

446.328 Mechanical System Analysis

기계시스템해석

- lecture 19, 20, 21 -

Dongjun Lee (이동준)

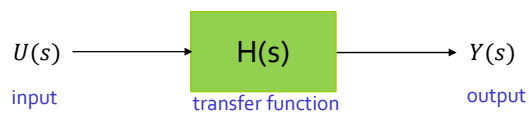
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Frequency Response

transfer function (input-to-output)



- often, we want to know (or design) system's response when the input signal contains certain frequency components, e.g.,

$$u(t) = A_1 \sin(\omega_1 t + \phi_1) + A_2 \sin(\omega_2 t + \phi_2) + \dots$$

← excitation mode

- e.g., microphone/headphone, engine mount, earthquake-proof structure, ...)

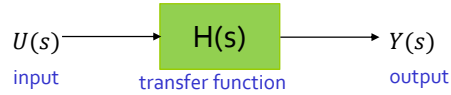
⇒ frequency response of $H(s)$

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Frequency Response of TF

transfer function (input-to-output)

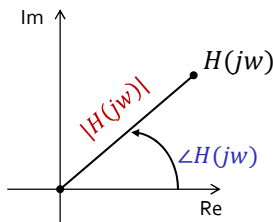


$$u(t) = A \sin \omega t$$

for stable $H(s)$,
in steady-state

$$y(t) = A|H(j\omega)| \sin(\omega t + \angle H(j\omega))$$

gain same frequency
as input phase



ex) $\ddot{x} + 3\dot{x} + 2x = u$

$$u(t) = A \sin t \Rightarrow y(t) \rightarrow \frac{A}{\sqrt{10}} \sin(t - \tan^{-1} 3)$$

$$u(t) = A \sin \sqrt{2}t \Rightarrow y(t) \rightarrow \frac{\sqrt{2}A}{6} \sin(t - \frac{\pi}{2})$$

ex) $\dot{x} + 2x = u$

$$u(t) = \sum_{k=1}^n A_k \sin(\omega_k t + \phi_k)$$

$$\Rightarrow y(t) \rightarrow \sum_{k=1}^n A_k |H(j\omega_k)| \sin(\omega_k t + \phi_k + \angle H(j\omega_k))$$

* if $H(s)$ is stable and in steady-state

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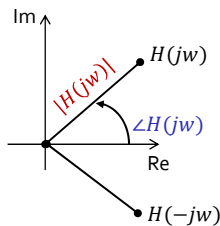
Derivation of Frequency Response

$$Y(s) = H(s)U(s) = H(s) \frac{Aw}{s^2 + \omega^2} = \frac{N(s)}{D(s)} \frac{Aw}{s^2 + \omega^2}$$

$$= \frac{c_1}{s + j\omega} + \frac{c_2}{s - j\omega} + \frac{\alpha(s + a) + \beta}{(s + a)^2 + b^2} + \frac{\gamma_1}{s + p} + \frac{\gamma_2}{(s + p)^2} + \dots + H(s)[initial]$$

if $H(s)$ is **stable** (i.e., all poles are in LHP), in **steady-state**,

$$Y(s) = \frac{c_1}{s + j\omega} + \frac{c_2}{s - j\omega} \quad c_2 = H(j\omega) \frac{A}{2j} \quad c_1 = H(-j\omega) \frac{A}{-2j}$$



$$H(j\omega) = Me^{j\phi} \rightarrow H(-j\omega) = \overline{H(j\omega)} = Me^{-j\phi}$$

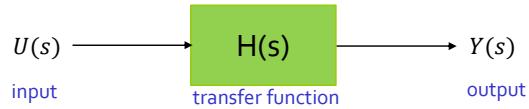
$$y(t) \rightarrow A|H(j\omega)| \sin(\omega t + \phi)$$

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Bode Plot

transfer function (input-to-output)



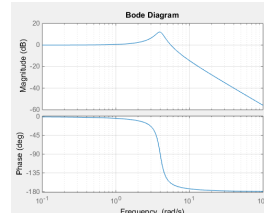
- steady-state frequency response when $H(s)$ is stable
- gain plot: $\log_{10} w$ vs $20 \log_{10} |H(jw)|$ [dB] (decibel)
- phase plot: $\log_{10} w$ vs $\angle H(jw)$
- decade: 10 times of w (e.g., from 5 [rad/s] to 50 [rad/s])
- decibel: $20 \log_{10} |H(jw)| \rightarrow$ allows us to "add" Bode plots
 - 0 [dB] $\rightarrow |H(jw)| = 1 \rightarrow$ no amplification/attenuation
 - 10 [dB] $\rightarrow |H(jw)| = 10^{0.5} \approx 3 \rightarrow$ 3 times amplification
 - 10 [dB] $\rightarrow |H(jw)| = 10^{-0.5} \approx 0.3 \rightarrow$ attenuation
 - 3 [dB] $\rightarrow |H(jw)| = 10^{-3/20} \approx 1/\sqrt{2}$
- if $u(t) = A \cos 0$, $y(t) \rightarrow A|H(j0)| \cos(\angle H(j0)) = AH(0)$, when $H(0) > 0$
↖ dc-gain

