

Continuity, Energy, and Momentum Equations







^{2/26} Chapter 4 Continuity, Energy, and Momentum Equations

Contents

4.1 Conservation of Matter in Homogeneous Fluids

4.2 The General Energy Equation

4.3 Linear Momentum Equation for Finite Control Volumes

4.4 The Moment of Momentum Equation for Finite Control Volumes

Objectives

- Apply finite control volume to get integral form of continuity, energy, and momentum equations
- Compare integral and point form equations
- Derive the simplified equations for continuity, energy, and momentum equations





4.2.1 The 1st law of thermodynamics

The 1st law of thermodynamics:

The difference between the <u>heat added to a system</u> of masses and the <u>work done by the system</u> depends only on the <u>initial and final states</u> of the system (\rightarrow <u>change in energy</u>).

 \rightarrow Conservation of energy







3/26

$$\delta Q - \delta W = dE \tag{4.14}$$



where δQ = heat added to the system from surroundings

- δW = work done by the system on its surroundings
- δE = increase in energy of the system





[Re]

- property of a system: position, velocity, pressure, temperature, mass, volume
- <u>state</u> of a system: condition as identified through properties of the system

Consider time rate of change

$$\frac{\delta Q}{dt} - \frac{\delta W}{dt} = \frac{dE}{dt}$$
(4.15)





- Work
 - $W_{pressure}$ = work of <u>normal stresses</u> acting on the system boundary
 - W_{shear} = work of <u>tangential stresses</u> done at the system boundary on adjacent external fluid in motion
 - W_{shaft} = shaft work done on a rotating element in the system
 - Energy

Consider *e* = energy per unit mass = *E/mass*

 e_u = internal energy associated with <u>fluid temperature</u> = u

 e_p = potential energy per unit mass = gh

where h = local elevation of the fluid

 e_q = kinetic energy per unit mass = $\frac{q^2}{2}$





$$u + \frac{p}{\rho} = \text{enthalpy}$$

$$e = e_u + e_p + e_q = u + gh + \frac{q^2}{2}$$

- Internal energy
 - = activity of the molecules comprising the substance
 - = force existing between the molecules
 - ~ depend on temperature and change in phase





4.2.2 General energy equation

$$\frac{\delta Q}{dt} - \frac{\delta W}{dt} = \frac{dE}{dt}$$
(4.15)

Consider work done

$$\frac{\delta W}{dt} = \frac{\delta W_{pressure}}{dt} + \frac{\delta W_{shaft}}{dt} + \frac{\delta W_{shear}}{dt}$$
(4.15a)

$$\delta W_{pressure}$$

dt

= <u>net rate</u> at which <u>work of pressure</u> is

<u>done by the fluid on the surroundings (유체 압력이 외부에 한 일)</u>

$$= \oint_{CS} p\left(\vec{q} \cdot d\vec{A}\right)$$





= <u>net flux of matter</u> through the control surface (외부에서 유입된 질량 _유출된 질량)

= flux in – flux out





 $\vec{q} \cdot d\vec{A} = \begin{pmatrix} \text{positive for outflow into CV} \\ \text{negative for inflow} \end{pmatrix}$

 $\vec{q} \cdot d\vec{A} = Q = L^3 / t$

$$p\left(\vec{q} \cdot d\vec{A}\right) = \frac{F}{L^2} \frac{L^3}{t} = FL/t = E/t$$

Thus, (4.15a) becomes

$$\frac{\delta W}{dt} = \oint_{CS} p\left(\vec{q} \cdot d\vec{A}\right) + \frac{\delta W_{shaft}}{dt} + \frac{\delta W_{shear}}{dt}$$



(4.15b)





Now, consider energy change term

- $\frac{dE}{dt}$ = total rate change of stored energy
 - = <u>net rate</u> of <u>energy flux</u> through C.V.

