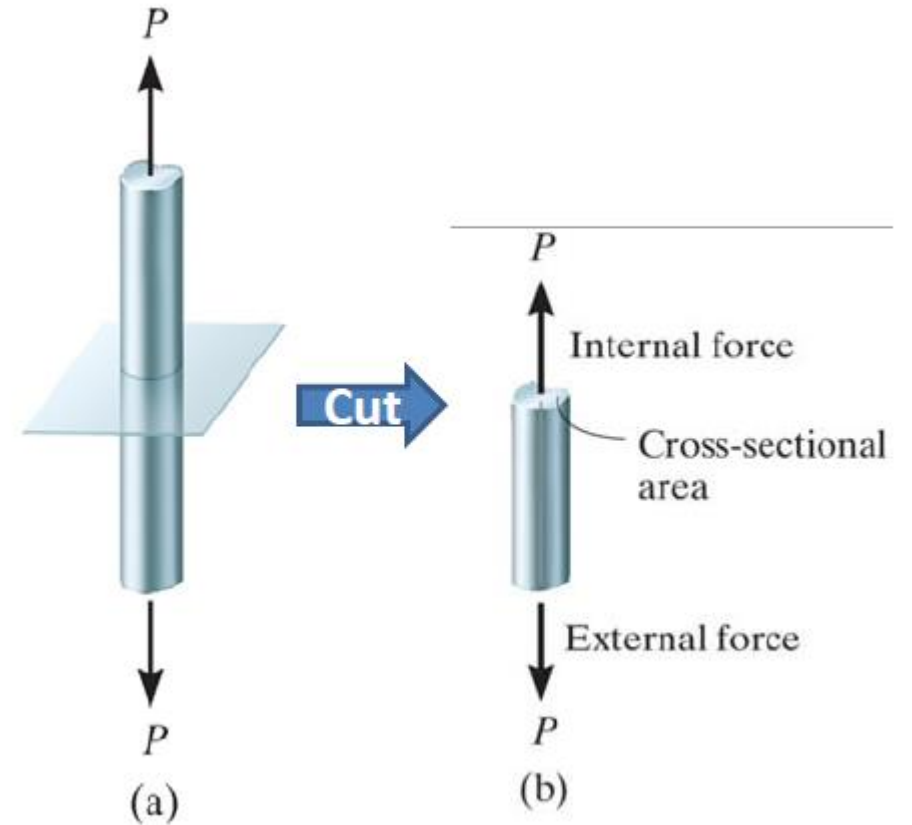


# Normal stress

- A bar is subjected to an axial load.
- The "average" normal stress is

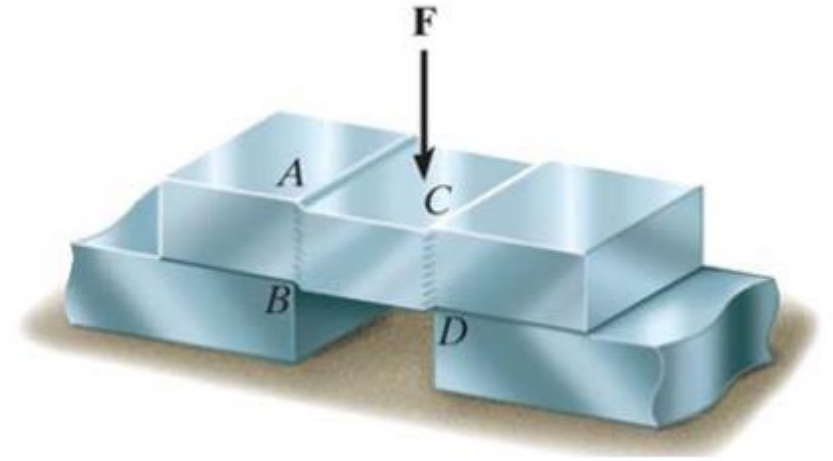
$$\sigma = \frac{P}{A}$$

- In general, the stress across the section is not uniform.
- Uniform stress can be assumed.

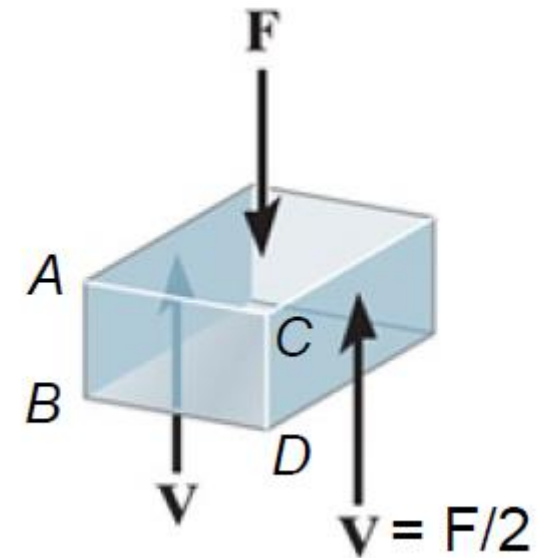
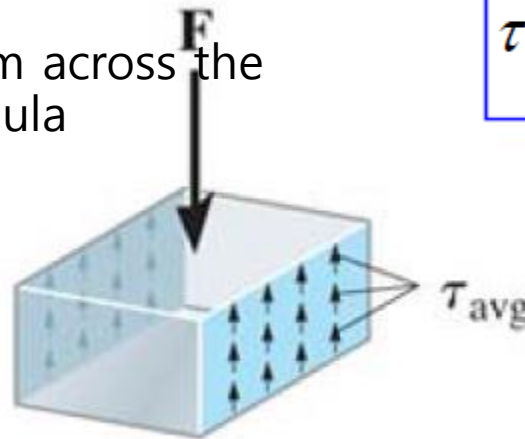


