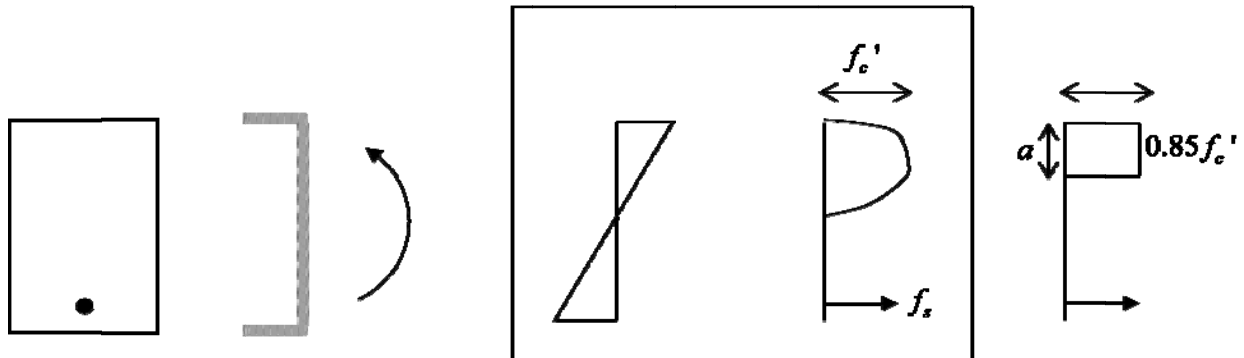


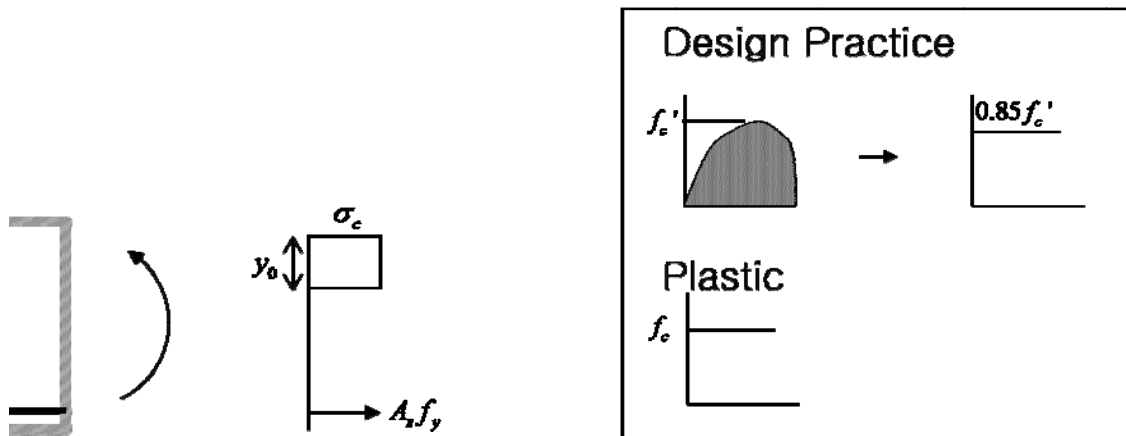
## 5. Beams

- bending
- shear
- torsion

### 5.1 Beams in Bending



Plastic approach



-equilibrium

$$\sigma_c b y_0 = A_s f_y$$

-yield condition

$$\sigma_c \leq f_c$$

$$y_0 = \frac{A_s f_y}{b f_c}$$

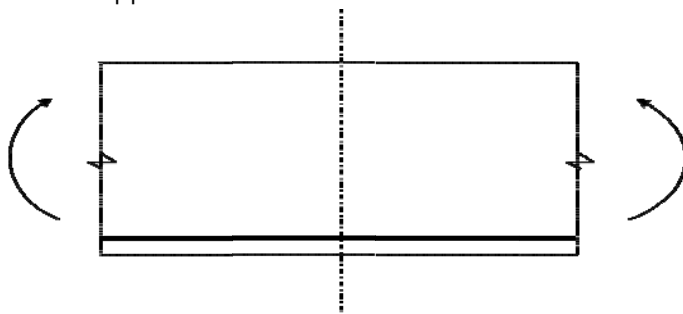
$$M = A_s f_y \left( d - \frac{y_0}{2} \right)$$

Let  $\phi = \frac{A_s}{bd} \cdot \frac{f_y}{f_c}$  (w: mechanical reinf. ratio)

