#### **Mechanics of Composite Materials**

#### CHAPTER 5. Failure of Composite

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Need elaborate 3-D analysis

 $au_{zy}$ , Important for edge effects and also plate bending



Layers slide over each other unless restrained by  $\tau_{zx}$ 

will discuss Interlaminar Stress later.

Now return to ply stresses from CLPT, and how they can cause failure For laminate, have found ply stresses and strains on ply-by-ply basis Try to use these to assess failure.

For a single ply: At least 5 failure mechanisms  $\rightarrow$  5 quantities need to be measured from unidirectional tests.

- $X_{t}$  longitudinal (L-direction) ultimate tensile stress
- $X_c$  longitudinal (L-direction) ultimate compressive stress
- $Y_t$  transverse (T-direction) ultimate tensile stress
- $Y_c$  transverse (T-direction) ultimate compressive stress
- S Ultimate shear stress
  - : Generally all different for a composite
- Basic Building Blocks

Try to extend these to combined stress states.

Several theories proposed.

(a) Maximum Stress Theory

Failure occurs if any one of following occurs

$$\sigma_{1} \geq X_{t}$$

$$\sigma_{1} \leq X_{c}$$

$$\sigma_{2} \geq Y_{t}$$

$$\sigma_{2} \leq Y_{c}$$

$$\sigma_{6} > S$$

where,  $\sigma_1, \sigma_2, \sigma_6 \rightarrow$  Stresses in ply coordinates  $X_c, Y_c \rightarrow \text{Negative}$ 

 $\sigma_6 \rightarrow$  Sign unimportant

Then, allowable stress values are

$$X_c \le \sigma_1 \le X_t$$
$$Y_c \le \sigma_2 \le Y_t$$
$$-S \le \sigma_6 \le S$$

Can draw failure envelope in  $\sigma_1, \sigma_2$  space



Also check  $-S \le \sigma_6 \le S$  (3-Dimensional) Previous Example with  $[0/\pm 45/90]_s$  laminate

ply, $\theta$	$\sigma_{_{1}}$	$\sigma_{_2}$	$\sigma_{_6}$	
$0^{\circ}$	-206	0.3	0	
$+45^{\circ}$	-73	-6.5	9.5	Ksi
-45°	-73	-6.5	-9.5	
$+90^{\circ}$	60	-13	-9.5	

Compare with strengths

$X_{t}$	190 Ksi
$X_{c}$	-160 Ksi
$Y_t$	6 Ksi
$Y_{c}$	- 25 Ksi
S	10 Ksi
$X_t >$	$> X_c$

Note

Crushed the tube.

if pulled tube, all stresses reversed

Failure of  $90^{\circ}$  plies (cracking)

May not be failed, but fiber failure in  $0^{\circ}$  much more fatal

(b) Maximum Strain TheoryLook at strains rather than stressesDefine,

- $\mathcal{E}_{xt}$  longitudinal (L-direction) ultimate tensile strain
- $\mathcal{E}_{xc}$  longitudinal (L-direction) ultimate compressive strain
- $\mathcal{E}_{yt}$  transverse (T-direction) ultimate tensile strain
- $\mathcal{E}_{yc}$  transverse (T-direction) ultimate compressive strain
- $\mathcal{E}_s$  Ultimate shear strain

Then allowable strain values are

 $\varepsilon_{xc} \le \varepsilon_1 \le \varepsilon_{xt}$  $\varepsilon_{xc} \le \varepsilon_2 \le \varepsilon_{yt}$  $-\varepsilon_s \le \varepsilon_6 \le \varepsilon_s$ 

Since stress-strain curves tend to be linear to failure, one can say roughly,

$$\varepsilon_{xt} = \frac{X_t}{E_t}, \quad \varepsilon_{xc} = \frac{X_c}{E_L}, \quad \varepsilon_{yt} = \frac{Y_t}{E_T}, \quad etc$$

#### For T300/934 this would give



Or, can use direct test values of strain

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Max. Strain criteria will be similar to Max. Stress

