

$$\tau = \mu \frac{d\langle u \rangle}{dy} - \rho \langle uv \rangle \quad \text{total shear stress } \tau_w$$

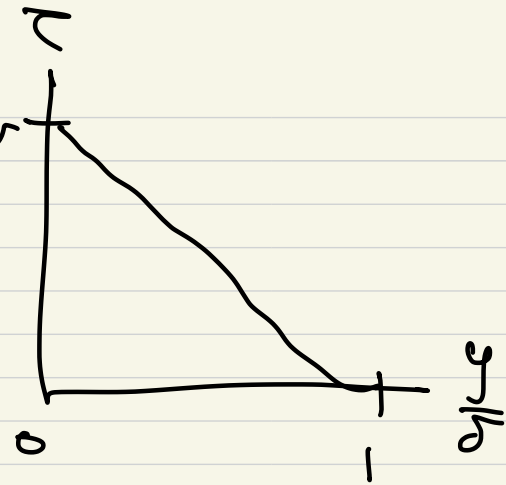
$$\tau(y) = \tau_w \left(1 - \frac{y}{\delta}\right)$$

$$u_\tau = \sqrt{\frac{\tau_w}{\rho}} \quad : \quad \begin{array}{l} \text{wall-shear velocity} \\ \text{friction} \end{array}$$

$$\delta_\nu = \frac{\nu}{u_\tau} \quad : \quad \text{viscous length scale}$$

$$Re_\tau = \frac{u_\tau \delta}{\nu} = \frac{\delta}{\delta_\nu} \quad : \quad \text{friction Reynolds number}$$

$$\left. \begin{array}{l} y^+ = \frac{y u_\tau}{\nu} \\ u^+ = \frac{\langle u \rangle}{u_\tau} \end{array} \right) \quad + : \quad \text{wall units}$$



viscous wall region : $y^+ < 50$ direct effect of molecular viscosity on the shear stress
 outer layer : $y^+ > 50$ direct effect of viscosity is negligible.
 viscous sublayer : $y^+ < 5$ Reynolds shear stress is negligible

As $Re \uparrow$, $\frac{\delta_v}{\delta} = \frac{\nu/u_c}{\delta} = \frac{\nu}{u_c \delta} = \frac{1}{Re_c} \downarrow$

fraction of viscous wall region decreases

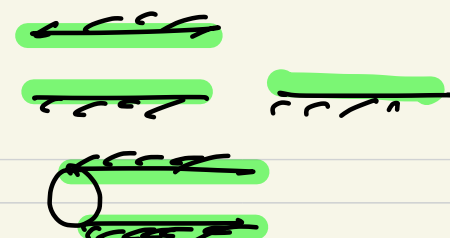
• Mean velocity profiles

channel flow is completely specified by

ρ, ν, δ and $-dp_w/dx$
 or " " " $u_c = \sqrt{\tau_w/\rho} = \left(-\frac{\delta}{\rho} \frac{dp_w}{dx}\right)^{\frac{1}{2}}$

$\Rightarrow \frac{\langle U \rangle}{u_c} = F_0\left(\frac{y}{\delta}, Re_c\right)$ from dimensional analysis

↑ universal non-dimensional ft.



$$\frac{d\langle u \rangle}{dy} / \frac{u_\tau}{y} = \Phi \left(\frac{y}{\delta}, \frac{y}{\delta^*} \right)$$

$$Re_\tau = \frac{u_\tau \delta}{\nu} = \frac{u_\tau \delta^*}{\nu}$$

$$\Phi \left(\frac{y}{\delta}, Re_\tau \right) \rightarrow \Phi \left(\frac{y}{\delta}, \frac{y}{\delta^*} \right)$$

• Law of the wall

Prandtl (1925): at high $Re \#$, close to the wall ($y/\delta \ll 1$), there is an inner layer in which $\langle u \rangle$ is determined by the viscous scales.

$$\rightarrow \frac{d\langle u \rangle}{dy} / \frac{u_\tau}{y} = \Phi_1 \left(\frac{y}{\delta^*} \right) \quad \text{for } \frac{y}{\delta} \ll 1$$

$$\left(u^+ = \langle u \rangle / u_\tau, \quad y^+ = y u_\tau / \nu \right)$$

universal ft.

