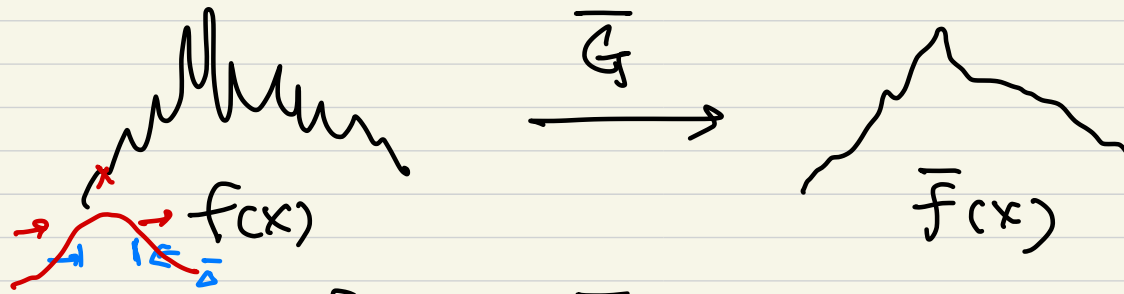


# Large eddy simulation

① Compromise between DNS and RANS

② Filtering

→ is an operation which damps out all spatial fluctuations of flow variables smaller than a prescribed length scale  
filter width



$\overline{(\cdot)}$  : filtered variable

•  $\bar{f}(\underline{x}) = \int f(\underline{x}') \underbrace{\bar{G}(\underline{x}, \underline{x}')}_{\text{filter}} d\underline{x}'$  : convolution integral

• Gaussian filter :  $\bar{G}(\underline{x}_i - \underline{x}'_i) = \left(\frac{6}{\pi \bar{\Delta}}\right)^{\frac{1}{2}} \exp\left[-6 \frac{(\underline{x}_i - \underline{x}'_i)^2}{\bar{\Delta}^2}\right]$



$\hat{G}(k) = \exp\left[-\frac{k^2 \bar{\Delta}^2}{2\pi}\right]$   $\bar{\Delta}$  : filter width  
↑  
wavenumber

(sharp) cut-off filter :  $\bar{G}(x_i - x_i') = \frac{\text{sim}[\pi(x_i - x_i')/\bar{\Delta}]}{\pi(x_i - x_i')}$

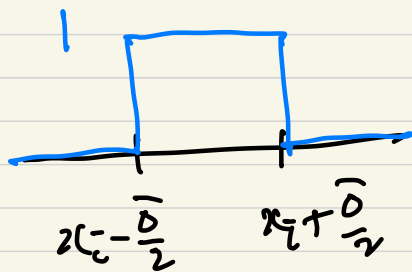


$\bar{G}(k) = \begin{cases} 1 & \text{for } k \leq k_c (= \frac{\pi}{\bar{\Delta}}) \\ 0 & \text{for } k > k_c \end{cases}$

cut-off wave-number

→ removes modes w/ wavenumber  $k > \frac{\pi}{\bar{\Delta}}$   
 appropriate for spectral method.

Box filter :  $\bar{G}(x_i - x_i') = \begin{cases} 1 & \text{for } x_i - \frac{\bar{\Delta}}{2} < x_i' < x_i + \frac{\bar{\Delta}}{2} \\ 0 & \text{otherwise} \end{cases}$



$\bar{G}(k) = \frac{\text{sim}(\frac{1}{2}k\bar{\Delta})}{\frac{1}{2}k\bar{\Delta}}$

appropriate for FDM, FVM, FEM  
 commonly used

③ Filtering the Navier-Stokes eqs.

Apply a filter to N-S eqs.