+ time rate of change inside C.V.

$$\frac{dE}{dt} = \oint_{CS} e\rho \left(\vec{q} \cdot d\vec{A} \right) + \frac{\partial}{\partial t} \int_{CV} \left(e\rho \, dV \right)$$



$$e = E / mass; \rho(\vec{q} \cdot d\vec{A}) = mass / time$$

 $e\rho(\vec{q} \cdot d\vec{A}) = E / t$





Substituting (4.15b) and (4.15c) into Eq. (4.15) yields

$$\frac{\delta Q}{dt} = \frac{\delta W_{shaft}}{dt} - \frac{\delta W_{shear}}{dt} - \oint_{cs} p(\vec{q} \cdot d\vec{A})$$

$$= \oint_{cs} e\rho(\vec{q} \cdot d\vec{A}) + \frac{\partial}{\partial t} \int_{cv} (e\rho \, dV)$$

$$\frac{\delta Q}{dt} - \frac{\delta W_{shaft}}{dt} - \frac{\delta W_{shear}}{dt}$$

$$= \oint_{cs} \left(\frac{p}{\rho} + e\right) \rho(\vec{q} \cdot d\vec{A}) + \frac{\partial}{\partial t} \int_{cv} (e\rho \, dV)$$
(4.16)





1*2/26*

Assume potential energy $e_p = gh$ (due to gravitational field of the earth)

Then
$$e = u + gh + \frac{q^2}{2}$$

Then, Eq. (4.17) becomes

$$\frac{\delta Q}{dt} - \frac{\delta W_{shaft}}{dt} - \frac{\delta W_{shear}}{dt}$$
$$= \oint_{CS} \left(\frac{p}{\rho} + u + gh + \frac{q^2}{2} \right) \rho \left(\vec{q} \cdot d\vec{A} \right) + \frac{\partial}{\partial t} \int_{CV} e\rho \, dV$$
(4.17)





Application: generalized apparatus

At boundaries normal to flow lines \rightarrow no shear

$$\rightarrow W_{shear} = 0$$

$$\frac{\delta Q}{dt} - \frac{\delta W_{shaft}}{dt} = \oint_{CS} \left(\frac{p}{\rho} + u + gh + \frac{q^2}{2} \right) \rho \left(\vec{q} \cdot d\vec{A} \right) + \frac{\partial}{\partial t} \int_{CV} e\rho \, dV$$

$$(4.18)$$

For steady motion,

$$\frac{\delta Q}{dt} - \frac{\delta W_{shaft}}{dt} = \oint_{CS} \left(\frac{p}{\rho} + u + gh + \frac{q^2}{2} \right) \rho \left(\vec{q} \cdot d\vec{A} \right)$$
(4.20)





- Effect of friction
- This effect is accounted for implicitly.
- This results in a <u>degradation of mechanical</u> <u>energy into heat</u> which may <u>be transferred</u> <u>away (*Q*, heat transfer)</u>, or may cause a temperature change
- → modification of internal energy
- Thus, Eq. (4.20) can be applied to both <u>viscous</u> <u>fluids and non-viscous fluids</u> (ideal frictionless processes).







4.2.3 1 D Steady flow equations

For flow through conduits, properties are uniform normal to the flow direction. \rightarrow one-dimensional steady flow

$$\begin{array}{c}
\frac{1}{1} & 2 \\
\downarrow \rightarrow v_{I} & \downarrow \rightarrow v_{2}
\end{array}$$

$$\begin{array}{c}
\frac{\partial Q}{\partial t} - \frac{\partial W_{shaft}}{\partial t} = \\
\frac{\partial Q}{\partial t} - \frac{\partial W_{shaft}}{\partial t} = \\
\oint_{CS} \left(u + \frac{p}{\rho} + gh + \frac{q^{2}}{2} \right) \rho \left(\vec{q} \cdot d\vec{A} \right)$$

Integrated form of Eq. (4.20) = (2) - (1)

$$\frac{heat}{dt} - \frac{\delta W_{shaft}}{dt} = \left[u + \frac{p}{\rho} + gh + \frac{V^2}{2}\right]_{\odot} \rho Q - \left[u + \frac{p}{\rho} + gh + \frac{V^2}{2}\right]_{\odot} \rho Q$$





where
$$\frac{V^2}{2}$$
 = average kinetic energy per unit mass
Section 1: $\int_1 \rho \left(\vec{q} \cdot d\vec{A} \right) = -\rho Q$ = mass flow rate into CV
Section 2: $\int_2 \rho \left(\vec{q} \cdot d\vec{A} \right) = \rho Q$ = mass flow from CV
Divide by ρQ (mass/time) $M = \rho Q dt$

$$\frac{heat \ transfer}{mass} = \left[u + \frac{p}{\rho} + gh + \frac{V^2}{2} \right]_{2} - \left[u + \frac{p}{\rho} + gh + \frac{V^2}{2} \right]_{1}$$





