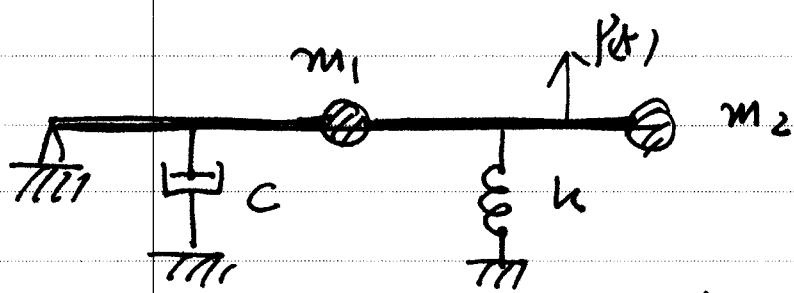


제 8 장 일반화된 관성계 시스템

딱 맞는 결과가 아닌 충분히 정확한 결과

8.1 일반화된 관성계 시스템

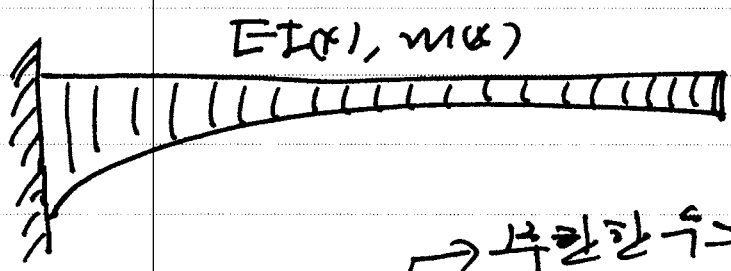
Case #1 : 강체 조립체



→ 일반화된 변위

변위 $u(x, t) = \psi(x) z(t)$
 ↑ 특성함수

Case #2 : 분포질량 시스템, 연속체 보철



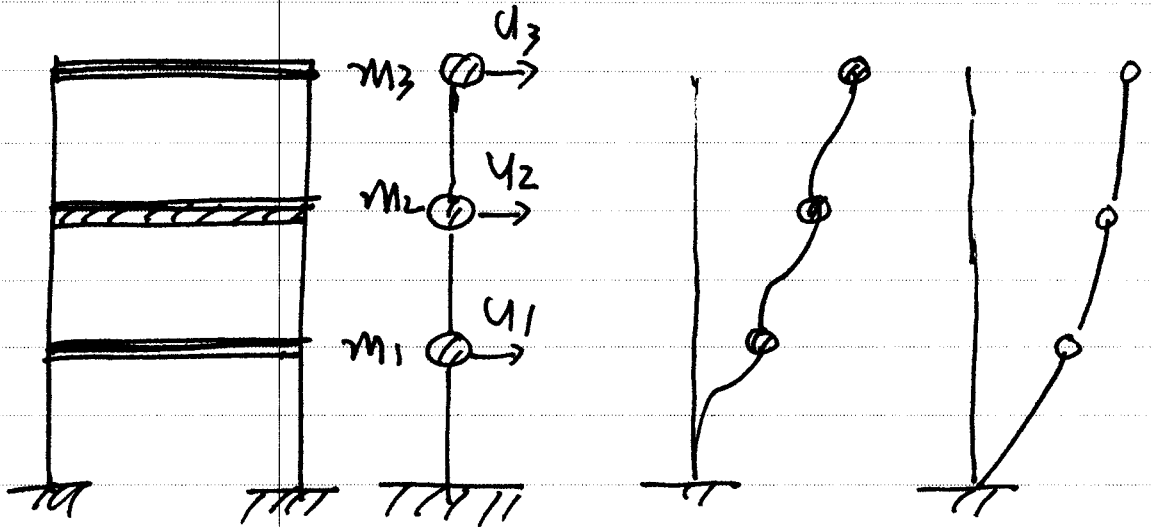
→ 변위함수

딱 맞는 해석 : 특성함수를 주어진 처단,

관성계 : 모드를 가정해서 관성계 보철링

$u(x, t) = \psi(x) z(t)$

Case #3: 질량(덩어리) 질량 시스템 (이산 보틀)