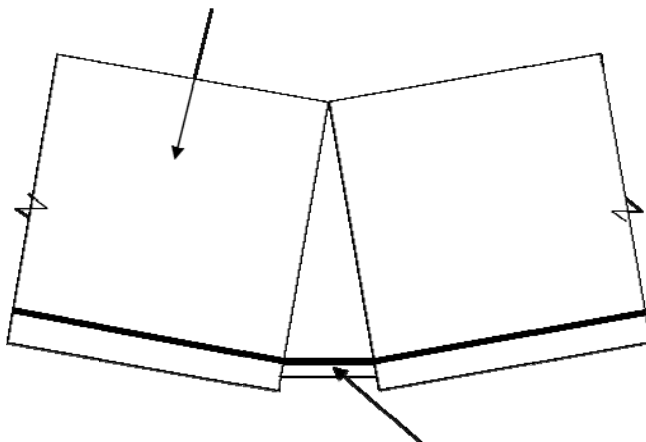
$$M_u = \left(1 - \frac{\phi}{2}\right) \phi b d^2 f_c$$

$$y_0 = \frac{A_s f_y}{b f_c} = \frac{A_s}{bd} \cdot \frac{f_y}{f_c} \cdot d = \phi d$$

- Upper bound solution



concrete strength =  $\infty$



For finite compressive strength

Assume two lines of discontinuity

OA&OB

1) Constant strain expansion of region OAB

2) 2<sup>nd</sup> mechanism : rigid body rotation

Dissipation work

We know that the first principal direction of strain is found to bisect the angle between the displacement vector and n-axis.

∴ direction of plastic displacement vector along OA is  $\alpha = 90 - 2\beta$

i.e.  $|v| = |v_2|$

$$W_l = \frac{1}{2} f_c v b (1 - \sin \alpha)$$

$$\text{적분} \rightarrow W_l = \frac{1}{2} b f_c \frac{y}{\sin \beta} w_1 [1 - \sin(90^\circ - 2\beta)] \frac{d_y}{\sin \beta}$$

$$W_{right} = \int_0^{y_0} W_l dl = \frac{1}{2} b f_c \frac{y_0^2}{2} \frac{1 - \cos 2\beta}{\sin^2 \beta} w_1 = \frac{1}{2} b f_c y_0^2 w_1$$

$$W_{total} = W_{right} + W_{rebar} = \frac{1}{2} b f_c y_0^2 w_1 + A_s f_y \left( d - \frac{y_0}{2} \right) w_1$$

$$W_{er} = M w_1$$

$$W_d = (w_1 + w_2) \left[ \frac{1}{2} b f_c y_0^2 + A_s f_y \left( d - \frac{y_0}{2} \right) \right]$$

$$W_l = M (w_1 + w_2)$$

$$W_d = W_l$$

$$\rightarrow M_p = \frac{1}{2} b f_c y_0^2 + A_s f_y (d - y_0)$$

$$\frac{\partial M_p}{\partial y_0} = b f_c y_0 - A_s f_y = 0$$

$$i\hat{A} \quad y_0 = \frac{A_s f_y}{b f_c}$$

Lowest upper bound solution

$$M_p = A_s f_y \left( d - \frac{y_0}{2} \right)$$

## 5.2 Beams in shear

ACI code

$$V_n = V_c + V_s = 2\sqrt{f'_c}bd + \frac{A_s f_y}{s}d \cot \theta \rightarrow \theta = 45^\circ$$

-axial forces: stringers

-shear force: compression field

“ç concrete stresses

$$\begin{cases} \sigma_{cx} = -\sigma_c \cos^2 \theta \rightarrow 1 \\ \sigma_{cy} = -\sigma_c \sin^2 \theta \rightarrow 2 \\ \tau_{cxy} = \sigma_c \sin \theta \cos \theta \rightarrow 3 \end{cases}$$

② steel stress

$$\sigma_{sy} = \frac{A_s}{cb} \sigma_s = r \sigma_s \rightarrow 4$$

● Total stresses

$$\begin{cases} \sigma_x = \sigma_{cx} + \sigma_{sx} = -\sigma_c \cos^2 \theta \rightarrow 5 \\ \sigma_y = \sigma_{cy} + \sigma_{sy} = -\sigma_c \sin^2 \theta + r \sigma_s \rightarrow 6 \\ \tau = \sigma_c \sin \theta \cos \theta \rightarrow 7 \end{cases}$$

From 7,

$$\sigma_c = \frac{\tau}{\sin \theta \cos \theta} = \frac{\tau(\sin^2 \theta + \cos^2 \theta)}{\sin \theta \cos \theta} = \tau(\tan \theta + \cot \theta) \rightarrow 8$$

● Boundary conditions at the top face

$$\left( \begin{array}{l} \sigma_y = 0 \\ r\sigma_s = \sigma_c \sin^2 \theta (\rightarrow 10) = \frac{\tau}{\sin \theta \cos \theta} \sin^2 \theta = \tau \tan \theta \rightarrow 10' \\ \sigma_x = -\frac{\tau}{\sin \theta \cos \theta} \cos^2 \theta = -\tau \cot \theta \rightarrow 11 \end{array} \right.$$

8, 10' & 11 → expressions I . t . o .  $\tau$

- Yield conditions

$$\left( \begin{array}{l} \sigma_c = \tau(\tan \theta + \cot \theta) \leq f_c \\ \sigma_s \leq f_Y \\ T \leq A_s f_Y \quad \& \quad c \leq c_f \end{array} \right.$$