# Shear stress

- Shear stress is the stress component that acts in the plane of the sectional area.
- If the supports are considered rigid and  $F$  is large enough, it will cause the material of the bar to fail along vertical planes (shear failure)
- The average shear stress (on each vertical plane) is
- Note: Shear stress is actually not uniform across the section. We will derive shear stress formula

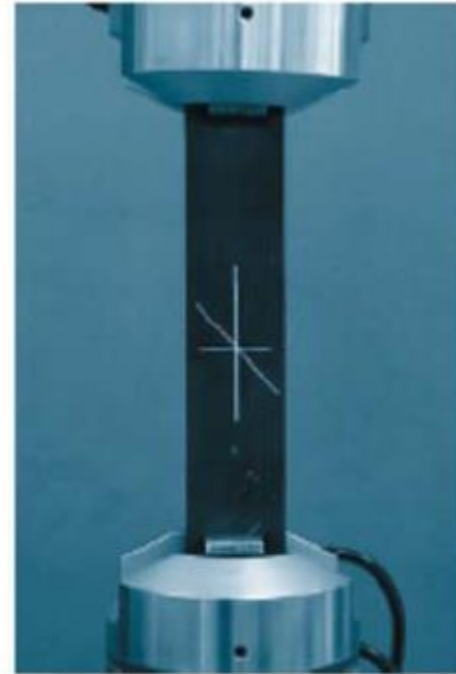


$$\tau_{avg} = \frac{V}{A}$$

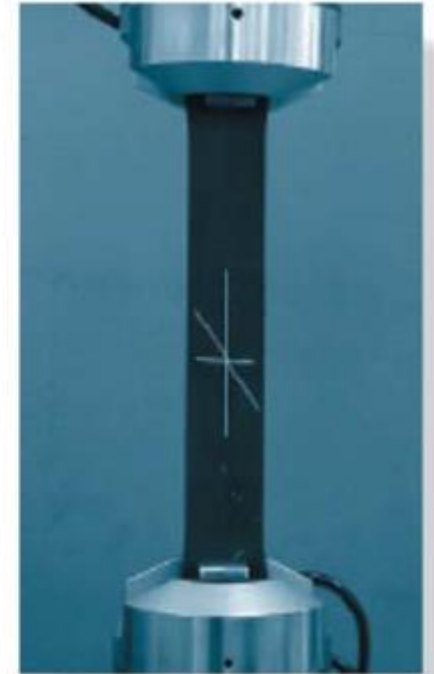


# Deformation

- When a force is applied to a body, it will change the body's shape and size. These changes are deformation.
- Example: The rubber strip (membrane) is subjected to tension.
- The vertical line is lengthened.
- The horizontal line is shortened.
- The inclined line changes its length and rotates.



Undeformed



Deformed due to tension

# Engineering stress and strain

- Engineering or nominal stress = Applied load  $P$  divided by the specimen's original cross-sectional area ( $A_0$ ).
- Engineering or nominal strain = Change in the specimen's gauge length by the specimen's original gauge length ( $L_0$ ).

