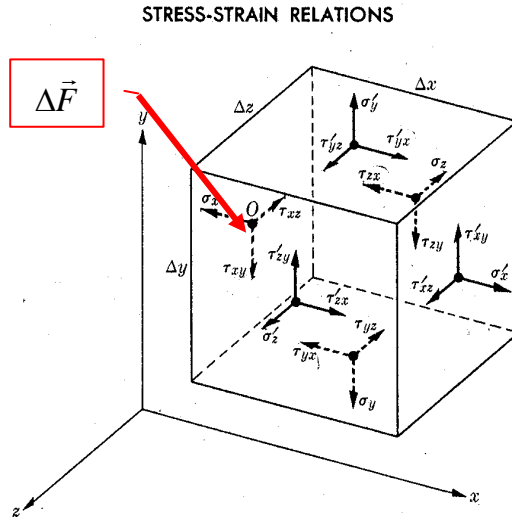


## Chapter 5 Stress – Strain Relation

### 5.1 General Stress – Strain system

#### Parallelepiped, cube



#### 5.1.1 Surface Stress

Surface stresses:  $\left\{ \begin{array}{l} \text{normal stress} - \sigma_x \\ \text{shear stress} - \tau_{xy}, \tau_{xz} \end{array} \right.$

$$\sigma_{xx} = \sigma_x = \lim_{\Delta A_x \rightarrow 0} \frac{\Delta F_x}{\Delta A_x}$$

$(\Delta A_x = \Delta y \Delta z)$

$$\tau_{xy} = \lim_{\Delta A_x \rightarrow 0} \frac{\Delta F_y}{\Delta A_x}$$

$$\tau_{xz} = \lim_{\Delta A_x \rightarrow 0} \frac{\Delta F_z}{\Delta A_x}$$

$$\tau_{yx} = \lim_{\Delta A_y \rightarrow 0} \frac{\Delta F_x}{\Delta A_y}$$

$(\Delta A_y = \Delta x \Delta z)$

$$\sigma_{yy} = \sigma_y = \lim_{\Delta A_y \rightarrow 0} \frac{\Delta F_y}{\Delta A_y}$$

$$\tau_{yz} = \lim_{\Delta A_y \rightarrow 0} \frac{\Delta F_z}{\Delta A_y}$$

$$\tau_{zx} = \lim_{\Delta A_z \rightarrow 0} \frac{\Delta F_x}{\Delta A_z}$$

$(\Delta A_z = \Delta x \Delta y)$

$$\tau_{zy} = \lim_{\Delta A_z \rightarrow 0} \frac{\Delta F_y}{\Delta A_z}$$

$$\sigma_{zz} = \sigma_z = \lim_{\Delta A_z \rightarrow 0} \frac{\Delta F_z}{\Delta A_z}$$

where  $\Delta F_x, \Delta F_y, \Delta F_z$  = component of force vector  $\vec{\Delta F}$

$\Delta F_x$  – acting in the direction of the x-axis

$$\Delta A_x = \text{area of the x- face of the element} = \Delta y \Delta z$$

$$\Delta A_y = \Delta x \Delta z$$

$$\Delta A_z = \Delta x \Delta y$$

•subscripts

$\sigma_x$ : subscript indicates the direction of stress

$\tau_{xy}$ : 1st - direction of the normal to the face on which  $\tau$  acts

2nd - direction in which  $\tau$  acts

•general stress system: stress tensor

~ 9 scalar components

$$\begin{pmatrix} \sigma_{xx} & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_{yy} & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_{zz} \end{pmatrix}$$

[Re] Tensor:

~ an ordered array of entities which is invariant under coordinate transformation; includes scalars & vectors

~  $3^n$

0th order – 1 component, scalar ( mass, length, pressure)

1st order – 3 components, vector (velocity, force, acceleration)

2nd order – 9 components (stress, rate of strain, turbulent diffusion)

At three other surfaces,

$$\begin{aligned}\sigma_x' &= \sigma_x + \frac{\partial \sigma_x}{\partial x} \Delta x \\ \sigma_y' &= \sigma_y + \frac{\partial \sigma_y}{\partial y} \Delta y \\ \sigma_z' &= \sigma_z + \frac{\partial \sigma_z}{\partial z} \Delta z \\ \tau_{xy}' &= \tau_{xy} + \frac{\partial \tau_{xy}}{\partial x} \Delta x \\ \tau_{yx}' &= \tau_{yx} + \frac{\partial \tau_{yx}}{\partial y} \Delta y \\ \tau_{zx}' &= \tau_{zx} + \frac{\partial \tau_{zx}}{\partial z} \Delta z\end{aligned}\quad (5.1)$$

◆ Shear stress is symmetric.

