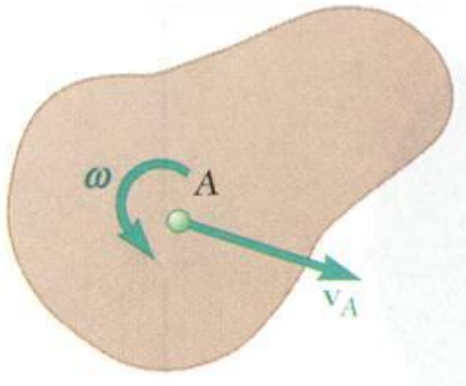


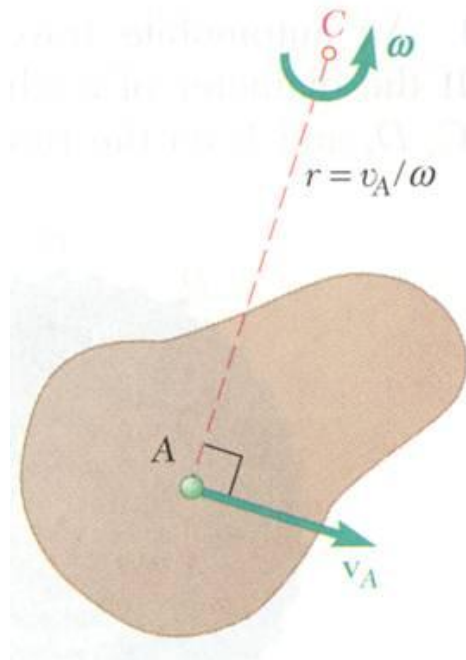
Kinematics of Rigid Bodies

Preview of 15.7- 15.9

15.7 Instantaneous Center of Rotation in Plane Motion

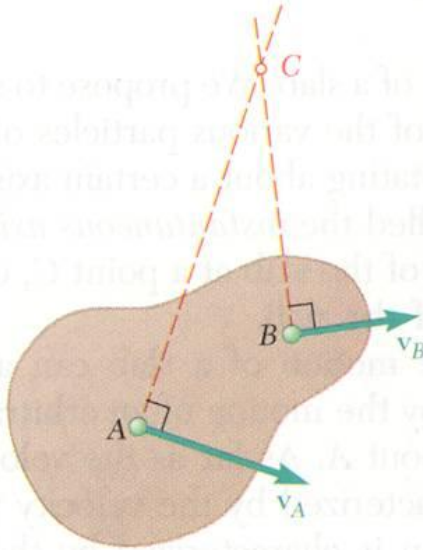


- Motion of a slab can always be replaced by the translation of an arbitrary point A + a rotation about A with an angular velocity that is **independent of the choice of A** .
- Now, Represent the motion with **an instantaneous center of rotation + angular velocity.**



- How do we find the instantaneous center of rotation?
On a line located perpendicular to V_A , distance of away from point A
- As far as the **velocities** are concerned, the slab seems to rotate about the *instantaneous center of rotation C* .

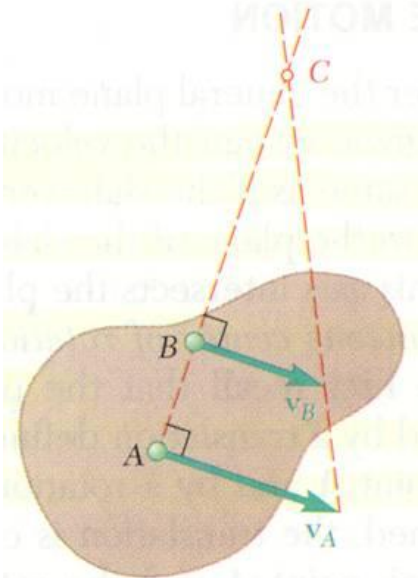
15.7 Instantaneous Center of Rotation in Plane Motion



- How do we find the center of rotation when the velocity at two points A and B are known?