$$\sigma = \frac{P}{A_0}$$

$$\varepsilon = \frac{\delta}{L_0}$$

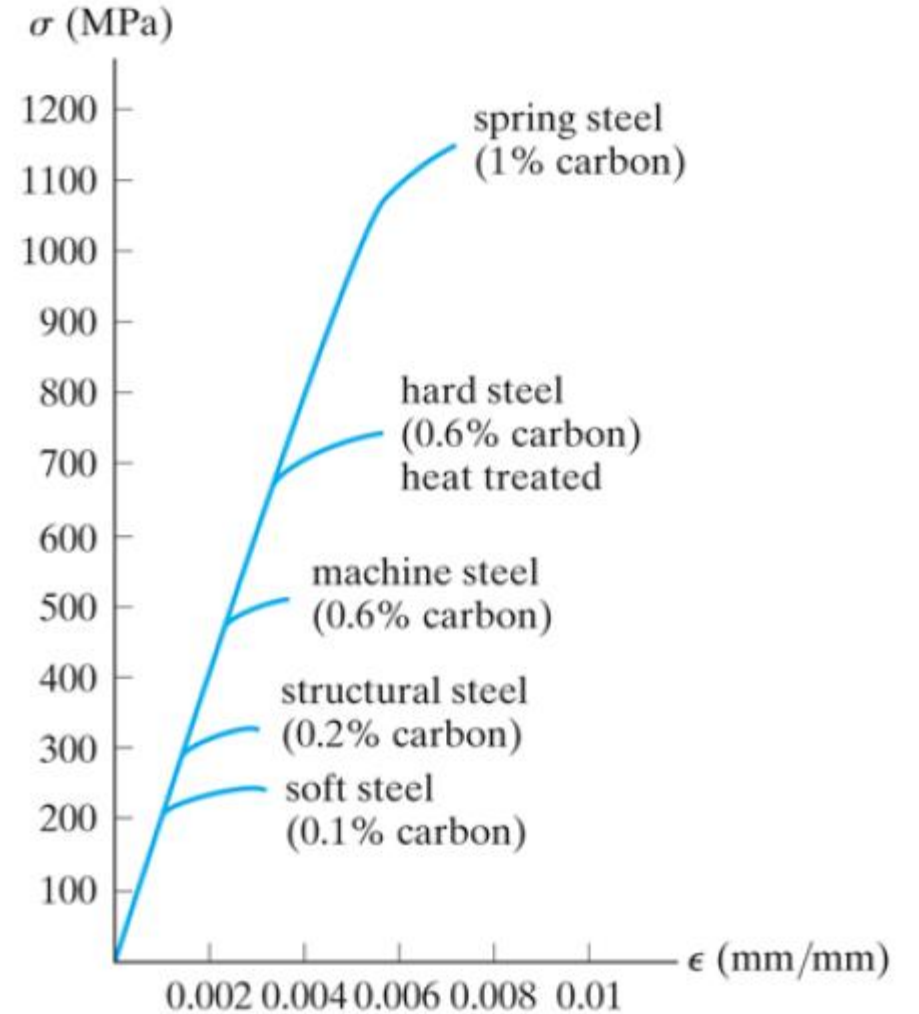
- The plot of  $\sigma$ – $\varepsilon$  based on the above definitions gives the conventional stress-strain diagram.
- If the actual cross-sectional area and specimen length are used, the plot is called the true stress-strain diagram.

# Hooke's law

- In the elastic region, a linear relationship exists between stress and strain as follows.

$$\sigma = E \varepsilon$$

- $\sigma$  = normal stress
- $\varepsilon$  = normal strain
- $E$  = modulus of elasticity or Young's modulus
- Note that steels of different grades have different proportional limits, but their Young's modulus values are practically the same.

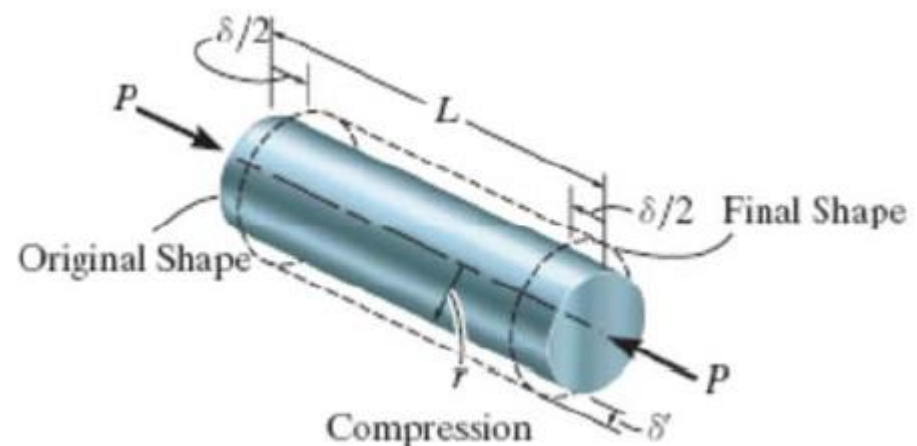
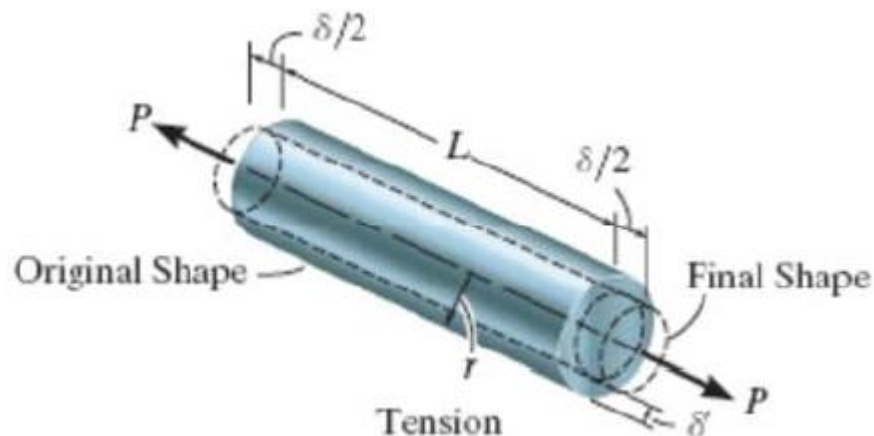


# Poisson's ratio

- When a deformable body is subjected to a tensile force, not only does it elongate in the direction of force but it also contracts laterally.
- Likewise, a compressive force acting on a body causes it to contract in the direction of force but its sides expand laterally.
- The ratio of lateral strain to the longitudinal strain is called Poisson's ratio:

$$\nu = -\frac{\epsilon_{lat}}{\epsilon_{long}}$$

Poisson's ratio is dimensionless.  
It ranges between 0 and 0.5.  
Typical values are about 1/4 to 1/3.  
(~0.5 for rubber & elastomer).



