{\partial z} + \rho b_x = 0$$

$$\frac{\partial \sigma_{yx}}{\partial x} + \frac{\partial \sigma_{yy}}{\partial y} + \frac{\partial \sigma_{yz}}{\partial z} + \rho b_y = 0$$

$$\frac{\partial \sigma_{zx}}{\partial x} + \frac{\partial \sigma_{zy}}{\partial y} + \frac{\partial \sigma_{zz}}{\partial z} + \rho b_z = 0$$

## **2.6 Plane problems and biaxial stress**

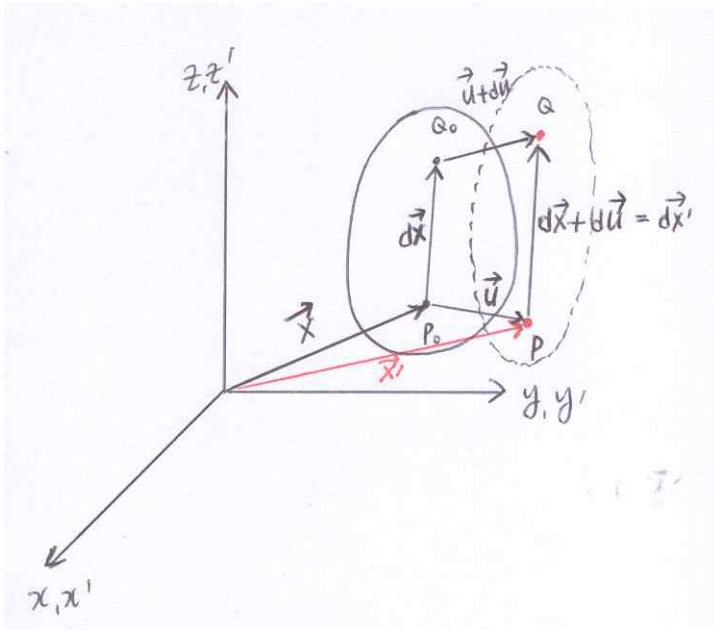
Apply the principles in 3D to 2D problems

# Problem

- Calculate traction, normal stress and shear stress in a surface whose normal vector is as follow ( $\Sigma$  is stress on the object).

$$\hat{n} = \left( \frac{2}{3}, -\frac{2}{3}, \frac{1}{3} \right) \quad \Sigma = \begin{bmatrix} 7 & 0 & -2 \\ 0 & 5 & 0 \\ -2 & 0 & 4 \end{bmatrix}$$

## 2.7 Displacement and strain



$$\vec{X} = (x, y, z) \quad \vec{X}' = (x', y', z')$$

$$\vec{u} = \vec{X}' - \vec{X} \quad d\vec{X}' = d\vec{X} + d\vec{u}$$

$$(dX')^2 = d\vec{X}' \cdot d\vec{X}' \rightarrow dX'_k dX'_k$$

$$dX'_k = \frac{\partial X'_k}{\partial X_j} dX_j$$

$$(dX')^2 - (dX)^2 = \frac{\partial X'_k}{\partial X_i} \frac{\partial X'_k}{\partial X_j} dX_i dX_j - \delta_{ij} dX_i dX_j$$

$$= \left( \frac{\partial X'_k}{\partial X_i} \frac{\partial X'_k}{\partial X_j} - \delta_{ij} \right) dX_i dX_j$$

## 2.7 Displacement and strain

$$L_{ij} = \frac{1}{2} \left( \frac{\partial X'_k}{\partial X_i} \frac{\partial X'_k}{\partial X_j} - \delta_{ij} \right)$$

$$X'_k = u_k + X_k$$

$$L_{ij} = \frac{1}{2} \left( \left( \frac{\partial u_k}{\partial X_i} + \delta_{ki} \right) \left( \frac{\partial u_k}{\partial X_j} + \delta_{kj} \right) - \delta_{ij} \right)$$

$$= \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} + \frac{\partial u_k}{\partial X_i} \frac{\partial u_k}{\partial X_j} + \delta_{ij} - \delta_{ij} \right)$$

$$\approx \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right) \quad \text{for infinitesimal displacement}$$

$$\therefore (dX')^2 - (dX)^2 = 2L_{ij} dX_i dX_j$$

## 2.7 Displacement and strain

$$L_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right)$$

$$[L_{ij}] = \begin{bmatrix} \frac{\partial u_x}{\partial X_x} & \frac{1}{2} \left( \frac{\partial u_x}{\partial X_y} + \frac{\partial u_y}{\partial X_x} \right) & \frac{1}{2} \left( \frac{\partial u_x}{\partial X_z} + \frac{\partial u_z}{\partial X_x} \right) \\ \frac{1}{2} \left( \frac{\partial u_y}{\partial X_x} + \frac{\partial u_x}{\partial X_y} \right) & \frac{\partial u_y}{\partial X_y} & \frac{1}{2} \left( \frac{\partial u_y}{\partial X_z} + \frac{\partial u_z}{\partial X_y} \right) \\ \frac{1}{2} \left( \frac{\partial u_z}{\partial X_x} + \frac{\partial u_x}{\partial X_z} \right) & \frac{1}{2} \left( \frac{\partial u_z}{\partial X_y} + \frac{\partial u_y}{\partial X_z} \right) & \frac{\partial u_z}{\partial X_z} \end{bmatrix}$$

