

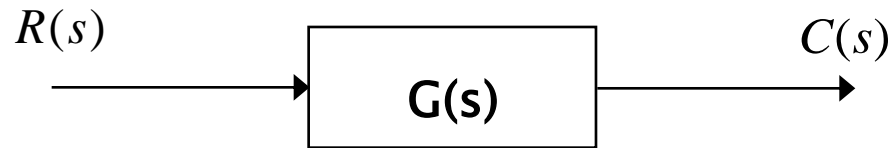
Linear Systems Analysis

in the Time Domain II

- Transient Response -



Second Order Systems



$$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$R(s) = \frac{1}{s} \text{ (step input), } C(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \cdot \frac{1}{s} = \frac{K_1}{s} + \frac{K_2 s + K_3}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

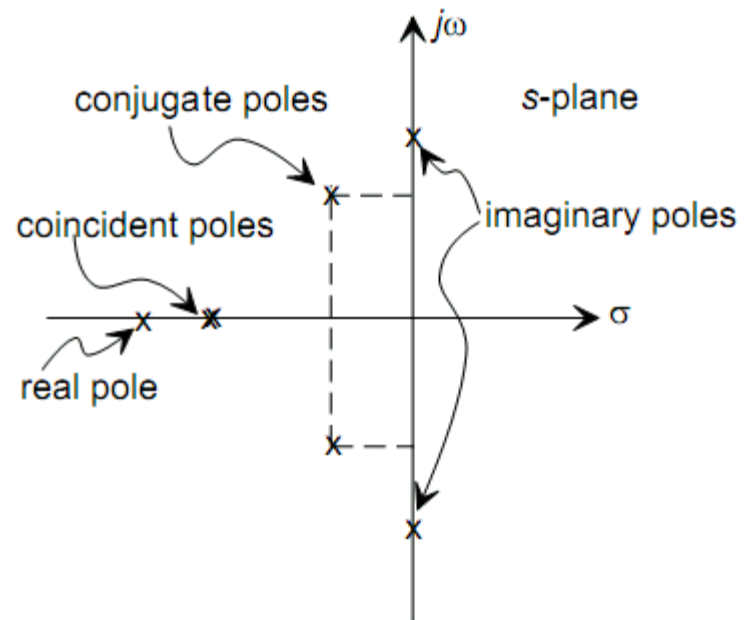
$$= \frac{1}{s} - \frac{(s + \zeta\omega_n) + \left(\zeta / \sqrt{1 - \zeta^2}\right) \omega_n \sqrt{1 - \zeta^2}}{(s + \zeta\omega_n)^2 + \omega_n^2 (1 - \zeta^2)}$$

$$c(t) = 1 - e^{-\zeta\omega_n t} \left(\cos \omega_d t + \frac{\zeta}{\sqrt{1 - \zeta^2}} \sin \omega_d t \right) = 1 - \frac{e^{-\zeta\omega_n t}}{\sqrt{1 - \zeta^2}} \cos(\omega_d t - \phi)$$

