Ship Stability

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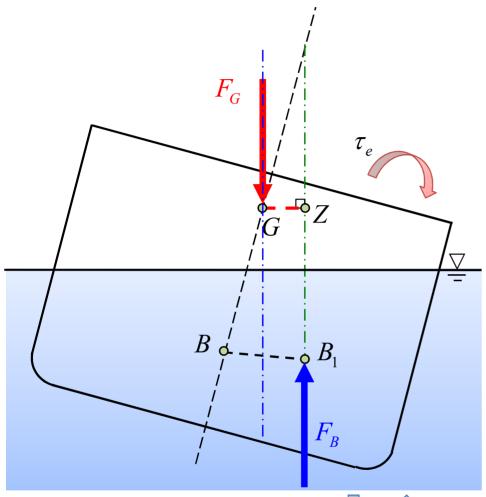
Ship Stability

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Ch. 4 Initial Transverse Stability

Transverse Metacentric Height (GM)

Righting Arm (GZ, Restoring Arm)



G: Center of mass of a ship

 F_G : Gravitational force of a ship

B: Center of buoyancy in the previous state (before inclination)

 F_R : Buoyant force acting on a ship

 B_1 : New position of center of buoyancy after the ship has been inclined

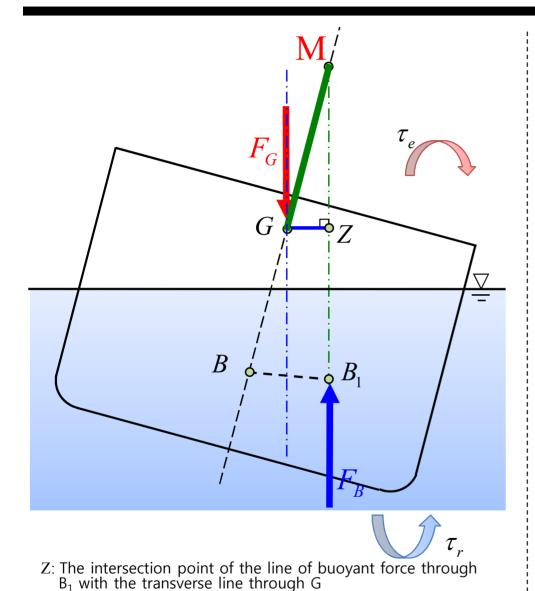
Z: The intersection point of a vertical line through the new position of the center of buoyancy(B₁) with the transversely parallel line to a waterline through the center of mass(G)

Transverse Righting Moment

$$\tau_{righting} = F_B \cdot \underline{GZ}$$

How can we find GZ in <u>small angle</u> of inclination?

$$\tau_{righting} = F_B \cdot GZ$$



Definition of M (Metacenter)

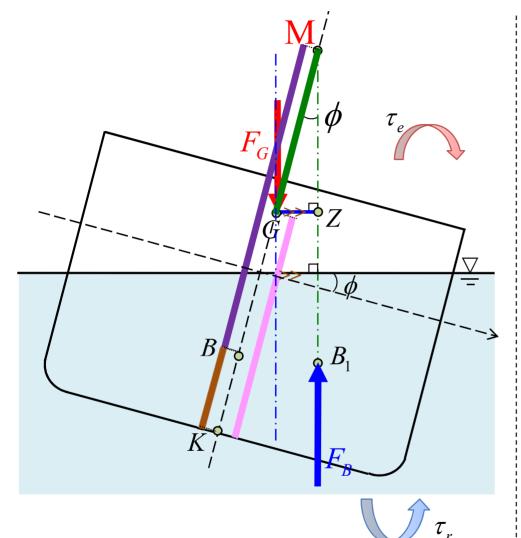
- The intersection point of the vertical line through the center of buoyancy at **previous position** (B) with the vertical line through the center of buoyancy at **new position** (B_1) **after inclination**
- The term **meta** was selected as a prefix for center because its Greek meaning implies **movement**. The **metacenter** therefore is a **moving center**.
- GM **→** Metacentric height
- From the figure, <u>GZ</u> can be obtained with assumption that *M* does not change within a <u>small angle of</u> inclination (about 7° to 10°), as below.

 $GZ \approx GM \cdot \sin \phi$

Metacentric Height (GM) of a Ship for Small Angle of Inclination

• Righting Moment

$$\tau_{righting} = F_B \cdot GZ$$



Z: The intersection point of the line of buoyant force through B₁ with the transverse line through G

M: Metacenter

GM: Metacentric height

 θ : Angle of heel

K: Keel, the lower most point on the ship vertical centre line

Righting Arm

$$GZ \approx GM \cdot \sin \phi$$

• From the geometrical configuration of the ship, *GM* is made up as follows:

$$GM = KB + BM - KG$$

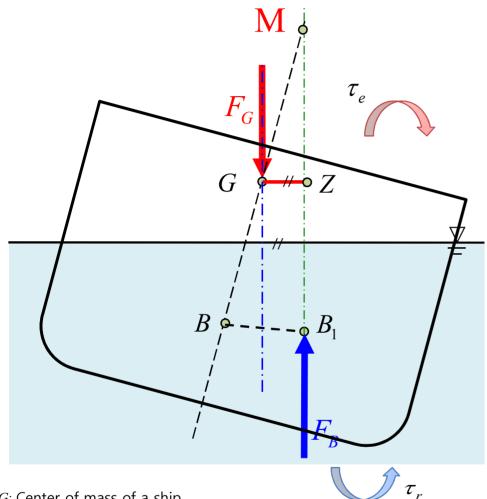
KB ≒ **51~52%** draft



Center of gravity of the ship

How can you get the value of the BM(metacentric radius)?

Righting Moment at Large Angle of Inclination



G: Center of mass of a ship

 F_G : Gravitational force of a ship

B: Center of buoyancy in the previous state (before inclination)

 F_R : Buoyant force acting on a ship

 B_i : New position of center of buoyancy after the ship has been inclined

Z: The intersection point of a vertical line through the new position of the center of buoyancy(B₁) with the transversely parallel line to a waterline through the center of mass(G)

- 1 Apply a large rotational angle to the ship by an external moment
- Transverse Righting Moment

$$\tau_{righting} = F_B \cdot GZ$$
Righting Arm

• The use of metacentric height(GM) as the righting arm is **not valid for a ship** at a large angle of inclination.



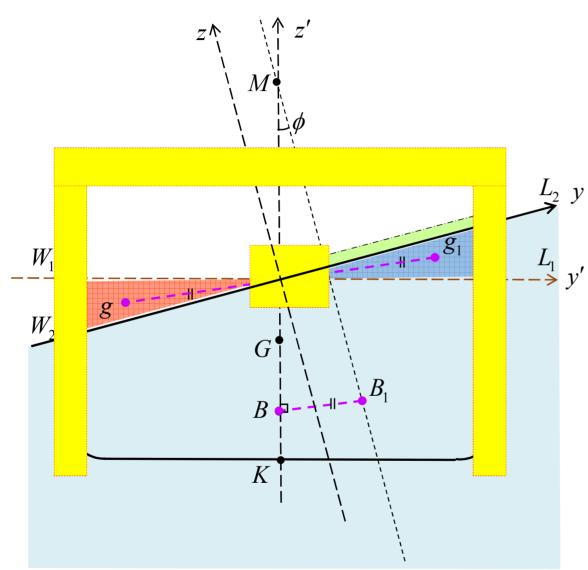
To determine the righting arm "GZ" of the ship at a large angle of inclination, it is necessary to know the accurate GZ which corresponds to the distance from the center of mass(G) to the vertical line through the new position of the center of buoyancy(B_1).

$$GZ \neq GM \cdot \sin \phi$$

for a large angle of inclination

Derivation of Transverse Metacentric Radius (BM_T)

Derivation of BM_T (1/12) (BM_T: Transverse Metacentric Radius)



B: Center of buoyancy in the previous state (before inclination)

 B_{i} : Center of buoyancy in the present state (after inclination)

M: Intersection of the line of the buoyant force through B_1 in the present state with the line of the buoyant force B in the previous state

Let us derive the transverse metacentric radius "BM" in case of a **wall-sided ship** with a simple section shape.

Wall sided ship

When a ship has a perpendicular side shell to water plane, the ship is called as "wall-sided ship".

Assumption

1. Wall sided ship

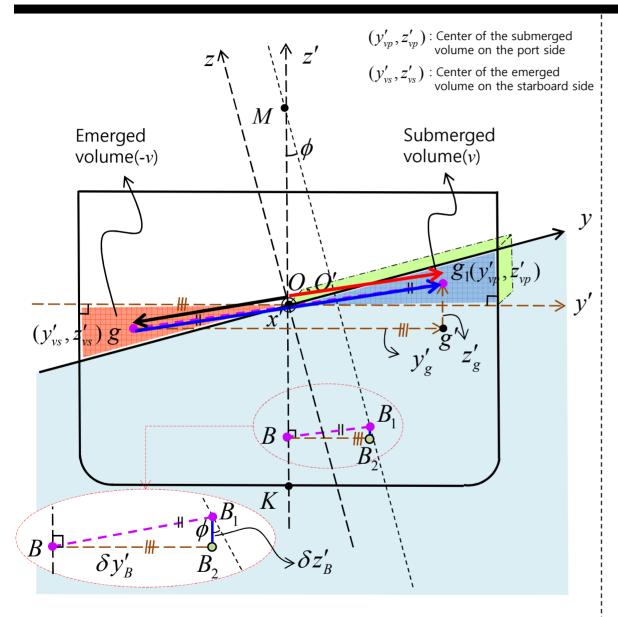
- When the ship is inclined, the submerged volume is the same as the emerged volume without any change in the displacement volume.
- 2. Main deck is not submerged.
- 3. Axis of inclination do not change.
 - In this case, the axis of inclination passes through the origin "O" of the inertial frame.

