# 465.211 Mechanics in Energy Resources Engineering, Spring 2010

### 1st Exam

### 09:00 - 11:00 31 March 2010

\* Be careful about the units that you use.

\* You may use Korean or English in answering above questions.

1. Indicate T(true) or F(false) for the following statements. <u>Note that an incorrect answer</u> receives -2 mark while a correct answer receives +2 mark. You may leave the question bla you wish.	<u>ank if</u> (10)
(1) When a material follows the Hooke's Law upon loading, this material can be said to be 'elastically'.	have (T, F)
(2) When a material have the same properties at every point, this material is said to be 'homogeneous'.	(T,F)
(3) Materials having the same properties in all directions are said to be 'isotropic'.	(T,F)
(4) While normal strain is related to the change in length, shear strain is related to the chan the shape.	nge of (T, F)
(5) When additional strains are generated with time even without stress increase, this phenomenon is called 'Creep'.	(T, F)

## 2.

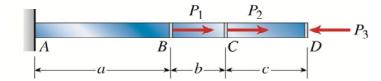
(1) Explain Saint Venant's Principle. Do you think the material properties such as modulus of elasticity (elastic modulus, E) affect the behavior explained by this principle? (5)

(2) Explain the 'statically indeterminate structures' and list an example of this type and another example of 'statically determinate structure' with brief explanations.(5)

(3) When a circular bar is subject to a torsion, shear stresses vary linearly with the distance from the center of the bar and maximum shear stress is generated at the outer surface of the bar.Explain why this is the case.(5)

(4) Explain the mechanism of how thermal stress is generated when temperature increases. List the important properties of materials that may affect the magnitude of the thermal stress with brief explanations. (5)

3. A homogeneous steel bar AD (see figure) has a cross-sectional area of 260 mm<sup>2</sup> and is loaded by forces  $P_1 = 12$  kN,  $P_2 = 10$  kN, and  $P_3 = 6$  kN. The lengths of the segments of the bar are a = 1.5 m, b = 0.6 m, and c = 0.9 m. By what amount P should the load  $P_3$  be increased so that the bar does not change in length when the three loads are applied? (10)



4. At room temperature (25 °C), a brass wire of diameter d = 3.0 mm is stretched tightly between rigid supports so that the tensile force is T = 200 N (see figure). The coefficient of thermal expansion for the wire is 19.5 x  $10^{-6}$ /°C and the modulus of elasticity is E = 110 GPa. (10)

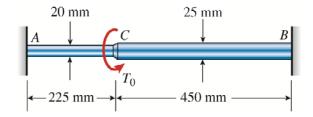
(a) At what temperature does the maximum shear stress in the wire reach 30 MPa?

(b) At what temperature does the wire go slack?

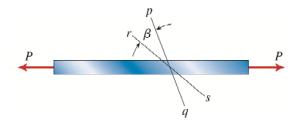
5. A solid aluminum bar of diameter d = 25 mm is subjected to torques T = 2500 Nm acting in the directions shown in the figure. Determine the maximum shear, tensile, and compressive stresses in the bar and show these stresses on sketches of properly oriented stress elements. (10)

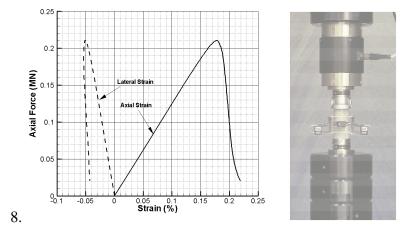


6. A stepped shaft ACB having solid circular cross sections with two different diameters is held against rotation at the ends (see figure). If the allowable shear stress in the shaft is 30 MPa, what is the maximum torque  $(T0)_{max}$  that may be applied at section C? (10)



7. The normal stress on plane pq of a prismatic bar in tension (see figure) is found to be 57 MPa. On plane *rs*, which makes an angle  $\beta = 30^{\circ}$  with plane pq, the stress is found to be 23 MPa. Determine the maximum normal stress  $\sigma_{max}$  and maximum shear stress  $\tau_{max}$  in the bar. (10)

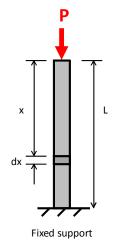




The above graph was obtained from a uniaxial compressive test on a rock (granite) from a site for the underground storage of carbon dioxide. The graph shows the response of axial strain and lateral strains against the axial force applied on a cylindrical rock sample whose diameter is 5.4 cm and length is 10.0 cm (as shown in the picture). What are the uniaxial compressive strength (also called ultimate stress), elastic modulus and Poisson's ratio of the rock measured from the above experiment? Use the relevant units when necessary. (10).