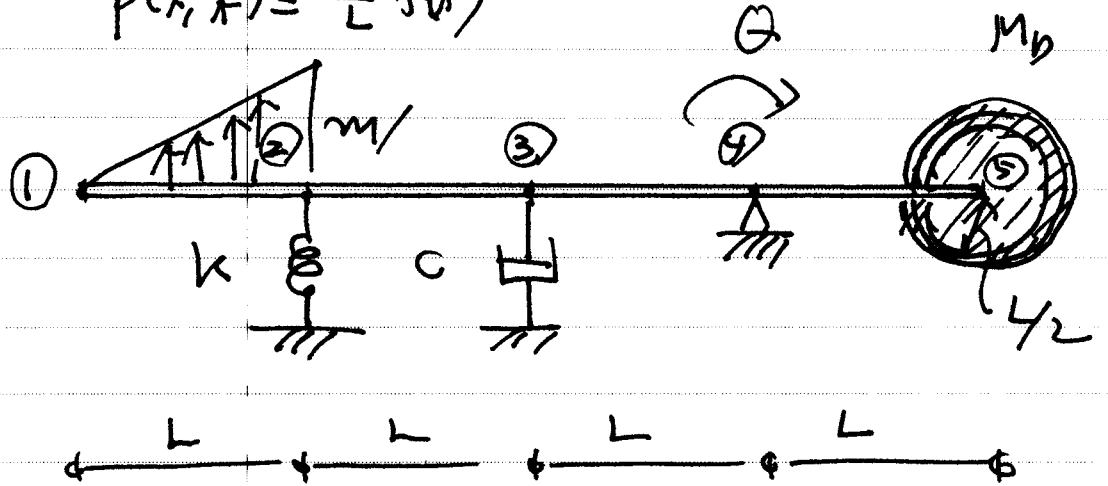
좌변의 보틀 벡터 $\rightarrow \phi$
 우변의 보틀 벡터 $\leftarrow \psi$

사용 방법 : > 보틀은 Energy method

1. Virtual work principle
2. Hamilton's principle
3. Lagrange Equation

8.2. 강체 조립체

$$P(x, t) = \frac{m}{L} f(x)$$



절점 2에서의 변위 변위 u_2 를 DFT로 결정

절점 4에서의 회전각 θ , $\theta = \frac{y}{2L}$

절점 1에서의 변위 변위, u_1

$$u_1 = (3L) \theta = \frac{3}{2} y$$

$$\begin{aligned} u(x) &= (3L - x) \theta = (3L - x) \frac{y}{2L} \\ &= \frac{3}{2} y - \frac{x}{2L} y \end{aligned}$$

절점 3에서의 변위 변위, $u_3 = \frac{y}{2}$

절점 5에서의 변위 변위 u_5

$$u_5 = \frac{3}{2} y - \frac{4L}{2L} y = -\frac{1}{2} y$$

질점 \odot 이의 디랙은 자유롭게 회전할 수
없도록 강하게 막아서 연결되어 있다.

Lagrange Equation

$$T_{\odot} = \frac{1}{2} (4mL) \left(\frac{\dot{\psi}}{2}\right)^2 = \frac{1}{2} mL (\dot{\psi})^2$$

$$\begin{aligned} T_{\ominus} &= \frac{1}{2} \left(\frac{1}{12} 4mL\right) (4L)^2 \dot{\theta}^2 \\ &= \frac{1}{2} I \cdot \left(\frac{\dot{\psi}}{2L}\right)^2 = \frac{1}{8L^2} I_0 \dot{\psi}^2 \end{aligned}$$

$$T_{\ominus} = \frac{1}{2} M_0 \left(-\frac{\dot{\psi}}{2}\right)^2 = \frac{M_0}{8} \dot{\psi}^2$$

