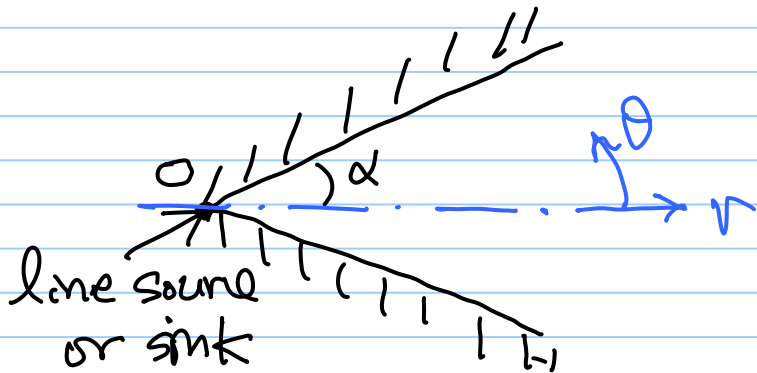


8-2) Flows in Wedge-shaped region (Jeffery-Hamel flows)



- radial flow is induced by source/sink @ $r=0$.

- $u_r = u_r(r, \theta)$, $u_r = \max$ @ $\theta=0$.

- $u_\theta = u_z = 0$.

- continuity: $\frac{1}{r} \frac{\partial}{\partial r} (r \cdot u_r) + \frac{1}{r} \frac{\partial}{\partial \theta} (u_\theta) + \frac{\partial}{\partial z} (u_z) = 0$.

- momentum: $u_r \frac{\partial u_r}{\partial r} = -\frac{1}{\rho} \frac{\partial p}{\partial r} + \nu \left(\frac{\partial^2 u_r}{\partial r^2} + \frac{1}{r} \frac{\partial u_r}{\partial r} - \frac{u_r}{r^2} + \frac{1}{r^2} \frac{\partial^2 u_r}{\partial \theta^2} \right)$

$$0 = -\frac{1}{\rho r} \frac{\partial p}{\partial \theta} + \frac{2\nu}{r^2} \frac{\partial u_r}{\partial \theta}$$

from continuity, $\frac{\partial}{\partial r}(ru_r) = 0 \rightarrow u_r = \frac{1}{r} \cdot F(\theta)$.

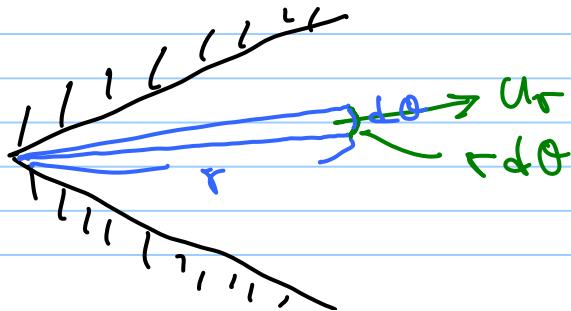
i) substitute $u_r = \frac{1}{r} F(\theta)$ into ntm eqns.

ii) $\frac{\partial}{\partial r}$ (θ -ntm eq), $\frac{\partial}{\partial \theta}$ (r -ntm eq.)

iii) $\frac{1}{\rho} \frac{\partial^2 p}{\partial r^2 \partial \theta}$ as a common term.

$$\Rightarrow \underline{2FF' + 2F'' + 4VF' = 0} \quad \text{BC) } F = 0 \text{ @ } \theta = \pm \alpha.$$

• Volume flow rate, $Q = \int_{-\alpha}^{\alpha} u_r \cdot r d\theta = \int_{-\alpha}^{\alpha} F \cdot d\theta$



$$\cdot Re \equiv \frac{UL}{\nu} \sim \frac{Q}{\nu} \rightarrow \text{Define } Re \equiv \frac{\alpha \cdot F_{\max}}{\nu}$$

$$= \frac{\alpha \cdot U_{\max} \cdot r}{\nu}$$

$Re < 0$: inflow (sink)
 $Re > 0$: outflow (source)