$$\rightarrow \frac{du^+}{dy^+} = \frac{1}{y^+} \Phi_1(y^+)$$

integration

$$u^+ = f_w(y^+) \quad \text{for } \frac{y}{\delta} \ll 1$$

inner layer

* viscous sublayer $u^+ = f_w(y^+)$

@ wall, $\langle u \rangle = 0 \rightarrow 0 = f_w(0) \rightarrow \underline{f_w(0) = 0}$
 ($y=0$)

$u^+ = f_w(y^+) \rightarrow \langle u \rangle = u_c f_w(y^+)$

$\frac{\partial \langle u \rangle}{\partial y} \Big|_{y=0} = u_c f_w'(y^+)|_{y^+=0} \cdot \frac{u_c}{\nu}$

$\rho \nu \frac{\partial \langle u \rangle}{\partial y} \Big|_{y=0} = \tau_w = \rho u_c^2 f_w'(0) \Rightarrow \underline{f_w'(0) = 1}$

$f_w(y^+) = f_w(0) + \underbrace{y^+}_{u^+} f_w'(0) + \frac{1}{2} y^{+2} f_w''(0) + \dots$

$\Rightarrow \boxed{u^+ = y^+} + O(y^{+2})$

$\boxed{\text{for } y^+ < 5}$
 viscous sublayer

* Log law

At high Re, outer part of inner layer \rightarrow large y^+

$y^+ = \frac{y u_c}{\nu} = 0.1 \frac{u_c \delta}{\nu}$ i.e. $\boxed{y^+ \approx 0.1 \frac{\delta}{\delta^*}} = 0.1 Re_c \gg 1$

$\frac{\nu}{u \tau} = 0.1$

then, the viscosity has little effect.

→ $\Phi_1\left(\frac{\nu}{u \tau}\right)$ does not depend on ν

→ Φ_1 is const. $\Phi_1 = \frac{1}{k}$ for $\frac{\nu}{u \tau} \ll 1$ and $y^+ \gg 1$

∴ $\frac{du^+}{dy^+} = \frac{1}{ky^+} \rightarrow u^+ = \frac{1}{k} \ln y^+ + B$

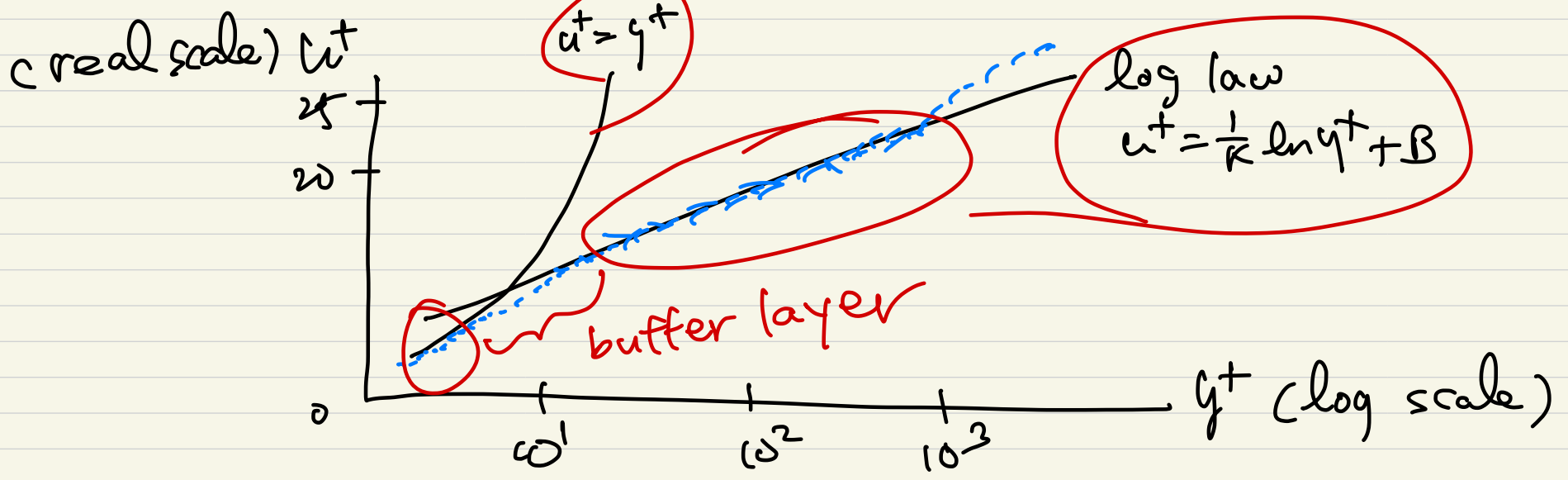
log law
logarithmic law
of the wall
von Karman (1930)