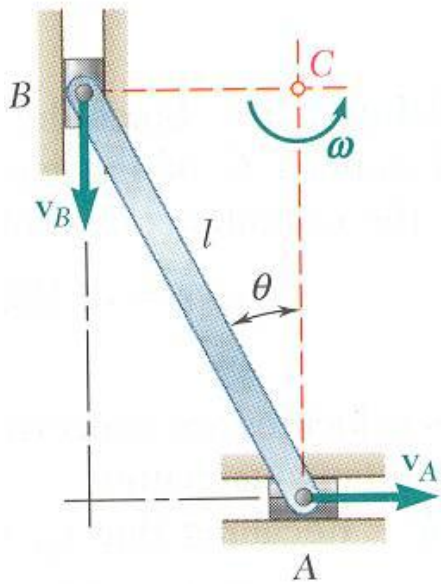
the instantaneous center of rotation lies at the intersection of the to \vec{v}_A and \vec{v}_B

- If the velocity vectors at A and B are perpendicular to the line AB , the instantaneous center of rotation lies at the intersection of the line AB with the line joining the of the velocity vectors at A and B .



- If the velocity magnitudes are equal, the instantaneous center of rotation is at infinity and the angular velocity is

15.7 Instantaneous Center of Rotation in Plane Motion

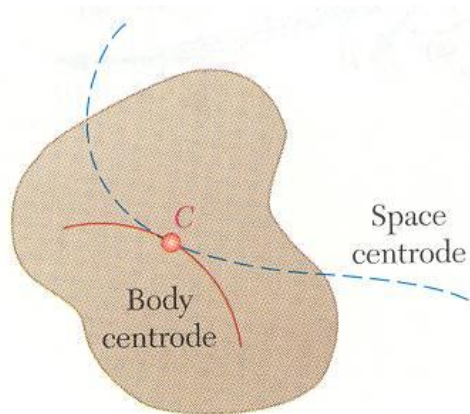


- The instantaneous center of rotation lies at the intersection of the perpendiculars to the velocity vectors through A and B .

$$\omega = \boxed{} = \boxed{}$$

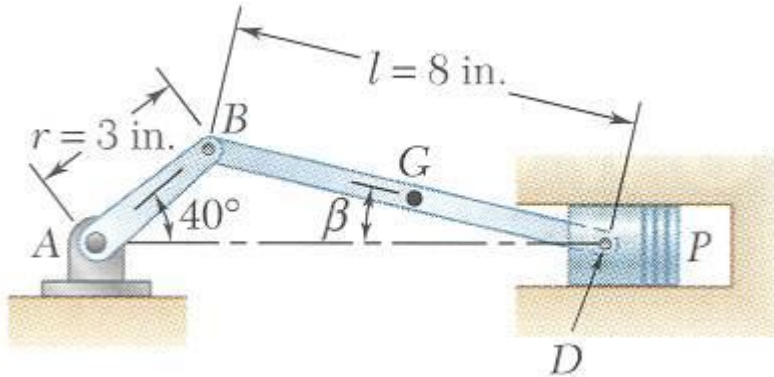
$$v_B = \boxed{} = \boxed{} \\ = \boxed{}$$

- The velocities of all particles on the rod are as if they were rotated about C .
- The particle **at the center of rotation** has velocity.
- The **particle coinciding with the center of rotation** changes with time and the **acceleration** of the particle at the instantaneous center of rotation is .
- The **acceleration** of the particles in the slab cannot be determined as if the slab were simply rotating about C .
- The trace of the locus of the center of rotation on the body is the **body centrode** and in space is the **space centrode**.



Kinematics of Rigid Bodies

Sample Problem 15.5



The crank AB has a constant clockwise angular velocity of 2000 rpm.