Let $\theta/\alpha \equiv \eta$, $F/F_{\max} = f = U_r/U_{r,\max}$

then, the gov. eqn becomes

$$f''' + 2\alpha Re f f' + 4\alpha^2 f' = 0.$$

↓ integrate

$$f = 0 \text{ @ } \eta = \pm 1.$$

$$f = 1, f' = 0 \text{ @ } \eta = 0.$$

$[f'' + \alpha \cdot \text{Re} f^2 + 4\alpha^2 f = \underline{b}.] \times f'$ and integrate

$$\rightarrow \frac{1}{2}(f')^2 + \frac{1}{3}\alpha \cdot \text{Re} f^3 + 2\alpha^2 f^2 = \underline{b}f + \underline{d}.$$

apply BC $\textcircled{a} \eta = 0, (f')^2 = (1-f)^2 \left\{ \frac{2}{3}\alpha \cdot \text{Re}(f^2+f) + 4\alpha^2 f + c \right\}$
" " $\eta = \pm 1, (f')^2 = c \textcircled{a} \eta = 1.$ 2d.

limiting case

① case I ($\alpha \ll 1, \alpha \cdot \text{Re} \ll 1$)

$$f''' = 0. \rightarrow f = 1 - \eta^2 = \frac{u_r}{u_{r,\max}} = 1 - \left(\frac{r}{\alpha}\right)^2.$$

\rightarrow similar to Poiseuille flow distribution.
(valid for in flow or outflow)

② case: $d \ll 1$, but $d \cdot Re \gg 1$.

(\nearrow most practical applications)

$$(f')^2 = \frac{2}{3} d \cdot Re \cdot f (1-f)^2 + c(1-f)$$

\downarrow

flow depends on the single parameter,

$$\left(\frac{2}{3} d \cdot Re\right)^{1/2} = \int_0^1 \frac{1}{(1-f)(f^2+f+k)} df. \quad k = \frac{c}{\frac{2}{3} d \cdot Re}$$

\hookrightarrow for a given $k = \frac{c}{\frac{2}{3} d \cdot Re}$, "c" is varying

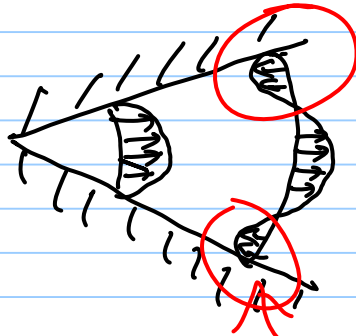
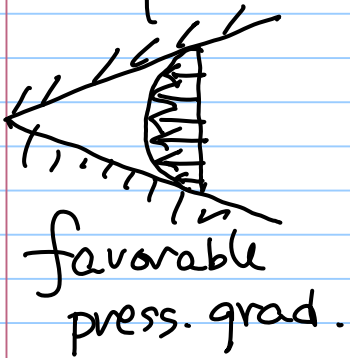
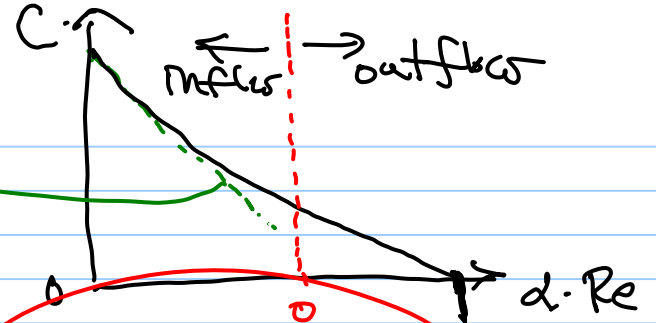
with respect to $d \cdot Re \rightarrow$ multiple solutions are possible.

if $Re < 0$ (inflow).