.....Linear (Infinitesimal) strain tensor

## 2.7 Displacement and strain

- Normal strain

$$\begin{aligned}(dX')^2 - (dX)^2 &= (dX' - dX)(dX' + dX) \\ &\approx 2(dX' - dX)dX = 2L_{ij}dX_i dX_j\end{aligned}$$

$$dX' - dX = \frac{L_{ij}dX_i dX_j}{dX}$$

$$\frac{dX' - dX}{dX} = \frac{dX_i}{dX} L_{ij} \frac{dX_j}{dX} = \hat{n} \cdot L \cdot \hat{n} \quad (\text{normal strain})$$

$$\hat{e}_x \cdot L \cdot \hat{e}_x = L_{xx}$$

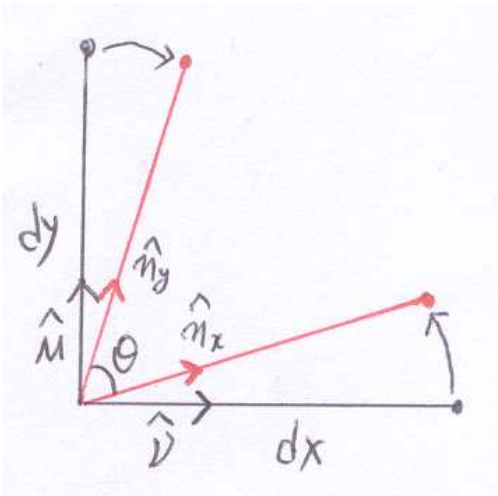
$$\hat{e}_y \cdot L \cdot \hat{e}_y = L_{yy}$$

$$\hat{e}_z \cdot L \cdot \hat{e}_z = L_{zz}$$



## 2.7 Displacement and strain

### • Shear strain



$$\hat{n}_x \cdot \hat{n}_y = \cos \theta = \sin \left( \frac{\pi}{2} - \theta \right) = \sin \gamma \approx \gamma \quad (\text{shear strain})$$

$$L_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right) = \frac{1}{2} (J + J_c)$$

$$d\bar{u} = \frac{\partial u_i}{\partial X_j} dX_j = J \cdot d\vec{X}$$

$$\hat{n}_x \approx \hat{v} + J \cdot \hat{v}, \quad \hat{n}_y \approx \hat{\mu} + J \cdot \hat{\mu}$$

$$\begin{aligned} \gamma &= \hat{n}_x \cdot \hat{n}_y = (\hat{v} + J \cdot \hat{v}) \cdot (\hat{\mu} + J \cdot \hat{\mu}) \\ &= \hat{v} \cdot \hat{\mu} + \hat{v} \cdot (J + J_c) \cdot \hat{\mu} + \hat{v} \cdot J_c \cdot J \cdot \hat{\mu} \\ &= \hat{v} \cdot 2L \cdot \hat{\mu} \end{aligned}$$

$$\hat{e}_x \cdot L \cdot \hat{e}_y = L_{xy} = \frac{1}{2} \gamma_{xy} \quad \hat{e}_y \cdot L \cdot \hat{e}_z = L_{yz} = \frac{1}{2} \gamma_{yz}$$

$$\hat{e}_z \cdot L \cdot \hat{e}_x = L_{zx} = \frac{1}{2} \gamma_{zx}$$

## 2.7 Displacement and strain

$$L_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right)$$

$$[L_{ij}] = \begin{bmatrix} \varepsilon_x & \frac{1}{2} \gamma_{xy} & \frac{1}{2} \gamma_{xz} \\ \frac{1}{2} \gamma_{xy} & \varepsilon_y & \frac{1}{2} \gamma_{yz} \\ \frac{1}{2} \gamma_{xz} & \frac{1}{2} \gamma_{yz} & \varepsilon_z \end{bmatrix}$$