For the previous example



Similar results as Max. Stress

#### (c) Tsai-Wu Interaction Theory

Want to account for potential interactions between failure mechanisms in  $\sigma_1$ ,  $\sigma_2$ ,  $\sigma_6$ Think back to von Mises criterion for isotropic materials von Mises (isotropic)

$$(\sigma_{x} - \sigma_{y})^{2} + (\sigma_{y} - \sigma_{z})^{2} + (\sigma_{z} - \sigma_{x})^{2} + 3\tau_{xy}^{2} + 3\tau_{yz}^{2} + 3\tau_{zx}^{2} = 2\sigma_{yield}^{2}$$

For 2-D Plane Stress

$$\sigma_{z} = \tau_{zx} = \tau_{yz} = 0$$

$$\sigma_{x}^{2} - 2\sigma_{x}\sigma_{y} + \sigma_{y}^{2} + \sigma_{y}^{2} + \sigma_{x}^{2} + 3\tau_{xy}^{2} = 2\sigma_{yield}^{2}$$
or
$$\sigma_{x}^{2} - \sigma_{x}\sigma_{y} + \sigma_{y}^{2} + \frac{3}{2}\tau_{xy}^{2} = \sigma_{yield}^{2} = X^{2}$$

$$\left(\frac{\sigma_{x}}{X}\right)^{2} - \left(\frac{\sigma_{x}\sigma_{y}}{X}\right)^{2} + \left(\frac{\sigma_{y}}{X}\right)^{2} + \left(\frac{\tau_{xy}}{\sqrt{\frac{3}{2}}X}\right)^{2} = 1$$

one failure value X

For Principal Stresses,

$$\sigma_x = \sigma_1, \ \sigma_y = \sigma_2, \ \tau_{xy} = 0,$$
 this gives





Hill generalized von Mises for Orthotropic material.

$$\frac{(\sigma_1)^2 - (\sigma_1 \sigma_2)^2 + (\sigma_2)^2 + (\sigma_2)^2 + (\sigma_2)^2 + (\sigma_2)^2 = 1}{\uparrow}$$

Interaction term

- X: strength in longitudinal
- Y: strength in Transverse

Tsai-Wu proposed more general criteria

$$\sigma_1 F_1 + \sigma_2 F_2 + \sigma_1^2 F_{11} + \sigma_2^2 F_{22} + \sigma_6^2 F_{66} + 2F_{12}\sigma_1\sigma_2 = 1$$

where  $F_{i}$ ,  $F_{ij}$  are coefficients to be found from tests

To find coefficients

a) consider  $\sigma_1$  only

$$\sigma_1 F_1 + \sigma_1^2 F_{11} = 1$$

for tension  $X_t F_1 + X_t^2 F_{11} = 1$ 

for compression  $X_{c}F_{1} + X_{c}^{2}F_{11} = 1$ 

Solving gives

$$F_{1} = \frac{1}{X_{t}} + \frac{1}{X_{c}}$$
$$F_{11} = -\frac{1}{X_{t}X_{c}}$$

Similarly for 
$$\sigma_2$$
 only  $F_1 = \frac{1}{Y_t} + \frac{1}{Y_c}$   
 $F_{11} = -\frac{1}{Y_tY_c}$ 

for  $\sigma_6$  only  $F_6 = 0$  $F_{66} = \frac{1}{S_2}$ 

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Correction on von Mises criterion

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1) 
$$(\sigma_x - \sigma_y)^2 + (\sigma_y - \sigma_z)^2 + (\sigma_z - \sigma_x)^2 + 6\tau_{xy}^2 + 6\tau_{yz}^2 + 6\tau_{zx}^2 = 2\sigma_{yield}^2$$
  
2)  $\sigma_x^2 - 2\sigma_x\sigma_y + \sigma_y^2 + \sigma_y^2 + \sigma_x^2 + 6\tau_{xy}^2 = 2\sigma_{yield}^2$   
3)  $\left(\frac{\sigma_x}{X}\right)^2 - \left(\frac{\sigma_x\sigma_y}{X}\right)^2 + \left(\frac{\sigma_y}{X}\right)^2 + \left(\frac{\tau_{xy}}{\frac{1}{\sqrt{3}}X}\right)^2 = 1$ 

$$F_{1}\sigma_{1} + F_{2}\sigma_{2} + F_{6}\sigma_{6} + F_{11}\sigma_{2}^{2} + F_{66}\sigma_{6}^{2} + 2F_{12}\sigma_{1}\sigma_{2} = 1$$

$$\begin{cases} \sigma_{1} = X_{t} \\ \sigma_{1} = X_{c} \end{cases} \qquad \begin{cases} \sigma_{2} = Y_{t} \\ \sigma_{2} = Y_{c} \end{cases} \qquad \begin{cases} \sigma_{6} = +S \\ \sigma_{6} = -S \end{cases}$$

