

convective mass transfer

Parameters in convective mass transfer

$$\frac{\text{momentum diffusivity}}{\text{mass diffusivity}} = \text{Sc} \equiv \frac{\nu}{D_{AB}} = \frac{\mu}{\rho D_{AB}}$$

$$\frac{\text{thermal diffusivity}}{\text{mass diffusivity}} = \text{Le} \equiv \frac{k}{\rho c_p D_{AB}}$$

$$\frac{\text{molecular mass transport resistance}}{\text{convective mass transfer resistance}} = Sh = \frac{k_c L}{D_{AB}} = \frac{-d(c_A - c_{A,s})/dy|_{y=0}}{(c_{A,s} - c_{A,\infty})/L}$$

Dimensional analysis

transfer of mass from the walls of a circular conduit to a fluid

variables: tube diameter, fluid density, fluid viscosity, fluid velocity, fluid diffusivity, mass-transfer coefficient (6)

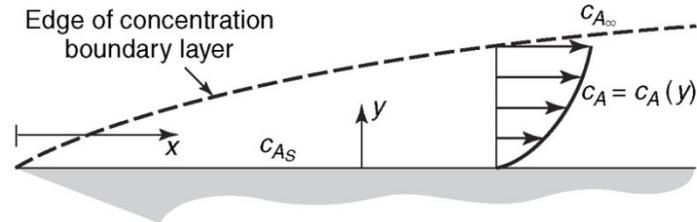
$$Sh = f(Re, Sc)$$

transfer of mass from vertical wall to adjacent fluid with natural convection

variables: characteristic length, fluid diffusivity, fluid density, fluid viscosity, buoyant force, mass-transfer coefficient (6)

$$Sh = f(Gr_{AB}, Sc) \quad Gr_{AB} = \frac{L^3 \rho g \Delta \rho_A}{\mu^2}$$

Laminar concentration boundary layer



$$\nu_x \frac{\partial v_x}{\partial x} + \nu_y \frac{\partial v_x}{\partial y} = \nu \frac{\partial^2 v_x}{\partial y^2}$$

momentum: $\frac{v_x}{v_\infty} = 0$ at $y = 0$ and $\frac{v_x}{v_\infty} = 1$ at $y = \infty$

$$\nu_x \frac{\partial T}{\partial x} + \nu_y \frac{\partial T}{\partial y} = \alpha \frac{\partial^2 T}{\partial y^2}$$

thermal: $\frac{T - T_s}{T_\infty - T_s} = 0$ at $y = 0$ and $\frac{T - T_s}{T_\infty - T_s} = 1$ at $y = \infty$

$$\nu_x \frac{\partial c_A}{\partial x} + \nu_y \frac{\partial c_A}{\partial y} = D_{AB} \frac{\partial^2 c_A}{\partial y^2}$$

concentration: $\frac{c_A - c_{A,s}}{c_{A,\infty} - c_{A,s}} = 0$ at $y = 0$ and $\frac{c_A - c_{A,s}}{c_{A,\infty} - c_{A,s}} = 1$ at $y = \infty$

The rate at which mass enters or leaves the boundary layer at the surface is so small that it does not alter the velocity profile.

heat

mass

$$\left. \frac{\partial T}{\partial y} \right|_{y=0} = (T_\infty - T_s) \left[\frac{0.332}{x} \text{Re}_x^{1/2} \right]$$

$$\frac{q_y}{A} = h_x (T_s - T_\infty) = -k \left. \frac{\partial T}{\partial y} \right|_{y=0}$$

$$h_x = -\frac{k}{T_s - T_\infty} \left. \frac{\partial T}{\partial y} \right|_{y=0} = \frac{0.332k}{x} \text{Re}_x^{1/2}$$

$$\frac{h_x x}{k} = \text{Nu}_x = 0.332 \text{Re}_x^{1/2}$$

Pr=1,
Sc=1

Pr \neq 1,
Sc \neq 1

$$\left. \frac{dc_A}{dy} \right|_{y=0} = (c_{A,\infty} - c_{A,s}) \left[\frac{0.332}{x} \text{Re}_x^{1/2} \right]$$

$$N_{A,y} = k_c (C_{A,s} - C_{A,\infty}) = -D_{AB} \left. \frac{\partial c_A}{\partial y} \right|_{y=0}$$

$$k_c = \frac{D_{AB}}{x} \left[0.332 \text{Re}_x^{1/2} \right]$$

$$\frac{k_c x}{D_{AB}} = \text{Sh}_x = 0.332 \text{Re}_x^{1/2}$$

$$\frac{\delta}{\delta_c} = \text{Sc}^{1/3} \frac{k_c x}{D_{AB}} = \text{Sh}_x = 0.332 \text{Re}_x^{1/2} \text{Sc}^{1/3}$$

$$\frac{\bar{k}_c L}{D_{AB}} = \text{Sh}_L = 0.664 \text{Re}_L^{1/2} \text{Sc}^{1/3}$$