$$C \approx -\frac{4}{3} \cdot (d \cdot Re)$$

if $Re > 0$ (outflow)

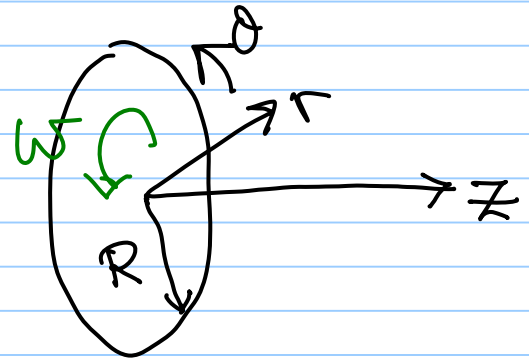
$C = 0$ at $d \cdot Re \approx 10.3$ \Rightarrow flow separation occurs.



* Characteristics of wedge-shaped flow

- ① convergent (accelerating) case \rightarrow thin viscous layer
 - ② divergent (diffusing) case \rightarrow at critical Re ,
($2 \cdot Re \approx 10.3$),
flow separates.
(everywhere viscous)
- similar to conventional boundary-layer behavior.

\rightarrow 3) Flow near an infinite rotating disc



$R \rightarrow \infty$

Von Karman viscous pump.
(refer to "white")

\rightarrow viscous drag at disc sets up a

Axisymmetric: $\frac{\partial}{\partial \theta} = 0$ - swirling (3D) flow toward the disc.

Steady, incompressible, constant p .

from continuity & 3 mfm eqns $\rightarrow p, u_r, u_\theta, u_z$.

BC) (a) $z=0, u_r = u_z = 0, u_\theta = r\omega, p = \text{const.}$

(b) $z \rightarrow \infty, u_r = 0, u_\theta = 0, \frac{\partial u_z}{\partial z} = 0$.

assume non-dimensional form of u_θ .

$$\frac{u_\theta}{r\omega} = f(z)$$

$\rightarrow u_r$: induced by centrifugal force, should be

scaled in the same way.

$$\frac{u_r}{rw} = F(z)$$

- since the viscous diffusion works in z -direction,
→ introduce viscous length $\sqrt{\nu/w}$.

to make $z^* = z/\sqrt{\nu/w}$.

- from the continuity, $H(z) = Uz/\sqrt{\nu w}$, also
define, $P^*(z) = P/\rho\nu w$.

$$\Rightarrow \begin{cases} H' + 2F = 0 \\ F^2 + F'H - G^2 = F'' \\ FG + H'G' + FG' = G'' \\ 2HF - 2F' = \dot{p}^* \end{cases}$$

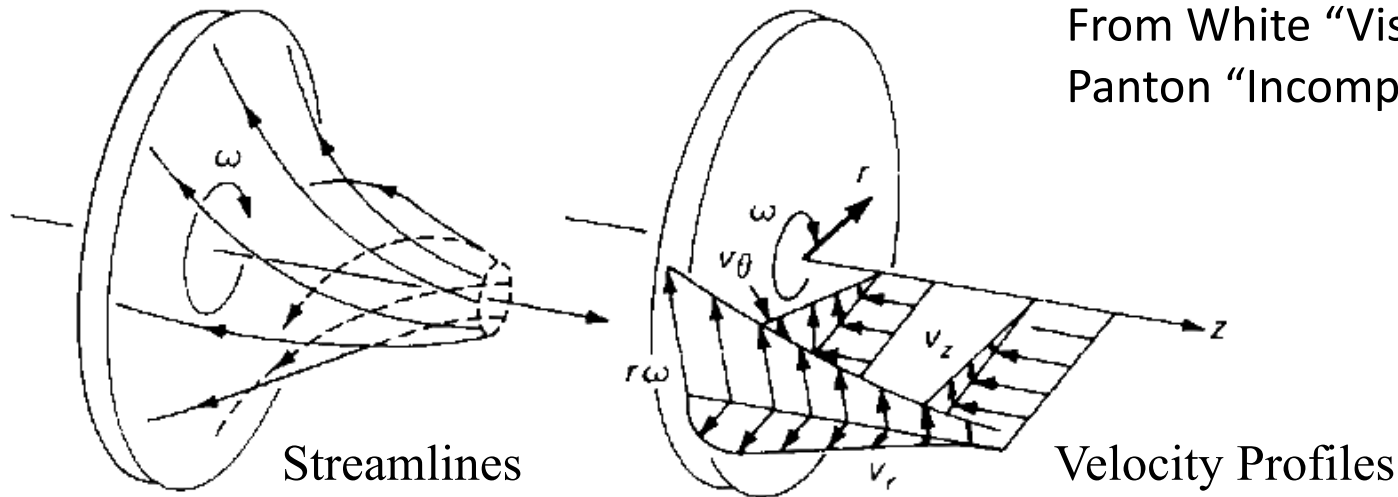
↓ numerical integration.