$$\begin{aligned} T_{\ominus} &= \left(\frac{1}{2}\right) \left(\frac{1}{6}\right) M_0 \left(\frac{L}{2}\right)^2 \left(\frac{\dot{\psi}}{2L}\right)^2 \\ &= \frac{1}{2} I_D \cdot \frac{\dot{\psi}^2}{4L^2} = \frac{I_D}{8L^2} \dot{\psi}^2 \end{aligned}$$

$$I_0 = \frac{1}{12} (4mL) (4L)^2 = \frac{16}{3} mL^3$$

$$I_D = \frac{1}{2} M_0 \left(\frac{L}{2}\right)^2 = \frac{1}{8} M_0 L^2$$

$$\begin{aligned}
 T &= T_{(1)} + T_{(2)} + T_{(3)} + T_{(4)} \\
 &= \left(\frac{mL}{2} + \frac{I_0}{8L^2} + \frac{I_0}{8L^2} + \frac{M_D}{8} \right) \dot{u}^2 \\
 &= \frac{1}{2} \left[mL + \frac{M_D}{4} + \frac{I_0}{4L^2} + \frac{I_0}{4L^2} \right] \dot{u}^2
 \end{aligned}$$

$$V = \frac{1}{2} k u^2$$

$$F = \frac{1}{2} c \left(\frac{\dot{u}}{2} \right)^2 = \frac{1}{2} \left(\frac{c}{4} \right) \dot{u}^2$$

$$W = p(x) \int_0^L \left(\frac{x}{L} \right) \left(\frac{3}{2} - \frac{x}{2L} \right) u \, dx$$

$$= \frac{p u}{L} \int_0^L \left(\frac{x}{L} \right) \left(\frac{3}{2} - \frac{x}{2L} \right) dx$$

$$= \left(\frac{p u}{L} \right) \left[\frac{3}{4} x^2 - \frac{x^3}{6L} \right]_0^L$$

$$= \left(\frac{p u}{L} \right) \left[\frac{3}{4} L^2 - \frac{L^3}{6} \right] = \frac{17}{12} p u$$

Lagrange Equation

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{u}} \right) + \frac{\partial T}{\partial u} + \frac{\partial V}{\partial u} = \frac{17}{12} p L$$

$$\dot{u} = u$$

$$M^* \ddot{u} + C^* \dot{u} + K^* u = P^*$$

$$M^* = \left(mL + \frac{M_D}{4} + \frac{I_0}{4L^2} + \frac{I_D}{4L^2} \right)$$

$$C^* = \frac{c}{4}$$

$$K^* = K$$

8.3. 분포 질량-라-분포 탄성을 가진 시스템

$$m(x) \frac{\partial^2 u}{\partial t^2} + \frac{\partial^2}{\partial x^2} \left[EI(x) \frac{\partial^2 u}{\partial x^2} \right] = p(x, t)$$

Rayleigh-Ritz method (Assumed mode shape method)

$$u(x, t) = \sum_{i=1}^N \psi_i(x) q_i(t)$$

$$\dot{u}(x, t) = \sum_{i=1}^N \psi_i(x) \dot{q}_i(t)$$

Kinetic Energy T

$$T(t) = \frac{1}{2} \int_0^l m(x) \left(\frac{\partial u(x,t)}{\partial t} \right)^2 dx$$

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Potential Energy, V

$$V(t) = \frac{1}{2} \int_0^l EI(x) \left(\frac{\partial^2 u}{\partial x^2} \right)^2 dx$$

External work, W_e

$$W_e = \int_0^l p(x,t) u(x,t) dx$$

Assumed mode shapes

$$\begin{aligned} T(t) &= \frac{1}{2} \int_0^l m(x) \left(\sum_{i=1}^N \psi_i(x) \dot{q}_i \right) \left(\sum_{j=1}^N \psi_j(x) \dot{q}_j \right) dx \\ &= \sum_{i=1}^N \sum_{j=1}^N \dot{q}_i \dot{q}_j \left(\frac{1}{2} \int_0^l m(x) \psi_i(x) \psi_j(x) dx \right) \\ &= \sum_{i=1}^N \sum_{j=1}^N \left(\frac{1}{2} \right) m_{ij} \dot{q}_i \dot{q}_j \end{aligned}$$