Derivation of BM_T (2/12)

∇: Displacement volume

v: Changed displacement volume (wedge)

 $BB_1(\delta y_B^i, \delta z_B^i)$: Distance between the initial center of buoyancy and the changed B_1 $gg_1(y_a^i, z_a^i)$: Distance between the center of wedges



Translation of the center of buoyancy caused by the movement of the small volume ν

$$\begin{cases} \delta y_B' \cdot \nabla = y_g' \cdot v & \text{(1)} \\ \delta z_B' \cdot \nabla = z_g' \cdot v & \text{(2)} \end{cases}$$

, where $\,v\,$ is the each volume of the submerged and emerged volume.

 ∇ is total volume of the ship.

$$\overrightarrow{O'g} + \overrightarrow{gg_1} = \overrightarrow{O'g_1}$$

$$\overrightarrow{gg_1} = \overrightarrow{O'g_1} - \overrightarrow{O'g}$$

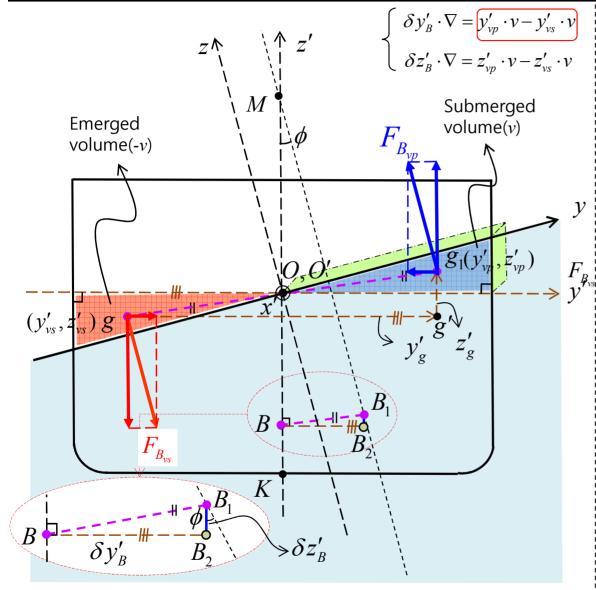
$$(y'_g, z'_g) = (y'_{vp}, z'_{vp}) - (y'_{vs}, z'_{vs})$$

$$y'_{g} = y'_{vp} - y'_{vs}$$
 (3)
 $z'_{g} = z'_{vp} - z'_{vs}$ (4)

Substituting Eq. (3) and (4) into the Eq. (1) and (2), respectively.

$$\begin{cases} \delta y_B' \cdot \nabla = y_{vp}' \cdot v - y_{vs}' \cdot v \\ \delta z_B' \cdot \nabla = z_{vp}' \cdot v - z_{vs}' \cdot v \end{cases}$$

Derivation of BM_T (3/12)



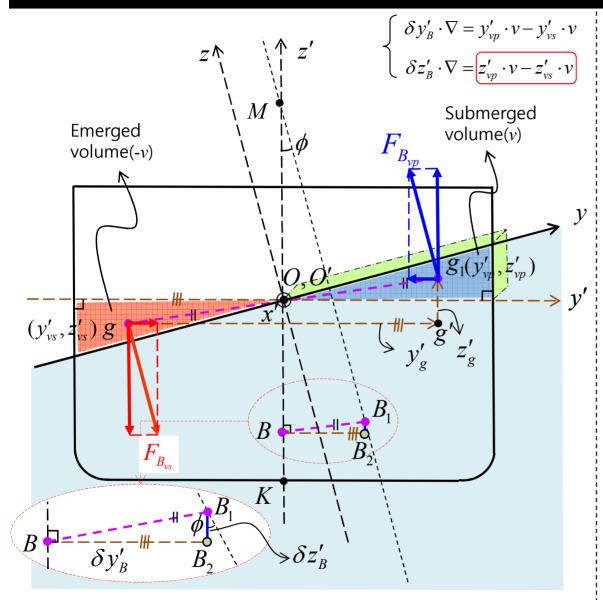
 (y'_{vp}, z'_{vp}) : The center of the changed volume on the port side

 (y'_{vs}, z'_{vs}) : The center of the changed volume on the starboard side

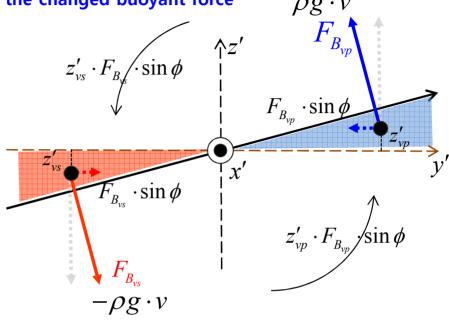
Moment about x' axis due to the z' component of the changed buoyant force $\cdot\cos\phi$ y'_{vs} $F_{R_{vs}} \cdot \cos \phi$ $-\rho g \cdot v$ $\delta y_{B}' \cdot \rho g \cdot \nabla \cdot \cos \phi = y_{vp}' \cdot F_{B_{vs}} \cdot \cos \phi + y_{vs}' \cdot F_{B_{vs}} \cdot \cos \phi$ $\delta y_B' \cdot \rho g \cdot \nabla = y_{vp}' \cdot F_{B_{vp}} + y_{vs}' \cdot F_{B_{vs}}$ $\int_{\mathcal{S}} F_{B_{vs}} = -\rho g \cdot v, F_{B_{vp}} = \rho g \cdot v$ $\rho g \cdot \delta y_B' \cdot \nabla = y_{vp}' \cdot \rho g \cdot v - y_{vs}' \cdot \rho g \cdot v$ $\delta y_B' \cdot \nabla = y_{vp}' \cdot v - y_{vs}' \cdot v$

> : Moment about x' axis due to the z' component of the changed volume

Derivation of BM_T (4/12)



Moment about x' axis due to the y' component of the changed buoyant force $Q \circ V$



$$\delta z_{\scriptscriptstyle B}' \cdot \rho g \cdot \nabla \cdot \sin \phi = z_{\scriptscriptstyle vp}' \cdot F_{\scriptscriptstyle B_{\scriptscriptstyle vp}} \cdot \sin \phi + z_{\scriptscriptstyle vs}' \cdot F_{\scriptscriptstyle B_{\scriptscriptstyle vs}} \cdot \sin \phi$$

$$\delta z_{B}' \cdot \rho g \cdot \nabla = z_{vp}' \cdot F_{B_{vp}} + z_{vs}' \cdot F_{B_{vs}}$$

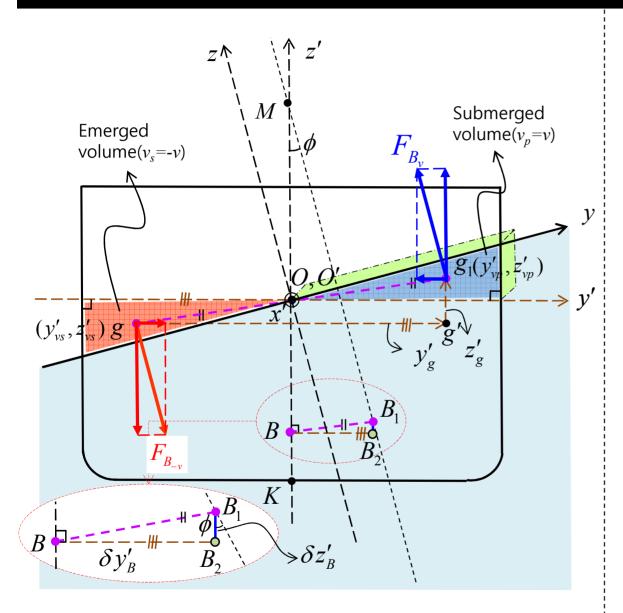
$$\downarrow F_{B_{vs}} = -\rho g \cdot v, F_{B_{vp}} = \rho g \cdot v$$

$$\delta z_B' \cdot \rho g \cdot \nabla = \rho g \cdot z_{vp}' \cdot v - \rho g \cdot z_{vs}' \cdot v$$

$$\delta z_B' \cdot \nabla = z_{vp}' \cdot v - z_{vs}' \cdot v$$

: Moment about x' axis due to the y' component of the changed volume

Derivation of BM_T (5/12)



$$\delta y_B' \cdot \nabla = y_{vp}' \cdot v - y_{vs}' \cdot v$$

: Moment about x' axis due to the z' component of the changed volume

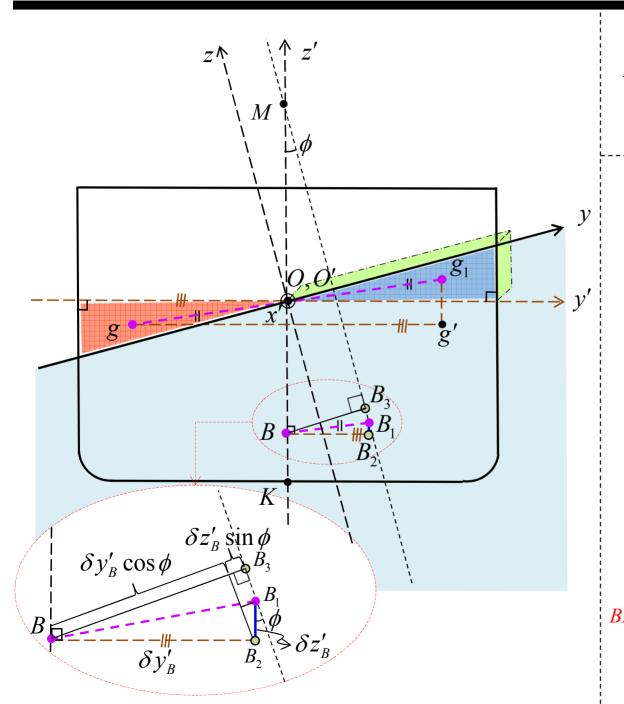
$$\delta z_B' \cdot \nabla = z_{vp}' \cdot v - z_{vs}' \cdot v$$