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Bode Plot Example

$$\ddot{x} + \dot{x} + 16x = 16u(t), \quad u(t) = \sin wt$$



$$1. w = 10 \text{ rad/s} \Rightarrow M \approx -14.5 \text{ dB} (\approx 0.2), \phi \approx -173^\circ$$

$$2. w = 1 \text{ rad/s} \Rightarrow M \approx 0 \text{ dB} (\approx 1), \phi \approx 0^\circ$$

$$3. w = 4 \text{ rad/s} \Rightarrow M \approx 12 \text{ dB} (\approx 4), \phi \approx -90^\circ$$

$$4. w = 0 \text{ rad/s} \Rightarrow M = 0 \text{ dB} (= 1), \phi = 0^\circ$$

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Simple Bode Plot Examples

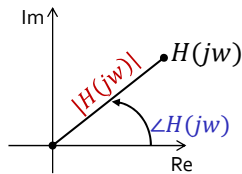
1. $H(s) = K \rightarrow H(j\omega) = K \rightarrow |H(j\omega)| = K, \angle H(j\omega) = 0$

2. $H(s) = \frac{1}{s} \rightarrow H(j\omega) = \frac{1}{j\omega} \rightarrow |H(j\omega)| = \frac{1}{\omega}$
 $\rightarrow 20 \log_{10}|H(j\omega)| = -20 \log_{10} \omega$
 $\angle H(j\omega) = -\frac{\pi}{2}$

-20dB/decade roll-off:
low-freq. attenuation

3. $H(s) = s \rightarrow H(j\omega) = j\omega \rightarrow |H(j\omega)| = \omega$
 $\rightarrow 20 \log_{10}|H(j\omega)| = 20 \log_{10} \omega$
 $\angle H(j\omega) = \frac{\pi}{2}$

+20dB/decade: noisy
high-freq. amplification



phase-lead?
acausal system

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Bode Plot of 1st Order System

$$H(s) = \frac{p}{s+p} \rightarrow H(j\omega) = \frac{p}{p+j\omega} = \frac{p}{p^2 + \omega^2}(p - j\omega)$$

$$|H(j\omega)| = \frac{p}{\sqrt{\omega^2 + p^2}}, \quad \angle H(j\omega) = -\tan^{-1}\left(\frac{\omega}{p}\right)$$

Bode Plot

- gain: $|H(j\omega)| = 20 \log_{10} \frac{p}{\sqrt{\omega^2 + p^2}} = 20 \log_{10} p - 10 \log_{10}(p^2 + \omega^2) \leq 0$ [dB]

- phase: $-\frac{\pi}{2} \leq -\tan^{-1}\left(\frac{\omega}{p}\right) \leq 0$ [rad]

- asymptotes

(i) $\omega \ll p$: gain ≈ 0 [dB], phase ≈ 0

(ii) $\omega \gg p$: gain $\approx 20 \log_{10} p - 20 \log_{10} \omega$, phase $\approx -90^\circ$

(iii) $\omega = p$: gain $\approx -20 \log_{10} \sqrt{2} \approx -3$ [dB], phase = -45°

roll-off: -20dB/decade

bandwidth $\omega \leq p$
 $\omega_c = p$: cut-off frequency

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Bode Plot of 2nd Order System w/ $\zeta < 1$

$$H(s) = \frac{w_n^2}{s^2 + 2\zeta w_n s + w_n^2} \rightarrow H(jw) = \frac{w_n^2}{-w^2 + 2\zeta w_n w j + w_n^2} = w_n^2 \frac{(w_n^2 - w^2) - 2\zeta w_n w j}{(w_n^2 - w^2)^2 + 4\zeta^2 w_n^2 w^2}$$

$$|H(jw)| = \frac{w_n^2}{\sqrt{(w_n^2 - w^2)^2 + 4\zeta^2 w_n^2 w^2}} \quad -\pi \leq \angle H(jw) = -\tan^{-1} \frac{2\zeta w_n w}{w_n^2 - w^2} \leq 0$$