For obtaining  $F_{12'}$  should use a biaxial  $\sigma_1 = \sigma_2 = X_B$ 

$$(F_{1} + F_{2})X_{B} + (F_{11} + F_{22})X_{B}^{2} + 2F_{12}X_{B}^{2} = 1$$
  
$$F_{12} = \frac{1}{2X_{B}}[1 - (F_{1} + F_{2})X_{B} - (F_{11} + F_{22})X_{B}^{2}]$$

convenient to express  $F_{12}$  as  $F_{12} = \eta / \sqrt{X_t X_c Y_t Y_c}$ 

where  $\eta$  is between 0 and -1



Biaxial Tests for  $F_{12}$  hard to do. For simplicity, sometimes assume

 $F_{12} \cong -0.5 \, / \, \sqrt{X_t X_c Y_t Y_c}$ 

comes from analogy with von Mises criteria for isotropic materials.

Summarizing, Tsai-Wu Criteria becomes

 $F_1\sigma_1 + F_2\sigma_2 + F_6\sigma_6 + F_{11}\sigma_2^2 + F_{66}\sigma_6^2 + 2F_{12}\sigma_1\sigma_2 = 1$ Based on tests for  $X_t, X_c, Y_t, Y_c, S$ Gives reasonable empirical fit to data See Jones p. 80, Tsai and Hahn



• Word of Caution

For two materials, different  $Y_t$ 



- Quads I II IV reasonable
- Quads III increased strength in biaxial

compression?

Careful, if designing submarine hulls

- Max. Stress Theory for Quad III
- Discussion of Failure Criteria given by

Hart – Smith, Composite (24), 1, 1993, p. 53-



- Remarks on Failure Criteria
- 1. Failure criteria useful to interpolate experimental data.
- 2. Don't use to extrapolate particularly into quadrants without test data
- 3. Useful for preliminary laminate design
- 4. Failure is complicated.
  - Ply behavior in a laminate may differ from simple lamina
     (delamination, edge effects, ··· )
  - Size effects, holes
  - Fatigue
  - Environment

5. But Failure criteria have their uses, easily employed, simple equations. Don't want to do full set of strength tests for each layup.

Calculating Laminate Failure
Up to here, looked at failure of a ply in isolation.
Combine with laminate analysis (CLPT)
Procedure

- 1. Analyze structure  $\rightarrow$  get  $N, A, \varepsilon^{\circ}$  (also  $B, D, M, \kappa$ )  $\rightarrow$  later on
- 2. Determine ply stresses (strains) in ply axes.
- 3. Apply appropriate criteria to each ply
- 4. Determine first ply to fail. (First Ply Failure)
- 5. To work out, ply stress

$$\sigma = T_{\sigma} \overline{\sigma} = T_{\sigma} \overline{Q} \varepsilon^{\circ} = T_{\sigma} \overline{Q} a N$$

Ply coords. Laminate coords. Linear eqn.

Using Max. Stress Theory one part of failure criterion

$$\sigma_{1} \leq X_{t}$$

$$\frac{\sigma_{1}(N)}{X_{t}} = 1$$
1 equation, 3 unknowns,  $N_{x'}$ ,  $N_{y'}$ ,  $N_{xy}$ 

To solve for failure loads



Failure occurs at  $\tilde{N} = \lambda \tilde{N}^{o}$ Repeat for others Using Tsai-Wu

 $\lambda F_{1}\sigma_{1}^{o} + \lambda F_{2}\sigma_{2}^{o} + \lambda^{2}\sigma_{1}^{o^{2}}F_{11} + \lambda^{2}\sigma_{2}^{o^{2}}F_{22} + \lambda^{2}\sigma_{6}^{o^{2}}F_{66} + \lambda^{2}\sigma_{1}^{o}\sigma_{2}^{o}F_{12} = 1$ 