: Moment about x' axis due to the y' component of the changed volume

$$\nabla \leftarrow v = v_p, -v = v_s$$

$$\begin{cases}
\delta y_B' \cdot \nabla = y_{vp}' \cdot v_p + y_{vs}' \cdot v_s \\
\delta z_B' \cdot \nabla = z_{vp}' \cdot v_p + z_{vs}' \cdot v_s
\end{cases}$$

Derivation of BM_{T} (6/12)



$$\int \delta y_B' \cdot \nabla = y_{vp}' \cdot v_p + y_{vs}' \cdot v_s$$
: Moment about x' axis due to the z' component of the changed volume

: Moment about x' axis of the changed volume

$$\delta z_B' \cdot \nabla = z_{vp}' \cdot v_p + z_{vs}' \cdot v_s$$
: Moment about x' axis

due to the y' component of the changed volume

What we want to find is BM.

$$BM \sin \phi = BB_{3}$$

$$BM = \frac{BB_{3}}{\sin \phi}$$

$$= \frac{1}{\sin \phi} (\delta y'_{B} \cos \phi + \delta z'_{B} \sin \phi)$$

$$= \frac{\cos \phi}{\sin \phi} (\delta y'_{B} + \delta z'_{B} \frac{\sin \phi}{\cos \phi})$$

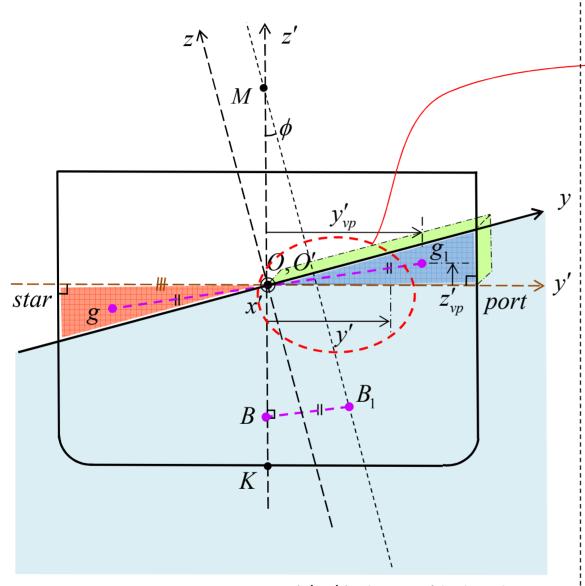
$$= \frac{1}{\tan \phi} (\delta y'_{B} + \delta z'_{B} \tan \phi)$$

$$\int \delta z'_{B} = (y'_{vp} \cdot v_{p} + y'_{vs} \cdot v_{s}) / \nabla$$

$$= \frac{1}{\tan \phi} (\frac{y'_{vp} \cdot v_{p} + y'_{vs} \cdot v_{s} + z'_{vs} \cdot v_{s}}{\nabla} + z'_{vs} \cdot v_{s}} \cdot \tan \phi)$$

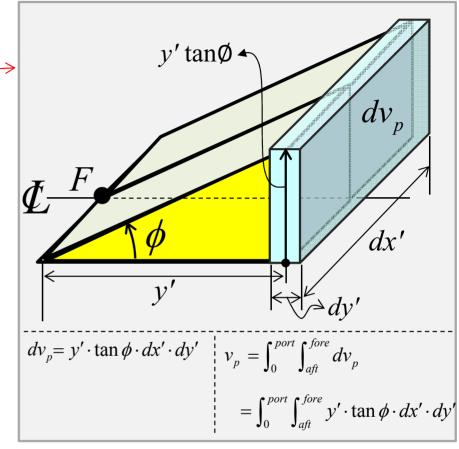
$$BM = \frac{1}{\nabla \cdot \tan \phi} (y'_{vp} \cdot v_{p} + y'_{vs} \cdot v_{s} + (z'_{vp} \cdot v_{p} + z'_{vs} \cdot v_{s}} \cdot \tan \phi)$$
Find!

Derivation of BM_T (7/12)
$$BM = \frac{1}{\nabla \cdot \tan \phi} \left(\frac{y'_{vp} \cdot v_p}{(A)} + \frac{y'_{vs} \cdot v_s}{(B)} + \left(\frac{z'_{vp} \cdot v_p}{(C)} + \frac{z'_{vs} \cdot v_s}{(D)} \right) \cdot \tan \phi \right)$$



 (y'_{vp}, z'_{vp}) : The center of the changed volume on the port side

 (y'_{vs}, z'_{vs}) : The center of the changed volume on the starboard side



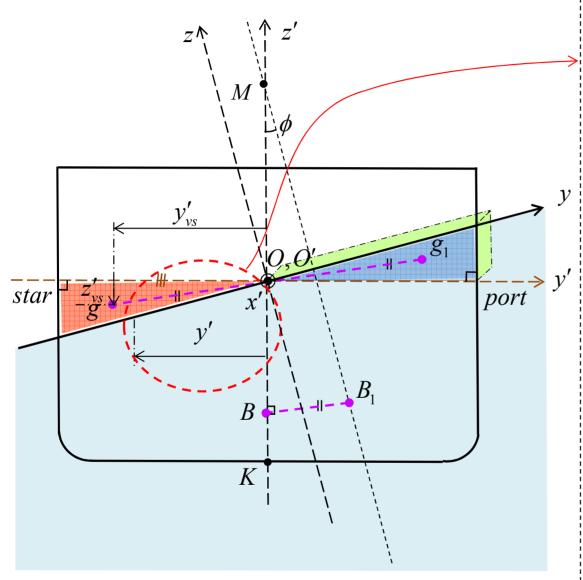
(A) $v_p \cdot y_{vp}'$: The moment about x'-axis due to the z' component of the changed volume of port side

$$= \int_{0}^{port} \int_{aft}^{fore} y' \cdot dv_{p}$$

$$= \int_{0}^{port} \int_{aft}^{fore} y' \cdot y' \cdot \tan \phi \cdot dx' \cdot dy'$$

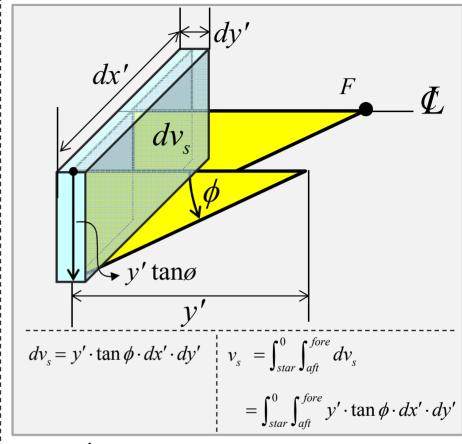
$$= \tan \phi \int_{0}^{port} \int_{aft}^{fore} y' \cdot y' \cdot dx' dy'$$

Derivation of BM_T (8/12)
$$\frac{BM}{\nabla \cdot \tan \phi} = \frac{1}{\nabla \cdot \tan \phi} \left(\frac{y'_{vp} \cdot v_p}{(A)} + \frac{y'_{vs} \cdot v_s}{(B)} + \left(\frac{z'_{vp} \cdot v_p}{(C)} + \frac{z'_{vs} \cdot v_s}{(D)} \right) \cdot \tan \phi \right)$$



 (y'_{vp}, z'_{vp}) : The center of the changed volume on the port side

 (y'_{vs}, z'_{vs}) : The center of the changed volume on the starboard side



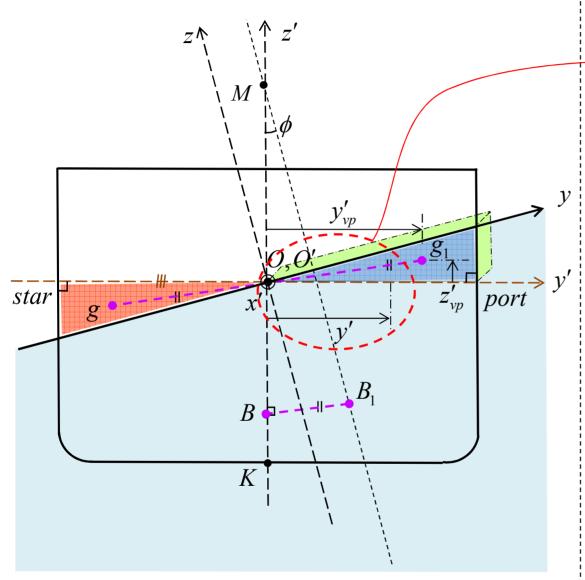
(B) $v_s \cdot y_{vs}'$: The moment about x'-axis due to the z' component of the changed volume of **starboard**

$$= \int_{star}^{0} \int_{aft}^{fore} y' \cdot dv_{s}$$

$$= \int_{star}^{0} \int_{aft}^{fore} y' \cdot y' \cdot \tan \phi \cdot dx' \cdot dy'$$

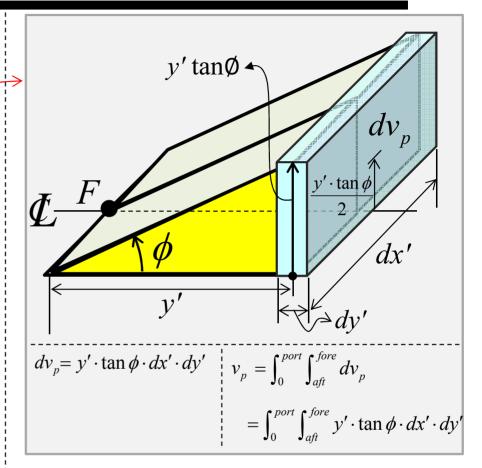
$$= \tan \phi \int_{star}^{0} \int_{aft}^{fore} y' \cdot y' \cdot dx' dy'$$