→ Shear stress pairs with subscripts differing in order are equal.

$$\rightarrow \tau_{xy} = \tau_{yx}$$

**[Proof]**

In static equilibrium, sum of all moments and sum of all forces equal zero for the element.

First, apply Newton's 2nd law

$$\sum F = m \frac{du}{dt}$$

Then, consider Torque (angular momentum),  $T$

$$\sum T = \frac{d}{dt}(rmu) = \frac{d}{dt}(r^2 m \omega) = \frac{d}{dt}(I \omega) = I \frac{d\omega}{dt}$$

where  $I = \text{moment of inertia} = r^2 m$

$r = \text{radius of gyration}$

$$\frac{d\omega}{dt} = \text{angular acceleration}$$

Thus,

$$\sum T = mr^2 \frac{d\omega}{dt} \tag{A}$$

Now, take a moment about a centroid axis in the z-direction

$$LHS = \sum T = (\Delta y \Delta z \tau_{xy}) \frac{\Delta x}{2} - (\tau_{yx} \Delta x \Delta z) \frac{\Delta y}{2} = \frac{\Delta x \Delta y \Delta z}{2} (\tau_{xy} - \tau_{yx})$$

$$RHS = \rho dvol r^2 \frac{d\omega}{dt} = \Delta x \Delta y \Delta z \rho r^2 \frac{d\omega}{dt}$$

$$\therefore (\tau_{xy} - \tau_{yx}) \Delta x \Delta y \Delta z = 2 \Delta x \Delta y \Delta z \rho r^2 \frac{d\omega}{dt}$$

After canceling terms, this gives

$$\tau_{xy} - \tau_{yx} = 2 \rho r^2 \frac{d\omega}{dt}$$

$$\lim_{\Delta x, \Delta y, \Delta z \rightarrow 0} r^2 \rightarrow 0$$

$$\tau_{xy} - \tau_{yx} = 0$$

$$\therefore \tau_{xy} = \tau_{yx}$$

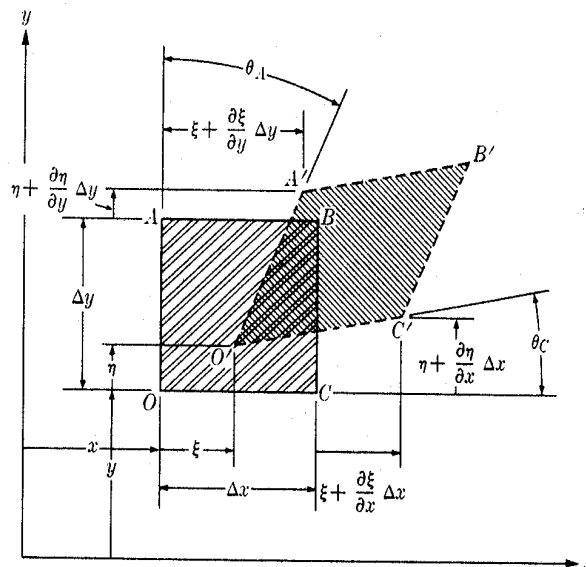
**[Homework Assignment-Special work]**

**Due: 1 week from today**

1. Make your own “Stress Cube” using paper box.

5.1.2 Strain components

- Strain
  - normal strain:  $\epsilon \leftarrow$  linear deformation
  - shear strain:  $\gamma \leftarrow$  angular deformation