## 2.7 Displacement and strain

- Decomposition of relative displacement ( $du$ )

$$\begin{aligned}
 du_i &= \frac{\partial u_i}{\partial X_j} dX_j \\
 &= \left[ \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right) + \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} - \frac{\partial u_j}{\partial X_i} \right) \right] dX_j \\
 &= [L_{ij} + \Omega_{ij}] dX_j \\
 [\Omega_{ij}] &= \begin{bmatrix} 0 & \omega_{xy} & \omega_{xz} \\ -\omega_{xy} & 0 & \omega_{yz} \\ -\omega_{xz} & -\omega_{yz} & 0 \end{bmatrix} \quad \dots \text{rotation tensor} \\
 &= \begin{bmatrix} 0 & -\Omega_z & \Omega_y \\ \Omega_z & 0 & -\Omega_x \\ -\Omega_y & \Omega_x & 0 \end{bmatrix} \left( \vec{\Omega} = \frac{1}{2} \vec{\nabla} \times \vec{u} = \frac{1}{2} \begin{bmatrix} \hat{e}_x & \hat{e}_y & \hat{e}_z \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ u_x & u_y & u_z \end{bmatrix} \right)
 \end{aligned}$$

## 2.8 Principal strains, strain transformation, volumetric strain and deviator strain

Principal strains: Eigen values/vectors of strain tensor

Strain transformation: analogous to the stress transformation

$$\begin{aligned} \text{Volumetric strain} &= \frac{V - V_0}{V_0} = \frac{(1 + \varepsilon_1)(1 + \varepsilon_2)(1 + \varepsilon_3)dX_1dX_2dX_3}{dX_1dX_2dX_3} - 1 \\ &= \varepsilon_1 + \varepsilon_2 + \varepsilon_3 + \varepsilon_1\varepsilon_2 + \varepsilon_2\varepsilon_3 + \varepsilon_3\varepsilon_1 + \varepsilon_1\varepsilon_2\varepsilon_3 \\ &\approx \varepsilon_1 + \varepsilon_2 + \varepsilon_3 \end{aligned}$$

$$\text{Deviator strain} = \begin{bmatrix} \varepsilon_x - \Delta/3 & \gamma_{xy} & \gamma_{xz} \\ \gamma_{xy} & \varepsilon_y - \Delta/3 & \gamma_{yz} \\ \gamma_{xz} & \gamma_{yz} & \varepsilon_z - \Delta/3 \end{bmatrix} \quad \Delta : \text{volumetric strain}$$

## 2.9 Strain compatibility equations

Independent strain components: 6 ( $\varepsilon_x, \varepsilon_y, \varepsilon_z, \gamma_{xy}, \gamma_{yz}, \gamma_{zx}$ )

Independent displacement components : 3 ( $u_x, u_y, u_z$ )

Random  $u_x, u_y, u_z \rightarrow \varepsilon_x, \varepsilon_y, \varepsilon_z, \gamma_{xy}, \gamma_{yz}, \gamma_{zx}$  (o)

Random  $\varepsilon_x, \varepsilon_y, \varepsilon_z, \gamma_{xy}, \gamma_{yz}, \gamma_{zx} \rightarrow u_x, u_y, u_z$  (x)

Finding restricting conditions:

$$\varepsilon_{xx} = \frac{\partial u_x}{\partial X_x}, \quad \varepsilon_{yy} = \frac{\partial u_y}{\partial X_y}, \quad \varepsilon_{xy} = \frac{1}{2} \left( \frac{\partial u_x}{\partial X_y} + \frac{\partial u_y}{\partial X_x} \right)$$

$$\frac{\partial^2 \varepsilon_{xx}}{\partial X_y^2} = \frac{\partial^3 u_x}{\partial X_x \partial X_y^2}, \quad \frac{\partial^2 \varepsilon_{yy}}{\partial X_x^2} = \frac{\partial^3 u_y}{\partial X_x^2 \partial X_y}, \quad \frac{\partial^2 \varepsilon_{xy}}{\partial X_x \partial X_y} = \frac{1}{2} \left( \frac{\partial^3 u_x}{\partial X_x \partial X_y^2} + \frac{\partial^3 u_y}{\partial X_x^2 \partial X_y} \right)$$

$$\therefore \frac{\partial^2 \varepsilon_{xx}}{\partial X_y^2} + \frac{\partial^2 \varepsilon_{yy}}{\partial X_x^2} = 2 \frac{\partial^2 \varepsilon_{xy}}{\partial X_x \partial X_y}$$