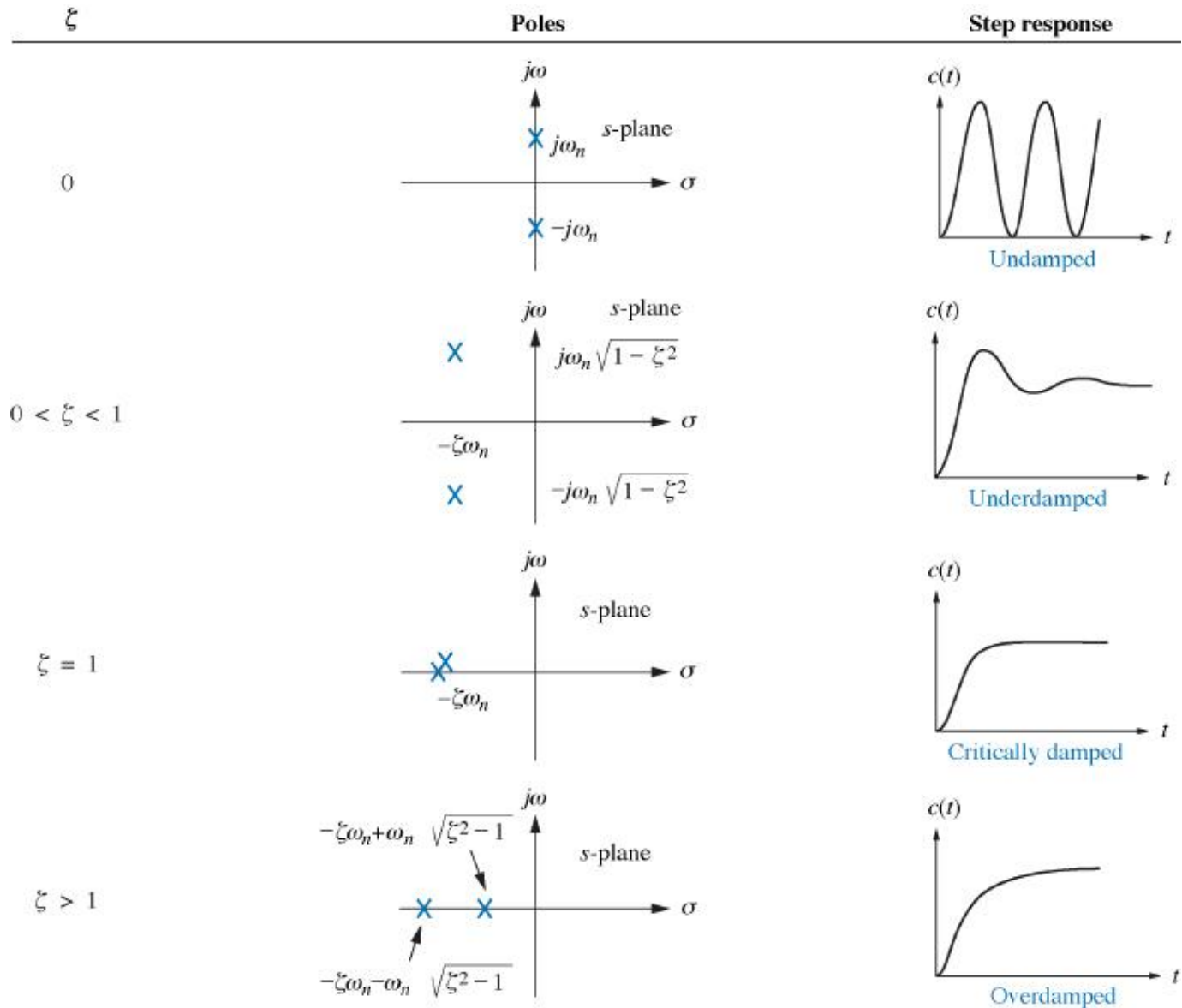
$$\phi = \tan^{-1} \frac{\zeta}{\sqrt{1 - \zeta^2}}$$

Damping ratio and Pole placement

- i) $\zeta > 1$: poles are real and distinct (over damped)
- ii) $\zeta = 1$: poles are real and coincident (critically damped)
- iii) $0 < \zeta < 1$: pole are complex conjugates (under damped)
- iv) $\zeta = 0$: The pole are purely imaginary (undamped)