Failure modes

- ① stirrup yielding
- ② web concrete crushing

Max. shear capacity

→ web crushing failure mode (stirrup yielding first)

$$\left( \begin{array}{l} \sigma_c = \tau(\tan \theta + \cot \theta) \\ r\sigma_s = \tau \tan \theta \end{array} \right. \rightarrow \left( \begin{array}{l} f_c = \tau(\tan \theta + \cot \theta) \rightarrow 16 \\ r f_Y = \tau \tan \theta \rightarrow 17 \end{array} \right.$$

16→17

$$\tan \theta = \frac{r f_Y}{\tau} = \frac{r f_Y}{f_c} \left( \tan \theta + \frac{1}{\tan \theta} \right) \rightarrow 18$$

$$\text{Let, } \psi = \frac{r f_Y}{f_c} w$$

Eq.18 is written as

$$\psi = \frac{\tan \theta}{\tan \theta + \frac{1}{\tan \theta}} = \frac{\tan^2 \theta}{\tan^2 \theta + 1}$$

$$\psi \tan^2 \theta + \psi = \tan^2 \theta$$

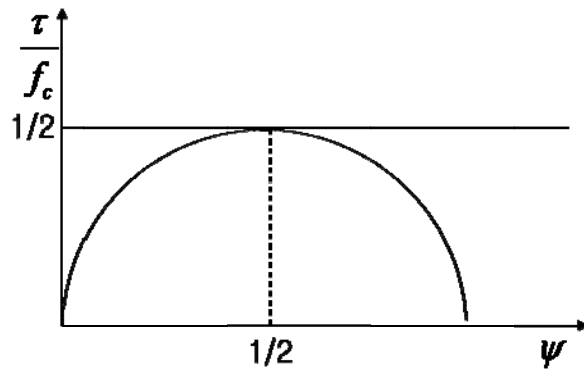
$$\tan^2 \theta = \frac{\psi}{1 - \psi} \rightarrow \tan \theta = \sqrt{\frac{\psi}{1 - \psi}} \rightarrow 19 \quad (5.2.14)$$

Hence eq. 16 is expressed as

$$\frac{\tau}{f_c} = \frac{1}{\tan \theta + \frac{1}{\tan \theta}} = \frac{\tan \theta}{\tan^2 \theta + 1} \rightarrow 20$$

19→20

$$\frac{\tau}{f_c} = \sqrt{\frac{\psi}{1-\psi}} \cdot \frac{1}{\frac{\psi}{1-\psi} + 1} = \sqrt{\psi(1-\psi)}$$



Design practice

-Min. stirrup:  $s < d$

-Max. stirrup:  $V_s \leq 8\sqrt{f'_c}bd$

$$f_d = \tau(\tan \theta + \cot \theta) < f_c$$

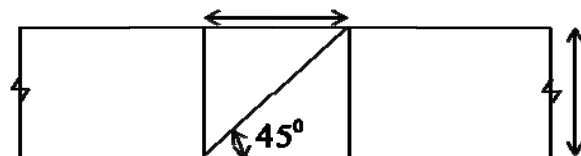
$$= \frac{V}{\sin \theta \cos \theta} \cdot \frac{1}{bd}$$

$$= \frac{A_s f_y d \cot \theta}{s} \cdot \frac{1}{\sin \theta \cos \theta} \cdot \frac{1}{bd}$$

Assume  $\theta = 45^\circ$

$$f_d = \frac{A_s f_y d}{s} \cdot \frac{2}{bd} < f_c$$

$$V_s = \frac{A_s f_y d}{s} < \frac{1}{2} f_c bd = 8\sqrt{f'_c}bd$$



$$D = \sigma_c bd \cos \theta$$

$$V = D \sin \theta = \sigma_c bd \sin \theta \cos \theta$$

- Shift rule

$$C = \frac{M}{d} - \frac{V}{2} \cot \theta$$

$$T = \frac{M}{d} + \frac{V}{2} \cot \theta$$

Let,  $\cot \theta = k$

Free body diagram



-concrete stresses(1)

$$\begin{cases} \sigma_{cx} = -\sigma_c \cos^2 \theta \\ \sigma_{cy} = -\sigma_c \sin^2 \theta \\ \tau_{cxy} = \sigma_c \sin \theta \cos \theta \end{cases}$$

-steel stresses(2)

$$\begin{cases} \sigma_{sx} = 0 \\ \sigma_{sy} = r\sigma_s \\ \tau_{sxy} = 0 \end{cases}$$

-total stresses(3) iç (1)+(2)

$$\begin{cases} \sigma_x = -\sigma_c \cos^2 \theta \\ \sigma_y = -\sigma_c \sin^2 \theta + r\sigma_s \\ \tau_{cxy} = \sigma_c \sin \theta \cos \theta \end{cases}$$