Bode Plot

- asymptotes

(i) $w \ll w_n$: gain = 0 [dB], phase = 0 roll-off: -40dB/decade

(ii) $w \gg w_n$: gain = $40 \log_{10} w_n - 40 \log_{10} w$, phase = -180°

(iii) $w = w_n$: gain $\approx -20 \log_{10} \left(\frac{1}{2\zeta}\right) \geq 0$ [dB] if $\zeta \leq 0.5$, phase = -90°

- resonance (i.e., peak of $|H(jw)|$): $w(w^2 - w_n^2(1 - 2\zeta^2)) = 0$ no resonance if $\zeta > \frac{1}{\sqrt{2}}$

$$w_r = w_n \sqrt{1 - 2\zeta^2}, \quad M_{pw} = \frac{1}{2\zeta \sqrt{1 - \zeta^2}} \quad \text{if } \zeta \leq \frac{1}{\sqrt{2}} \approx 0.707$$

$$\text{recall } w_d = w_n \sqrt{1 - \zeta^2} \quad \text{and} \quad M_p = e^{-\frac{\pi\zeta}{\sqrt{1 - \zeta^2}}}$$

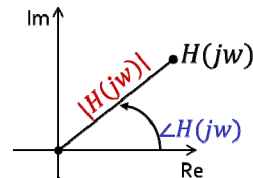
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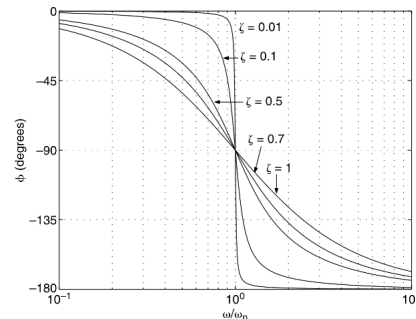
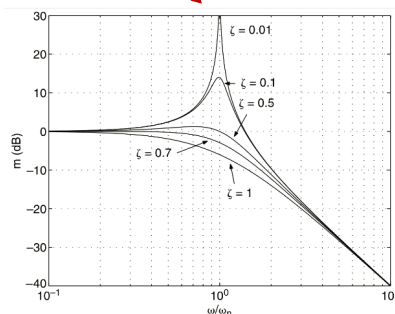
Bode Plot of 2nd Order System w/ $\zeta < 1$

$$H(s) = \frac{w_n^2}{s^2 + 2\zeta w_n s + w_n^2}$$

➔ polar plot



$M_{pw} \rightarrow \infty$, if $\zeta = 0$



$w = w_n$: gain $\approx -20 \log_{10} \left(\frac{1}{2\zeta}\right) \geq 0$ [dB] if $\zeta \leq 0.5$, phase = -90°

$$w_r = w_n \sqrt{1 - 2\zeta^2}, \quad M_{pw} = \frac{1}{2\zeta \sqrt{1 - \zeta^2}} \quad \text{if } \zeta \leq \frac{1}{\sqrt{2}} \approx 0.707$$

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Additive Property of Bode Plot

$$H(s) = \frac{k(s+z)}{s(s+p)(s^2+\alpha s+\beta)}$$



$$\begin{aligned} \text{bode}[H(s)] = & \text{bode}[k] - \text{bode}\left[\frac{1}{s+z}\right] + \text{bode}\left[\frac{1}{s}\right] \\ & + \text{bode}\left[\frac{1}{s+p}\right] + \text{bode}\left[\frac{1}{s^2+\alpha s+\beta}\right] \end{aligned}$$

$$\begin{aligned} H(s) &= H_1(s)H_2(s) \dots H_n(s) \\ \Rightarrow 20 \log_{10}|H(j\omega)| &= 20 \log_{10}|H_1(j\omega)| + 20 \log_{10}|H_2(j\omega)| \dots + 20 \log_{10}|H_n(j\omega)| \\ \Rightarrow \angle H(j\omega) &= \angle H_1(j\omega) + \angle H_2(j\omega) + \dots + \angle H_n(j\omega) \end{aligned}$$

$$\therefore \text{bode}[H] = \text{bode}[H_1] + \text{bode}[H_2] + \dots + \text{bode}[H_n]$$

$$\begin{aligned} H(s) &= 1/H'(s) \\ \Rightarrow 20 \log_{10}|H(j\omega)| &= -20 \log_{10}|H'(j\omega)|, \quad \angle H(j\omega) = -\angle H'(j\omega) \end{aligned}$$

$$\therefore \text{bode}[1/H] = -\text{bode}[H]$$

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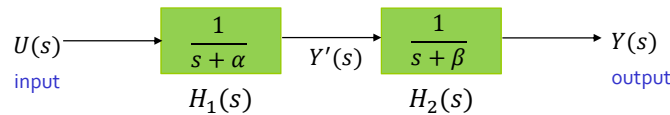