Derivation of BM_T (9/12)
$$BM = \frac{1}{\nabla \cdot \tan \phi} \left(\frac{y'_{vp} \cdot v_p}{(A)} + \frac{y'_{vs} \cdot v_s}{(B)} + \left(\frac{z'_{vp} \cdot v_p}{(C)} + \frac{z'_{vs} \cdot v_s}{(D)} \right) \cdot \tan \phi \right)$$



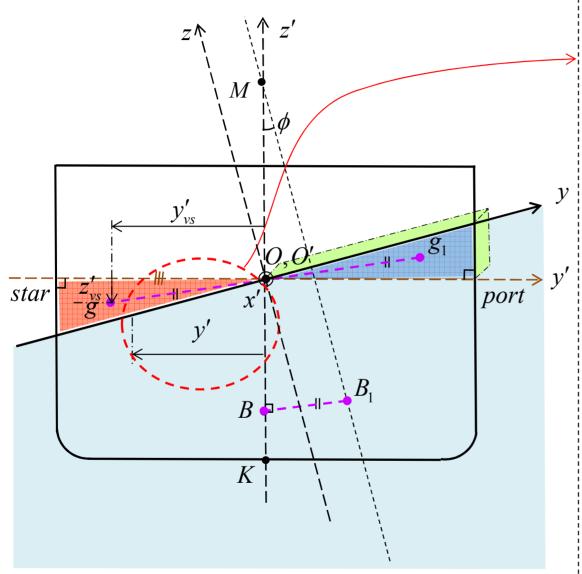
 (y'_{vp}, z'_{vp}) : The center of the changed volume on the port side

 (y'_{vs}, z'_{vs}) : The center of the changed volume on the starboard side



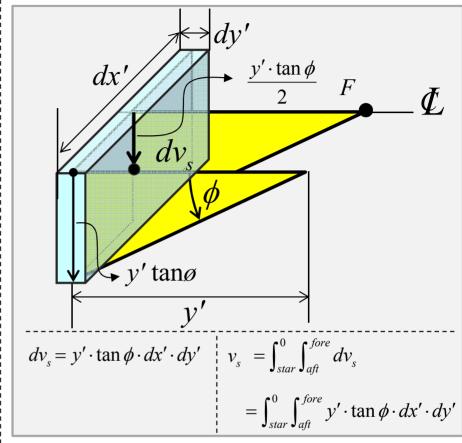
(C) $v_p \cdot z'_{vp}$: The moment about x'-axis due to the y' component of the changed volume of port side $= \int_0^{port} \int_{aft}^{fore} \frac{y' \cdot \tan \phi}{2} \cdot dv_p$ $= \int_0^{port} \int_{aft}^{fore} \frac{y' \cdot \tan \phi}{2} \cdot y' \cdot \tan \phi \cdot dx' \cdot dy'$ $= \frac{\tan^2 \phi}{2} \int_0^{port} \int_{aft}^{fore} y' \cdot y' \cdot dx' \cdot dy'$

Derivation of BM_T (10/12)^{BM} = $\frac{1}{\nabla \cdot \tan \phi} \left(\frac{y'_{vp} \cdot v_p}{(A)} + \frac{y'_{vs} \cdot v_s}{(B)} + \left(\frac{z'_{vp} \cdot v_p}{(C)} + \frac{z'_{vs} \cdot v_s}{(D)} \right) \cdot \tan \phi \right)$



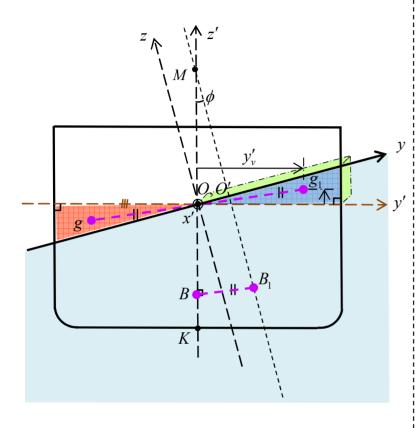
 (y'_{vp}, z'_{vp}) : The center of the changed volume on the port side

 (y'_{vs}, z'_{vs}) : The center of the changed volume on the starboard side



(D) $v_s \cdot z'_{vs}$: The moment about x'-axis due to the y' component of the changed volume of starboard side $= \int_{star}^{0} \int_{aft}^{fore} \frac{y' \cdot \tan \phi}{2} \cdot dv_s$ $= \int_{star}^{0} \int_{aft}^{fore} \frac{y' \cdot \tan \phi}{2} \cdot y' \cdot \tan \phi \cdot dx' \cdot dy'$ $= \frac{\tan^2 \phi}{2} \int_{star}^{0} \int_{aft}^{fore} y' \cdot y' \cdot dx' \cdot dy'$

Derivation of BM_T (11/12)^{BM} = $\frac{1}{\nabla \cdot \tan \phi} \left(\frac{y'_{vp} \cdot v_p}{(A)} + \frac{y'_{vs} \cdot v_s}{(B)} + \left(\frac{z'_{vp} \cdot v_p}{(C)} + \frac{z'_{vs} \cdot v_s}{(D)} \right) \cdot \tan \phi \right)$



(A)+(B) Moment about x' axis due to the z' component of the changed volume

$$y'_{vp} \cdot v_p + y'_{vs} \cdot v_s = \tan \phi \int_0^{port} \int_{aft}^{fore} y' \cdot y' \cdot dx' dy' + \tan \phi \int_{star}^0 \int_{aft}^{fore} y' \cdot y' \cdot dx' dy'$$

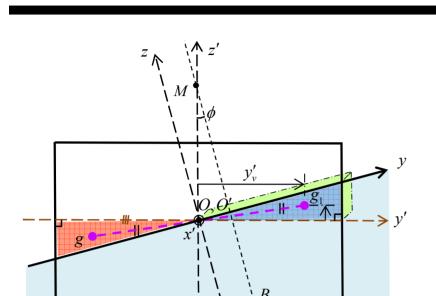
$$= \tan \phi \int_{star}^{port} \int_{aft}^{fore} y' \cdot y' \cdot dx' dy'$$

$$= \tan \phi \int_{star}^{port} \int_{aft}^{fore} y'^2 \cdot dx' dy'$$

$$= \tan \phi \cdot I_T$$

(C)+(D) Moment about x' axis due to the y' component of the changed volume

$$\begin{aligned} z'_{vp} \cdot v_p + z'_{vs} \cdot v_s &= \frac{\tan^2 \phi}{2} \int_0^{port} \int_{aft}^{fore} y' \cdot y' \cdot dx' \cdot dy' + \frac{\tan^2 \phi}{2} \int_{star}^0 \int_{aft}^{fore} y' \cdot y' \cdot dx' \cdot dy' \\ &= \frac{\tan^2 \phi}{2} \int_{star}^{port} \int_{aft}^{fore} y' \cdot y' \cdot dx' \cdot dy' \\ &= \frac{\tan^2 \phi}{2} \int_{star}^{port} \int_{aft}^{fore} y'^2 \cdot dx' \cdot dy' \\ &= \frac{\tan^2 \phi}{2} \cdot I_T \end{aligned}$$



Derivation of BM_T (12/12)^{BM} =
$$\frac{1}{\nabla \cdot \tan \phi} \left(\frac{y'_{vp} \cdot v_p}{(A)} + \frac{y'_{vs} \cdot v_s}{(B)} + \left(\frac{z'_{vp} \cdot v_p}{(C)} + \frac{z'_{vs} \cdot v_s}{(D)} \right) \cdot \tan \phi \right)$$

(A)+(B) Moment about
$$x'$$
 axis due to the z' component of the changed volume $y'_{vp} \cdot v_p + y'_{vs} \cdot v_s = \tan \phi \cdot I_T$

(C)+(D) Moment about x' axis due to the y' component of the changed volume $z'_{vp} \cdot v_p + z'_{vs} \cdot v_s = \frac{\tan^2 \phi}{2} \cdot I_T$

$$\mathbf{BM} = \frac{1}{\nabla \cdot \tan \phi} \left(\tan \phi \cdot I_T + \frac{1}{2} \cdot \tan^3 \phi \cdot I_T \right)$$

$$BM = \frac{I_T}{\nabla} \left(1 + \frac{1}{2} \tan^2 \phi \right)$$

$$BM = \frac{I_T}{\nabla} \left(1 + \frac{1}{2} \tan^2 \phi \right) \xrightarrow{\text{if } \phi \text{ is small}} BM = \frac{I_T}{\nabla}$$

which is generally known as BM.

This BM does not consider the change of the center of buoyancy in vertical direction.

In order to distinguish those, we will indicate two BM as follows.

$$BM_0 = \frac{I_T}{\nabla} (1 + \frac{1}{2} \tan^2 \phi)$$

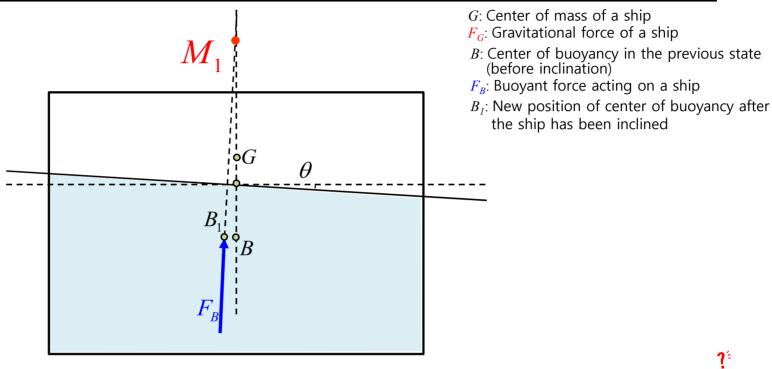
(Considering the change of the center of buoyancy in vertical direction)

$$BM = \frac{I_T}{\nabla}$$

(Without considering the change of the center of buoyancy in vertical direction)

Change of the Metacenter for Large Angle of Inclination

Metacenter (M)



Definition

Which vertical line is used to define the metacenter "M"?