$$\textcircled{a} z^* = 0, \quad \begin{aligned} u_\theta = r\omega &\rightarrow G(0) = 1 \\ u_r = 0 &\rightarrow F(0) = 0 \\ u_z = 0 &\rightarrow H(0) = 0 \\ \phi = 0 &\rightarrow \dot{p}^*(0) = 0 \end{aligned}$$

$$\textcircled{a} z^* \rightarrow \infty, \quad \begin{aligned} u_r = 0 &\rightarrow F(\infty) = 0 \\ u_\theta = 0 &\rightarrow G(\infty) = 0 \end{aligned}$$

Von Karman's Viscous Pump Principle

1



From White "Viscous Fluid Flow" and Panton "Incompressible Flow"

$$\frac{1}{r} \frac{\partial}{\partial r} (rv_r) + \frac{\partial}{\partial z} (v_z) = 0$$

$$v_r \frac{\partial v_r}{\partial r} + v_z \frac{\partial v_r}{\partial z} - \frac{v_\theta^2}{r} = -\frac{1}{\rho} \frac{\partial p}{\partial r} + \nu \left(\frac{\partial^2 v_r}{\partial r^2} + \frac{1}{r} \frac{\partial v_r}{\partial r} + \frac{\partial^2 v_r}{\partial z^2} - \frac{v_r}{r^2} \right)$$