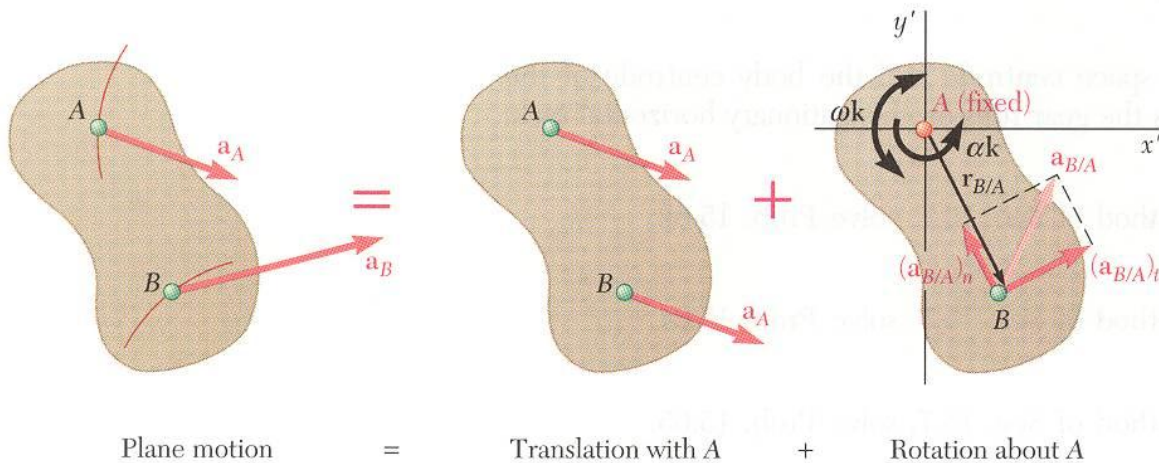
For the crank position indicated, determine (a) the angular velocity of the connecting rod BD , and (b) the velocity of the piston P .

Kinematics of Rigid Bodies

Sample Problem 15.5

Kinematics of Rigid Bodies

15.8 Absolute and Relative Acceleration in Plane Motion

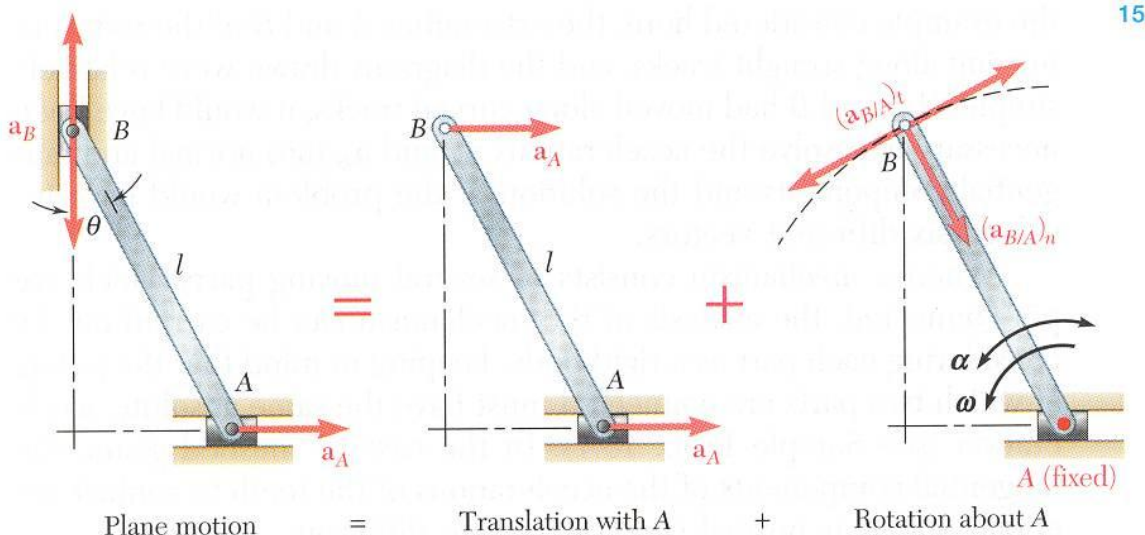


- Absolute acceleration of a particle of the slab,

- Relative acceleration $\vec{a}_{B/A}$ associated with rotation about A includes tangential and normal components,