Bode Plot of 2nd Order System w/ $\zeta \geq 1$

$$\begin{aligned} H(s) &= \frac{1}{s^2 + as + b} \\ &= \frac{1}{\alpha\beta} \frac{\alpha}{s+\alpha} \frac{\beta}{s+\beta} \end{aligned}$$



$$\text{bode}[H(s)] = \text{bode}\left[\frac{\alpha}{s+\alpha}\right] + \text{bode}\left[\frac{\beta}{s+\beta}\right] - \text{bode}\left[\frac{1}{\alpha\beta}\right]$$



$$\begin{aligned} u(t) = A \sin \omega t &\Rightarrow y'(t) \rightarrow A|H_1(j\omega)| \sin(\omega t + \angle H_1(j\omega)) \\ &\Rightarrow y(t) \rightarrow A|H_1(j\omega)||H_2(j\omega)| \sin(\omega t + \angle H_1(j\omega) + \angle H_2(j\omega)) \end{aligned}$$

$$|H(j\omega)| = |H_1(j\omega)||H_2(j\omega)| \Rightarrow 20 \log_{10}|H| = 20 \log_{10}|H_1| + 20 \log_{10}|H_2|$$

$$\angle H(j\omega) = \angle H_1(j\omega) + \angle H_2(j\omega)$$

$$\Rightarrow \text{bode}[H] = \text{bode}[H_1] + \text{bode}[H_2]$$

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Example 7.3

$$H(s) = \frac{s + 1}{s(s^2 + s + 16)}$$



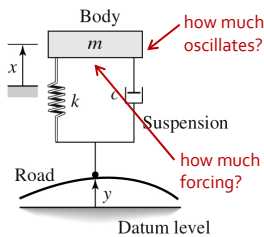
$$\begin{aligned} \text{bode}[H(s)] = & -\text{bode}\left[\frac{1}{s+1}\right] + \text{bode}\left[\frac{1}{s}\right] \\ & + \text{bode}\left[\frac{16}{s^2+s+16}\right] - \text{bode}[16] \end{aligned}$$

- low frequency: $H(s)$ behaves like an integrator
- high frequency: $H(s)$ behaves like a standard 2nd order system

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Base Motion & Transmissibility



system dynamics

$$m\ddot{x} = f_t = -c(\dot{x} - \dot{y}) - k(x - y)$$

displacement transmissibility

$$T_D(s) = \frac{X(s)}{Y(s)} = \frac{cs + k}{ms^2 + cs + k} \quad \leftarrow \text{body motion}$$

force transmissibility

$$T_F(s) = \frac{F_t(s)}{kY(s)} = \frac{ms^2(cs + k)}{k(ms^2 + cs + k)} \quad \leftarrow \text{chassis force}$$

ex) $m = 1\text{kg}, c = \frac{1\text{Ns}}{m}, k = \frac{4\text{N}}{m}$

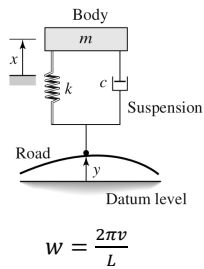
$$T_D(s) = \frac{cs + k}{k} \frac{k}{ms^2 + cs + k} \quad T_F(s) = \frac{m}{k} s^2 \cdot T_D(s)$$

$T_D(s)$ peak 7.16dB ≈ 2.28 @ $\omega = 1.87\text{rad/s} \Rightarrow$ approx: $|T_D(j\omega_n)| = \sqrt{5} \approx 2.24$