# Shear stress-strain relation

When subjected to shear, most engineering materials will exhibit linear-elastic behaviour in shear distortion until a proportional limit.

Hooke's law applies for shear stress-strain relation in the elastic range.

$$\tau = G\gamma$$

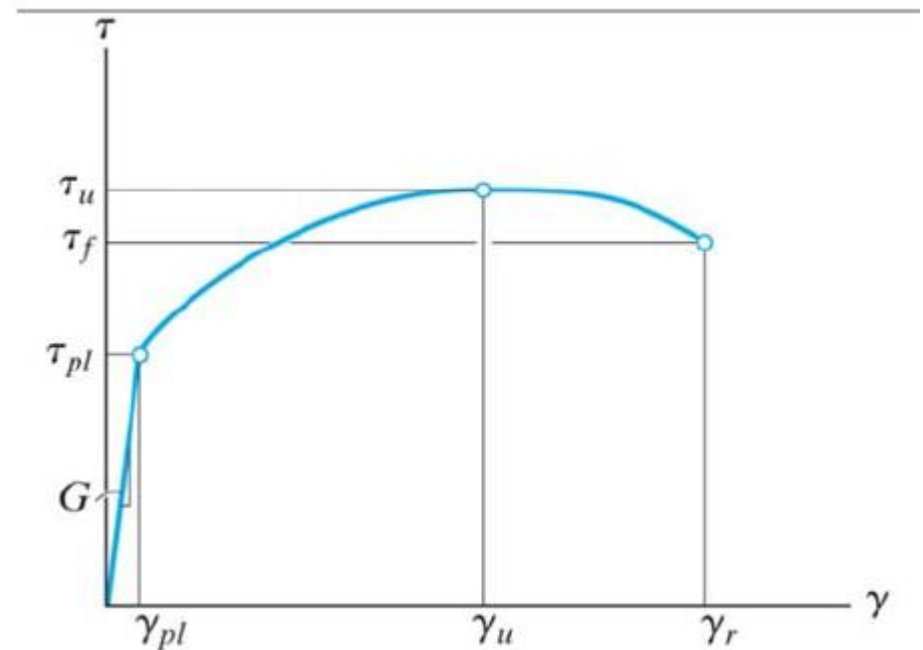
$\tau$  = shear stress

$\gamma$  = (engineering) shear strain

$G$  = shear modulus of elasticity

$E$  and  $G$  are related through Poisson's ratio:

$$G = \frac{E}{2(1+\nu)}$$

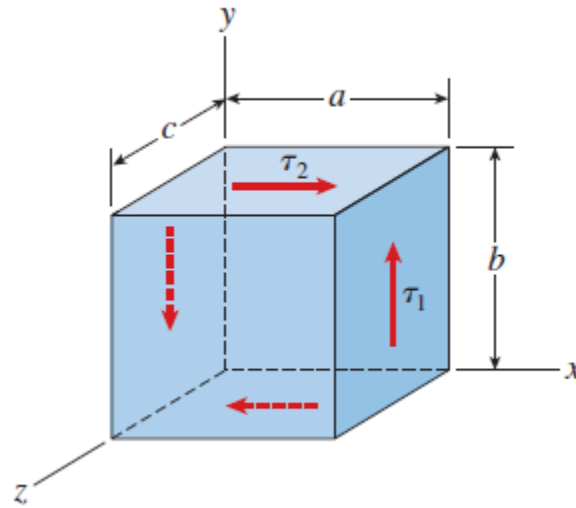


# Shear strain

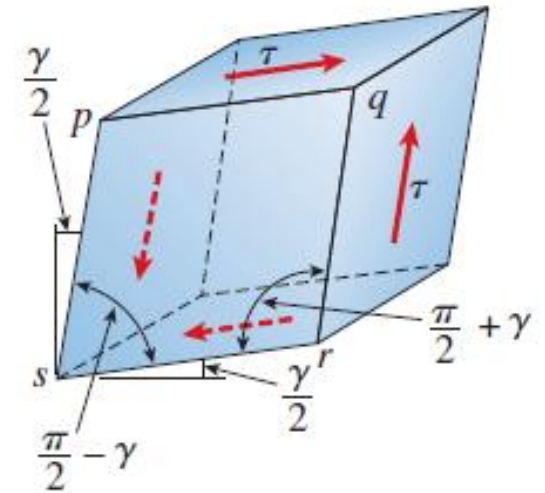
Shear strains cause a change in its shape (distortion)