Metacenter (M)

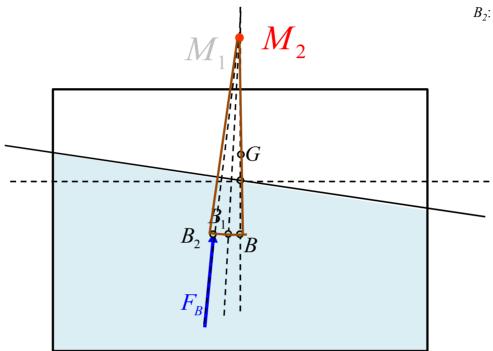
* Metacenter "M" is valid for small angle of inclination.

The **intersection point** of

a vertical line through the center of buoyancy at a previous position (B)

with a vertical line through the center of buoyancy at the present position (B_1)

Metacenter (M) at Small Angles



 B_2 : New position of center of buoyancy after the ship has been inclined with small angle

M remains at the same position for small angles of inclination, up to about $7\sim10$ degrees.

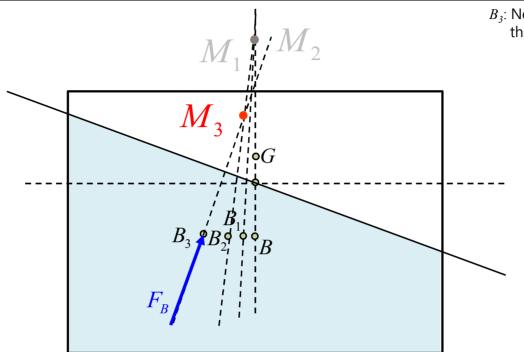
As the ship is inclined with a small angle, **B** moves **along the arc of a circle** whose center is at **M**.

The BM is **metacentric radius**.

The GM is **metacentric height**.

* The term **meta** was selected as a prefix for center because its Greek meaning implies **movement**. Therefore, the **metacenter** is a **"moving center"**.

Metacenter (M) at Large Angles

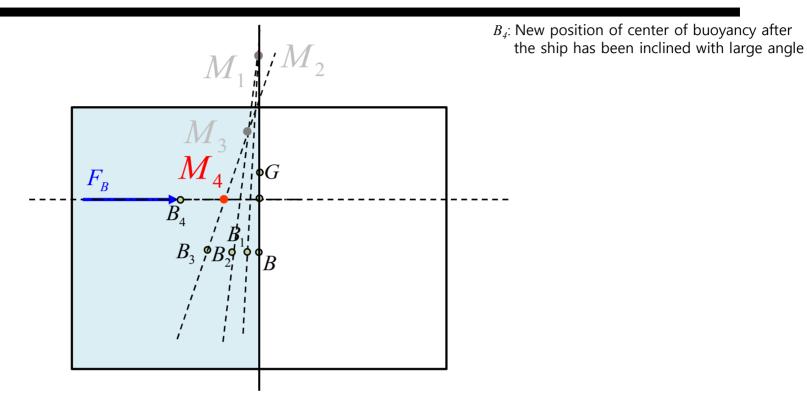


 B_3 : New position of center of buoyancy after the ship has been inclined with large angle

M does not remain in the same position for large angles of inclination over 10 degrees.

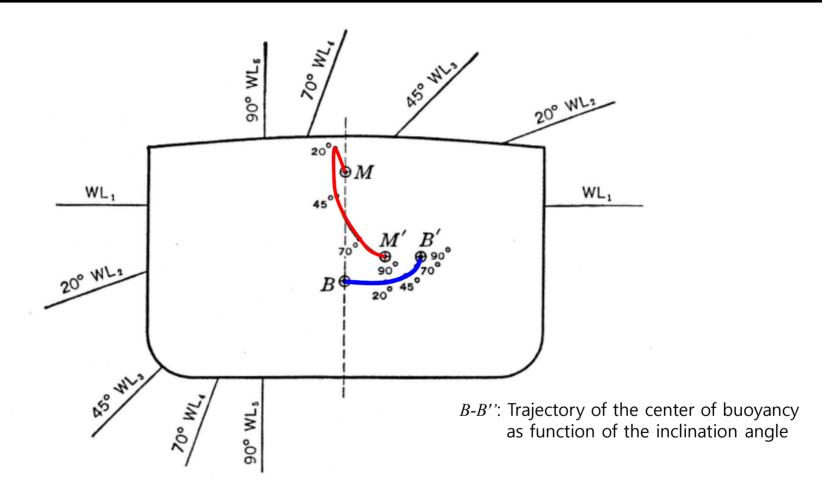
Thus, the metacenter, M, is only valid for a small angle of inclination.

Meaning of Metacenter (M)



The term **meta** was selected as a prefix for center because its Greek meaning implies **movement**. The **metacenter** therefore is a "**moving center**".

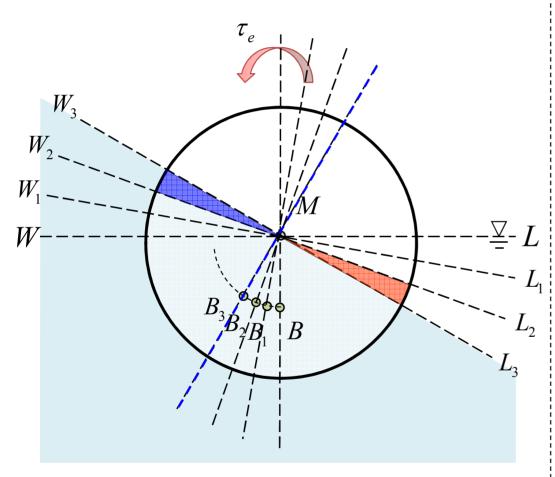
Example of Metacenter (M)



Typical locus of metacenter and centers of buoyancy for an average form merchant ship

Metacenter (M) of Circular Section

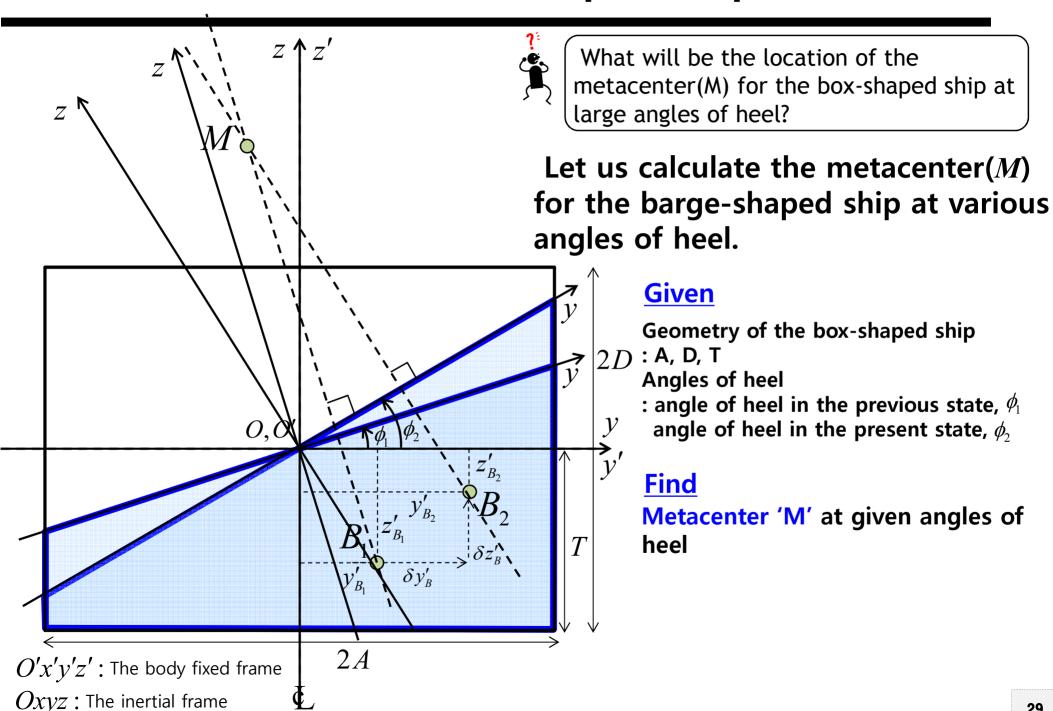
Metacenter (*M***)** : The <u>intersection point</u> of a <u>vertical line through the center of buoyancy in the <u>previous</u> <u>state</u> with a <u>vertical line through the center of buoyancy in the <u>present state</u>.</u></u>

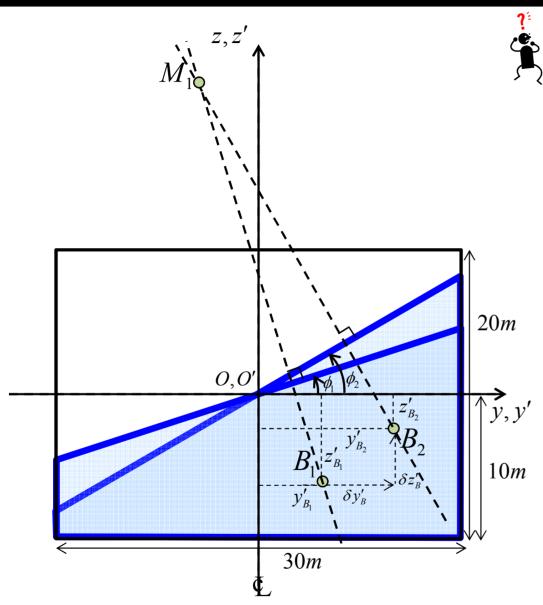


This figure shows a ship with a circular section.

The M does not move either vertically or off the center line.

Since the shape of the immersed section (under water plane) remains the same, the position of the metacenter \mathbf{M} has the same value regardless of the inclination.





What will be the location of the metacenter(M) for the box-shaped ship at large angles of heel?