[Re] displacement vs. deformation

i) Displacement (translation):  $\xi, \eta, \zeta$

$$O(x, y, z) \rightarrow O'(x + \xi, y + \eta, z + \zeta)$$

ii) Deformation: due to system of external forces

$$OABC \rightarrow O'A'B'C'$$

[Cf] Motion

- translation
- rotation

[Cf] Deformation

- Linear deformation
- Angular deformation

(1) Deformation

1) Normal strain,  $\varepsilon$

$$\varepsilon = \frac{\text{change in length}}{\text{original length}}$$

$$\varepsilon_x = \lim_{\Delta x \rightarrow 0} \frac{O'C' - OC}{OC} = \lim_{\Delta x \rightarrow 0} \frac{\left\{ \left( x + \Delta x + \xi + \frac{\partial \xi}{\partial x} \Delta x \right) - (x + \xi) \right\} - \Delta x}{\Delta x} = \frac{\partial \xi}{\partial x}$$

$$\varepsilon_y = \lim_{\Delta y \rightarrow 0} \frac{O'A' - OA}{OA} = \lim_{\Delta y \rightarrow 0} \frac{\left\{ \left( y + \Delta y + \eta + \frac{\partial \eta}{\partial y} \Delta y \right) - (y + \eta) \right\} - \Delta y}{\Delta y} = \frac{\partial \eta}{\partial y}$$

$$\varepsilon_x = \frac{\partial \zeta}{\partial z}$$

~  $\varepsilon$  is positive when element elongates under deformation

2) Shear strain,  $\gamma$

~ **change in angle** between two originally perpendicular elements

For  $xy$ -plane

$$\begin{aligned} \gamma_{xy} &= \lim_{\Delta x, \Delta y \rightarrow 0} (\theta_c + \theta_A) \cong \lim_{\Delta x, \Delta y \rightarrow 0} (\tan \theta_c + \tan \theta_A) \\ &= \lim_{\Delta x, \Delta y \rightarrow 0} \left\{ \frac{\frac{\partial \eta}{\partial x} \Delta x}{\Delta x + \frac{\partial \xi}{\partial x} \Delta x} + \frac{\frac{\partial \xi}{\partial y} \Delta y}{\Delta y + \frac{\partial \eta}{\partial y} \Delta y} \right\} = \frac{\partial \eta}{\partial x} + \frac{\partial \xi}{\partial y} \end{aligned}$$

(∴  $\Delta x \cdot \frac{\partial \xi}{\partial x} < \Delta x$ )

$$\gamma_{xy} = \frac{\partial \eta}{\partial x} + \frac{\partial \xi}{\partial y}$$

$$\gamma_{yz} = \frac{\partial \zeta}{\partial y} + \frac{\partial \eta}{\partial x} \tag{5.4}$$

$$\gamma_{zx} = \frac{\partial \xi}{\partial z} + \frac{\partial \zeta}{\partial x}$$

(2) displacement vector  $\vec{\delta}$

$$\vec{\delta} = \xi \vec{i} + \eta \vec{j} + \zeta \vec{k}$$

(3) Volume dilation

$$e = \frac{\text{change of volume of deformed element}}{\text{original volume}}$$

$$e = \frac{d(\Delta V)}{\Delta V} = \frac{\left(\Delta x + \frac{\partial \xi}{\partial x} \Delta x\right) \left(\Delta y + \frac{\partial \eta}{\partial y} \Delta y\right) \left(\Delta z + \frac{\partial \zeta}{\partial z} \Delta z\right) - \Delta x \Delta y \Delta z}{\Delta x \Delta y \Delta z}$$

$$\cong \frac{\partial \xi}{\partial x} + \frac{\partial \eta}{\partial y} + \frac{\partial \zeta}{\partial z} = \varepsilon_x + \varepsilon_y + \varepsilon_z$$

$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z \quad (5.6)$$

$$e = \frac{\partial \xi}{\partial x} + \frac{\partial \eta}{\partial y} + \frac{\partial \zeta}{\partial z} = \nabla \cdot \vec{\delta} \quad \text{--- divergence} \quad (5.7)$$

## 5.2 Relations between Stress and Strain for Elastic Solids

### 5.2.1 Normal Stresses

Hooke's law: stress is linear with strain

$$\sigma_x = E \cdot \varepsilon_x^\circ$$

$$\varepsilon_x^\circ = \frac{1}{E} \sigma_x$$

in which  $E$  = Young's modulus of elasticity



$\varepsilon_x^\circ$  = elongation in the  $x$  – *dir* . due to normal stress,  $\sigma_x$

$$y - \text{dir.} : \varepsilon_y^\circ = \frac{\sigma_y}{E}$$

$$z - \text{dir.} : \varepsilon_z^\circ = \frac{\sigma_z}{E}$$

Now, we have to consider other elongations because of lateral contraction of matter under tension.