Step Response of Second-Order Systems



Step Response of Second-Order Systems

1. Over damped Case $p_1, p_2 = -\zeta\omega_n \pm \omega_n\sqrt{\zeta^2 - 1}$

$$y_{step}(t) =$$

2. Critically damped Case $p_1, p_2 = -\zeta\omega_n$

$$y_{step}(t) =$$

3. Under damped Case $p_1, p_2 = -\zeta\omega_n \pm j\omega_n\sqrt{1 - \zeta^2}$

$$y_{step}(t) =$$

4. Undamped Case $p_1, p_2 = \pm j\omega_n$

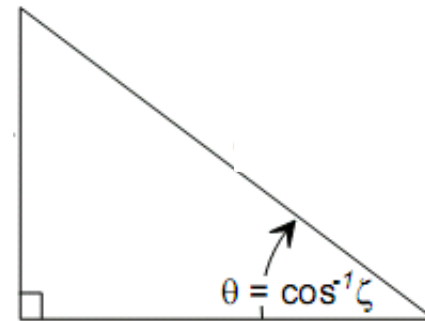
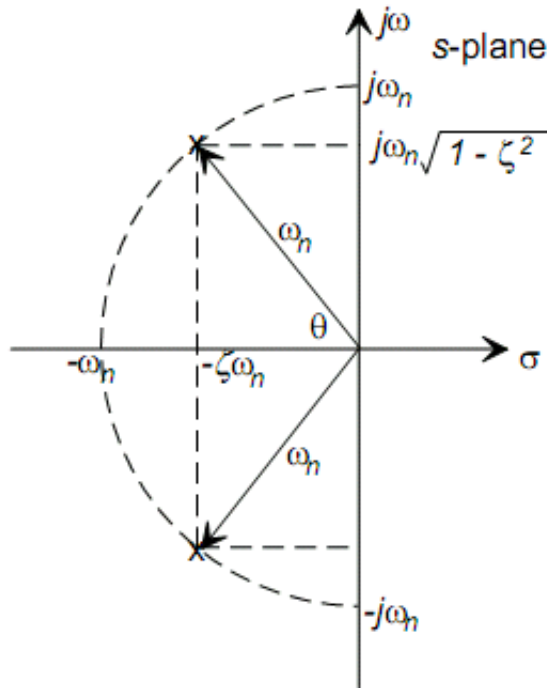
$$y_{step}(t) =$$



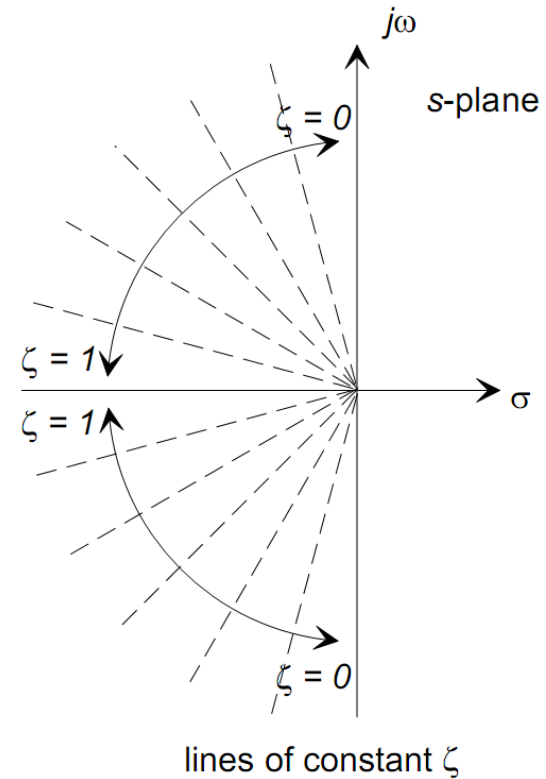
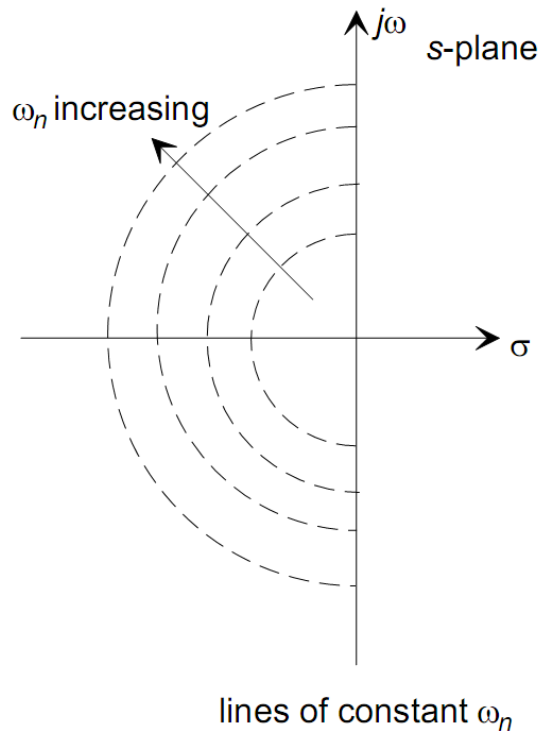
Under-damped Second-Order System