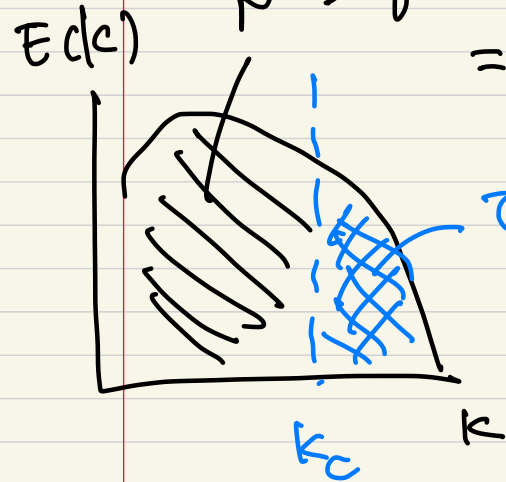
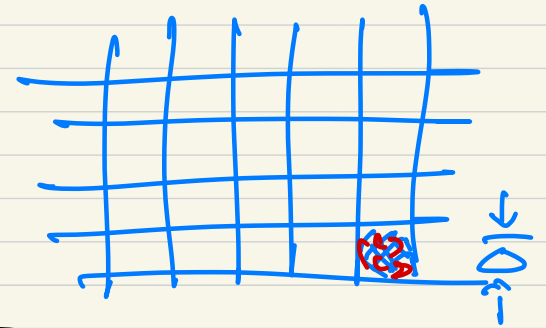
$$\overline{\frac{\partial u}{\partial x}} = \frac{\partial \bar{u}}{\partial x}$$

$$\left\{ \begin{aligned} \frac{\partial \bar{u}_i}{\partial x_i} &= 0 \end{aligned} \right.$$

$\bar{u}_i$ : filtered velocity

$\bar{p}$ : filtered pressure

$$\rho \frac{\partial \bar{u}_i}{\partial t} + \rho \frac{\partial}{\partial x_j} \overbrace{u_i u_j} = - \frac{\partial \bar{p}}{\partial x_i} + \mu \nabla^2 \bar{u}_i$$



$$= \frac{\partial}{\partial x_j} (\bar{u}_i \bar{u}_j) + \frac{\partial}{\partial x_j} (\bar{u}_i \bar{u}_j - \bar{u}_i \bar{u}_j)$$

$$\tau_{ij} \equiv \bar{u}_i \bar{u}_j - \bar{u}_i \bar{u}_j$$

should be modeled

residual stress tensor  
(subgrid scale stress tensor)

$$\Rightarrow \rho \frac{\partial \bar{u}_i}{\partial t} + \rho \frac{\partial}{\partial x_j} (\bar{u}_i \bar{u}_j) = - \frac{\partial \bar{p}}{\partial x_i} + \mu \nabla^2 \bar{u}_i \quad \left( - \rho \frac{\partial \tau_{ij}}{\partial x_j} \right)$$

$$\Downarrow$$

$$\frac{\partial}{\partial x_j} \left( \mu \frac{\partial \bar{u}_i}{\partial x_j} \right)$$

④ Smagorinsky model - eddy viscosity model

$$\tau = 2\mu S_{ij}$$

$$\tau_{ij} = -2\nu_t \bar{S}_{ij}$$

↑ eddy viscosity of residual motions  
subgrid-scale

$$\bar{S}_{ij} = \frac{1}{2} \left( \frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right)$$

$$\bar{(\cdot)} = \overline{(\cdot)}(\underline{x}, t)$$

Mixing-length hypothesis

$$\nu_t = l_s^2 \bar{S} \quad \bar{S} = \sqrt{2\bar{S}_{ij}\bar{S}_{ij}} : \text{filtered rate of strain}$$

$$= (C_s \bar{\Delta})^2 \bar{S} \geq 0 \quad \bar{\Delta} : \text{filter width ex) grid size}$$

↑ Smagorinsky constant

$l_s$  : Smagorinsky length scale

(similar to mixing length)

The rate of transfer of energy to the residual motions

$$P_t = -\tau_{ij} \bar{S}_{ij} = 2\nu_t \bar{S}_{ij} \bar{S}_{ij} = \nu_t \bar{S}^2 \geq 0$$

⇒ no backscatter

energy transfer from small scales to large scales

with  $\bar{\Delta}$  in the inertial subrange

(i.e.,  $l_{Df} < \bar{\Delta} < l_{EF}$ )

$P_{\tau}$  is balanced by the dissipation  $\epsilon$

$$\rightarrow \epsilon = \langle P_{\tau} \rangle = \langle \nu_{\tau} \bar{S}^2 \rangle = l_s^2 \langle \bar{S}^3 \rangle$$