Remember the "slip" behavior of atoms

Internal force equilibrium condition



$$\tau_1 = \tau_2$$



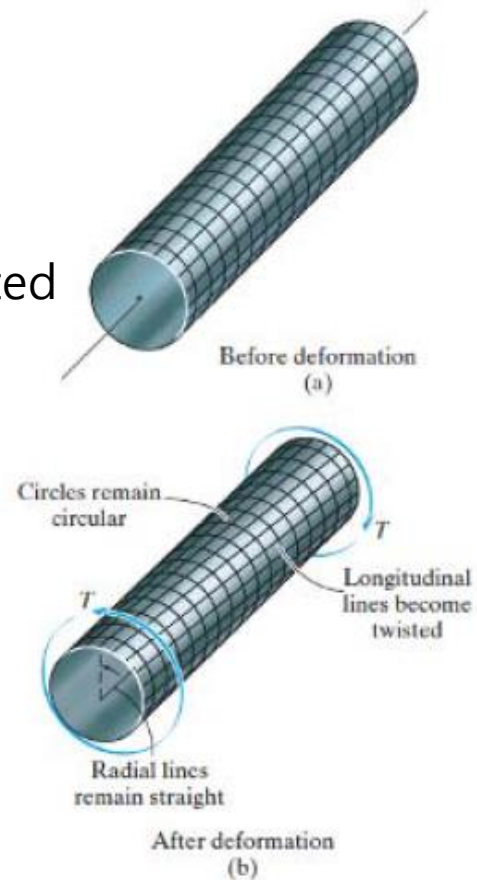


# Torsion

Torque is a moment that twists a member about its longitudinal axis.

Assumptions:

- Linear and elastic deformation
- Plane section remains plane and undistorted



Notice the deformation of the rectangular element when this rubber bar is subjected to a torque.

# Static Review

**Static equilibrium** of a body requires **balance of forces** to prevent the body from translating, and **balance of moments** to prevent the body from rotating.

Equilibrium equations in vector form:

In scalar form, there are 6 equilibrium equations for 3D problems:

$$\text{Vector equations:} \quad \underline{\Sigma \mathbf{F}} = \underline{\mathbf{0}} \quad ; \quad \underline{\Sigma \mathbf{M}} = \underline{\mathbf{0}}$$

$$\text{Scalar equations in 3D:} \quad \Sigma F_x = 0 \quad ; \quad \Sigma F_y = 0 \quad ; \quad \Sigma F_z = 0$$

$$\Sigma M_x = 0 \quad ; \quad \Sigma M_y = 0 \quad ; \quad \Sigma M_z = 0$$

## **Free Body Diagram (FBD):**


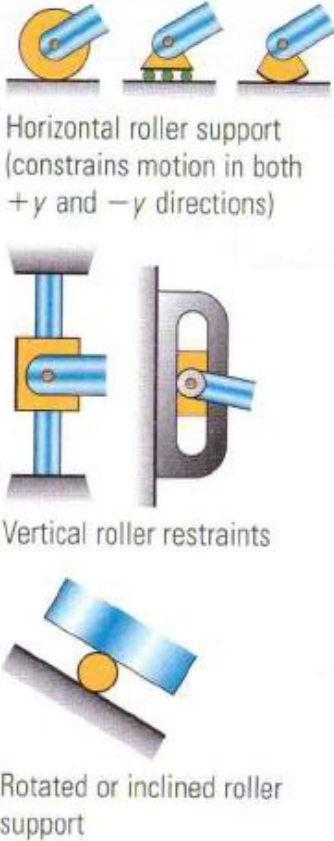
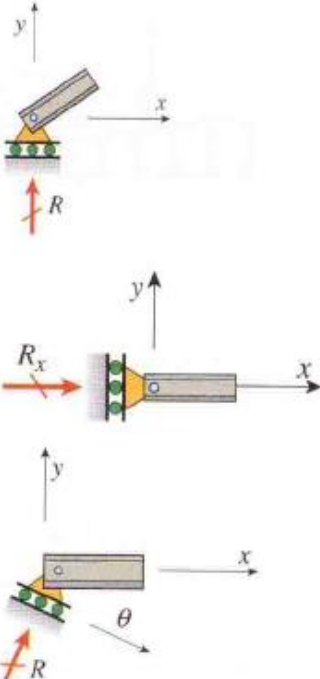
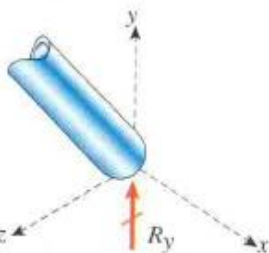
This is a very important and useful tool in most engineering disciplines!

Pictorial representation to analyze forces acting on a body of interest – which could be part of the structure or even an “infinitesimal” element (with dimension  $\rightarrow 0$ ).

Use equilibrium to relate forces and moments acting on the FB.

# Static Review

## Roller support

Type of support or connection	Simplified sketch of support or connection	Display of restraint forces and moments, or connection forces
<p>(1) Roller support—horizontal, vertical, or inclined</p>  <p>Bridge with roller support (The Earthquake Engineering Online Archive)</p>	 <p>Horizontal roller support (constrains motion in both <math>+y</math> and <math>-y</math> directions)</p> <p>Vertical roller restraints</p> <p>Rotated or inclined roller support</p>	<p>(a) Two-dimensional roller support</p>  <p>(b) Three-dimensional roller support</p> 

# Static Review

## Pin support

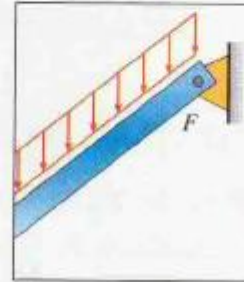
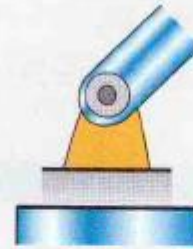
(2) Pin support



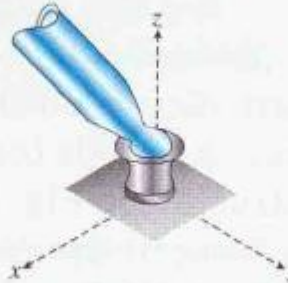
Bridge with pin support  
(Courtesy of Joel Kerkhoff,  
P.Eng.)