$\varepsilon_x'$  = elongation in the  $x$  – *dir* . due to  $\sigma_y$

$\varepsilon_x''$  = elongation in the  $x$  – *dir* . due to  $\sigma_z$

Now, define

$$\varepsilon_x' = -n\varepsilon_y^\circ = -n\frac{\sigma_y}{E} \quad (5.9)$$

$$\varepsilon_x'' = -n\varepsilon_z^\circ = -n\frac{\sigma_z}{E} \quad (5.10)$$

where  $n$  = **Poisson's ratio**

Thus, total strain  $\varepsilon_x$  is

$$\begin{aligned} \varepsilon_x &= \varepsilon_x^\circ + \varepsilon_x' + \varepsilon_x'' = \frac{\sigma_x}{E} - \frac{n}{E}(\sigma_y + \sigma_z) = \frac{1}{E}[\sigma_x - n(\sigma_y + \sigma_z)] \\ \varepsilon_y &= \frac{1}{E}[\sigma_y - n(\sigma_z + \sigma_x)] \\ \varepsilon_z &= \frac{1}{E}[\sigma_z - n(\sigma_x + \sigma_y)] \end{aligned} \quad (5.12)$$

## 5.2.2 Shear Stress

~ Hooke's law  $\tau_{xy} = G\gamma_{xy}$

$$\gamma_{xy} = \frac{\tau_{xy}}{G} = \frac{\partial \eta}{\partial x} + \frac{\partial \xi}{\partial y}$$

$$\gamma_{yz} = \frac{\tau_{yz}}{G} = \frac{\partial \xi}{\partial y} + \frac{\partial \eta}{\partial z}$$

$$\gamma_{zx} = \frac{\tau_{zx}}{G} = \frac{\partial \xi}{\partial z} + \frac{\partial \zeta}{\partial x}$$

where  $G =$  shear modulus of elasticity

$$G = \frac{E}{2(1+n)} \quad (5.14)$$

■ Volume dialation

$$\begin{aligned} \varepsilon &= \varepsilon_x + \varepsilon_y + \varepsilon_z = \frac{1}{E} \left[ \sigma_x - n(\sigma_y + \sigma_z) \right] \\ &\quad + \frac{1}{E} \left[ \sigma_y - n(\sigma_z + \sigma_x) \right] \\ &\quad + \frac{1}{E} \left[ \sigma_z - n(\sigma_x + \sigma_y) \right] \\ &= \frac{1}{E} \left[ (1-2n)(\sigma_x + \sigma_y + \sigma_z) \right] \end{aligned} \quad (5.15)$$

■  $\bar{\sigma}$  = arithmetic mean of 3 normal stresses

$$\bar{\sigma} = \frac{1}{3}(\sigma_x + \sigma_y + \sigma_z) \quad (5.16)$$

From Eq. (5.12), (5.14) and (5.15)

$$\sigma_x = 2G \left[ \varepsilon_x + \frac{ne}{1-2n} \right] \quad (5.17)$$

Therefore

$$\begin{aligned} \sigma_x - \bar{\sigma} &= 2G \left( \varepsilon_x - \frac{e}{3} \right) \\ \sigma_y - \bar{\sigma} &= 2G \left( \varepsilon_y - \frac{e}{3} \right) \\ \sigma_z - \bar{\sigma} &= 2G \left( \varepsilon_z - \frac{e}{3} \right) \end{aligned} \quad (5.18)$$

$$\begin{aligned} \tau_{xy} = \tau_{yx} &= G \left( \frac{\partial \eta}{\partial x} + \frac{\partial \xi}{\partial y} \right) \\ \tau_{zy} = \tau_{yz} &= G \left( \frac{\partial \zeta}{\partial y} + \frac{\partial \eta}{\partial z} \right) \\ \tau_{xz} = \tau_{zx} &= G \left( \frac{\partial \xi}{\partial z} + \frac{\partial \zeta}{\partial x} \right) \end{aligned} \quad (5.19)$$