Using the Kolmogorov spectrum ( $\epsilon \sim k^{-5/3}$ )

$$\text{and } \langle \bar{S}^3 \rangle \cong \langle \bar{S}^2 \rangle^{3/2}$$

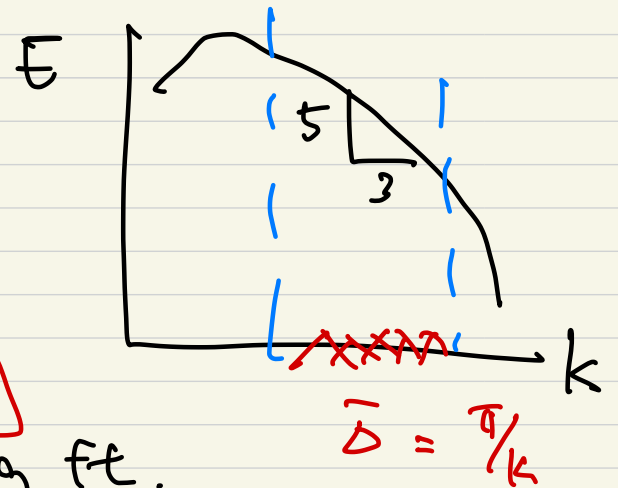
↑ isotropic turb.

$$C_s = l_s / \bar{\Delta} \cong 0.17$$

Damping ft. is required near the wall for  $C_s$  (Moin & Kim 1982)

$$C_s \bar{\Delta} \rightarrow C_s \bar{\Delta} [1 - \exp(-y^+ / A^+)]$$

van Driest damping ft.

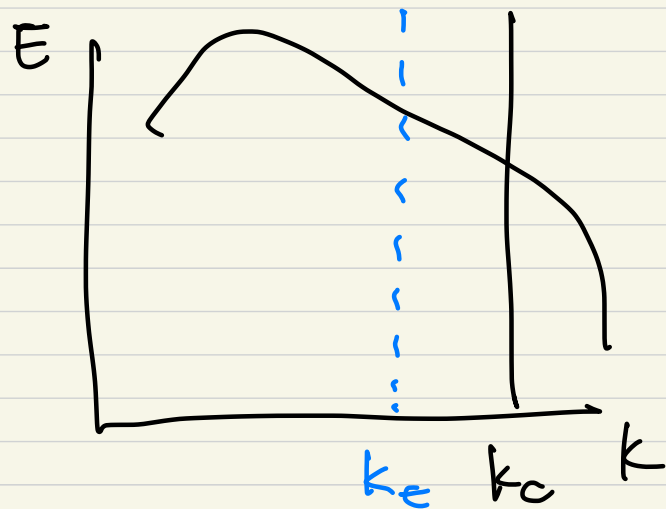


Problems : (1) We don't know  $C_s$

(2)  $\tau_{ij} \neq 0$  for laminar flow.

(5) Dynamic procedure of obtaining  $C_s$

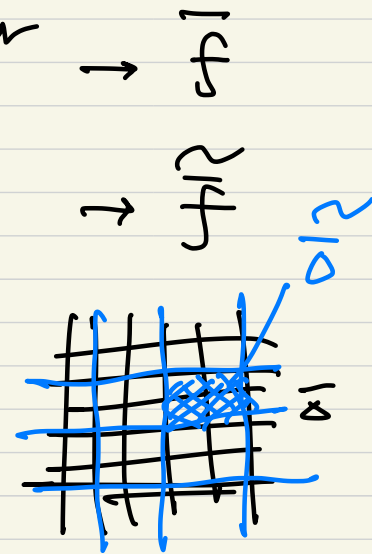
Dynamic subgrid-scale model (Germano et al. 1991)



$\pi/\Delta = k_c$  : cut-off filter  
grid "  $\rightarrow \Delta$

$\pi/\Delta_2 = k_t$  : test "  $\rightarrow \Delta_2$

$k_t < k_c \rightarrow \Delta_2 > \Delta$



Apply two filters to N-S eqs.

$$\frac{\partial \bar{u}_i}{\partial x_i} = 0$$

$$\frac{\partial \bar{u}_i}{\partial t} + \frac{\partial}{\partial x_j} \bar{u}_i \bar{u}_j = -\frac{\partial \bar{p}}{\partial x_i} + \mu \nabla^2 \bar{u}_i - \frac{\partial \bar{\tau}_{ij}}{\partial x_j}$$

$$T_{ij} = \overline{u_i u_j} - \overline{u_i} \overline{u_j}$$

Let  $L_{ij} \equiv \overline{u_i u_j} - \overline{u_i} \overline{u_j}$  : computable quantity

$$\tau_{ij} = \overline{u_i u_j} - \overline{u_i} \overline{u_j}$$

$$\rightarrow L_{ij} = T_{ij} - \tau_{ij}$$

Germano identity

②

$$C C_s \delta^2$$

$$C = C_s^2$$

modeling:  $\tau_{ij} - \frac{1}{3} \delta_{ij} \tau_{kk} = -2 C \overline{\delta^2} |\overline{S}| \overline{S}_{ij}$

$$T_{ij} - \frac{1}{3} \delta_{ij} T_{kk} = -2 C \overline{\delta^2} |\overline{S}| \overline{S}_{ij} \quad \text{--- ③}$$