Quadratic for  $\lambda$ , solve Failure at  $N = \lambda N^{o}$ 

Alternatively, do incremental computation

• First-ply Failure for the previous example



Using Max. Stress 
$$\Rightarrow \lambda = \frac{X_c}{\sigma_1} = \frac{-160}{-206} = 0.78$$
  
So permissible loading is  
 $\tilde{N} = 0.78 \begin{cases} -3183 \\ 0 \\ 0 \end{cases} = \begin{cases} -2483 \\ 0 \\ 0 \end{cases}$   
 $N_x = \frac{P}{2\pi r}, P = 2\pi(1)(-2483) = 15,600 \text{ compression}$ 

Note if applied Tsai-Wu

$$\lambda = 0.76$$
  
 $N_x = 0.76(-3183) = -2419$   
 $P = 15,190 \ lbs$   $\leftarrow$  small effect of transverse tension on  $O^\circ$  ply

Generally repeat to check for  $Y_{c'}$   $S_{r}$  etc.

(above for  $\sigma$  is max. stress)

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• Progressive Failure

Does failure of a ply = failure of the laminate?

Not necessarily.

If tube loaded in tension



same stress state with all signs changed

	ply, $ heta$	$\sigma_{_1}$	$\sigma_{_2}$	$\sigma_{_6}$
	$0^{\circ}$	206	-0.3	0
	$+45^{\circ}$	+73	+6.5	-9.5
	-45°	+73	+6.5	+9.5
	$+90^{\circ}$	+60	+13	0
Allowables +190		+6	± 10	
		l -160	-25	

90°s would crack in tension at low load.

$$\sigma_{2} = +13 \ ksi, \ Y_{t} = +6,$$
  

$$\lambda = Y_{t} / \sigma_{2} = 6 / 13 = 0.46$$
  

$$N_{x} = 0.46(+3183) = 1464 \ lb / in$$
  
or  $P = 2\pi r N_{x} = 9,213 \ lb$ 

But does this kill the laminate?

Assume 90° ply cracks and looses transverse stiffness  $E_t = kE_T$ 

knock down factor

More generally assume all properties at cracked ply are knocked down.

$$E_{t}' = k_{T}E_{T}, E_{L}' = k_{L}E_{L}, v_{LT}' = K_{v}v_{LT}, G_{LT}' = K_{G}G_{LT}$$

Select K's somewhat arbitrary

Туре	$K_{L}$	$K_{T}$	$K_{v}$	$K_{G}$
Drastic crack	1	0.001	0.001	0.001
Typical crack	1	0.5	1	0.5
Some Research	1	0.5	1	1
Fiber	10 <sup>-6</sup>	10 <sup>-6</sup>	10 <sup>-6</sup>	10 <sup>-6</sup>



- Do all calculation once again
- Recompute stresses, get new  $\lambda$

new failure loads

- Automate on computer

Typical Progression for  $[0/\pm 45/90]_s$ 

λ	$   N_x = 3183\lambda ) $	$P = (= 628N_x)$	Failu	re	$\overline{E_x}$ (=7.7 <i>Msi</i> , orig)
0.42	1340	8400	cracking	90° plies	7.5
0.65	2070	13,000	cracking	$\pm$ 45° plies	7.2
1 0.86	2740	17,200	cracking	0° plies	0

Last ply to fail /

Jones Sec 4

• Other Remarks on Failure

Failure of laminates is complicated phenomenon.

Many modes of failure

- 1. Fiber failure
- 2. Matrix cracking
- 3. Delamination
- 4. Interlaminar stress effects near free edge
- 5. Effects of holes and notches
- 6. Residual initial stresses due to thermal contraction

From cure temperature, some observations

• Interlaminar stresses at free edge can cause delamination. Tests on  $[\pm \theta]_s$  laminate show



For  $\theta > 30^{\circ}$ , Delamination failure

"Brooming" strong  $\sigma_z$  develops

For  $\theta > 30^{\circ}$ , Regular fracture



- Holes in composites lower strength



Holes in composites behave like cracks in metals.

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Much work on failure still going on.

Research on Impact, Fatigue, Environment (temperature, moisture) For example with conservative safety factor, can design reasonable composite structure.

Better understanding → lower safety factors

more efficient design See Jones Sections 6.1 ~ 6.4