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$$\begin{aligned}
 V(t) &= \frac{1}{2} \int_0^l EI(x) \left(\sum_{i=1}^N \psi_i'' q_i \sum_{j=1}^N \psi_j'' q_j \right) dx \\
 &= \frac{1}{2} \sum_{i=1}^N \sum_{j=1}^N \left(\int_0^l EI(x) \psi_i''(x) \psi_j''(x) dx \right) q_i q_j \\
 &= \frac{1}{2} \sum_{i=1}^N \sum_{j=1}^N k_{ij} q_i q_j
 \end{aligned}$$

$$W_e = \int_0^l p(x,t) \sum_{i=1}^N \psi_i(x) q_i(t) dx$$

$$= \sum_{i=1}^N \left(\int_0^l p(x,t) \psi_i(x) dx \right) q_i(t)$$

$$= \sum_{i=1}^N p_i(t) q_i(t)$$

Lagrange Equation

- generalized coordinates $q_i(t)$
- variational approach

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$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}_m} + \frac{\partial V}{\partial q_m} = \frac{\partial W_e}{\partial q_m}$$

$$\frac{d}{dt} \frac{\partial}{\partial \dot{q}_m} \sum_{i=1}^N \sum_{j=1}^N \frac{1}{2} m_{ij} \dot{q}_i \dot{q}_j$$

$$+ \frac{\partial}{\partial q_m} \sum_{i=1}^N \sum_{j=1}^N \frac{1}{2} k_{ij} q_i q_j = \frac{\partial}{\partial q_m} \sum_{i=1}^N P_i(t) q_i(t)$$

$$\sum_{n=1}^N m_{mn} \ddot{q}_n + \sum_{n=1}^N k_{mn} q_n = P_m(t)$$

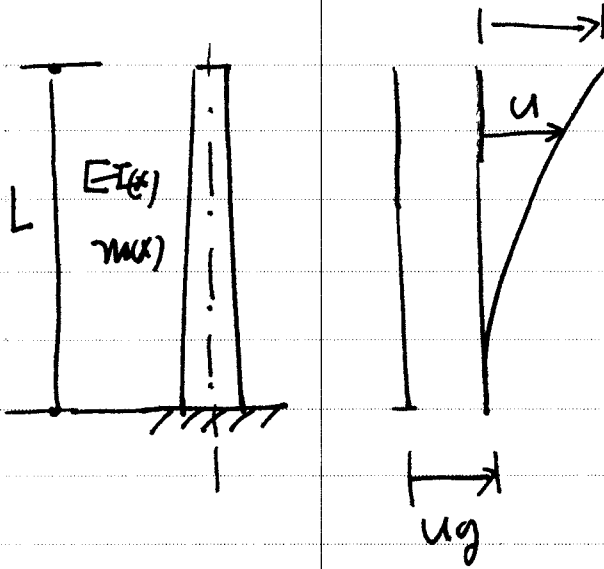
$m=1, 2, \dots, N$

$$\begin{bmatrix} m_{11} & \dots & m_{1N} \\ \vdots & & \vdots \\ m_{N1} & \dots & m_{NN} \end{bmatrix} \begin{Bmatrix} \ddot{q}_1 \\ \vdots \\ \ddot{q}_N \end{Bmatrix} = \begin{Bmatrix} P_1 \\ \vdots \\ P_N \end{Bmatrix}$$

$$+ \begin{bmatrix} k_{11} & \dots & k_{1N} \\ \vdots & & \vdots \\ k_{N1} & \dots & k_{NN} \end{bmatrix} \begin{Bmatrix} q_1 \\ \vdots \\ q_N \end{Bmatrix} = \begin{Bmatrix} P_1 \\ \vdots \\ P_N \end{Bmatrix}$$

w/8

NO



$$u^*(x, t) = u(x, t) + u_g(t)$$

↑
상대변위

$$u(x, t) = \psi(x) f(t)$$

$$\psi(x) = \frac{3}{2} \frac{x^2}{L^2} - \frac{1}{2} \frac{x^3}{L^3}$$

(중간값정리
역사 곡선)