O'x'y'z': The body fixed frame

- Center of Buoyancy at a Given Angle of Heel

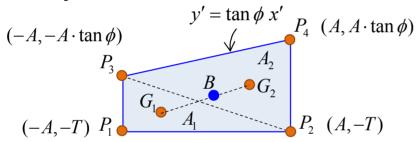


What will be the location of the metacenter(M) for the box-shaped ship at large angles of heel?

(1) Center of Buoyancy(B) at a given angle of heel ϕ

X Assumption: Deck will not be immersed and the bottom will not emerge.

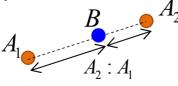
In the body fixed reference frame,

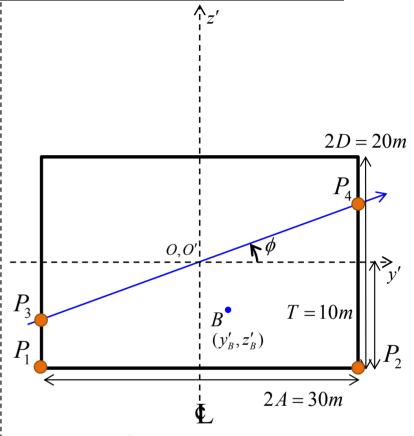


- Area of $A_1 = (T A \cdot \tan \phi) \cdot 2A \cdot \frac{1}{2}$ Area of $A_2 = (T + A \cdot \tan \phi) \cdot 2A \cdot \frac{1}{2}$
- Centroid of A₁(G₁) = $\left(\frac{-A+A-A}{3}, \frac{-T-T-A \cdot \tan \phi}{3}\right)$
- Centroid of $A_2(G_2) = \left(\frac{A-A+A}{3}, \frac{-T+A\cdot\tan\phi-A\cdot\tan\phi}{3}\right)$
- Centroid of $P_1P_2P_3P_4$ = Center of Buoyancy (B)

$$= \frac{A_1}{A_1 + A_2} G_1 + \frac{A_2}{A_1 + A_2} G_2$$

$$= \left(\frac{A^2 \tan \phi}{3T}, \frac{-3T^2 + A^2 \tan \phi}{6T} \right)$$





- Geometry of the box-shaped ship
- : 2A=30m, 2D=20m, T=10m

O'x'y'z': The body fixed frame

- Vertical Line through the Center of Buoyancy at a Given Angle of Heel

(2) Vertical line through the center of buoyancy(B) at a given angle of heel ϕ

X Assumption: Deck will not be immersed and the bottom will not emerge.

- Slope of the vertical line = $-\frac{1}{\tan \phi}$
- Vertical line through the center of buoyancy(B)

$$z' = -\frac{1}{\tan \phi} (y' - y'_B) + z'_B$$

$$z' = -\frac{1}{\tan \phi} y' + \frac{1}{\tan \phi} y'_B + z'_B$$

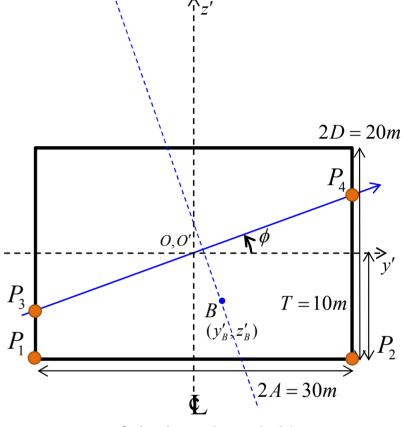
$$z' = -\frac{1}{\tan \phi} y' + \frac{1}{\tan \phi} \frac{A^2 \tan \phi}{3T} + \frac{-3T^2 + A^2 \tan \phi}{6T}$$

$$z' = -\frac{1}{\tan \phi} y' + \frac{A^2}{3T} + \frac{-3T^2 + A^2 \tan \phi}{6T}$$

$$z' = -\frac{1}{\tan \phi} y' + \frac{2A^2 - 3T^2 + A^2 \tan \phi}{6T}$$

 If the angle of heel is zero degree, the vertical line is given as follows.

$$z' = 0$$



- Geometry of the box-shaped ship
- : 2A=30m, 2D=20m, T=10m
- Center of buoyancy(B) at a angle of heel ϕ

$$= \left(\frac{A^2 \tan \phi}{3T}, \frac{-3T^2 + A^2 \tan \phi}{6T}\right) \cdot \cdot \cdot \cdot (1)$$

O'x'y'z': The body fixed frame

- Metacenter at Given Angles of Heel (1/4)

$M_{\rm 0}$: metacenter at upright condition

(3) Metacenter(M) at given angles of heel ϕ_1, ϕ_2

where, ϕ_1 is the angle of heel in the previous state ϕ_2 is the angle of heel in the present state

X Assumption: Deck will not be immersed and the bottom will not emerge.

From the equation (2), the vertical lines at each position are obtained as follows.

- Vertical line through the center of buoyancy(B₁) at a given angle of heel $\phi_{\rm l}$:

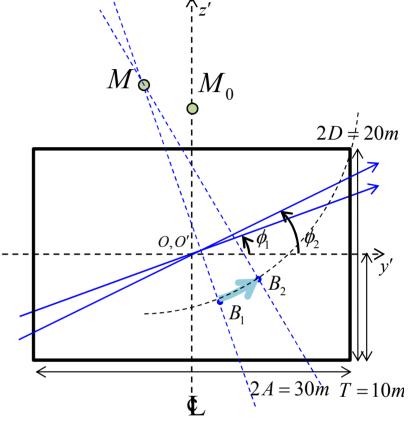
$$z' = -\frac{1}{\tan \phi_1} y' + \frac{2A^2 - 3T^2 + A^2 \tan \phi_1}{6T}$$

- Vertical line through the center of buoyancy(B₂) at a given angle of heel ϕ_2 :

$$z' = -\frac{1}{\tan \phi_2} y' + \frac{2A^2 - 3T^2 + A^2 \tan \phi_2}{6T}$$

If the two lines are described as z' = ay' + b and z' = cy' + d, the intersection point is obtained as follows.

- Intersection point:
$$\left(\frac{d-b}{a-c}, \frac{ad-bc}{a-c}\right)$$



- Geometry of the box-shaped ship
- : 2A=30m, 2D=20m, T=10m
- Center of buoyancy(B) at a angle of heel ϕ

$$= \left(\frac{A^2 \tan \phi}{3T}, \frac{-3T^2 + A^2 \tan \phi}{6T}\right) \cdot \cdot \cdot \cdot (1)$$

- Vertical line through the center of buoyancy (B) at a angle of heel ϕ

$$z' = -\frac{1}{\tan \phi} y' + \frac{2A^2 - 3T^2 + A^2 \tan \phi}{6T} \cdot \dots (2)$$

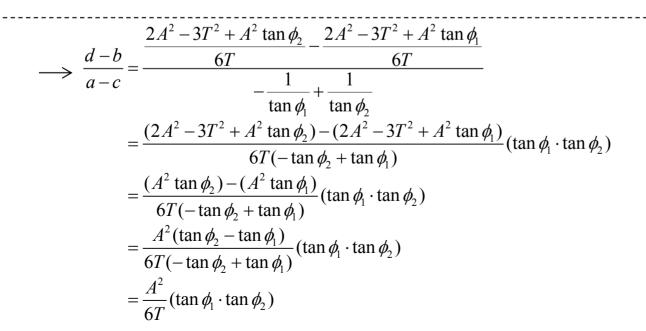
O'x'y'z': The body fixed frame

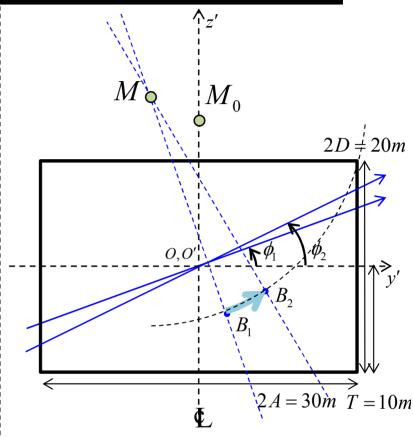
- Metacenter at Given Angles of Heel (2/4)

 M_0 : metacenter at upright condition

(3) Metacenter(M) at given angles of heel ϕ_1, ϕ_2 ,

where, ϕ_1 is the angle of heel in the previous state ϕ_2 is the angle of heel in the present state





- Geometry of the box-shaped ship
- : 2A=30m, 2D=20m, T=10m
- Center of buoyancy(B) at a angle of heel ϕ

$$= \left(\frac{A^2 \tan \phi}{3T}, \frac{-3T^2 + A^2 \tan \phi}{6T}\right) \cdot \cdot \cdot \cdot (1)$$

- Vertical line through the center of buoyancy (B) at a angle of heel ϕ

$$z' = -\frac{1}{\tan \phi} y' + \frac{2A^2 - 3T^2 + A^2 \tan \phi}{6T} \cdot \dots (2)$$

O'x'y'z': The body fixed frame

- Metacenter at Given Angles of Heel (3/4)

M_0 : metacenter at upright condition

(3) Metacenter(M) at given angles of heel ϕ_1,ϕ_2 ,

where, ϕ_1 is the angle of heel in the previous state ϕ_2 is the angle of heel in the present state

$$\left(\frac{d-b}{a-c}, \frac{ad-bc}{a-c}\right), \frac{d-b}{a-c} = \frac{A^2}{6T}(\tan\phi_1 \cdot \tan\phi_2)$$
 where , $a = -\frac{1}{\tan\phi_1}$, $b = \frac{2A^2 - 3T^2 + A^2 \tan\phi_1}{6T}$, $c = -\frac{1}{\tan\phi_2}$, $d = \frac{2A^2 - 3T^2 + A^2 \tan\phi_2}{6T}$

$$\Rightarrow \frac{ad - bc}{a - c} = \frac{\left(-\frac{1}{\tan\phi_{1}}\right) \frac{2A^{2} - 3T^{2} + A^{2} \tan\phi_{2}}{6T} - \frac{2A^{2} - 3T^{2} + A^{2} \tan\phi_{1}}{6T} \left(-\frac{1}{\tan\phi_{2}}\right)}{-\frac{1}{\tan\phi_{1}} + \frac{1}{\tan\phi_{2}}}$$

$$= \frac{\left(-\tan\phi_{2}\right) \frac{2A^{2} - 3T^{2} + A^{2} \tan\phi_{2}}{6T} - \frac{2A^{2} - 3T^{2} + A^{2} \tan\phi_{1}}{6T} \left(-\tan\phi_{1}\right)}{\left(-\tan\phi_{2} + \tan\phi_{1}\right)}$$