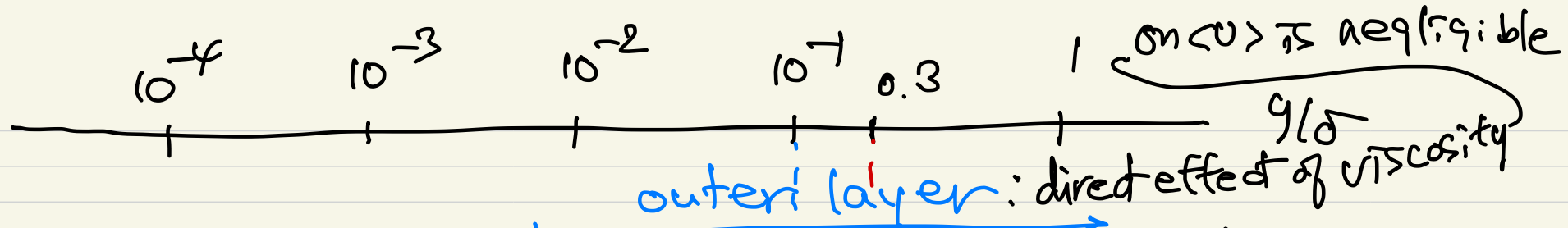
k : von Karman constant

$k = 0.41, B = 5.2$

$k = 0.41 \ \& \ B = 5.0$

$k = 0.4 \ \& \ B = 5.5$





$\langle u \rangle$ determined by u_c & y^+ indep. of u_0 & δ

outer layer: direct effect of viscosity
 overlap layer (region): between inner ($y/\delta < 0.1$) and outer ($y^+ > 50$) layers
 $u^+ = \frac{1}{\kappa} \ln y^+ + B$

significant contribution of viscosity on shear stress

inner layer
 viscous wall region

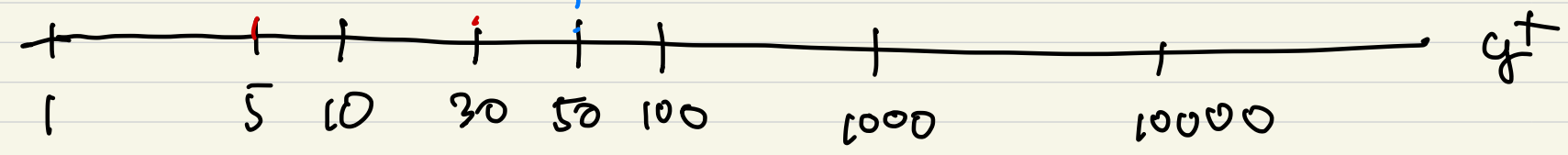
$$y^+ = \frac{y u_\tau}{\nu}$$

$$= \frac{y}{\delta} \cdot \frac{\delta u_\tau}{\nu}$$

$$= \frac{y}{\delta} \cdot Re_\tau$$

buffer layer: region between viscous sublayer ($y^+ < 5$) and log-law region ($y^+ > 30$)

viscous sublayer



$u^+ = y^+$
 Reynolds shear stress is negligible

* Velocity-defect law

in outer layer ($y^+ > 50$), $\Phi\left(\frac{y}{\delta}, \frac{y}{\delta}\right)$ is indep. of ν .

$$\rightarrow \frac{d\langle u \rangle}{dy} \bigg/ \frac{u_\tau}{y} = \Phi_0\left(\frac{y}{\delta}\right)$$

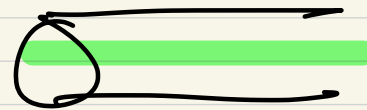
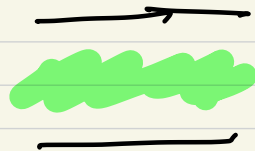
integrate
from y to δ

$$\frac{U_0 - \langle u \rangle}{u_\tau} = \Phi_0\left(\frac{y}{\delta}\right)$$

velocity-defect law
 $y^+ > 50$

not universal fit.

is different for diff. flows



At sufficiently high $Re \#$, there is an overlap region
between inner layer ($y/\delta < 0.1$) and outer layer ($y^+ > 50$).