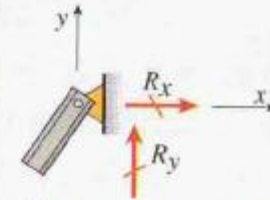
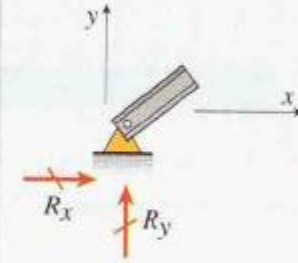
Pin support on old truss  
bridge  
(© Barry Goodno)



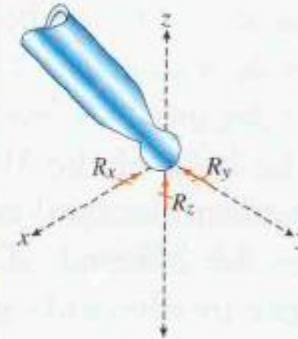
Pin support at  $F$  in Fig. 1-1



(a) Two-dimensional pin  
support

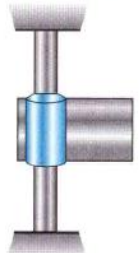
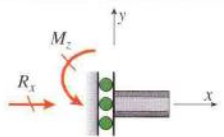
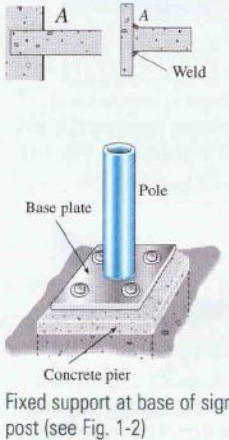
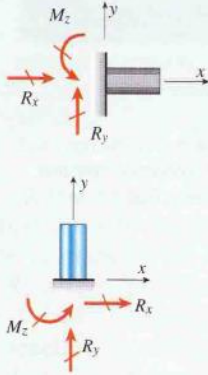
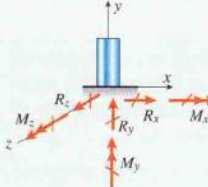
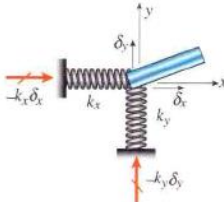


(b) Three-dimensional pin  
support



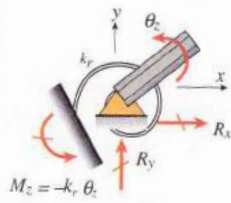

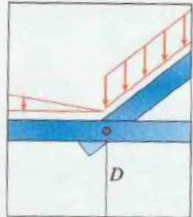
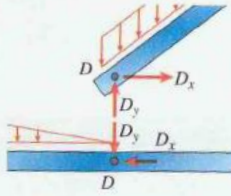
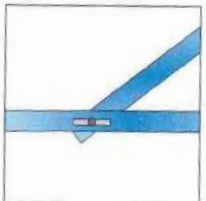
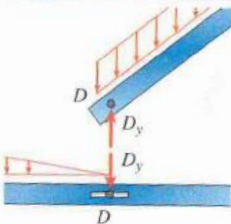
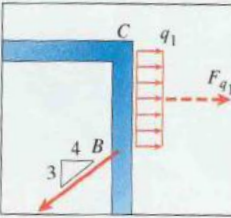
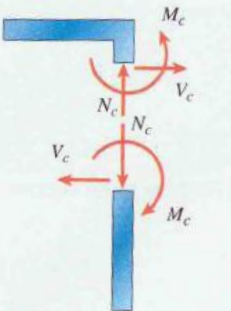
# Static Review

## Other supports

<p>(3) Sliding support</p>	 <p>Frictionless sleeve on vertical shaft</p>	
<p>(4) Clamped or fixed support</p>	 <p>Fixed support at base of sign post (see Fig. 1-2)</p>	<p>(a) Two-dimensional fixed support</p>  <p>(b) Three-dimensional fixed support</p> 
<p>(5) Elastic or spring supports</p>		<p>(a) Translational spring (<math>k</math>)</p> 