**[Proof]** Eq. (5.17) & (5.18)

$$(5.15) \rightarrow e = \frac{1}{E} (1-2n) (\sigma_x + \sigma_y + \sigma_z) \quad (A)$$

$$(5.12) \rightarrow \varepsilon_x = \frac{1}{E} [\sigma_x - n(\sigma_y + \sigma_z)] \quad (B)$$

$$(5.14) \rightarrow G = \frac{E}{2(1+n)} \rightarrow E = 2G(1+n) \quad (C)$$

i) Combine (A) and (B)

$$+ \left[ \begin{aligned} \frac{n}{(1+2n)} \times e &= \frac{n}{(1-2n)} \frac{(1-2n)}{E} (\sigma_x + \sigma_y + \sigma_z) = \frac{n}{E} (\sigma_x + \sigma_y + \sigma_z) \\ \varepsilon_x &= \frac{1}{E} [\sigma_x - n(\sigma_y + \sigma_z)] \end{aligned} \right]$$

$$\frac{n}{(1-2n)} e + \varepsilon_x = \frac{1+n}{E} \sigma_x$$

$$\therefore \sigma_x = \frac{E}{1+n} \left[ \varepsilon_x + \frac{n}{(1-2n)} e \right] \quad (D)$$

Substitute (C) into (D)

$$\therefore \sigma_x = 2G \left[ \varepsilon_x + \frac{n}{(1-2n)} e \right] \rightarrow \text{Eq. (5.17)}$$

ii) Subtract (5.16) from (5.17)

$$\sigma_x - \bar{\sigma} = 2G \left[ \varepsilon_x + \frac{n}{(1-2n)} e \right] - \frac{1}{3} (\sigma_x + \sigma_y + \sigma_z) \quad (E)$$

Substitute (A) into (E);  $\sigma_x + \sigma_y + \sigma_z = \frac{E}{(1-2n)} e$

$$\therefore \text{RHS of (E)} = 2G \left[ \varepsilon_x + \frac{n}{(1-2n)} e \right] - \frac{1}{3} \frac{E}{(1-2n)} e$$

$$= 2G \varepsilon_x + \left[ \frac{2Gn}{(1-2n)} - \frac{1}{3} \frac{2G(1+n)}{(1-2n)} \right] e = 2G \left\{ \varepsilon_x \left[ \frac{n}{(1-2n)} - \frac{\frac{1+n}{3}}{(1+2n)} \right] e \right\}$$

$$= 2G \left\{ \varepsilon_x + \frac{-\frac{1}{3}(1-2n)}{(1-2n)} e \right\}$$

$$= 2G \left( \varepsilon_x - \frac{1}{3} e \right) \quad \rightarrow \text{Eq. (5.18)}$$

### 5.3 Relations between Stress and Rate of Strain for Newtonian Fluids

Experimental evidence suggests that, in fluid, stress is linear with time rate of strain.

$$\rightarrow \text{stress} \propto \frac{\partial}{\partial t}(\text{strain})$$

$\rightarrow$  Newtonian fluid (**Newton's law of viscosity**)

[Cf] For solid,

$$\text{stress} \propto \text{strain}$$

#### 5.3.1 Normal stress

For solid, Eq. (5.18) can be used as

$$\text{Hookeian elastic solid: } \sigma_x - \bar{\sigma} = 2 \left( \frac{F}{L^2} \right) \left( \varepsilon_x - \frac{e}{3} \right)$$

By analogy,

$$\text{Newtonian fluid: } \sigma_x - \bar{\sigma} = 2 \left( \frac{Ft}{L^2} \right) \frac{\partial}{\partial t} \left( \varepsilon_x - \frac{e}{3} \right) \quad (5.20)$$

$$\text{Now set } \mu \equiv \frac{Ft}{L^2} = \text{dynamic viscosity}$$

$$\sigma_x - \bar{\sigma} = 2\mu \frac{\partial \varepsilon_x}{\partial t} - \frac{2}{3}\mu \frac{\partial e}{\partial t} \quad (5.21)$$