$$\psi(x) = \frac{x^2}{L^2}$$

$$\psi(x) = 1 - \cos \frac{\pi x}{2L}$$

가동한역시

$$P_{\text{eff}} = -m\ddot{y}_g(t)$$

$$\tilde{m} = \int_0^L m(x) \psi(x)^2 dx$$

$$\tilde{k} = \int_0^L EI(x) (\psi'(x))^2 dx$$

$$\tilde{L} = \int_0^L m(x) \psi(x) dx$$

$$\tilde{m} \ddot{z} + \tilde{k} z = -\tilde{L} \ddot{y}_g(t)$$

$$\omega_n^2 = \frac{\tilde{k}}{\tilde{m}} = \frac{\int_0^L EI(x) [\psi'(x)]^2 dx}{\int_0^L m(x) [\psi(x)]^2 dx}$$

이력계산 : 등가-경적하중변형

$$\begin{aligned} f_s(x, t) &= [EI(x) u''(x, t)]'' \\ &= [EI(x) \psi''(x, t)]'' q(t) \end{aligned}$$

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$$\begin{aligned}
 \int_0^L \psi(x) f_S(x, t) dx &= \int_0^L \psi(x) [E_L(x) \psi''(x)]' g(A) dx \\
 &= \left\{ \psi(x) [E_L(x) \psi''(x)] \right\}_0^L - \left\{ \psi'(x) [E_L(x) \psi'(x)] \right\}_0^L \\
 &\quad + \int_0^L E_L(x) (\psi'')^2 dx \quad g(A)
 \end{aligned}$$

$\psi(x)$ 가 경계조건을 만족할 때 이 가 된다.

$$\int_0^L \psi(x) f_S(x, t) dx = \int_0^L E_L(x) (\psi'')^2 g(A) dx$$

$$= \int_0^L \omega_n^2 [m(x) \psi^2] dx \quad g(A)$$

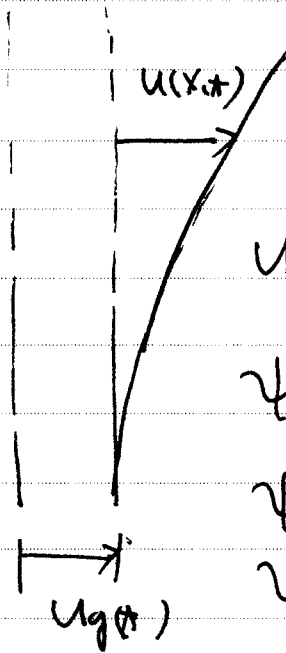
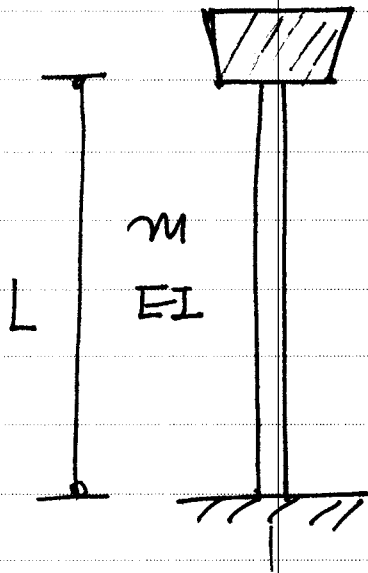
$$= \int_0^L \psi(x) (\omega_n^2 m(x) \psi(x) g(A)) dx$$