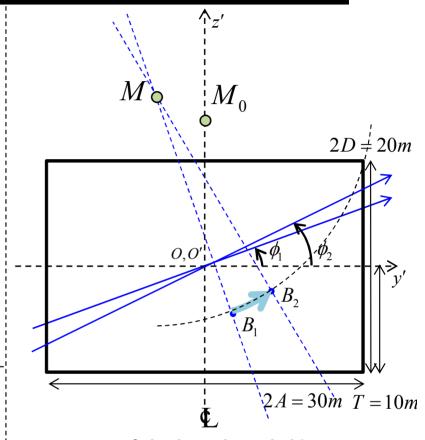
$$= \frac{-\tan\phi_{2}(2A^{2} - 3T^{2} + A^{2} \tan\phi_{2}) + \tan\phi_{1}(2A^{2} - 3T^{2} + A^{2} \tan\phi_{1})}{6T(-\tan\phi_{2} + \tan\phi_{1})}$$

$$= \frac{-\tan\phi_{2}(2A^{2} - 3T^{2} + A^{2} \tan\phi_{2}) + \tan\phi_{1}(2A^{2} - 3T^{2} + A^{2} \tan\phi_{1})}{6T(-\tan\phi_{2} + \tan\phi_{1})}$$

$$= \frac{\left(-\tan\phi_{2} + \tan\phi_{1}\right)(2A^{2} - 3T^{2}) + A^{2}\left(-\tan\phi_{2} + \tan\phi_{1}\right)}{6T(-\tan\phi_{2} + \tan\phi_{1})}$$

$$= \frac{\left(-\tan\phi_{2} + \tan\phi_{1}\right)(2A^{2} - 3T^{2}) + A^{2}\left(-\tan\phi_{2} + \tan\phi_{1}\right)(\tan\phi_{2} + \tan\phi_{1})}{6T(-\tan\phi_{2} + \tan\phi_{1})}$$

$$= \frac{\left(2A^{2} - 3T^{2}\right) + A^{2}\left(\tan\phi_{2} + \tan\phi_{1}\right)}{6T}$$



- Geometry of the box-shaped ship
- : 2A=30m, 2D=20m, T=10m
- Center of buoyancy(B) at a angle of heel ϕ

$$= \left(\frac{A^2 \tan \phi}{3T}, \frac{-3T^2 + A^2 \tan \phi}{6T}\right) \cdot \cdot \cdot \cdot (1)$$

- Vertical line through the center of buoyancy (B) at a angle of heel ϕ

$$z' = -\frac{1}{\tan \phi} y' + \frac{2A^2 - 3T^2 + A^2 \tan \phi}{6T} \cdot \dots \cdot (2)$$

O'x'y'z': The body fixed frame

- Metacenter at Given Angles of Heel (4/4)

 M_0 : metacenter at upright condition

(3) Metacenter(M) at given angles of heel ϕ_1, ϕ_2 , where, ϕ_1 is the angle of heel in the previous state ϕ_2 is the angle of heel in the present state

- Intersection point:

$$\left(\frac{d-b}{a-c}, \frac{ad-bc}{a-c}\right) = \left(\frac{A^2}{6T} (\tan \phi_1 \cdot \tan \phi_2), \frac{(2A^2 - 3T^2) + A^2 (\tan \phi_2 + \tan \phi_1)}{6T}\right)$$
where, $a = -\frac{1}{\tan \phi_1}$, $b = \frac{2A^2 - 3T^2 + A^2 \tan \phi_1}{6T}$, $c = -\frac{1}{\tan \phi_2}$, $d = \frac{2A^2 - 3T^2 + A^2 \tan \phi_2}{6T}$

By definition of the metacenter(M), the metacenter is the intersection point.

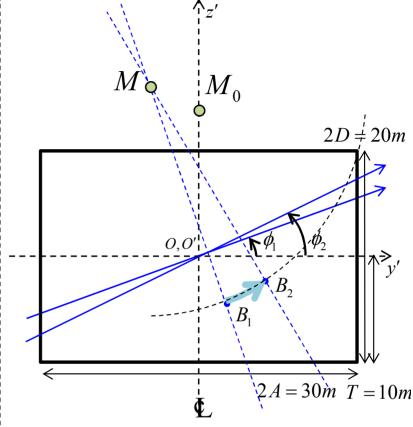
$$\therefore \text{ Metacenter } (M) = \left(\frac{A^2}{6T} (\tan \phi_1 \cdot \tan \phi_2), \frac{(2A^2 - 3T^2) + A^2 (\tan \phi_2 + \tan \phi_1)}{6T}\right)$$



If
$$\phi_1, \phi_2 \ll 1$$
, $\tan \phi_1 \simeq 0$, $\tan \phi_2 \simeq 0$

Thus, y' component of metacenter is approximately zero.

$$\frac{A^2}{6T}(\tan\phi_1\cdot\tan\phi_2)\approx 0$$



- Geometry of the box-shaped ship
- : 2A=30m, 2D=20m, T=10m
- Center of buoyancy(B) at a angle of heel ϕ

$$= \left(\frac{A^2 \tan \phi}{3T}, \frac{-3T^2 + A^2 \tan \phi}{6T}\right) \cdot \cdot \cdot \cdot (1)$$

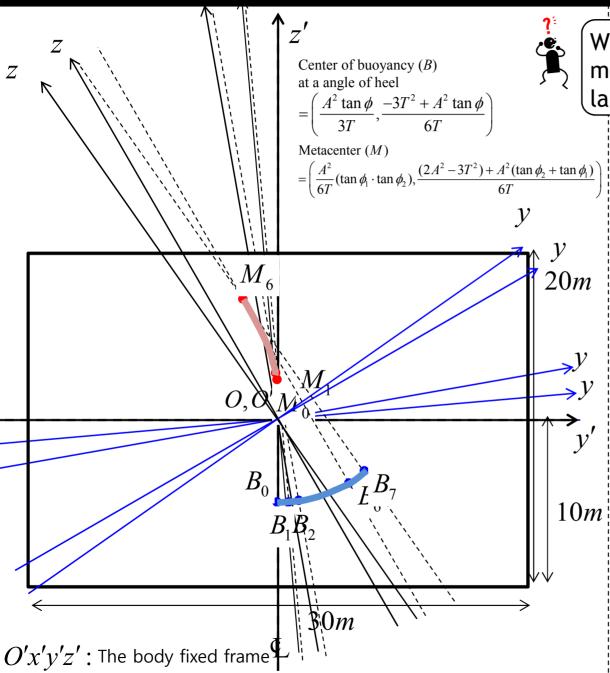
 Vertical line through the center of buoyancy (B) at a angle of heel ϕ

$$z' = -\frac{1}{\tan \phi} y' + \frac{2A^2 - 3T^2 + A^2 \tan \phi}{6T} \cdot \dots (2)$$

O'x'y'z': The body fixed frame

- Result (1/2) z

Oxvz: The inertial frame



What will be the location of the metacenter(M) for the box-shaped ship at large angles of heel?

 ϕ : angle of heel at present position

 ϕ_2 : angle of heel at next position

 $\downarrow(y_B', z_B')$: center of buoyancy at present position(B)

 (y'_{M}, z'_{M}) : metacenter(M)

The calculation result of the metacenter(M) and center of buoyancy(B) is as follows;

$$\phi_1 = 0^{\circ}, \phi_2 = 5^{\circ}: (y'_{B_0}, z'_{B_0}) = (0.000, -5.000), (y'_{B_1}, z'_{B_1}) = (0.656, -4.971)$$

$$(y'_{M_0}, z'_{M_0}) = (0.000, 2.529)$$

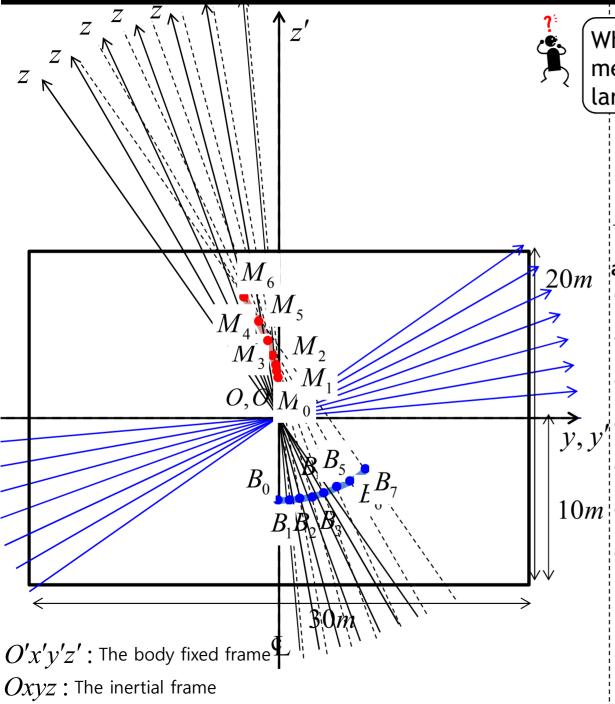
$$\phi_1 = 5^{\circ}, \phi_2 = 10^{\circ}: \quad (y'_{B_1}, z'_{B_1}) = (0.656, -4.971), \quad (y'_{B_2}, z'_{B_2}) = (1.322, -4.883)$$
$$(y'_{M_1}, z'_{M_1}) = (-0.015, 2.703)$$

: At small angle of heel, y' component of metacenter is approximately zero.

$$\phi_1 = 30^{\circ}, \phi_2 = 35^{\circ}: (y'_{B_6}, z'_{B_6}) = (4.330, -3.750), (y'_{B_7}, z'_{B_7}) = (5.252, -3.161)$$
$$(y'_{M_6}, z'_{M_6}) = (-3.392, 9.182)$$

: At large angle of heel, y' component of metacenter is not negligible.