$$\begin{aligned}
 (\vec{a}_{B/A})_t &= \boxed{} & (a_{B/A})_t &= \boxed{} \\
 (\vec{a}_{B/A})_n &= \boxed{} & (a_{B/A})_n &= \boxed{}
 \end{aligned}$$

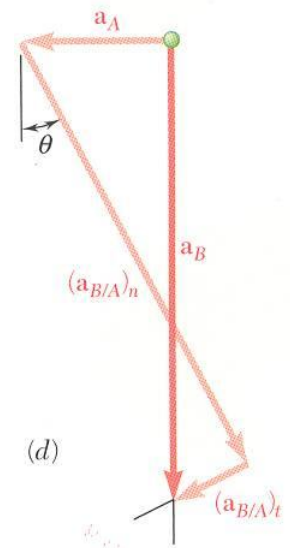
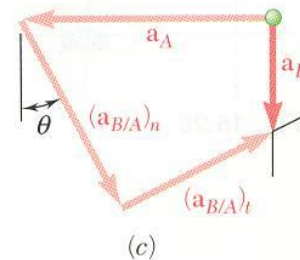
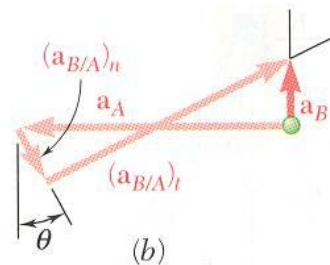
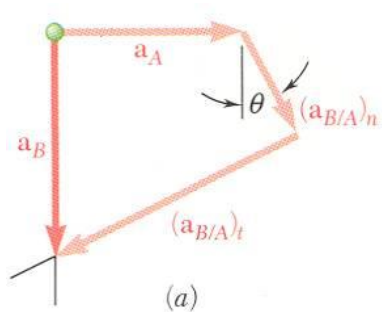
15.8 Absolute and Relative Acceleration in Plane Motion



- Given \vec{a}_A and \vec{v}_A , determine \vec{a}_B and $\vec{\alpha}$.