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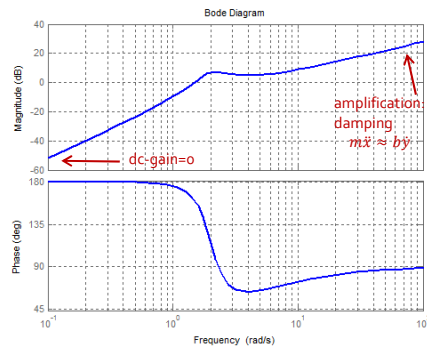
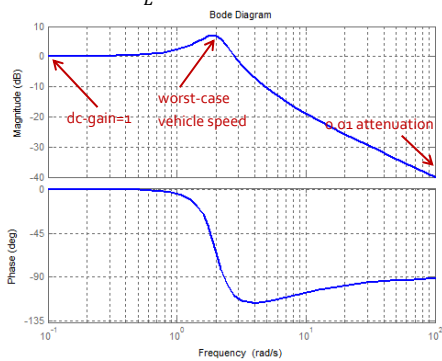
Base Motion & Transmissibility



$$m\ddot{x} = -b(\dot{x} - \dot{y}) - k(x - y) = f_t$$

$$T_D(s) = \frac{X(s)}{Y(s)} = \frac{bs + k}{ms^2 + bs + k}$$

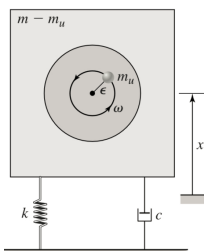
$$T_F(s) = \frac{F_t(s)}{kY(s)} = \frac{ms^2(bs + k)}{k(ms^2 + bs + k)}$$



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Rotating Unbalance



system dynamics

$$m\ddot{x} + c\dot{x} + kx = m_u\epsilon\omega^2 \sin \omega t$$

vibration displacement

$$X(s) = \frac{1}{ms^2 + cs + k} L[m_u\epsilon\omega^2 \sin \omega t]$$

$$x(t) \rightarrow m_u\epsilon\omega^2 |H_D(j\omega)| \sin(\omega t + \angle H_D(j\omega))$$

$$|H_D(j\omega)| = \left| \frac{1}{ms^2 + cs + k} \right|_{s=j\omega}$$

ε: eccentricity
 m_u : unbalance mass
 e.g.: lathe, rotor, etc...

transmitted force

$$F_t(s) = \frac{cs + k}{ms^2 + cs + k} L[m_u\epsilon\omega^2 \sin \omega t]$$

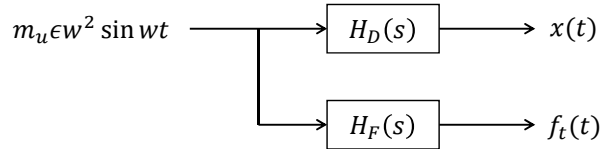
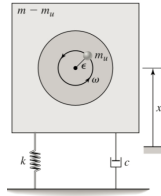
$$f_t(t) \rightarrow m_u\epsilon\omega^2 |H_F(j\omega)| \sin(\omega t + \angle H_F(j\omega))$$

$$|H_F(j\omega)| = \left| \frac{cs + k}{ms^2 + cs + k} \right|_{s=j\omega}$$

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Rotating Unbalance



what is $|x(t)|_{max}$ and $|f_t(t)|_{max}$ given rotation speed w ?

vibration displacement

$$x(t) \rightarrow m_u \epsilon |w^2 H_D(jw)| \sin(\omega t + \angle H_D(jw)) \quad |H_D(jw)| = \left| \frac{1}{ms^2 + cs + k} \right|_{s=jw}$$

transmitted force

$$f_t(t) \rightarrow m_u \epsilon |w^2 H_F(jw)| \sin(\omega t + \angle H_F(jw)) \quad |H_F(jw)| = \left| \frac{cs + k}{ms^2 + cs + k} \right|_{s=jw}$$

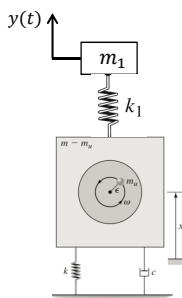
- bode plot of $s^2 H_D(s)$ and $s^2 H_F(s)$ to take into account w^2
- effect of damping c ?
- problematic if c is small, yet, $w \approx w_n$

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Dynamic Vibration Absorber

absorber.m
rotate_m_u_absorber.m



$$m\ddot{x} + b\dot{x} + kx = -k_1(x - y) + u, \quad u = m_u \epsilon \omega^2 \sin \omega t$$

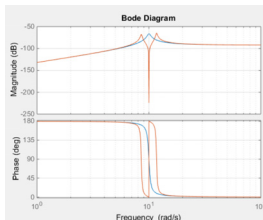
$$m_1 \ddot{y} = -k_1(y - x)$$

$$X(s) = \frac{m_1 s^2 + k_1}{(ms^2 + bs + k + k_1)(m_1 s^2 + k_1) - k_1^2} U(s)$$

$$\sin \omega t \longrightarrow m_u \epsilon \omega^2 H_V(s) \longrightarrow x(t)$$