$$\int_0^L \psi(x) (f_S(x, t) - \omega_n^2 m(x) \psi(x) g(A)) dx = 0$$

$$f_S(x, t) = \omega_n^2 m(x) \psi(x) g(A)$$

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(511) 물렁크



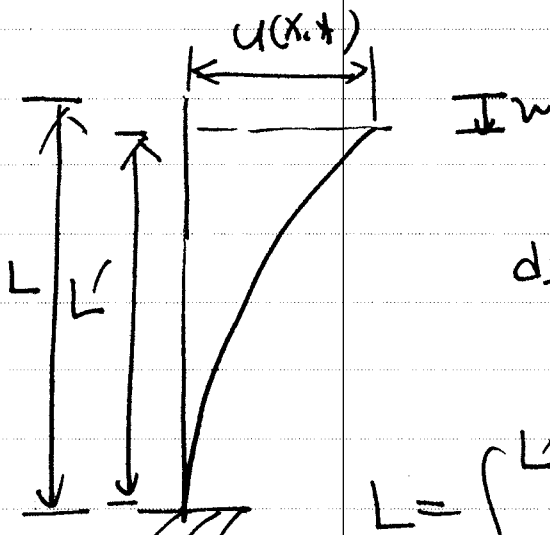
$$u(x,t) = \psi(x) q(t)$$

$$\psi(x) = 1 - \cos \frac{\pi x}{L}$$

$$\psi(0) = 0$$

$$\psi(L) = 1$$

$$N = Mg$$



$$ds = \sqrt{dx^2 + du^2}$$

$$= \sqrt{1 + \left(\frac{du}{dx}\right)^2} dx$$

$$L = \int_0^{L'} ds = \int_0^L \sqrt{1 + \left(\frac{du}{dx}\right)^2} dx$$

$$\approx \int_0^{L'} \left(1 + \frac{1}{2} \left(\frac{du}{dx}\right)^2\right) dx$$

$$L = L' + \int_0^{L'} \frac{1}{2} \left(\frac{dy}{dx} \right)^2 dx$$

$$v = L - L' = \int_0^L \frac{1}{2} \left(\frac{dy}{dx} \right)^2 dx$$

$$= q_0^2 \int_0^L \left(\frac{1}{2} \right) \psi^2 dx$$

$$T = \frac{1}{2} M (\psi(L) \dot{q} + \dot{u}q)^2$$

$$+ \frac{1}{2} \int_0^L m (\psi(x) \dot{q} + \dot{u}q)^2 dx$$

$$= \frac{1}{2} M [\dot{q}^2 + 2\dot{u}q\dot{q} + \dot{u}q^2]$$

$$+ \frac{1}{2} \int_0^L m [\psi^2 \dot{q}^2 + 2\psi \dot{q} \dot{u}q + \dot{u}q^2] dx$$

$$V = \frac{1}{2} \int_0^L EI \left(\frac{d^2y}{dx^2} \right)^2 dx$$

$$= \frac{1}{2} \left(\int_0^L EI (\psi'')^2 dx \right) q^2$$

$$W_e = Nv = \frac{1}{2} N \int_0^L (\psi)^2 dx q^2$$

Lagrange Equation

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}} + \frac{\partial V}{\partial q} = \frac{\partial W_e}{\partial q}$$

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}} = M \ddot{q} + M \ddot{u}_g + \left(\int_0^L m \psi''^2 dx \right) \ddot{q} \\ + \left(\int_0^L m \psi(x) dx \right) \ddot{u}_g$$