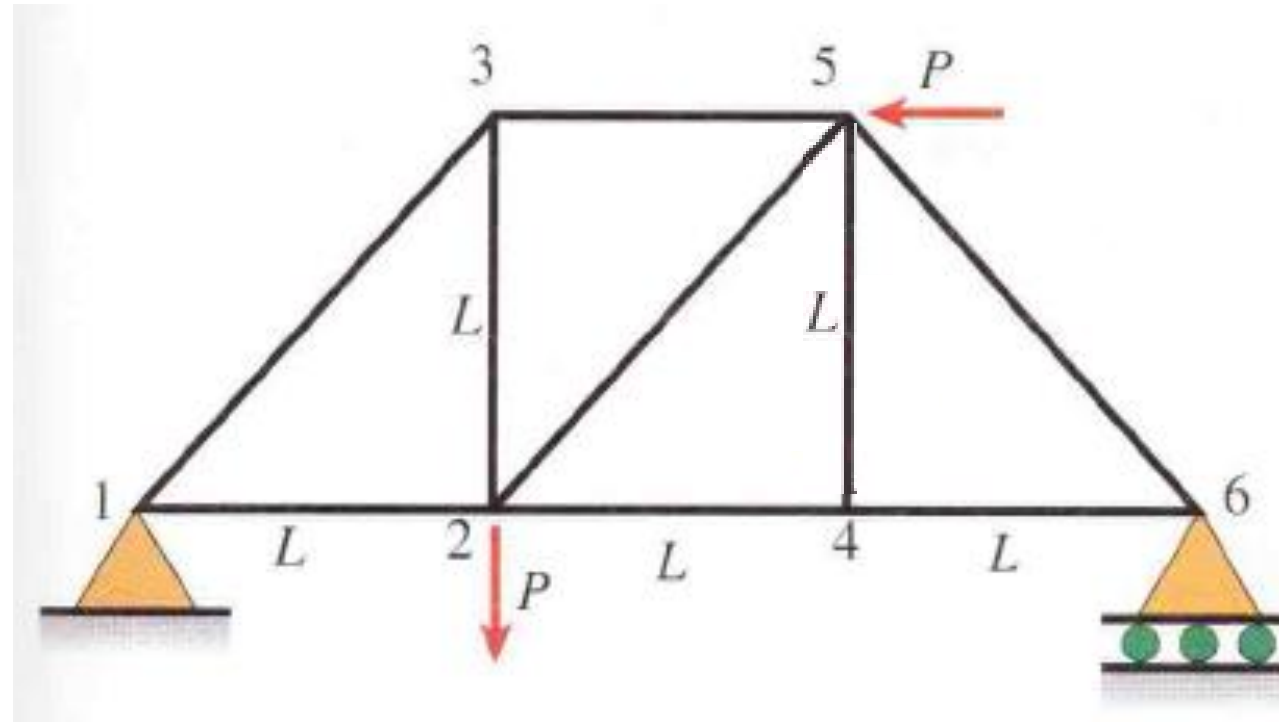
# Static Review

## Other supports

		<p>(b) Rotational spring (<math>k_r</math>)</p>  <p><math>M_z = -k_r \theta_z</math></p>
<p>(6) Pinned connection (from Figs. 1-1 and 1-3)</p>  <p>Pin connection on old bridge (© Barry Goodno)</p>	 <p>Pinned connection at <math>D</math> between members <math>EDC</math> and <math>DF</math> in plane frame (Fig. 1-1)</p>	
<p>(7) Slotted connection (modified connection from that shown in Figs. 1-1 and 1-3)</p>	 <p>Alternate slotted connection at <math>D</math> on plane frame (Note that the plane frame in Fig. 1-1 is <i>unstable</i> if this slotted connection is used instead of a pin at <math>D</math>.)</p>	
<p>(8) Rigid connection (internal forces and moment in members joined at <math>C</math> of plane frame in Fig. 1-1)</p>	 <p>Rigid connection at <math>C</math> on plane frame</p>	