Eq's (3) are rewritten (@ top)

$$\begin{cases} \sigma_x = -\tau_{xy} \frac{\cos \theta}{\sin \theta} = -\tau_{xy} \cot \theta (\leftarrow \tau = p \cot \theta) = -p \cot^2 \theta \\ \sigma_y = -\tau_{xy} \tan \theta + r\sigma_s = -p + r\sigma_s \\ \sigma_c = \tau (\tan \theta + \cot \theta) = p(1 + \cot^2 \theta) \end{cases}$$

Boundary along (1')-(2)-(2') & (2')-(2)-(3)

$$\left( \begin{array}{l} \sigma_s = 0 \\ \sigma_x = -p \cot^2 \theta \\ \sigma_y = -p \\ \sigma_c = p(1 + \cot^2 \theta) = p(1 + k^2) < f_c \end{array} \right. \quad \text{since } \left( \begin{array}{l} \sigma_y = 0, -p + r\sigma_s = 0 \\ r\sigma_s = p \\ \sigma_c = p(1 + k^2) \end{array} \right.$$

Region (2')-(3)-(3')

$$\left( \begin{array}{l} \sigma_x = -2pk^2 \\ \sigma_y = -p + r\sigma_s = -2p \\ \sigma_c = 2p(1 + k^2) \end{array} \right.$$

$$\tau = 2p \cot \theta$$

$$\tau = \frac{pa}{h} \quad (39)$$

$$\sigma_s = \frac{\tau \tan \theta - p}{r} \quad (40)$$

$$\sigma_x = -\tau \cot \theta \quad (41)$$

$$\sigma_y = -\tau \tan \theta \quad (42)$$

$$\sigma_c = \tau(\tan \theta + \cot \theta) \quad (43)$$

For web crushing failure

$$\begin{cases} \sigma_c = f_c \\ \sigma_s = f_y \end{cases} \quad (a)$$

Substituting (a) into (40) yields

$$f_y = \frac{\tau \tan \theta - p}{r} \quad (b)$$

Eq(43) becomes

$$\frac{f_c h}{pa} = \tan \theta + \cot \theta \quad (c)$$

Sub. (39) into (b) yields

$$f_y = \frac{\frac{pa}{h} \tan \theta - p}{r} \rightarrow \tan \theta = (f_y r + p) \frac{h}{pa}$$

$$\text{Let } \psi = \frac{rf_y}{f_c}$$

$$\tan \theta = \frac{f_c h}{p a} \left( \psi + \frac{p}{f_c} \right) \quad (d)$$

Sub. (c) into (d) yields

$$\tan \theta = (\tan \theta + \cot \theta) \left( \psi + \frac{p}{f_c} \right)$$

$$\psi + \frac{p}{f_c} = \frac{\tan^2 \theta}{\tan^2 \theta + 1}$$

$$\tan^2 \theta \left[ \left( \psi + \frac{p}{f_c} \right) - 1 \right] = - \left( \psi + \frac{p}{f_c} \right) \quad (e)$$

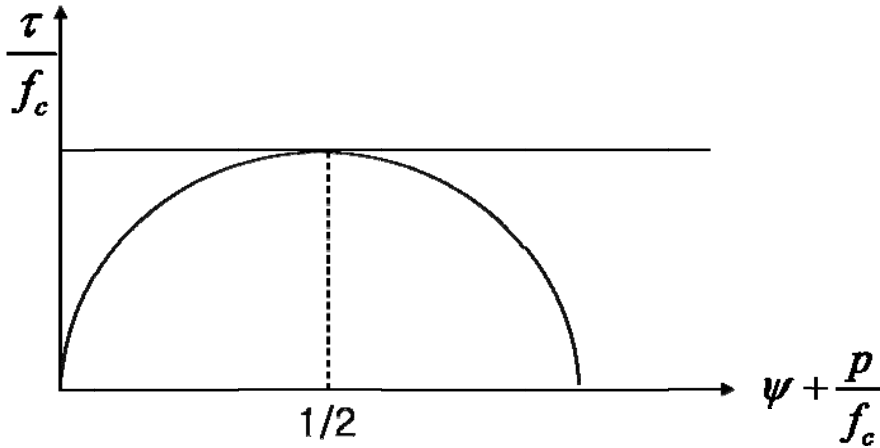
Eq(5.2.33) becomes

$$\frac{\tau}{f_c} = \frac{1}{\tan \theta + \cot \theta} = \frac{\tan \theta}{\tan^2 \theta + 1} \quad (f)$$

Sub. (e) into (f) yields

$$\frac{\tau}{f_c} = \left( \psi + \frac{p}{f_c} \right) \frac{1}{\tan \theta} = \left( \psi + \frac{p}{f_c} \right) \sqrt{\frac{1 - \left( \psi + \frac{p}{f_c} \right)}{\psi + \frac{p}{f_c}}}$$

$$\therefore \frac{\tau}{f_c} = \sqrt{\left(\psi + \frac{p}{f_c}\right) \left[1 - \left(\psi + \frac{p}{f_c}\right)\right]} \quad (5.2.34)$$



Derivation of (5.2.37)

We know  $\tau = \frac{pa}{h}$  (\*)