Divide by g

$$\frac{heat \ transfer}{weight} - \frac{W_{shaft}}{weight} = \left[\frac{u}{g} + \frac{p}{\gamma} + h + \frac{V^2}{2g}\right]_{\odot} - \left[\frac{u}{g} + \frac{p}{\gamma} + h + \frac{V^2}{2g}\right]_{\odot}$$
(4.21)

- Energy Equation for 1-D steady flow: Eq. (4.21)
 - use average values for p, γ , h, u, and V at each flow section
 - use K_e (energy correction coeff.) to account for non-uniform velocity distribution over flow cross section





18/26

$$K_{e} \frac{\rho}{2} V^{2} Q = \int \frac{\rho}{2} q^{2} dQ \quad \text{---- kinetic energy/time} = \frac{1}{2} \frac{mV^{2}}{t}$$

$$K_{e} = \frac{\int \frac{\rho}{2} q^{2} dQ}{\frac{\rho}{2} V^{2} Q} \ge 1 \quad (4.22)$$

$$\frac{heat \ transfer}{weight} - \frac{W_{shaft}}{weight} = \left[\frac{p}{\gamma} + h + K_{e} \frac{V^{2}}{2g}\right]_{\odot} - \left[\frac{p}{\gamma} + h + K_{e} \frac{V^{2}}{2g}\right]_{\odot} + \frac{u_{2} - u_{1}}{g}$$

 $K_e = \begin{cases} 2, \text{ for laminar flow (parabolic velocity distribution)} \\ 1.06, \text{ for turbulent flow (smooth pipe)} \end{cases}$





(4.23)

For a fluid of uniform density γ

$$\frac{p_1}{\gamma} + h_1 + K_{e_1} \frac{V_1^2}{2g} = \frac{p_2}{\gamma} + h_2 + K_{e_2} \frac{V_2^2}{2g} + \frac{W_{shaft}}{weight} - \frac{heat \ transfer}{weight} + \frac{u_2 - u_1}{g}$$
(4.24)

→ unit: m (energy per unit weight)
 For viscous fluid;

$$-\frac{heat \ transfer}{weight} + \frac{u_2 - u_1}{g} = H_{L_{1-2}}$$

- \rightarrow loss of mechanical energy
- ~ irreversible in liquid





Then, Eq. (4.24) becomes

$$\frac{p_1}{\gamma} + h_1 + K_{e_1} \frac{V_1^2}{2g} = \frac{p_2}{\gamma} + h_2 + K_{e_2} \frac{V_2^2}{2g} + \Delta H_M + \Delta H_{L_{1-2}}$$
(4.24a)

where ΔH_M = shaft work transmitted from the system to the outside

$$H_1 = H_2 + \Delta H_M + \Delta H_{L_{1-2}}$$
(4.24b)

where H_1 , H_2 = weight flow rate average values of total head





Bernoulli Equation

Assume

- (1) ideal fluid \rightarrow friction losses are negligible
- ② no shaft work $\rightarrow \Delta H_M = 0$
- ③ no heat transfer and internal energy is constant $\rightarrow \Delta H_{L_{1-2}} = 0$

$$\frac{p_1}{\gamma} + h_1 + K_{e_1} \frac{V_1^2}{2g} = \frac{p_2}{\gamma} + h_2 + K_{e_2} \frac{V_2^2}{2g}$$
(4.25)

 $H_1 = H_2$





If $K_{e_1} = K_{e_2} = 1$, then Eq. (4.25) reduces to



~ total head along a conduct is constant





- Grade lines
- 1) Energy (total head) line (E.L) ~ H above datum

2) Hydraulic (piezometric head) grade line (H.G.L.)

$$=\left(\frac{p}{\gamma}+h\right)$$
above datum

For flow through a pipe with a constant diameter

$$V_1 = V_2 \quad \rightarrow \quad \frac{V_1^2}{2g} = \frac{V_2^2}{2g}$$











25/26

1) If the fluid is <u>real (viscous fluid)</u> and if no energy is being added, then the energy line may never be horizontal or slope upward in the direction of flow.

 Vertical drop in energy line represents the <u>head loss or energy</u> <u>dissipation</u>.