$$p_1, p_2 = -\zeta\omega_n \pm j\omega_n\sqrt{1-\zeta^2}$$

Damped Natural Frequency: $\omega_d = \omega_n\sqrt{1-\zeta^2}$



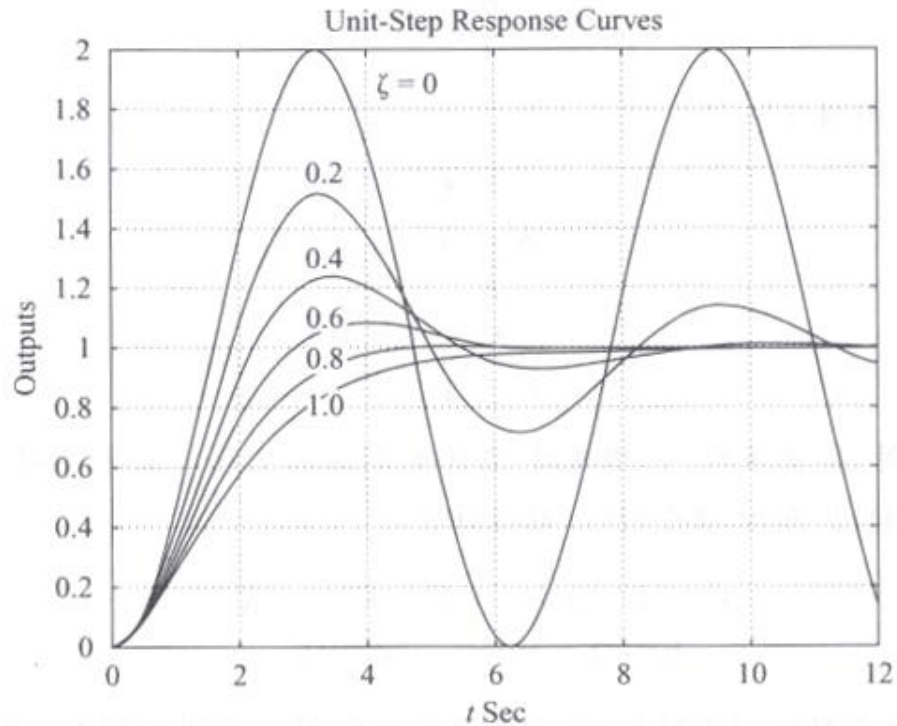
Influence of ω_n and ζ on the pole locations



Influence of ζ

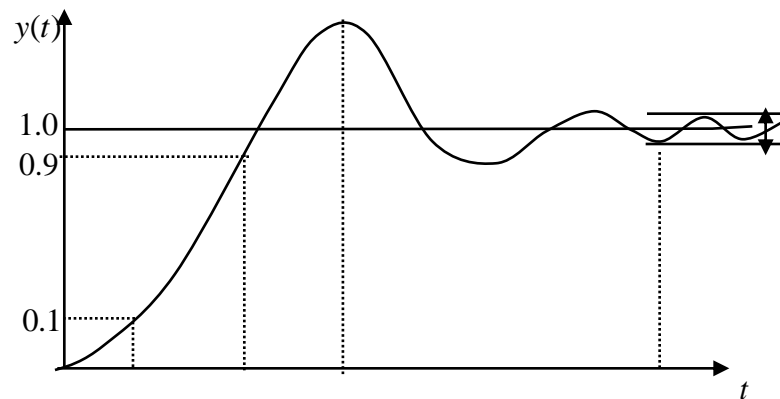
$$y_{step}(t) = 1 - e^{-\zeta\omega_n t} \cos \omega_d t - \frac{\zeta}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin \omega_d t = 1 - \frac{e^{-\zeta\omega_n t}}{\sqrt{1-\zeta^2}} \sin(\omega_d t + \eta)$$

$$\eta = \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta}$$



Step Response Based Second -Order System Specifications

- 1) Peak Time: T_p The time required to reach the first or maximum peak
- 2) Settling Time: T_s The time required for the transients' damped oscillations to reach and stay within $\pm 2\%$ of the steady-state value.
- 3) Rise time : T_r The time required to go from 0.1 to 0.9 of the final value
- 4) Percent Overshoot: % OS The amount that the waveform overshoots the steady-state at the peak time, expressed as a percentage of the steady-state value



Step Response Based Second -Order System Specifications

1) Peak Time T_p

$$sY(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} = \frac{\omega_n^2}{(s + \zeta\omega_n)^2 + \omega_n^2(1 - \zeta^2)} = \frac{\frac{\omega_n}{\sqrt{1 - \zeta^2}} \omega_n \sqrt{1 - \zeta^2}}{(s + \zeta\omega_n)^2 + \omega_n^2(1 - \zeta^2)}$$

$$\dot{y}(t) = \frac{\omega_n}{\sqrt{1 - \zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_n \sqrt{1 - \zeta^2} t)$$