② & ③  $\rightarrow$  ① :

$$L_{ij} - \frac{1}{3} \delta_{ij} L_{kk} = -2 C \left[ \overline{\delta^2} |\overline{S}| \overline{S}_{ij} - \overline{\delta^2} |\overline{S}| \overline{S}_{ij} \right]$$

$$= M_{ij}$$

$$\rightarrow C(\underline{x}, t) = \frac{L_{ij} - \frac{1}{3} \delta_{ij} L_{kk}}{M_{ij}} \quad ?$$

Least square method :  $Q = \left( L_{ij} - \frac{1}{3} \delta_{ij} L_{kk} + 2c M_{ij} \right)^2$

$$\frac{\partial Q}{\partial c} = 0 \rightarrow$$

$$c = -\frac{1}{2} \frac{L_{ij} M_{ij}}{M_{ij} M_{ij}}$$

dynamic  
Smagorinsky  
constant

Lilly (1992 PoF)

$(x, t)$

very large fluctuations of  $c$   
including negative values  $\rightarrow$  unstable

$$\Rightarrow c = -\frac{1}{2} \frac{\langle L_{ij} M_{ij} \rangle_h}{\langle M_{ij} M_{ij} \rangle_h}$$

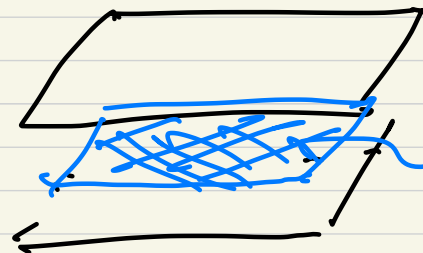
$h$ : homogeneous directions

averaging

clipping  $(c < 0$

$\rightarrow c = 0)$

or  $c < 0$  but  $\nu_t + \nu > 0$



$\Rightarrow c(y, t)$

~~$c(x, y, z, t)$~~



• Vreman model (2004, PoF)

$$\tau_{ij} = -2\nu_t \bar{S}_{ij}$$

$$\nu_t = C_v \sqrt{\frac{\mathbb{I}_\beta}{d_{ij} d_{ij}}}$$

zero for 13 laminar shear flows  
among 320 possible velocity  
derivative matrices

$$\bar{d}_{ij} = \partial \bar{u}_j / \partial x_i, \quad \mathbb{I}_\beta = \beta_{11} \beta_{22} - \beta_{12}^2 + \beta_{11} \beta_{33} - \beta_{13}^2 + \beta_{22} \beta_{33} - \beta_{23}^2,$$

$$\beta_{ij} = \sum_{m=1}^3 \frac{-2}{\Delta_m} \bar{d}_{me} \bar{d}_{nj}$$

$C_v = 0.07$  for isotropic  
turbulence

is not universal

$$\bar{d}_{ij} = \begin{bmatrix} \bar{d}_{11} & \bar{d}_{12} & \bar{d}_{13} \\ \bar{d}_{21} & \bar{d}_{22} & \bar{d}_{23} \\ \bar{d}_{31} & \bar{d}_{32} & \bar{d}_{33} \end{bmatrix}$$

→ dynamic procedure to determine  $C_v$  is required  
as done for Smagorinsky model

Park et al. (PoF 2006), Lee et al. (PoF 2010)

$C_s(x, t) \rightarrow \text{unstable} \rightarrow \langle C_s \rangle_{\text{homo}} \Leftarrow$  cannot be used

To overcome this problem,

dynamic localization model

(Ghosal et al. 1995)

for complex geometry because there is no homo. direction

$\rightarrow t$

Lagrangian dynamic model (Meneveau et al. 1996)

Vreman model  $\rightarrow C_v \rightarrow \text{dynamic } C_v \rightarrow \langle C_v \rangle_{\text{volume}}$

dynamic global model  $\leftarrow \overset{C_v(t)}{\text{stable sol.}}$

many subgrid-scale models are available!