$$\vec{a}_B = \vec{a}_A + \vec{a}_{B/A}$$

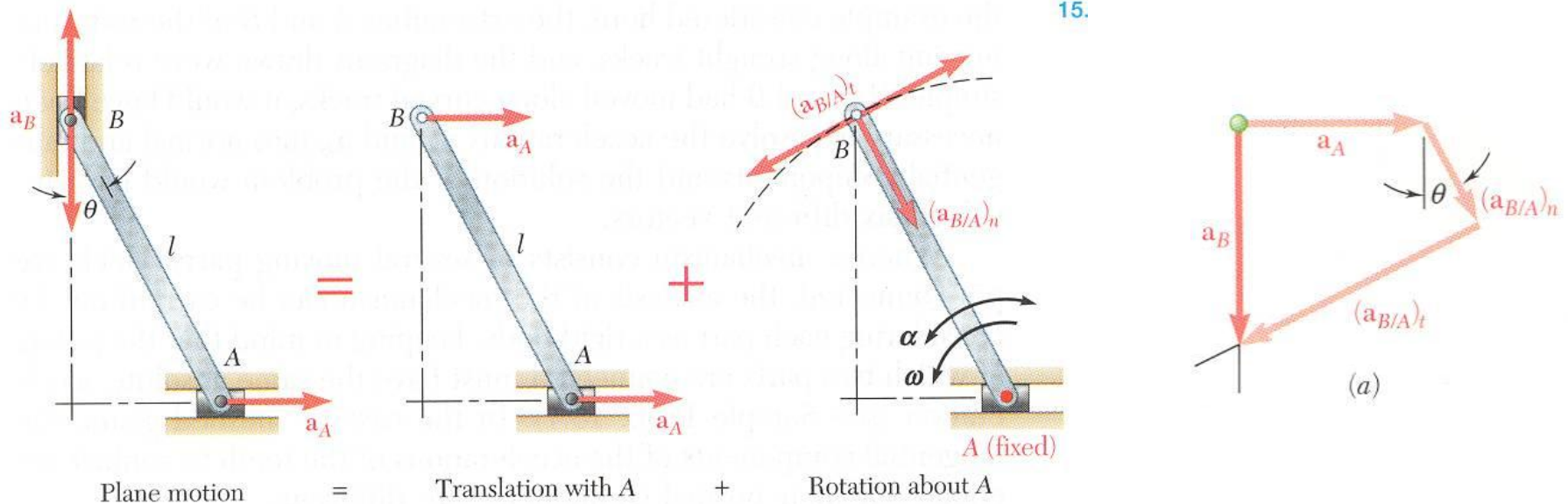
=



- Vector result depends on sense of \vec{a}_A and the relative magnitudes of a_A and $(a_{B/A})_n$
- Must also know angular velocity ω .

Kinematics of Rigid Bodies

15.8 Absolute and Relative Acceleration in Plane Motion



- Write $\vec{a}_B = \vec{a}_A + \vec{a}_{B/A}$ in terms of the two component equations,

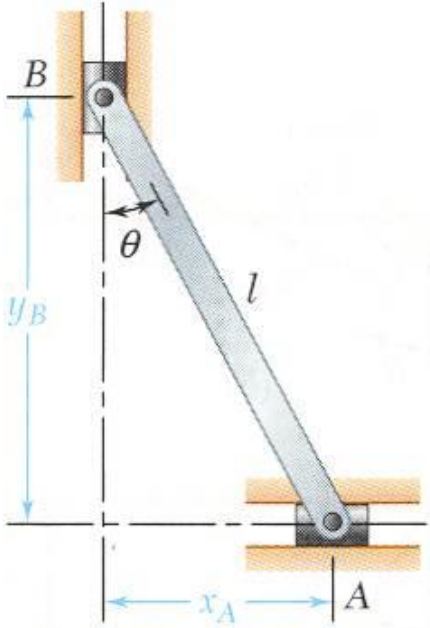
+ \rightarrow x components:

+ \uparrow y components:

- Solve for a_B and α .

Kinematics of Rigid Bodies

15.9 Analysis of Plane Motion in Terms of a Parameter



- In some cases, it is advantageous to determine the absolute velocity and acceleration of a mechanism directly.

$$x_A = \boxed{}$$

$$y_B = \boxed{}$$

$$\begin{aligned}v_A &= \dot{x}_A \\ &= l\dot{\theta} \cos \theta \\ &= l\omega \cos \theta\end{aligned}$$

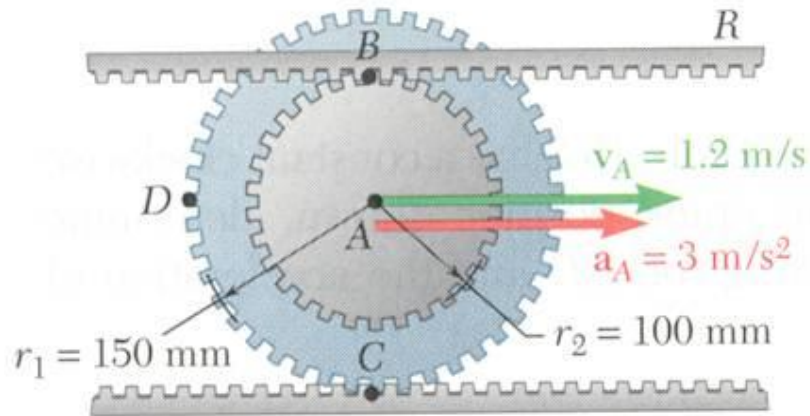
$$\begin{aligned}v_B &= \dot{y}_B \\ &= -l\dot{\theta} \sin \theta \\ &= -l\omega \sin \theta\end{aligned}$$