By the way,

$$\varepsilon_x = \frac{\partial \xi}{\partial x}; e = \nabla \cdot \vec{\delta}$$

$$u = \frac{\partial \xi}{\partial t}, v = \frac{\partial \eta}{\partial x}, w = \frac{\partial \zeta}{\partial t} \quad (\xi, \eta, \zeta = \text{displacement})$$

Therefore,

$$\frac{\partial \xi}{\partial t} = \frac{\partial}{\partial t} \left( \frac{\partial \xi}{\partial x} \right) = \frac{\partial}{\partial x} \left( \frac{\partial \xi}{\partial t} \right) = \frac{\partial u}{\partial x} \quad (5.22)$$

$$\frac{\partial e}{\partial t} = \nabla \cdot \frac{\partial \vec{\delta}}{\partial t} = \nabla \cdot \vec{q} = \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} \quad (5.23)$$

$$\begin{aligned} \vec{\delta} &= \xi \vec{i} + \eta \vec{j} + \zeta \vec{k} \\ \vec{q} &= \frac{\partial \vec{\delta}}{\partial t} = u \vec{i} + v \vec{j} + w \vec{k} \\ \nabla \cdot \vec{q} &= \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} \end{aligned}$$

Eq. (5.21) becomes

$$\sigma_x = \bar{\sigma} + 2\mu \frac{\partial u}{\partial x} - \frac{2}{3}\mu (\nabla \cdot \vec{q})$$

For compressible fluid,

$$\sigma_x = \bar{\sigma} + 2\mu \frac{\partial u}{\partial x} - \frac{2}{3}\mu (\nabla \cdot \vec{q})$$

$$\sigma_y = \bar{\sigma} + 2\mu \frac{\partial v}{\partial y} - \frac{2}{3}\mu (\nabla \cdot \vec{q})$$

$$\sigma_z = \bar{\sigma} + 2\mu \frac{\partial w}{\partial z} - \frac{2}{3}\mu(\nabla \cdot \vec{q}) \quad (5.24)$$

For incompressible fluid,

$$\frac{de}{dt} = \nabla \cdot \vec{q} = 0 \quad \leftarrow \text{time rate of volume expansion} = 0$$

$$\rightarrow \nabla \cdot \vec{q} = 0 \rightarrow \text{Continuity Eq.}$$

Therefore, Eq. (5.24) becomes

$$\sigma_x = \bar{\sigma} + 2\mu \frac{\partial u}{\partial x}$$

$$\sigma_y = \bar{\sigma} + 2\mu \frac{\partial v}{\partial y}$$

$$\sigma_z = \bar{\sigma} + 2\mu \frac{\partial w}{\partial z}$$

### 5.3.2. Shear stress

By following the same analogy

$\mu$

$$\tau_{xy} = G \left( \frac{\partial \eta}{\partial x} + \frac{\partial \xi}{\partial y} \right) = \left( \frac{Ft}{L^2} \right) \frac{\partial}{\partial t} \left( \frac{\partial \eta}{\partial x} + \frac{\partial \xi}{\partial y} \right)$$

$$= \mu \frac{\partial}{\partial x} \left( \frac{\partial \eta}{\partial t} \right) + \frac{\partial}{\partial y} \left( \frac{\partial \xi}{\partial t} \right) = \mu \left( \frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right)$$

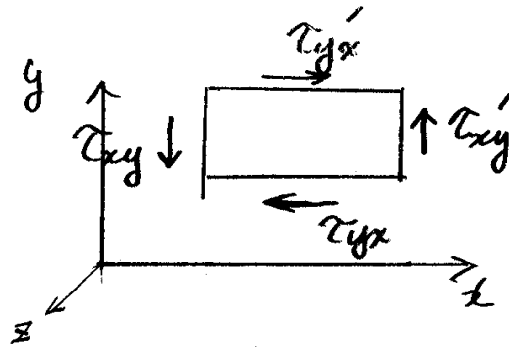
$$\frac{\partial \eta}{\partial t} = v$$