$$\begin{aligned} \text{In this region, } \frac{d\langle u \rangle}{dy} \bigg/ \frac{u_\tau}{y} &= \Phi_1\left(\frac{y}{\delta}\right) \text{ inner layer} \\ &= \Phi_0\left(\frac{y}{\delta}\right) \text{ outer layer} \end{aligned}$$

= const

$$\rightarrow \frac{d\langle u \rangle}{dy} \bigg/ \frac{u_\tau}{y} = \frac{1}{\kappa} \quad \text{for } \delta_y \ll y \ll \delta$$

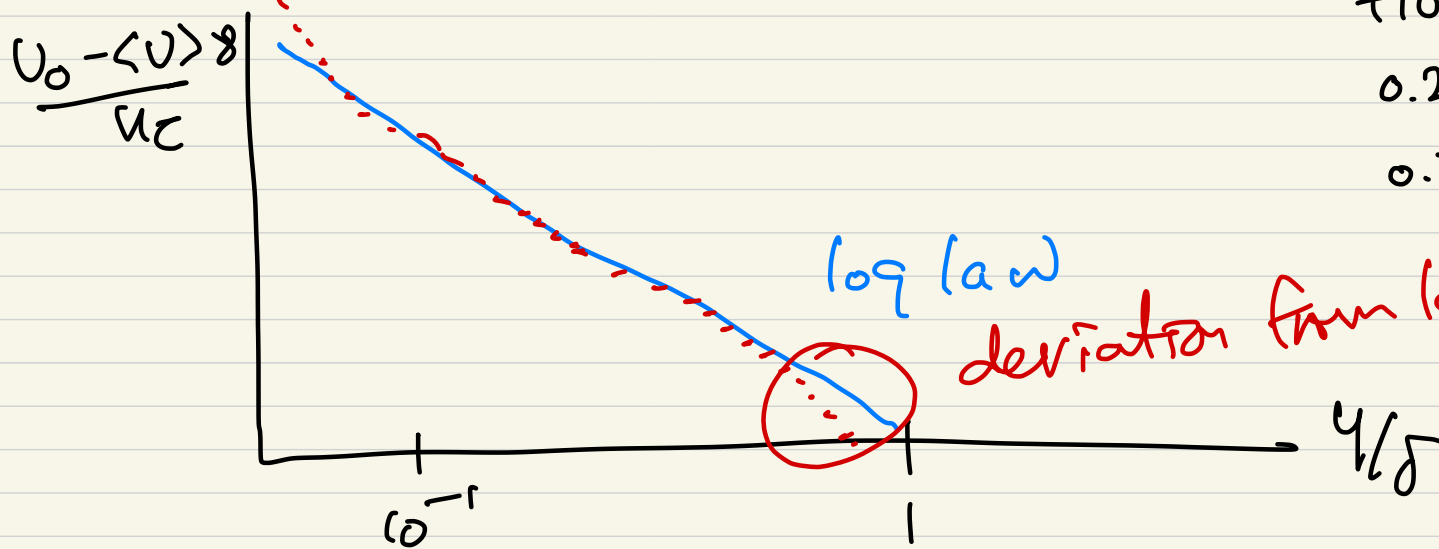
Millikan (1938)

$$\hookrightarrow \text{log-law again} \rightarrow u^+ = \frac{1}{\kappa} \ln y^+ + B$$

$$\text{or, } \frac{U_0 - \langle u \rangle}{u_\tau} = F_D\left(\frac{y}{\delta}\right) = -\frac{1}{\kappa} \ln \frac{y}{\delta} + B_1 \quad \text{for } \frac{y}{\delta} \ll 1$$

flow-dependent const.

0.2 (ONS) } small
0.7 (exps.) }



log law

deviation from log law exists.

y/δ

① Friction law & Reynolds number

Relationships among U_0 , \bar{U} and u_τ

- bulk velocity \bar{U} from log law

$$\frac{U_0 - \langle U \rangle}{u_\tau} = -\frac{1}{K} \ln \frac{y}{\delta} + B_1$$

② $y = \delta$, $\langle U \rangle = U_0 \rightarrow B_1 = 0$

$$\frac{U_0 - \bar{U}}{u_\tau} = \frac{U_0 - \frac{1}{\delta} \int_0^\delta \langle U \rangle dy}{u_\tau} = \frac{1}{\delta} \int_0^\delta \frac{U_0 - \langle U \rangle}{u_\tau} dy$$

$$\approx \frac{1}{\delta} \int_0^\delta \left(-\frac{1}{K} \ln \frac{y}{\delta} \right) dy = \frac{1}{K} \approx 2.4$$

agrees well with exp. data.