$$\frac{hL}{k} = \text{Nu}_L = 0.664 \text{Re}_L^{1/2} \text{Pr}^{1/3}$$

Reynolds analogy (Pr=1, Sc=1)

heat

$$f' = 2 \frac{v_x}{v_\infty} = 2 \frac{T - T_s}{T_\infty - T_s}$$

$$\left. \frac{d}{dy} \frac{v_x}{v_\infty} \right|_{y=0} = \left. \frac{d}{dy} \left(\frac{T - T_s}{T_\infty - T_s} \right) \right|_{y=0}$$

$$h(T_s - T_\infty) = -k \left. \frac{\partial}{\partial y} (T - T_s) \right|_{y=0}$$

$$h = \left. \frac{\mu c_p}{v_\infty} \frac{dv_x}{dy} \right|_{y=0}$$

$$C_f \cong \frac{\tau_0}{\rho v_\infty^2 / 2} = \left. \frac{2\mu}{\rho v_\infty^2} \frac{dv_x}{dy} \right|_{y=0}$$

$$\frac{h}{\rho v_\infty c_p} = \frac{C_f}{2}$$

mass

$$\left. \frac{\partial}{\partial y} \left(\frac{c_A - c_{A,s}}{c_{A,\infty} - c_{A,s}} \right) \right|_{y=0} = \left. \frac{\partial}{\partial y} \left(\frac{v_x}{v_\infty} \right) \right|_{y=0}$$

$$N_{A,y} = -D_{AB} \left. \frac{\partial}{\partial y} (c_A - c_{A,s}) \right|_{y=0} = k_c (c_{A,s} - c_{A,\infty})$$

$$k_c = \left. \frac{\mu}{\rho v_\infty} \frac{\partial v_x}{\partial y} \right|_{y=0}$$

$$C_f \cong \frac{\tau_0}{\rho v_\infty^2 / 2} = \left. \frac{2\mu}{\rho v_\infty^2} \frac{dv_x}{dy} \right|_{y=0}$$

$$\frac{k_c}{v_\infty} = \frac{C_f}{2}$$

Turbulent flow

heat

$$\tau_{yx} = \mu \frac{d\bar{v}_x}{dy} - \overline{\rho v'_x v'_y}$$

$$\frac{q_y}{A} = -\rho c_p (\alpha + \varepsilon_H) \frac{d\bar{T}}{dy}$$

$$C_f \equiv \frac{\tau_s}{\rho(v_\infty^2 / 2)}$$

Prandtl
analogy

$$\text{Nu} = \frac{(C_f / 2) \text{Re} \text{Pr}}{1 + 5\sqrt{C_f / 2}(\text{Pr} - 1)}$$

von Karman
analogy

$$\text{Nu} = \frac{(C_f / 2) \text{Re} \text{Pr}}{1 + 5\sqrt{C_f / 2} \left\{ \text{Pr} - 1 + \ln \left[(1 + 5\text{Pr}) / 6 \right] \right\}}$$

mass

$$N_{A,y} = -(D_{AB} + \varepsilon_D) \frac{d\bar{c}_A}{dy}$$

$$\text{Sh} = \frac{(C_f / 2) \text{Re} \text{Sc}}{1 + 5\sqrt{C_f / 2}(\text{Sc} - 1)}$$

$$\text{Sh} = \frac{(C_f / 2) \text{Re} \text{Sc}}{1 + 5\sqrt{C_f / 2} \left\{ \text{Sc} - 1 + \ln \left[(1 + 5\text{Sc}) / 6 \right] \right\}}$$

Chilton-Colburn analogy

$$j_H = j_D$$

$$\frac{h}{\rho v_\infty c_p} (\text{Pr})^{2/3} = \frac{k_c}{v_\infty} (\text{Sc})^{2/3}$$

for turbulent pipe flow

$$\frac{hD}{k} = \text{Nu} = 0.023 \text{Re}^{0.8} \text{Pr}^{1/3}$$

$$\frac{\bar{k}_c D}{D_{AB}} = \text{Sh} = 0.023 \text{Re}^{0.8} \text{Sc}^{1/3}$$

Table 28.1 Models for convective mass-transfer coefficients (dilute systems)

Model	Basic form	$f(D_{AB})$	Notes
Film theory	$k_c = \frac{D_{AB}}{\delta}$	$k_c \propto D_{AB}$	δ unknown, may be found when solute has high Sc
Penetration theory	$k_c = \sqrt{\frac{D_{AB} v_\infty}{\pi \delta}}$	$k_c \propto D_{AB}^{1/2}$	δ unknown, good model when homogeneous reaction within boundary layer or when solute has low Sc
Boundary-layer theory	$k_c = 0.664 \frac{D_{AB}}{L} (\text{Re})^{1/2} (\text{Sc})^{1/3}$	$k_c \propto D_{AB}^{2/3}$	Best way to scale k_c from one solute to another exposed to same hydrodynamic flow