- Result (2/2) z



What will be the location of the metacenter(M) for the box-shaped ship at large angles of heel?

 ϕ : angle of heel at present position

 ϕ_2 : angle of heel at next position

 (y'_B, z'_B) : center of buoyancy at present position(B)

 (y'_M, z'_M) : metacenter(M)

The calculation result of the metacenter(M) and center of buoyancy(B) is as follows;

$$\phi_1 = 0^{\circ}, \phi_2 = 5^{\circ}: \quad (y'_{B_0}, z'_{B_0}) = (0.000, -5.000), \quad (y'_{B_1}, z'_{B_1}) = (0.656, -4.971)$$

$$(y'_{M_0}, z'_{M_0}) = (0.000, 2.529)$$

$$\phi_1 = 5^{\circ}, \phi_2 = 10^{\circ}: (y'_{B_1}, z'_{B_1}) = (0.656, -4.971), (y'_{B_2}, z'_{B_2}) = (1.322, -4.883)$$

$$(y'_{M_1}, z'_{M_1}) = (-0.015, 2.703)$$

$$|\phi_1 = 10^\circ, \phi_2 = 15^\circ : (y'_{B_2}, z'_{B_2}) = (1.322, -4.883), (y'_{B_3}, z'_{B_3}) = (2.010, -4.731)$$

$$(y'_{M_2}, z'_{M_2}) = (-0.079, 3.063)$$

$$\phi_{1} = 15^{\circ}, \phi_{2} = 20^{\circ} : (y'_{B_{3}}, z'_{B_{3}}) = (2.010, -4.731), (y'_{B_{4}}, z'_{B_{4}}) = (2.730, -4.503)$$

$$(y'_{M_{5}}, z'_{M_{5}}) = (-0.231, 3.632)$$

$$[\phi_1 = 20^\circ, \phi_2 = 25^\circ : (y'_{B_4}, z'_{B_4}) = (2.730, -4.503), (y'_{B_5}, z'_{B_5}) = (3.497, -4.185)$$

$$(y'_{M_4}, z'_{M_4}) = (-1.054, 5.575)$$

$$10m | \phi_1 = 25^{\circ}, \phi_2 = 30^{\circ} : (y'_{B_5}, z'_{B_5}) = (3.497, -4.185), (y'_{B_6}, z'_{B_6}) = (4.330, -3.750) (y'_{M_5}, z'_{M_5}) = (-1.937, 7.105)$$

$$[\phi_1 = 30^\circ, \phi_2 = 35^\circ : (y'_{B_6}, z'_{B_6}) = (4.330, -3.750), (y'_{B_7}, z'_{B_7}) = (5.252, -3.161)$$

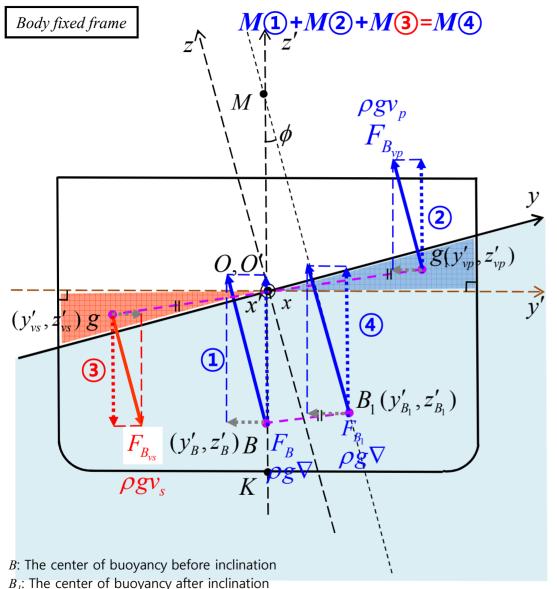
$$(y'_{M_6}, z'_{M_6}) = (-3.392, 9.182)$$

Another Approach to Derive the Following Formula

$$\begin{cases} \delta y_B' \cdot \nabla = y_{vp}' \cdot v_p + y_{vs}' \cdot v_s \\ \delta z_B' \cdot \nabla = z_{vp}' \cdot v_p + z_{vs}' \cdot v_s \end{cases}$$

Derivation of BM_T (1/2)

$$M = \begin{bmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ r_x & r_y & r_z \\ F_x & F_y & F_z \end{bmatrix} = \underbrace{\mathbf{i}(r_y \cdot F_z - r_z \cdot F_y)}_{\mathbf{M}_{O,x'}} + \mathbf{j}(-r_x \cdot F_z + r_z \cdot F_x) + \mathbf{k}(r_x \cdot F_y - r_y \cdot F_x)$$



For the convenience of calculation, the forces are decomposed in body fixed frame.

Moment about x' axis through point O

1. Moment about x' axis due to the z' component of the changed buoyant force

$$M(4) = M(1) + M(2) + M(3)$$

$$y'_{B_1} \cdot F_{B_1,z'} = y'_B \cdot F_{B,z'} + y'_{vp} \cdot F_{B_{vp},z'} + y'_{vs} \cdot F_{B_{vs},z'}$$

$$y'_{B_1} \cdot (\rho g \nabla \cdot \cos \phi)$$

$$= y'_B \cdot (\rho g \nabla \cdot \cos \phi) + y'_{vp} \cdot (\rho g v_p \cdot \cos \phi) + y'_{vs} (\rho g v_s \cdot \cos \phi)$$

$$(y'_{B_1} - y'_B) \cdot (\rho g \nabla \cdot \cos \phi) = y'_{vp} \cdot (\rho g v_p \cdot \cos \phi)$$

$$= \delta y'_B + y'_{vs} (\rho g v_s \cdot \cos \phi)$$

$$(\delta y'_B) \cdot (\rho g \nabla \cdot \cos \phi) = y'_{vp} \cdot (\rho g v_p \cdot \cos \phi)$$

$$+ y'_{vs} (\rho g v_s \cdot \cos \phi)$$

$$\delta y'_B \cdot \nabla = y'_{vp} \cdot v_B + y'_{vs} \cdot v_S$$

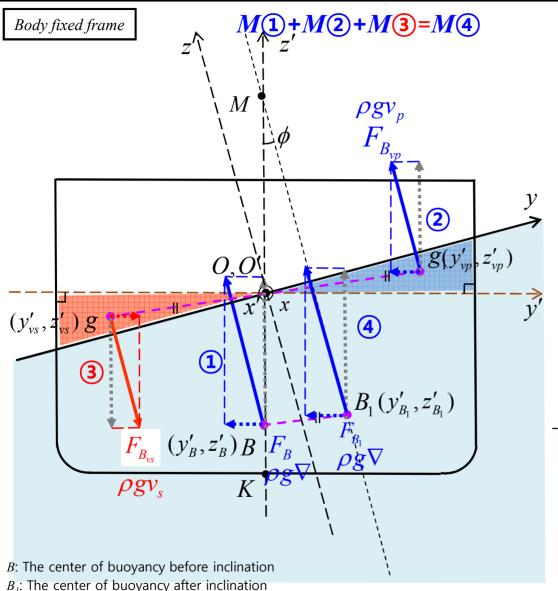
v: Changed displacement volume (wedge)

BB₁: Distance of changed center of buoyancy

gg₁: Distance of changed center of wedge

Derivation of BM_T (2/2)

$$M = \begin{bmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ r_x & r_y & r_z \\ F_x & F_y & F_z \end{bmatrix} = \underbrace{\mathbf{i}(r_y \cdot F_z - r_z \cdot F_y)}_{\mathbf{M}_{O,x'}} + \mathbf{j}(-r_x \cdot F_z + r_z \cdot F_x) + \mathbf{k}(r_x \cdot F_y - r_y \cdot F_x)$$



For the convenience of calculation, the forces are decomposed in body fixed frame.

Moment about x' axis through point O

2. Moment about x' axis due to the y' component of the changed buoyant force

$$M(4) = M(1) + M(2) + M(3)$$

$$-z'_{B_1} \cdot F_{B_1, y'} = -z'_{B} \cdot F_{B, y'} - z'_{vp} \cdot F_{B_{vp}, y'} - z'_{vs} \cdot F_{B_{vs}, y'}$$

$$-z'_{B_1} \cdot (\rho g \nabla \cdot \sin \phi)$$

$$= -z'_{B} \cdot (\rho g \nabla \cdot \sin \phi) - z'_{vp} \cdot (\rho g v_p \cdot \sin \phi) - z'_{vs} (\rho g v_s \cdot \sin \phi)$$

$$= \delta z'_{B} \qquad -z'_{vs} (\rho g v_s \cdot \sin \phi)$$

$$= \delta z'_{B} \qquad -z'_{vs} (\rho g v_s \cdot \sin \phi)$$

$$(\delta z'_{B}) \cdot (\rho g \nabla \cdot \sin \phi) = z'_{vp} \cdot (\rho g v_p \cdot \sin \phi)$$

$$+ z'_{vs} (\rho g v_s \cdot \sin \phi)$$

$$\delta z'_{B} \cdot \nabla = z'_{vp} \cdot v_p + z'_{vs} \cdot v_s$$

∇: Displacement volume

v. Changed displacement volume (wedge)

BB_i: Distance of changed center of buoyancy gg_i: Distance of changed center of wedge