... inner layer $\frac{\langle U \rangle}{u_\tau} = \frac{1}{K} \ln \frac{y}{\delta} + B$

outer layer $\frac{U_0 - \langle U \rangle}{u_\tau} = -\frac{1}{K} \ln \frac{y}{\delta} + B_1$

$$\frac{\delta}{\delta y} = \frac{\delta}{y/u_\tau} = \frac{\delta u_\tau}{y}$$

$$\frac{U_0}{u_\tau} = \frac{1}{K} \ln \frac{\delta}{\delta y} + B + B_1$$

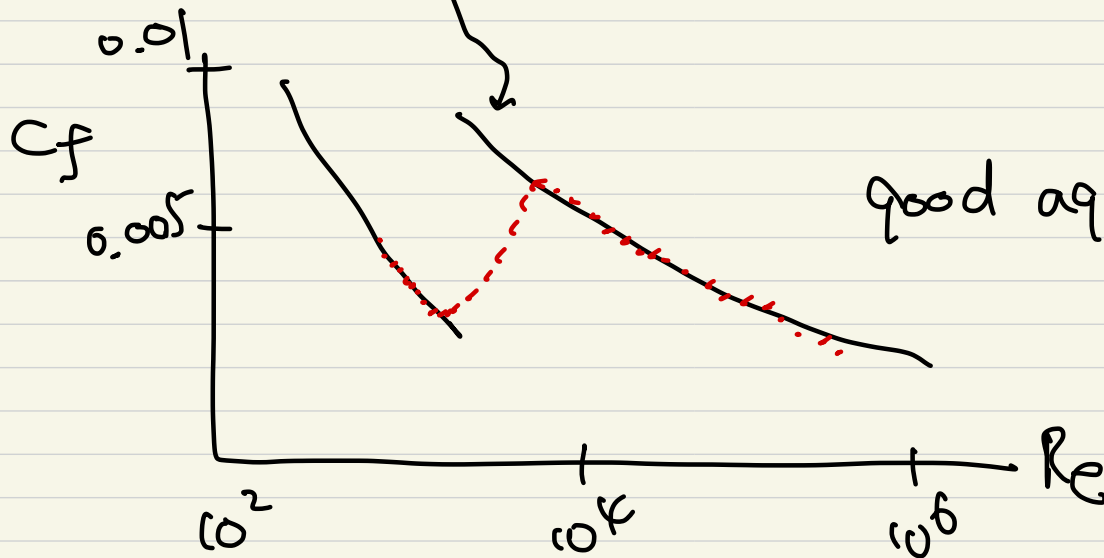
$$= \frac{\delta u_\tau}{y} \cdot \frac{u_\tau}{U_0}$$

$$\rightarrow \frac{U_o}{u_c} = \frac{1}{\kappa} \ln \left[Re_o \left(\frac{U_o}{u_c} \right)^{-1} \right] + B + B,$$