Set $\dot{y}(t) = 0$, $\omega_d t = \pi, 2\pi, \dots$

$$T_p =$$



Step Response Based Second -Order System Specifications

2) % OS M_o

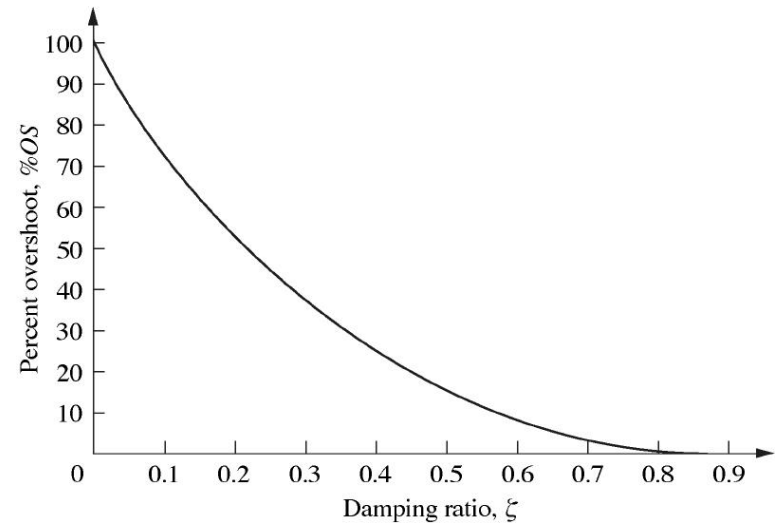
$$\% \text{ OS} = \frac{y(T_p) - y_{\text{steady-state}}}{y_{\text{steady-state}}} \times 100$$

$$M_p = y(T_p) = 1 - \frac{1}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n \cdot \frac{\pi}{\omega_n\sqrt{1-\zeta^2}}} \sin\left(\omega_n\sqrt{1-\zeta^2} \cdot \frac{\pi}{\omega_n\sqrt{1-\zeta^2}} + \phi\right)$$

=

percent overshoot

$$M_o = \frac{M_p - y_s}{y_s} \times 100 =$$



Step Response Based Second -Order System Specifications

3) Settling time :

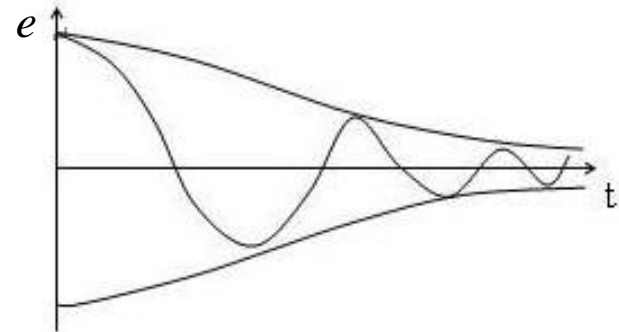
$$e = y - r = \frac{e^{-\zeta\omega_n t}}{\sqrt{1-\zeta^2}} \sin(\omega_d t + \phi)$$