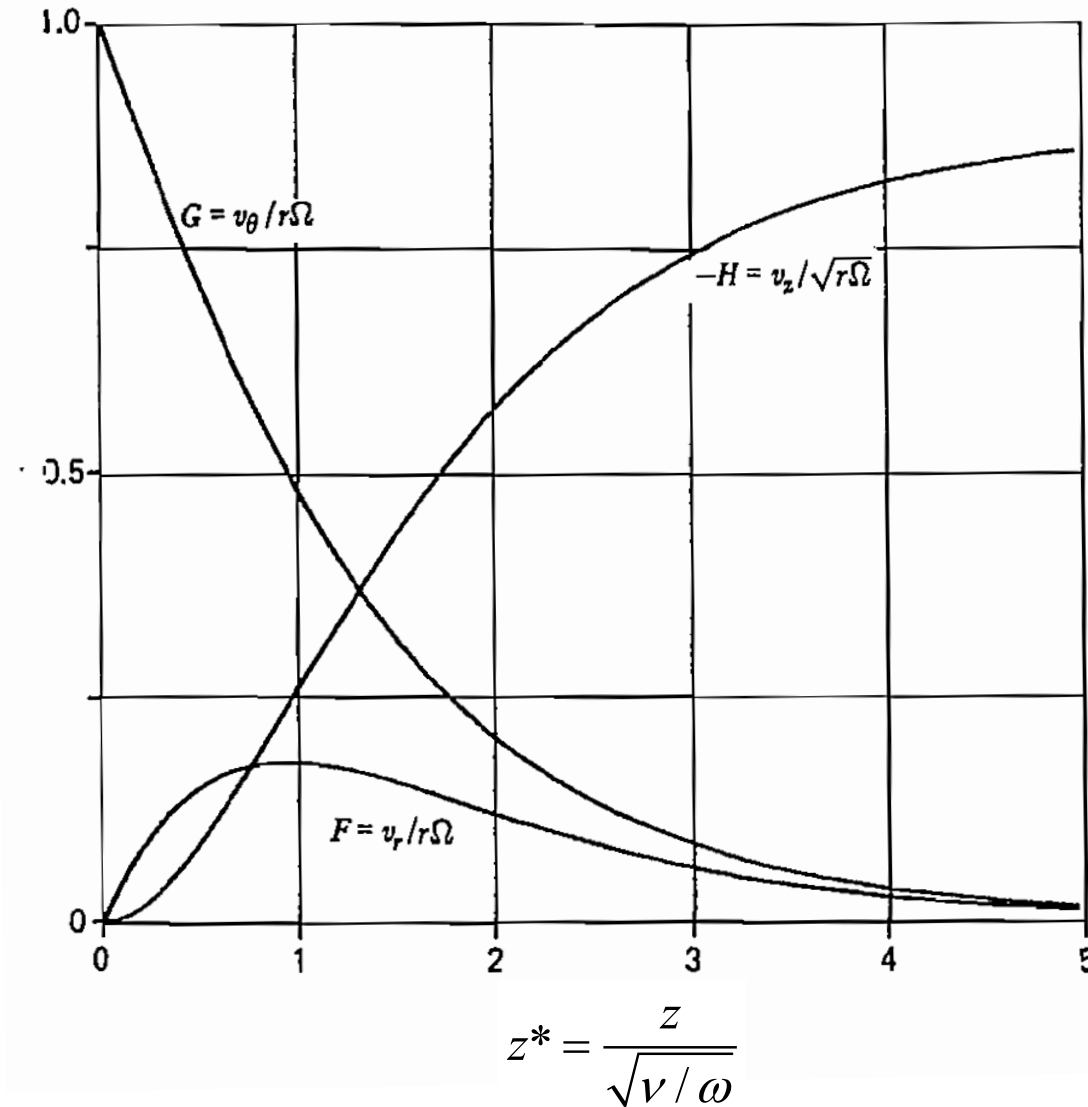
$$v_r \frac{\partial v_\theta}{\partial r} + v_z \frac{\partial v_\theta}{\partial z} + \frac{1}{r} v_r v_\theta = \nu \left(\frac{\partial^2 v_\theta}{\partial r^2} + \frac{1}{r} \frac{\partial v_\theta}{\partial r} + \frac{\partial^2 v_\theta}{\partial z^2} - \frac{v_\theta}{r^2} \right)$$

$$v_r \frac{\partial v_z}{\partial r} + v_z \frac{\partial v_z}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \left(\frac{\partial^2 v_z}{\partial r^2} + \frac{1}{r} \frac{\partial v_z}{\partial r} + \frac{\partial^2 v_z}{\partial z^2} \right)$$