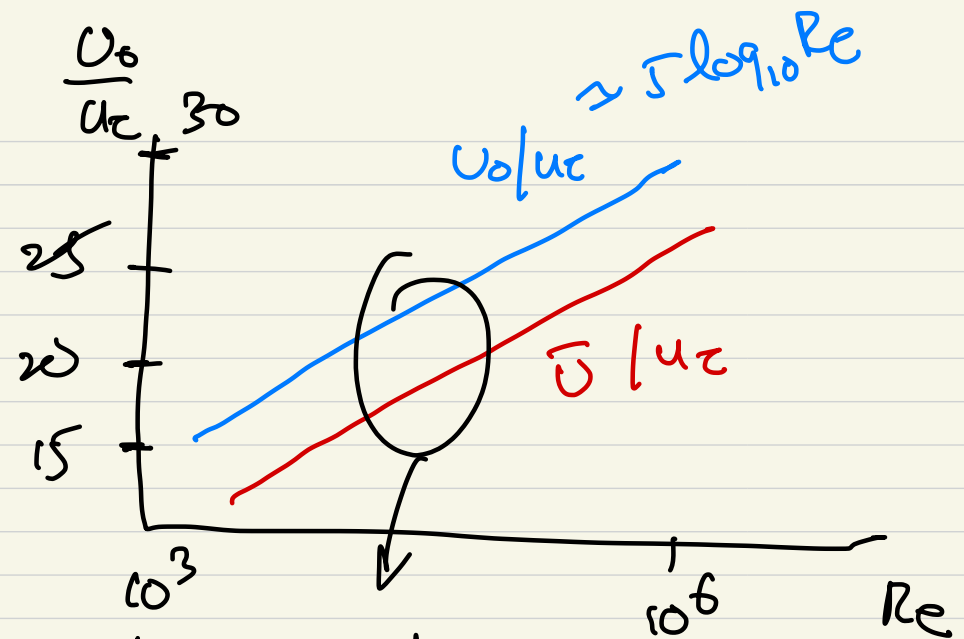
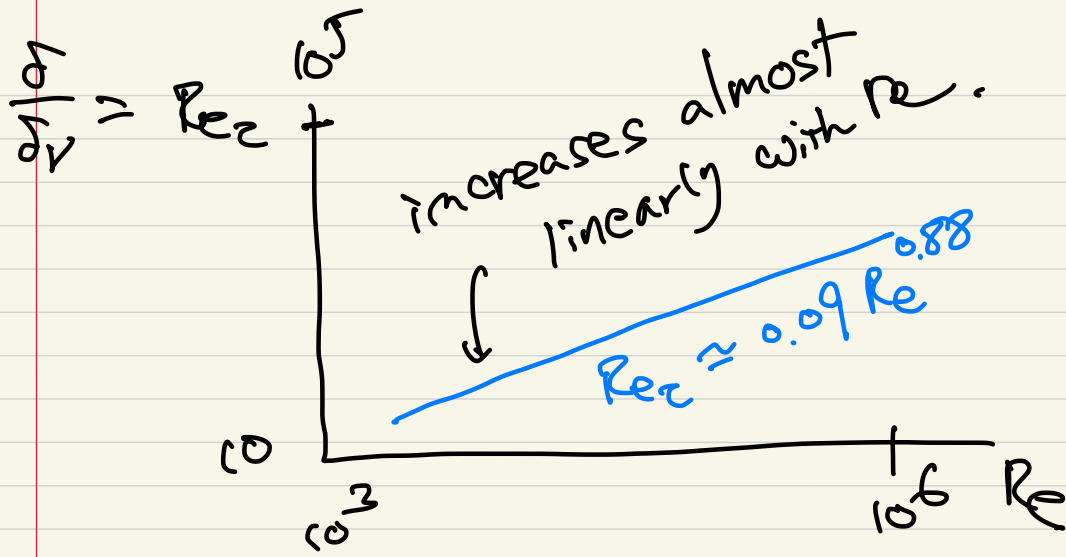
$$C_f = \frac{\tau_w}{\frac{1}{2} \rho U_o^2} = 2 \left(\frac{u_c}{U_o} \right)^2 \rightarrow \frac{U_o}{u_c} = \sqrt{\frac{2}{C_f}}$$

$$\rightarrow \sqrt{\frac{2}{C_f}} = \frac{1}{\kappa} \ln \left(Re_o \sqrt{\frac{C_f}{2}} \right) + B + B,$$

Given Re_o , C_f (or $\frac{U_o}{u_c}$) can be obtained,



good agreement at $Re > 3000$



velocity ratios
increase very slowly
with $Re \#$.

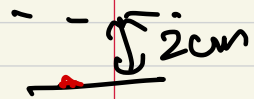
- At high Re , viscous length scale is very small.

e.g., $\delta = 2 \text{ cm}$, $Re = 10^5$

$$Re_z = 0.09 Re^{0.88} \approx 2260 = \frac{\delta}{\delta_y} \rightarrow \delta_y = 0.88 \times 10^{-5} \text{ m} \approx 10^{-5} \text{ m}$$

$$\underline{y^+ = 100} \rightarrow y = 1 \text{ mm}!$$

significant fraction of the increase in mean velocity bet wall & centerline occurs in the viscous wall region.



e.g., $\delta = 2 \text{ cm}$, $Re = 10^5 \rightarrow y^+ = 10 \rightarrow y = 0.1 \text{ mm}$

$\hookrightarrow \langle U \rangle_{y^+=10} \approx 0.3 U_0!$

$$y^+ = \frac{y u_\tau}{\nu} = \frac{y_0}{Re \nu} = \frac{y_0}{10.09 Re^{0.88}}$$

$$\ln\left(\frac{y}{\delta}\right) = \ln\left(\frac{y_0^+}{5.09}\right) - 0.88 \ln Re$$

log-law persists beyond the region suggested by the overlap argument.