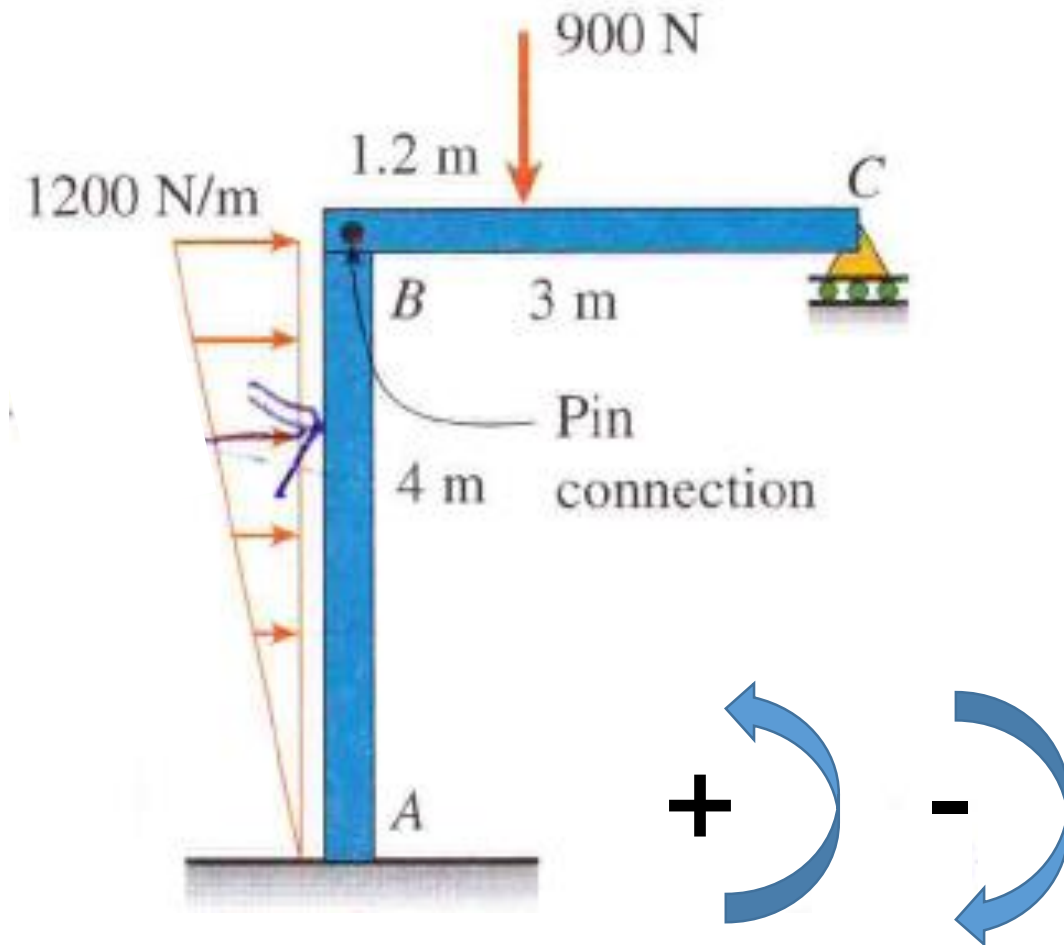
# Static Review

A plane truss has downward applied load  $P$  at joint 2 and another load  $P$  applied leftward at joint 5. The force in member 4-5 is:



# Static Review

The moment reaction at A in the plane frame below is approximately:

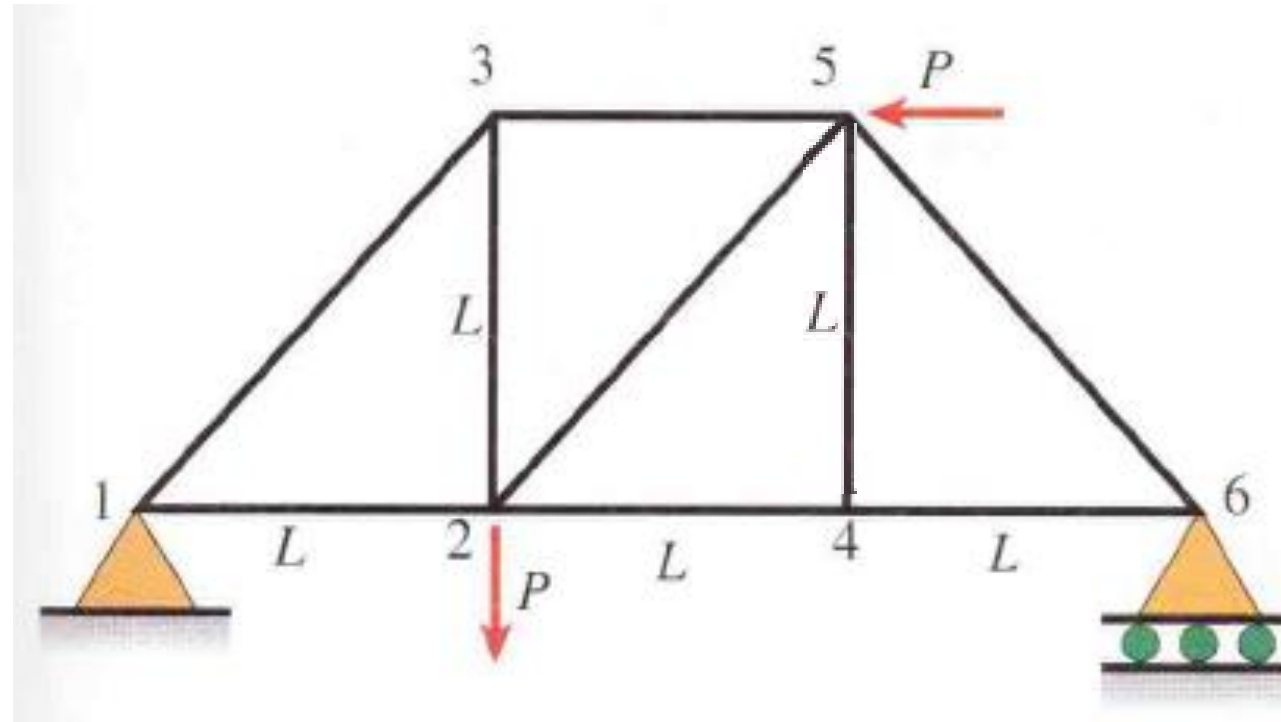




# Static Review

A plane truss has downward applied load  $P$  at joint 2 and another load  $P$  applied leftward at joint 5. The force in member 4-5 is:

- (A) 0
- (B)  $-P/2$
- (C)  $-P$
- (D)  $+ 1.5 P$



# Static Review

The moment reaction at A in the plane frame below is approximately:

- (A) +1400 N.m
- (B) -2280 N.m
- (C) -3600 N.m
- (D) +6400 N.m