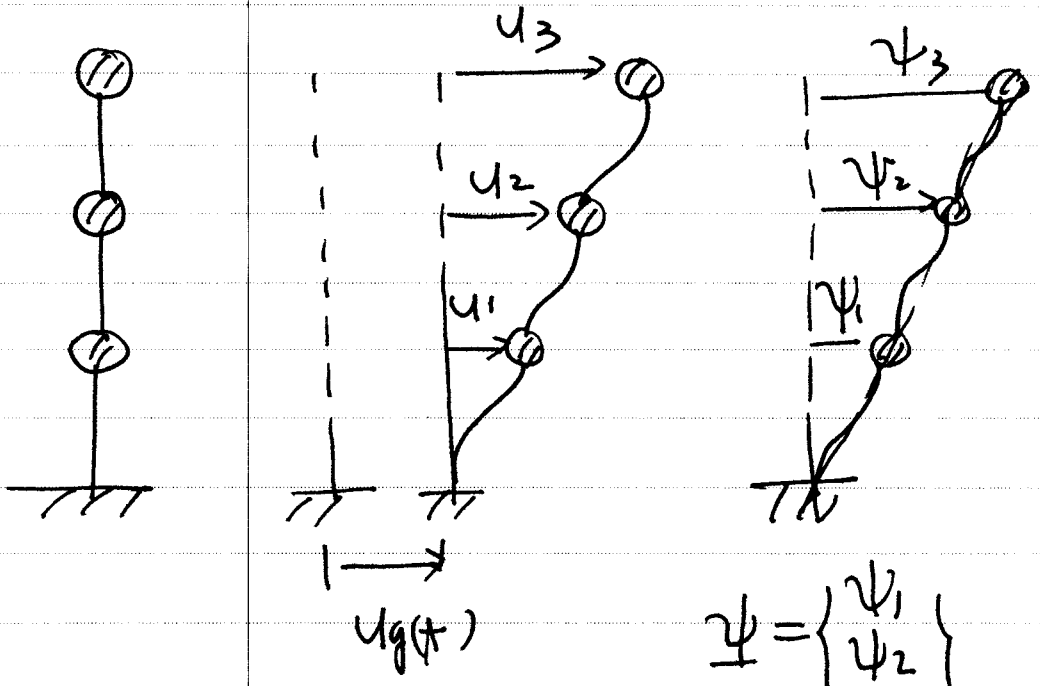
$$\frac{\partial V}{\partial q} = q \int_0^L EI (\psi'')^2 dx$$

$$\frac{\partial W_e}{\partial q} = \left(N \int_0^L (\psi(x))^2 dx \right) q$$

$$\left(M + \int_0^L m \psi^2 dx \right) \ddot{q} + \left[\int_0^L EI (\psi'')^2 dx \right. \\ \left. - N \int_0^L (\psi(x))^2 dx \right] q = -M \ddot{u}_g - \left(\int_0^L m \psi dx \right) \ddot{u}_g$$

$$M \ddot{q} + (\tilde{K} - \tilde{K}_G) q = P_{eff}$$

8.4 질량-질량 시스템: 전단-결합



$$u_j^*(t) = u_j(t) + u_g(t)$$

$$\underline{u}^*(t) = \underline{u}(t) + \{1\} u_g(t)$$

$$\underline{\psi} = \begin{Bmatrix} \psi_1 \\ \psi_2 \\ \psi_3 \end{Bmatrix}$$

↑
가장 큰
변위

$$T = \frac{1}{2} (\dot{\underline{u}}^*)^T \underline{m} \dot{\underline{u}}^*$$

$$= \frac{1}{2} (\dot{\underline{u}}(t) + \{1\} \dot{u}_g(t))^T \underline{m} (\dot{\underline{u}}(t) + \{1\} \dot{u}_g(t))$$

$$\underline{u}(t) = \underline{\psi} q(t) \text{ 라고 가정한다.}$$

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$$\begin{aligned}
 T &= \frac{1}{2} (\dot{q} \psi^T + \psi^T \dot{q}) m (\dot{q} \psi + \psi \dot{q}) \\
 &= \frac{1}{2} (\psi^T m \psi) \dot{q}^2 + \dot{q} \psi^T m \psi \dot{q} \\
 &\quad + \frac{1}{2} (\psi^T m \psi) (\dot{q})^2
 \end{aligned}$$

$$\frac{d}{dt} \frac{\partial T}{\partial \dot{q}} = (\psi^T m \psi) \ddot{q} + (\psi^T m \psi) \dot{q}$$

$$V = \frac{1}{2} u^T k u = \frac{1}{2} q \psi^T k \psi q$$

$$\frac{\partial V}{\partial q} = (\psi^T k \psi) q$$

$$(\psi^T m \psi) \ddot{q} + (\psi^T k \psi) q = -\psi^T m \psi \ddot{q}$$

$$\tilde{m} \ddot{q} + \tilde{k} q = -\tilde{L} q \ddot{q}$$

$$\omega_n^2 = \frac{\tilde{k}}{\tilde{m}} = \frac{\psi^T k \psi}{\psi^T m \psi}$$