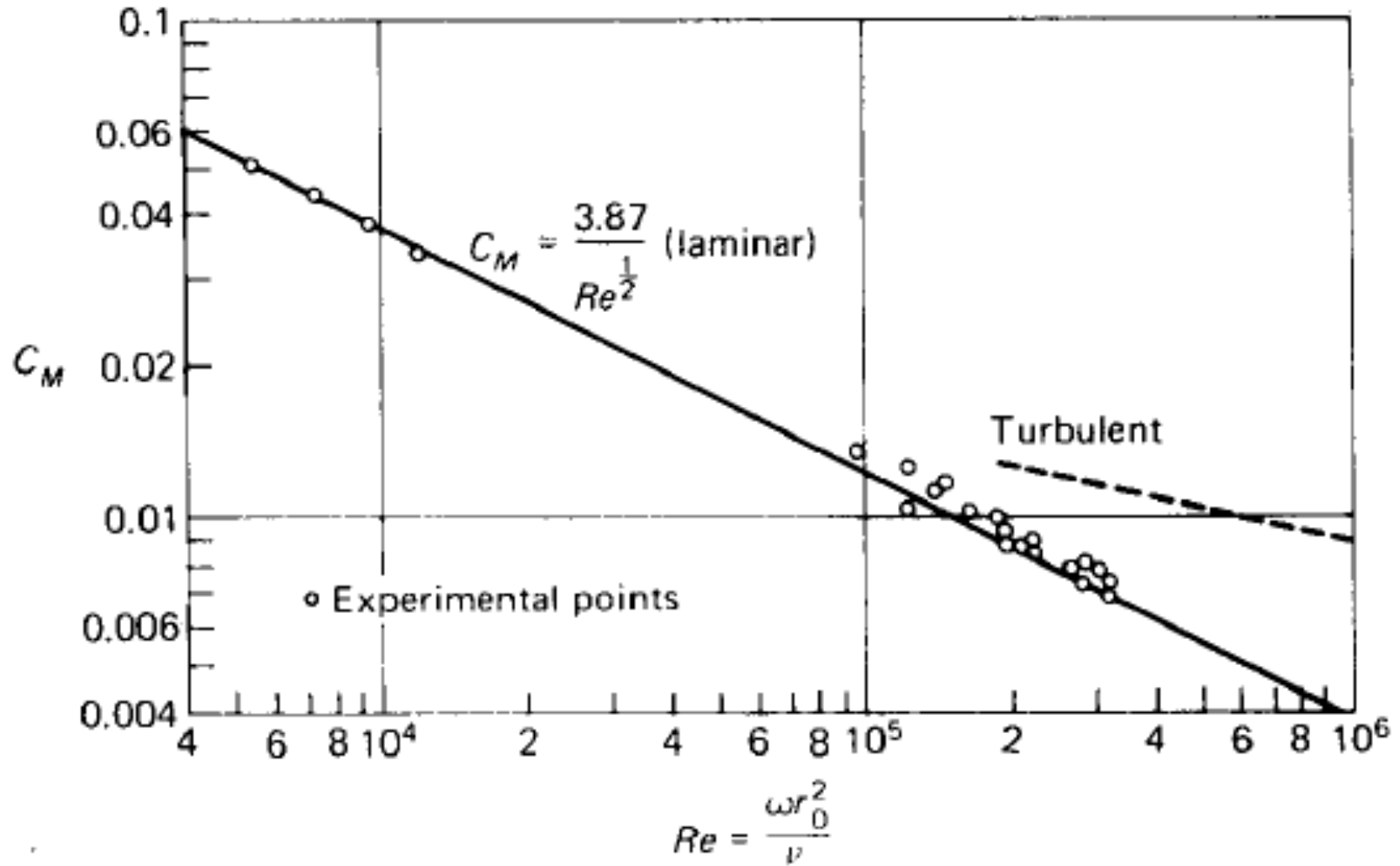


Von Karman's Viscous Pump Principle

2



Von Karman's Viscous Pump Principle



Let's define the thickness, δ (where the swirling flow near the disk is confined) as the point where u_θ drops 1% of its wall value. $\therefore \underbrace{\tau}_{\text{value at the}} \equiv u_\theta / r \Omega = 0.01$

$$\rightarrow \delta \approx 4.5 \sqrt{\frac{\nu}{\omega}}$$

• Circumferential wall shear stress, $\tau_{z\theta} = \mu \frac{\partial u_\theta}{\partial z} \Big|_{z=0}$.

\rightarrow total torque required to turn a disk of radius, r_0 .

$$M = \int_0^{r_0} \tau_{z\theta} \cdot r \cdot dA = \int_0^{r_0} \tau_{z\theta} \cdot r (2\pi r \cdot dr)$$