## 2.9 Strain compatibility equations

$$\frac{\partial^2 \varepsilon_{xx}}{\partial X_y^2} + \frac{\partial^2 \varepsilon_{yy}}{\partial X_x^2} = 2 \frac{\partial^2 \varepsilon_{xy}}{\partial X_x \partial X_y}$$

$$\frac{\partial^2 \varepsilon_{yy}}{\partial X_z^2} + \frac{\partial^2 \varepsilon_{zz}}{\partial X_x^2} = 2 \frac{\partial^2 \varepsilon_{yz}}{\partial X_y \partial X_z}$$

$$\frac{\partial^2 \varepsilon_{zz}}{\partial X_x^2} + \frac{\partial^2 \varepsilon_{xx}}{\partial X_z^2} = 2 \frac{\partial^2 \varepsilon_{zx}}{\partial X_x \partial X_z}$$

$$\frac{\partial}{\partial X_x} \left( -\frac{\partial \varepsilon_{yz}}{\partial X_x} + \frac{\partial \varepsilon_{zx}}{\partial X_y} + \frac{\partial \varepsilon_{xy}}{\partial X_z} \right) = \frac{\partial^2 \varepsilon_{xx}}{\partial X_y \partial X_z}$$

$$\frac{\partial}{\partial X_y} \left( \frac{\partial \varepsilon_{yz}}{\partial X_x} - \frac{\partial \varepsilon_{zx}}{\partial X_y} + \frac{\partial \varepsilon_{xy}}{\partial X_z} \right) = \frac{\partial^2 \varepsilon_{yy}}{\partial X_z \partial X_x}$$

$$\frac{\partial}{\partial X_z} \left( \frac{\partial \varepsilon_{yz}}{\partial X_x} + \frac{\partial \varepsilon_{zx}}{\partial X_y} - \frac{\partial \varepsilon_{xy}}{\partial X_z} \right) = \frac{\partial^2 \varepsilon_{zz}}{\partial X_x \partial X_y}$$

## 2.10 Stress-strain relations

$$\varepsilon_{xx} = \frac{1}{E} [\sigma_{xx} - \nu(\sigma_{yy} + \sigma_{zz})], \quad \varepsilon_{yy} = \frac{1}{E} [\sigma_{yy} - \nu(\sigma_{xx} + \sigma_{zz})], \quad \varepsilon_{zz} = \frac{1}{E} [\sigma_{zz} - \nu(\sigma_{xx} + \sigma_{yy})]$$

$$\gamma_{xy} = \frac{1}{G} \tau_{xy}, \quad \gamma_{yz} = \frac{1}{G} \tau_{yz}, \quad \gamma_{zx} = \frac{1}{G} \tau_{zx} \quad \left( G = \frac{E}{2(1+\nu)} \right)$$

$$\begin{bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{zz} \\ \gamma_{xy} \\ \gamma_{yz} \\ \gamma_{zx} \end{bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & -\nu & 0 & 0 & 0 \\ -\nu & 1 & -\nu & 0 & 0 & 0 \\ -\nu & -\nu & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 2(1+\nu) & 0 & 0 \\ 0 & 0 & 0 & 0 & 2(1+\nu) & 0 \\ 0 & 0 & 0 & 0 & 0 & 2(1+\nu) \end{bmatrix} \begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \tau_{xy} \\ \tau_{yz} \\ \tau_{zx} \end{bmatrix}$$

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \tau_{xy} \\ \tau_{yz} \\ \tau_{zx} \end{bmatrix} = \frac{E(1-\nu)}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1 & \nu/(1-\nu) & \nu/(1-\nu) & 0 & 0 & 0 \\ \nu/(1-\nu) & 1 & \nu/(1-\nu) & 0 & 0 & 0 \\ \nu/(1-\nu) & \nu/(1-\nu) & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{(1-2\nu)}{2(1-\nu)} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{(1-2\nu)}{2(1-\nu)} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{(1-2\nu)}{2(1-\nu)} \end{bmatrix} \begin{bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \varepsilon_{zz} \\ \gamma_{xy} \\ \gamma_{yz} \\ \gamma_{zx} \end{bmatrix}$$

## **2.11 Cylindrical polar co-ordinates**

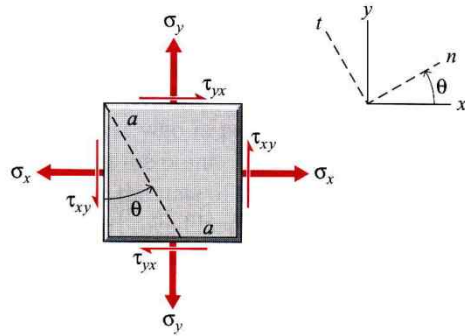
Refer to p.37, textbook

## **2.12 Geomechanics conventions for displacement, strain and stress**

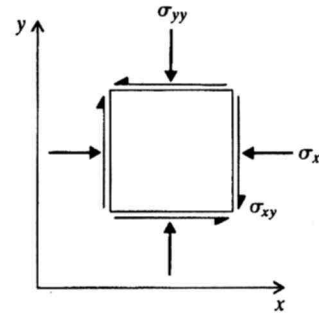
Compressive/contractile force/displacement are positive.  
Refer to Figure 2.12.



# Sign convention in rock mechanical problems



Two-dimensional



Three-dimensional