- if we set m_1, k_1 s.t. $\sqrt{k_1/m_1} = w_n$, $x(t) \rightarrow 0$ even if $w = w_n$

$$Y(s) = \frac{k_1}{(ms^2 + bs + k + k_1)(m_1 s^2 + k_1) - k_1^2} U(s)$$

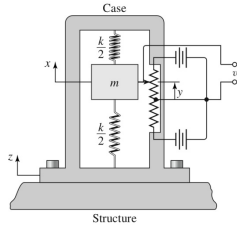


- zero vibration at $w_n = \sqrt{k_1/m_1}$; two extra peaks nearby though
- at $w = w_n$, $y(t) \rightarrow -\frac{1}{k_1} u(t) \Rightarrow$ force cancelation!

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Seismograph



system dynamics:

$$m\ddot{x} = -b\dot{y} - ky = m(\ddot{y} + \ddot{z})$$

$$Y(s) = -\frac{s^2}{s^2 + 2\zeta w_n s + w_n^2} Z(s)$$

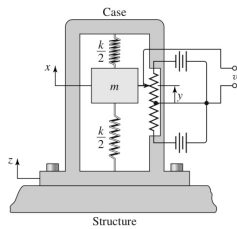
$y = x - z$: direct measurement
 $b/2$: damping top & bottom
 want to measure earthquake $z(t)$
 from $y(t)$

- if $w \gg w_n \Rightarrow y(t) \approx -z(t)$
- want to have a small $w_n \Rightarrow$ large m & soft k (i.e., $x(t) \approx 0$)

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Accelerometer



system dynamics:

$$m\ddot{y} + b\dot{y} + ky = -m\ddot{z} = -ma$$

$$Y(s) = -\frac{1}{s^2 + 2\zeta w_n s + w_n^2} A(s)$$

$y = x - z$: direct measurement
 $b/2$: damping top & bottom
 want to measure $\ddot{z}(t)$ from $y(t)$

- if $w \ll w_n \Rightarrow y(t) \approx -\frac{1}{w_n^2} a(t)$
- want to have a high $w_n \Rightarrow$ small m & stiff k

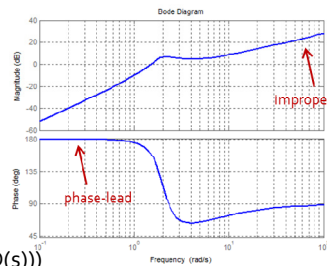
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More Comments on Bode Plots

- phase-lead isn't forecasting of a future:
merely a phase-shift in the sinusoid output
in steady-state

$$y(t) = A|H(j\omega)| \sin(\omega t + \angle H(j\omega))$$



- $H(s)$ is improper if $\lim_{\omega \rightarrow \infty} |H(j\omega)| \rightarrow \infty$ (i.e., $\alpha(N(s)) > \alpha(D(s))$)
⇒ not realizable since it requires differentiation
- we may define Bode plot purely mathematically by $(20\log_{10}|H(j\omega)|, \angle H(j\omega))$:
this then represents steady-state gain/phase if $H(s)$ is stable
- Bode gain-phase relation: $|G(j\omega)|$ uniquely determines "minimum" $\angle G(j\omega)$

$$\angle G(j\omega_0) = \frac{1}{\pi} \int_{-\infty}^{+\infty} \frac{d \ln |G(j\omega)|}{d \ln \omega} \ln \left| \frac{\omega + \omega_0}{\omega - \omega_0} \right| \frac{d\omega}{\omega}$$

↙ slope
↘ sampling around ω_0

- * can't design $|G(j\omega)|$ and $\angle G(j\omega)$ independent
- * non-minimum phase: $|G(j\omega)|$, but non-minimum $\angle G(j\omega)$

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Example: Non-Minimum Phase System

non_minimum.m

$$H(s) = \frac{s - 10}{s^2 + 6s + 10}$$



$$bode[H(s)] = bode\left[\frac{s-10}{10}\right] + bode\left[\frac{10}{s^2+6s+10}\right]$$

- * same gain with $H(s) = \frac{s+10}{s^2+6s+10}$, but non-minimum phase shift
- * $H(s)$ with RHP zeros ⇒ *non-minimum phase*
phase-shift is not minimum among the systems with the same gain plot
- * $H(s)$ without RHP zeros (& delay) ⇒ *minimum-phase* system

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Next Lecture

- control