$$\frac{\partial \xi}{\partial t} = u$$

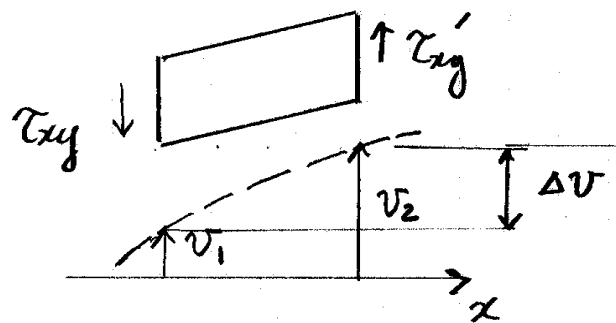
$$\begin{aligned}
 \tau_{xy} = \tau_{yx} &= \mu \left( \frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right) \\
 \tau_{zy} = \tau_{yz} &= \mu \left( \frac{\partial w}{\partial y} + \frac{\partial v}{\partial z} \right) \\
 \tau_{xz} = \tau_{zx} &= \mu \left( \frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right)
 \end{aligned}
 \tag{5.25}$$

[Appendix 1]

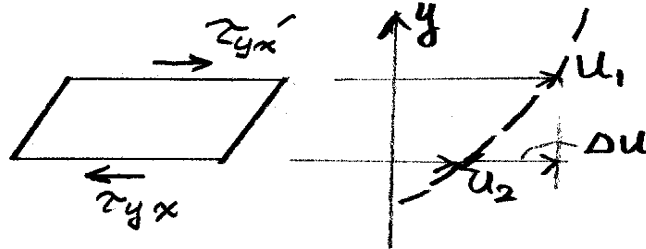
$$\tau_{xy} = \tau_{yx} = \mu \left( \frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right)$$



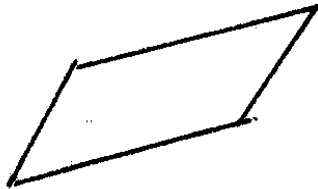
i)  $\tau_{xy}, \tau_{yx}'$





ii)  $\tau_{yx}, \tau_{yx}'$ 

iii) composition

5.3.3 Relation between thermodynamic pressure  $p$  and mean normal stress  $\bar{\sigma}$ 1) Assume viscous effects are completely represented by the viscosity  $\mu$  forincompressible fluid

$$\bar{\sigma} = -p = \frac{1}{3}(\sigma_x + \sigma_y + \sigma_z) \quad (5.26)$$

~ minus sign accounts for pressure (compression)

2) For compressible fluid

$$\bar{\sigma} = -p + \mu'(\nabla \cdot \vec{q})$$

in which  $\mu'$  = 2nd coefficient of viscosity associated solely with dilation

= bulk viscosity

Since, dilation effect is small for most cases

$$\mu'(\nabla \cdot \vec{q}) \rightarrow 0$$

$$\therefore \bar{\sigma} = -p$$

For zero-dilation viscosity effects ( $\mu' = 0$ ), (5.24) becomes

$$\begin{aligned} \sigma_x &= -p + 2\mu \frac{\partial u}{\partial x} - \left(\frac{2}{3}\right)\mu(\nabla \cdot \vec{q}) \\ \sigma_y &= -p + 2\mu \frac{\partial v}{\partial y} - \left(\frac{2}{3}\right)\mu(\nabla \cdot \vec{q}) \\ \sigma_z &= -p + 2\mu \frac{\partial w}{\partial z} - \left(\frac{2}{3}\right)\mu(\nabla \cdot \vec{q}) \end{aligned} \tag{5.29}$$



■ Shear stresses in a real fluid

$$\begin{aligned} \tau_{xy} = \tau_{yx} &= \mu \left( \frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right) \\ \tau_{zy} = \tau_{yz} &= \mu \left( \frac{\partial w}{\partial y} + \frac{\partial v}{\partial z} \right) \\ \tau_{xz} = \tau_{zx} &= \mu \left( \frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right) \end{aligned} \tag{5.30}$$