Sub. (8) into (5.2.34) yields

$$\left(\frac{\tau}{f_c}\right)^2 \left[1 + \left(\frac{h}{a}\right)^2\right] + \frac{\tau}{f_c} \left[2\frac{h}{a}\psi - \frac{h}{a}\right] + \psi^2 - \psi = 0$$

$$\frac{\tau}{f_c} = \frac{1}{2} \frac{\frac{a}{h}}{1 + \left(\frac{a}{h}\right)^2} \left[1 - 2\psi + \sqrt{1 + 4\left(\frac{h}{a}\right)^2 \psi(1-\psi)}\right]$$

Eq (5.2.37) is valid for  $\psi \leq \frac{1}{2} \left(1 - \frac{h}{a}\right)$

$$\psi + \frac{p}{f_c} = \psi + \frac{\tau}{f_c} \frac{a}{h} \leq 0.5$$

$$\psi \leq \frac{1}{2} \left(1 - \frac{2h}{a} \frac{\tau}{f_c}\right) \quad (*)$$

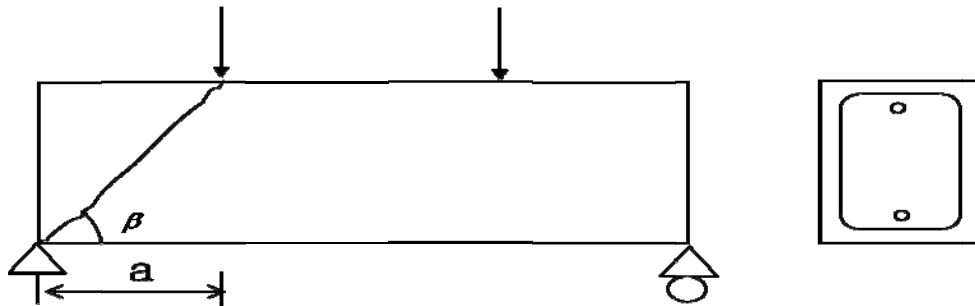
Sub.(47) into(\*) yields

$$\psi \leq \frac{1}{2} \left[ \frac{1 - 2\psi + \sqrt{1 + 4\left(\frac{h}{a}\right)^2 \psi(1-\psi)}}{\left(\frac{a}{h}\right)^2 + 1} \right]$$

The result is

$$\psi \leq \frac{1}{2} \left(1 - \frac{h}{a}\right)$$

Upper bound solutions



$$\textcircled{1} \quad \frac{h}{a} \leq \tan \beta \leq \infty$$

$$w_e = w_i$$

$$Pu = rf_y bh \cos \beta \cdot u + \frac{1}{2} f_c b (1 - \cos \beta) \frac{h}{\sin \beta} \cdot u$$

$$\left( \frac{\tau}{f_c} \right)_{\min} = \sqrt{\psi(1-\psi)}$$

$$\text{Where } \tan \psi = \frac{2\sqrt{\psi(1-\psi)}}{1-2\psi}$$

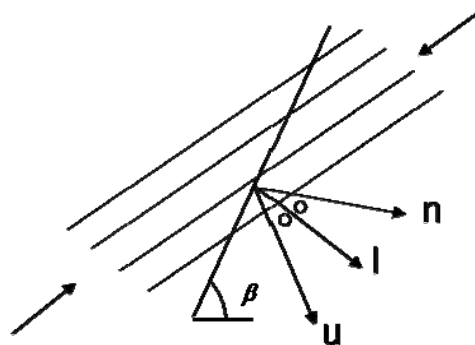
$$\beta = 2\theta$$

$\theta$  : angle of uniform diagonal stress field

$\beta$  : angle of crack

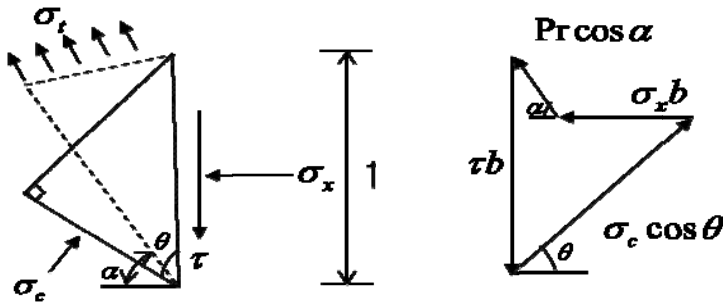
$$\textcircled{2} \quad \tan \beta = \frac{h}{a}$$

$$\frac{\tau}{f_c} = \frac{1}{2} \left[ \sqrt{1 + \left( \frac{h}{a} \right)^2} - \frac{a}{h} \right] + \psi \frac{a}{h}$$