9. Determine the strain energy of a cylindrical bar on a ground as shown in the below. Assume linearly elastic behavior and consider both the weight of the bar itself and a load P at the upper end. Dimensions and material properties are as follows. (10)

L: 10 m, Diameter: 20 cm, density of the bar: 2500 kg/m<sup>3</sup>, P: 100 kN, Elastic modulus: 50 GPa,



#### **Formula Sheet**

Chapter 1.

$$\sigma = \frac{P}{A} \qquad \varepsilon = \frac{\delta}{L} \qquad \sigma = E\varepsilon \qquad v = -\frac{lateral \ strain}{axial \ strain} = -\frac{\varepsilon'}{\varepsilon} \qquad \varepsilon' = -v\varepsilon$$
$$\sigma_b = \frac{F_b}{A_b} \qquad \tau_{aver} = \frac{V}{A} \qquad \tau_1 = \tau_2 \qquad \tau = G\gamma \qquad G = \frac{E}{2(1+v)}$$

Chapter 2.

$$P = k\delta \qquad \delta = fP \qquad f = \frac{1}{k} \qquad \delta = \frac{PL}{EA} \qquad k = \frac{EA}{L} \qquad f = \frac{L}{EA}$$
$$\delta = \sum_{i=1}^{n} \frac{N_i L_i}{E_i A_i} \qquad \delta = \int_0^L d\delta = \int_0^L \frac{N(x) dx}{EA(x)} \qquad \varepsilon_T = \alpha \left(\Delta T\right) \qquad \delta_T = \varepsilon_T L = \alpha \left(\Delta T\right) L$$
$$\sigma_\theta = \sigma_x \cos^2 \theta = \frac{\sigma_x}{2} \left(1 + \cos 2\theta\right) \qquad \tau_\theta = -\sigma_x \cos \theta \sin \theta = -\frac{\sigma_x}{2} \left(\sin 2\theta\right) \qquad U = W = \int_0^\delta P_1 d\delta dt$$
$$U = W = \frac{P\delta}{2} \qquad U = \frac{P^2 L}{2EA} \qquad U = \frac{EA\delta^2}{2L} \qquad U = \frac{P^2}{2k} \qquad U = \frac{k\delta^2}{2} \qquad \delta = \frac{2U}{P}$$
$$u = \frac{\sigma^2}{2E} \qquad u = \frac{E\varepsilon^2}{2} \qquad K = \frac{\sigma_{max}}{\sigma_{nom}}$$

Chapter 3. Torsion

 $\theta = \frac{d\phi}{dx}$   $\gamma_{\max} = \frac{rd\phi}{dx} = r\theta$   $\gamma_{\max} = r\theta = \frac{r\phi}{L}$   $\gamma = \rho\theta = \frac{\rho}{r}\gamma_{\max}$ 

$$\tau_{\max} = Gr\theta$$
  $\tau = G\rho\theta = \frac{\rho}{r}\tau_{\max}$ 

$$I_p = \int_A \rho^2 dA \qquad I_p = \frac{\pi r^4}{2} = \frac{\pi d^4}{32}$$
 (for a circle of radius r and diameter d)

$$\tau_{\max} = \frac{T_r}{I_p} \qquad \tau_{\max} = \frac{16T}{\pi d^3} \qquad \tau = \frac{\rho}{r} \tau_{\max} = \frac{T\rho}{I_p} \qquad \theta = \frac{T}{GI_p} \qquad \phi = \frac{TL}{GI_p}$$

 $I_{p} = \frac{\pi}{2} \left( r_{2}^{4} - r_{1}^{4} \right) = \frac{\pi}{32} \left( d_{2}^{4} - d_{1}^{4} \right)$  (for a tube with radius r1 and r2)

 $I_{p} = \frac{\pi rt}{2} \left( 4r^{2} + t^{2} \right) = \frac{\pi dt}{4} \left( d^{2} + t^{2} \right)$  (with average radius r, average diameter d, wall thickness t)

$$\phi = \sum_{i=1}^{n} \phi_i = \sum_{i=1}^{n} \frac{T_i L_i}{G_i (I_p)_i} \qquad \phi = \int_0^L d\phi = \int_0^L \frac{T dx}{GI_p(x)} \qquad \phi = \int_0^L d\phi = \int_0^L \frac{T(x) dx}{GI_p(x)}$$

$$\sigma_{\theta} = \tau \sin 2\theta$$
  $\tau_{\theta} = \tau \cos 2\theta$   $\gamma = \frac{\tau}{G}$   $\varepsilon_{\max} = \frac{\gamma}{2}$   $G = \frac{E}{2(1+\nu)}$ 

$$U = W = \frac{T\phi}{2} \qquad \qquad U = \frac{T^2L}{2GI_p} \qquad \qquad U = \frac{GI_p\phi^2}{2L} \qquad u = \frac{\tau^2}{2G} \qquad \qquad u = \frac{G\gamma^2}{2}$$

Additional formulas

 $\sin(A\pm B) = \sin A \cos B \pm \cos A \sin B$ 

 $\cos(A\pm B) = \cos A \cos B \mp \sin A \sin B$