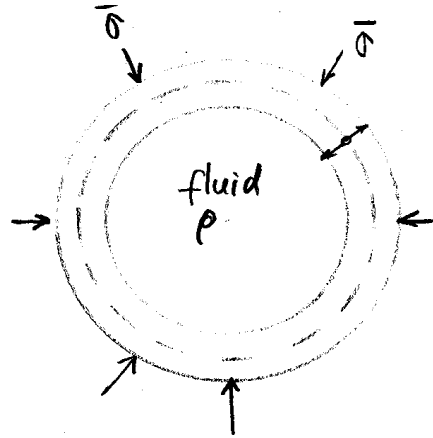
For zero viscous effects ( $\mu = 0$ )  $\rightarrow$  inviscid fluids in motion and for all fluids at rest

$$\sigma_x = \sigma_y = \sigma_z = \bar{\sigma} = -p$$

$$\tau_{xy} = \tau_{yz} = \tau_{zx} = 0$$

**[Appendix 2]** Bulk viscosity and thermodynamic pressure

→ Boundary-Layer Theory (Schlichting, 1979) pp. 61-63



$$\bar{\sigma} = -p + \mu'(\nabla \cdot \vec{q})$$

If fluid is compressed, expanded or made to oscillate at a finite rate, work done in a thermodynamically reversible process per unit volume is

$$W = p \nabla \cdot \vec{q} = P \frac{de}{dt} \sim \text{dissipation of energy}$$

where  $\mu'$  = bulk viscosity of fluid that represents that property which is responsible for energy dissipation in a fluid of uniform temperature during a change in volume at a finite rate  
= second property of a compressible, isotropic, Newtonian fluid

[Cf]  $\mu$  = shear viscosity = first property

$$\mu' = 0, \quad p = -\bar{\sigma}$$

$$\mu' \neq 0, \quad p \neq -\bar{\sigma}$$

Direct measurement of bulk viscosity is very difficult.

**[Appendix 3]** Normal stress

Normal stress = pressure + deviation from it

$$\sigma_x = -p + \sigma'_x$$

$$\sigma_y = -p + \sigma'_y$$

$$\sigma_z = -p + \sigma'_z$$

Thus, stress matrix becomes

$$\begin{pmatrix} -p & 0 & 0 \\ 0 & -p & 0 \\ 0 & 0 & -p \end{pmatrix} + \begin{pmatrix} \sigma'_x & \tau_{xy} & \tau_{xz} \\ \tau_{yz} & \sigma'_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma'_z \end{pmatrix}$$

Normal stresses are proportional to the volume change (compressibility) and corresponding components of linear deformation,  $a$ ,  $b$ ,  $c$ .

Thus,

$$\sigma_x = -p + \lambda(a + b + c) + 2\mu a$$

$$\sigma_y = -p + \lambda(a + b + c) + 2\mu b$$

$$\sigma_z = -p + \lambda(a + b + c) + 2\mu c$$

where  $\lambda$  = compressibility coefficient

**Homework Assignment # 3****Due: 1 week from today**

5-1. Verify Eq. (5-14)

$$G = \frac{E}{2(1+n)}$$

5-3. Consider a fluid element under a general state of stress as illustrated in Fig. 5-1. Given that the element is in a gravity field, show that the equilibrium requirement between surface, body and inertial forces leads to the equations

$$\frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z} + \rho g_x = \rho a_x$$

$$\frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} + \frac{\partial \tau_{zy}}{\partial z} + \rho g_y = \rho a_y$$

$$\frac{\partial \tau_{xz}}{\partial x} + \frac{\partial \tau_{yz}}{\partial y} + \frac{\partial \sigma_z}{\partial z} + \rho g_z = \rho a_z$$

5-4. Consider a fluid in two-dimensional motion. Using plane polar coordinates  $r$ ,  $\theta$ , and  $z$ , show that the rate of strain components are

$$\frac{\partial \varepsilon_r}{\partial t} = \frac{\partial v_r}{\partial r}, \quad \frac{\partial \varepsilon_\theta}{\partial t} = \frac{1}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r}{r}, \quad \frac{\partial \gamma_{r\theta}}{\partial t} = \frac{\partial v_\theta}{\partial r} + \frac{1}{r} \frac{\partial v_r}{\partial \theta} - \frac{v_\theta}{r}$$