### 5.2.2 Maximum shear capacity inclined shear reinforcement

I - infinitesimal element



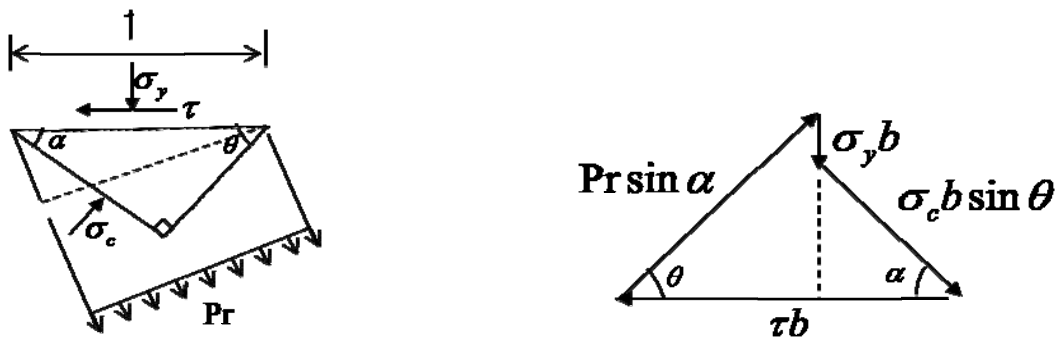
$$P_r \cos \alpha$$

$$P_r = \frac{A_s \sigma_s}{c}$$

$$\sigma_x b = -\sigma_c b \cos \theta + P_r \cos^2 \alpha$$

$$\sigma_x = -\sigma_c \cos^2 \theta + \frac{P_r}{b} \cos^2 \alpha = -\sigma_c \cos^2 \theta + \frac{A_s \sigma_s}{cb} \cos^2 \alpha = -\sigma_c \cos^2 \theta + r \sigma_s \cos^2 \alpha \quad (5.2.47)$$

II - infinitesimal element



$$\sigma_y b = -\sigma_c \sin \theta \cdot \sin \theta + pr \sin \alpha \cdot \sin \alpha$$

$$\sigma_y = -\sigma_c \sin^2 \theta + r \sigma_s \sin^2 \alpha \quad (5.2.48)$$

$$\tau = \sigma_c \cos \theta \cdot \sin \theta + r \sigma_s \cos \alpha \cdot \sin \alpha \quad (5.2.49)$$

Setting  $\sigma_y = 0$

$$r\sigma_s = \frac{\sigma_c \sin^2 \theta}{\sin^2 \alpha} \quad (5.2.50)$$

(50) → (49)

$$\tau = \sigma_c \left( \cos \theta \cdot \sin \theta + \frac{\sin^2 \theta}{\sin \alpha} \cos \alpha \right)$$

$$\sigma_c = \frac{\tau}{\sin^2 \theta (\cot \theta + \cot \alpha)} \quad (5.2.51)$$

(51) → (50)

$$r\sigma_s = \frac{\tau}{\sin^2 \theta} \cdot \frac{1}{\cot \theta + \cot \alpha} \cdot \frac{\sin^2 \theta}{\sin^2 \alpha} = \frac{\tau}{\sin^2 \alpha (\cot \theta + \cot \alpha)} \quad (5.2.52)$$

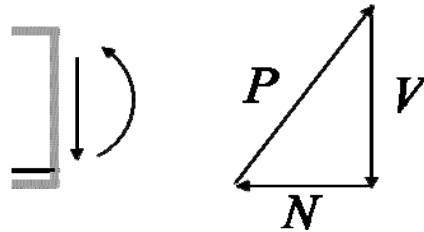
(51) & (52) → (47)

$$\sigma_x = -\tau (\cot \theta - \cot \alpha)$$

$$N = \sigma_x b d$$

$$T = \frac{M}{h} + \frac{1}{2} V (\cot \theta - \cot \alpha)$$

$$C = \frac{M}{h} - \frac{1}{2} V (\cot \theta - \cot \alpha)$$



When  $\sigma_c = f_c$

Maximum shear capacity

Eq(5.2.51) becomes

$$\frac{\tau}{f_c} = \sin^2 \theta (\cot \theta + \cot \alpha)$$

Eq(5.2.52)

$$r f_y = f_c \sin^2 \theta (\cot \theta + \cot \alpha) \cdot \frac{1}{\sin^2 \alpha (\cot \theta + \cot \alpha)}$$

### 5.2.3 Maximum shear capacity, Beams without shear reinforcement

- Deep beam

transfer = beam action+arch action

- Lower bound solution

Sigrist p.19

Marti (2.22) & (2.23)

$$\frac{\tau}{f_c} = \frac{1}{2} \left[ \sqrt{1 + \left(\frac{a}{h}\right)^2} - \frac{a}{h} \right]$$

The maximum load-carrying capacity corresponding to web crushing,  $\sigma_c = f_c$

Since  $\frac{\tau}{f_c} = \sin^2 \theta (\cot \theta + \cot \alpha)$  &

$$f_y = f_c \sin^2 \theta (\cot \theta + \cot \alpha) \cdot \frac{1}{\sin^2 \alpha (\cot \theta + \cot \alpha)}$$

We have to find  $\theta$  (free parameter)