$$\begin{aligned}a_A &= \ddot{x}_A \\ &= -l\dot{\theta}^2 \sin \theta + l\ddot{\theta} \cos \theta \\ &= -l\omega^2 \sin \theta + l\alpha \cos \theta\end{aligned}$$

$$\begin{aligned}a_B &= \ddot{y}_B \\ &= -l\dot{\theta}^2 \cos \theta - l\ddot{\theta} \sin \theta \\ &= -l\omega^2 \cos \theta - l\alpha \sin \theta\end{aligned}$$

Kinematics of Rigid Bodies

Sample Problem 15.6



The center of the double gear has a velocity and acceleration to the right of 1.2 m/s and 3 m/s^2 , respectively. The lower rack is stationary.

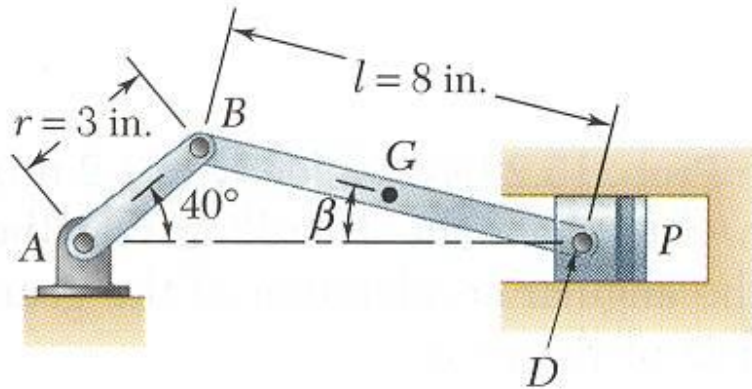
Determine (a) the angular acceleration of the gear, and (b) the acceleration of points B , C , and D .

Kinematics of Rigid Bodies

Sample Problem 15.6

Kinematics of Rigid Bodies

Sample Problem 15.7



Crank AG of the engine system has a constant clockwise angular velocity of 2000 rpm.

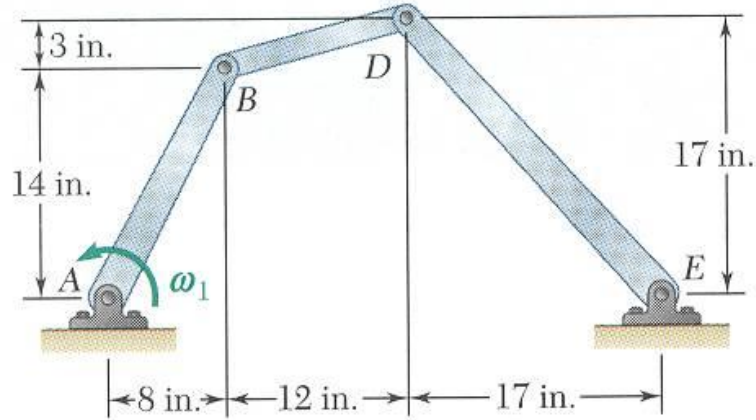
For the crank position shown, determine the angular acceleration of the connecting rod BD and the acceleration of point D .

Kinematics of Rigid Bodies

Sample Problem 15.7

Kinematics of Rigid Bodies

Sample Problem 15.8



In the position shown, crank AB has a constant angular velocity $\omega_1 = 20 \text{ rad/s}$ counterclockwise.

Determine the angular velocities and angular accelerations of the connecting rod BD and crank DE .

Kinematics of Rigid Bodies

Sample Problem 15.8