Since  $\psi = \frac{rf_y}{f_c}$

From (5.2.52)

$$\frac{rf_y}{f_c} = \psi = \frac{\sin^2 \theta}{\sin^2 \alpha}$$

$$\therefore \sin \theta = \sqrt{\psi \sin^2 \alpha} \quad - (A) \quad \text{or} \quad \cos \theta = \sqrt{1 - \psi \sin^2 \alpha} \quad - (B)$$

Then

$$\frac{\tau}{f_c} = \sin^2 \theta (\cot \theta + \cot \alpha) = \sin \theta \cdot \cos \theta + \sin^2 \theta \cdot \cot \alpha = \sqrt{\psi \sin^2 \alpha} \sqrt{1 - \psi \sin^2 \alpha} + \psi \sin^2 \alpha \cot \alpha$$



$$= \sqrt{\psi \sin^2 \alpha (1 - \psi \sin^2 \alpha)} + \psi \cos \alpha \sin \theta \quad (63)$$

From eqs (A) & (B)

$$\tan \theta = \frac{\sin \theta}{\cos \theta} = \sqrt{\frac{\psi \sin^2 \alpha}{1 - \psi \sin^2 \alpha}} \quad (64)$$

To find the max of eq (63)

$$\frac{\partial}{\partial \theta} \left( \frac{\tau}{f_c} \right) = \frac{\partial}{\partial \theta} [\sin \theta \cdot \cos \theta + \sin^2 \theta \cot \alpha] = \cos^2 \theta - \sin^2 \theta + 2 \sin \theta \cos \theta \cot \alpha = 0$$

$$\therefore -\cot \alpha = \frac{1}{2} \left( \frac{\cos \theta}{\sin \theta} - \frac{\sin \theta}{\cos \theta} \right)$$

$$\frac{2}{\tan \alpha} = \frac{\sin^2 \theta - \cos^2 \theta}{\sin \theta \cos \theta} \quad (C)$$

(A) & (B)  $\rightarrow$  (C)

$$\frac{2\psi \sin^2 \alpha - 1}{\sqrt{\psi \sin^2 \alpha (1 - \psi \sin^2 \alpha)}} = \frac{2}{\tan \alpha} \quad (C^*)$$

Let  $\lambda = \psi \sin^2 \alpha$

Eq (C\*) becomes

$$\lambda^2 - \lambda + \frac{\tan^2 \alpha}{4(1 + \tan^2 \alpha)} = 0$$

$$\lambda = \frac{1}{2} \left[ 1 \pm \sqrt{\frac{1}{1 + \tan^2 \alpha}} \right]$$

Since  $\frac{1}{1 + \tan^2 \alpha} = \cos^2 \alpha$

$$\lambda = \frac{1}{2} [1 \pm \cos \alpha]$$

Therefore, if

$$\psi \sin^2 \alpha \leq \frac{1}{2}(1 + \cos \alpha) \quad (5.2.55)$$

Eq (5.2.55) is valid

$$\text{When } \lambda = \frac{1}{2}(1 + \cos \alpha) \quad (D)$$

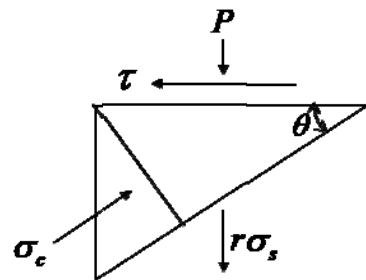
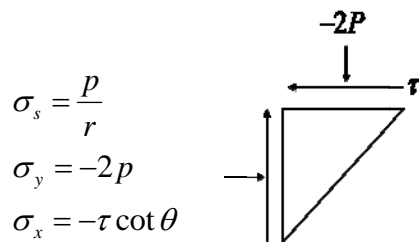
$$\frac{\tau}{f_c} = \sqrt{\frac{1}{2}(1 + \cos \alpha) \left( 1 - \frac{1}{2} - \frac{\cos \alpha}{2} \right)} + \frac{1}{2}(1 + \cos \alpha) \cot \alpha = \frac{1}{2} \sqrt{1 - \cos^2 \alpha} + \frac{1}{2}(1 + \cos \alpha) \cot \alpha$$

$$\begin{aligned}
&= \frac{1}{2} \sqrt{1 + \cos \alpha} \left[ \sqrt{1 - \cos \alpha} + \sqrt{1 + \cos \alpha} \frac{\cos \alpha}{\sin \alpha} \right] = \frac{1}{2} \sqrt{\frac{1 + \cos \alpha}{1 - \cos \alpha}} \left[ 1 - \cos \alpha + \sin \alpha \frac{\cos \alpha}{\sin \alpha} \right] \\
&= \frac{1}{2} \sqrt{\frac{1 + \cos \alpha}{1 - \cos \alpha}} = \frac{1}{2} \sqrt{\frac{1 + \cos^2 \frac{\alpha}{2}}{1 - \cos^2 \frac{\alpha}{2}}} = \frac{1}{2} \cot \frac{\alpha}{2} \quad (5.2.58)
\end{aligned}$$

$\cos 2\alpha = 2 \cos^2 \alpha - 1$ $\cos 2\alpha = 1 - 2 \sin^2 \alpha$
--

Sub  $\psi^2 \sin^2 \alpha = \frac{1}{2}(1 + \cos \alpha)$  into (5.2.59) yields

$$\tan \theta = \sqrt{\frac{\frac{1}{2}(1 + \cos \alpha)}{1 - \frac{1}{2}(1 + \cos \alpha)}} = \sqrt{\frac{1 + \cos \alpha}{1 - \cos \alpha}} = \sqrt{\frac{2 \cos^2 \frac{\alpha}{2}}{2 \sin^2 \frac{\alpha}{2}}} = \cot \frac{\alpha}{2} \quad (5.2.59)$$



$$\begin{aligned}
\sigma_c \sin^2 \theta &= p + r \sigma_s \\
\sigma_c \sin^2 \theta &= 2p \\
\sigma_c \sin \theta \cos \theta &= \tau
\end{aligned}$$

$$\cot \theta = \frac{\frac{a}{4}}{h} = \frac{a}{4h}$$

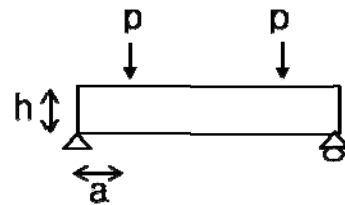
$$4p \cdot \frac{a}{4h} = \frac{pa}{h}$$

- Upper bound solution

- New flexural failure

### 5.2.4 Influence on shear capacity of longitudinal reinforcement

$$\begin{cases} T = \frac{M}{d} + \frac{V}{2} \cot \theta \\ C = -\frac{M}{d} + \frac{V}{2} \cot \theta \\ V = \frac{A_v f_y}{s} \cdot d \cot \theta \end{cases}$$



Yielding of longitudinal bars & stirrups

$$T_y = \frac{pa}{h} + \frac{p}{2} \cot \theta \quad (a)$$

$$\tau = \frac{V}{bh} = \frac{p}{bh} = \frac{A_v f_y}{bsh} \cdot h \cot \theta = r f_y \cot \theta \quad (b)$$

$$\text{Let } \phi = \frac{A_{sl} f_{rl}}{bh f_c} \quad \& \quad \psi = \frac{r f_y}{f_c}$$

Eq(a) is rewritten as

$$A_{sl} f_{rl} = \frac{\tau b h a}{h} + \frac{\tau b h}{2} \cdot \frac{\tau}{r f_y}$$

$$\phi = \frac{\tau}{f_c} \left( \frac{a}{h} \right) + \frac{\tau}{2 f_c} \cdot \frac{\tau}{f_c} \cdot \frac{f_c}{r f_y} = \frac{\tau}{f_c} \left( \frac{a}{h} \right) + \frac{1}{2} \left( \frac{\tau}{f_c} \right) \frac{1}{\psi}$$

$$\frac{\tau}{f_c} = \psi \cdot \frac{a}{h} \cdot \left[ \sqrt{1 + \frac{2\phi}{\psi \left( \frac{a}{h} \right)^2}} - 1 \right] = \psi \left[ \sqrt{\frac{2\phi}{\psi} + \left( \frac{a}{h} \right)^2} - \left( \frac{a}{h} \right) \right] = -\psi \cdot \frac{a}{h} + \sqrt{\left( \psi \frac{a}{h} \right)^2 + 2\psi\phi}$$

### 5.2.6 Design of shear reinforcement in beams

$$T = \frac{M}{d} + \frac{V}{2} \cot \theta$$

$$\sigma_c = \tau (\tan \theta + \cot \theta) \leq v f'_c$$

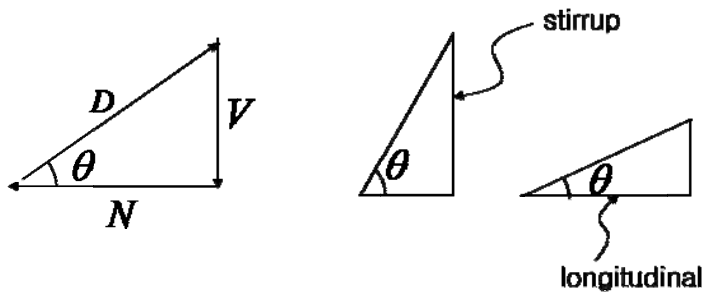
In ACI Code,

To avoid diagonal crushing failure

$$V_n = V_c + V_s$$

$$V_s \leq 8 \sqrt{f'_c} b_w d$$

$$\frac{50 b_w s}{f_y} \leq A_v \leq \left( V_s \leq 8 \sqrt{f'_c} b_w d \right)$$



$$D = f_d \cos \theta b d$$

$$V = D \sin \theta = f_d \cos \theta \sin \theta b d$$

$$\boxed{\frac{3}{5} \leq \cot \theta \leq \frac{5}{3}}$$