$$\frac{e^{-\zeta\omega_n t}}{\sqrt{1-\zeta^2}} = 0.2$$

$$T_s = \frac{-\ln(0.2\sqrt{1-\zeta^2})}{\zeta\omega_n}$$

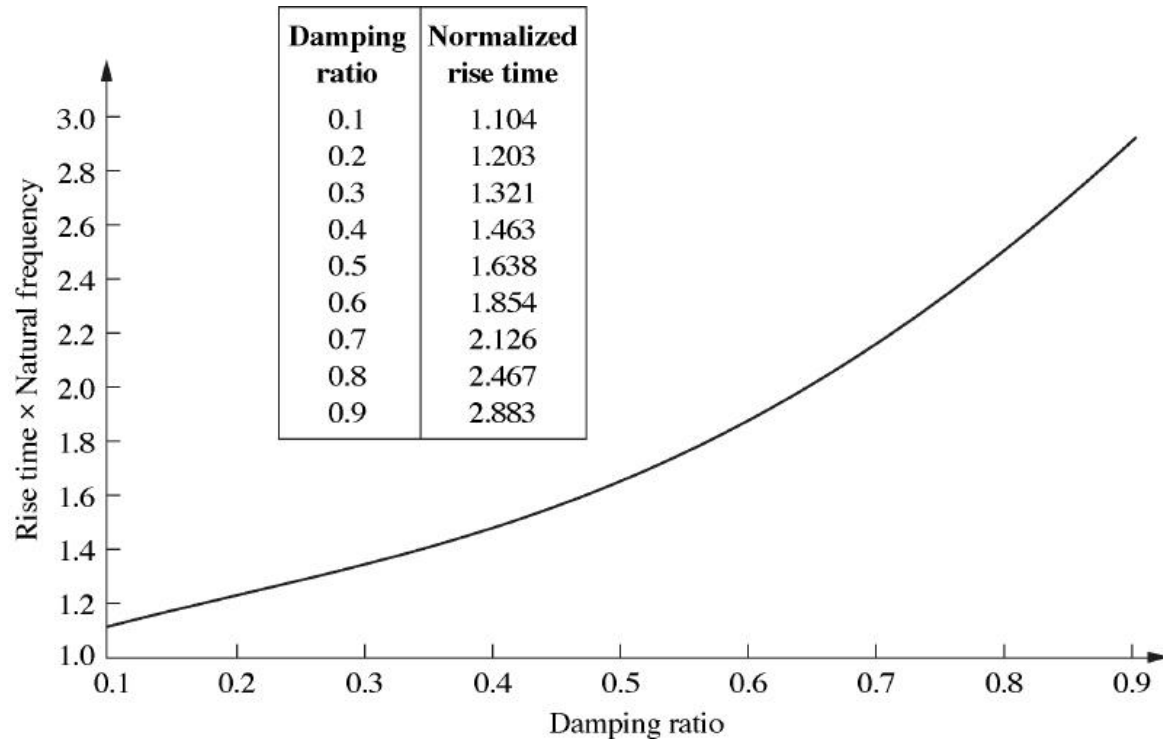
$$2\% \text{ case, } T_s \cong$$

$$5\% \text{ case, } T_s \cong$$



Step Response Based Second -Order System Specifications

4) Rise time :



Experimental Determination of Damping Ratio

$$m\ddot{x} + b\dot{x} + kx = 0, \quad \dot{x}(0) = 0$$

$$\ddot{x} + 2\zeta\omega_n\dot{x} + \omega_n^2x = 0$$

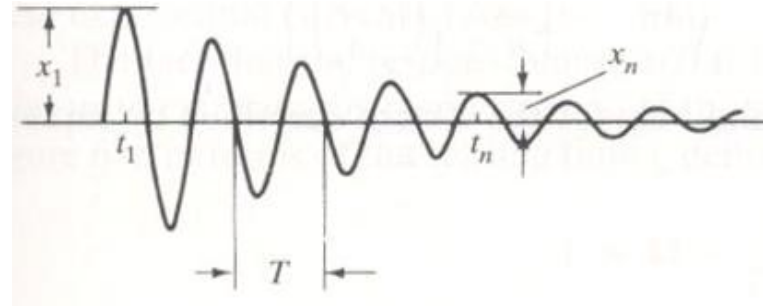
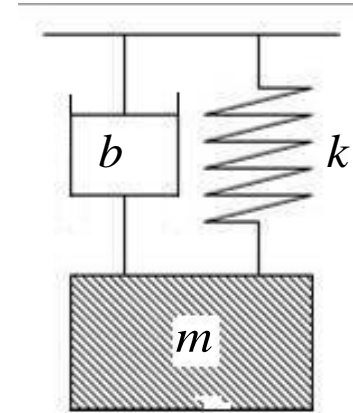
$$\zeta = \frac{1}{2\omega_n} \frac{b}{m} = \frac{b}{2\sqrt{mk}}$$

$$\left[s^2 X(s) - sx(0) - \dot{x}(0) \right] + 2\zeta\omega_n \left[sX(s) - x(0) \right] + \omega_n^2 X(s) = 0$$

$$X(s) = \frac{(s + 2\zeta\omega_n)x(0)}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$x(t) = e^{-\zeta\omega_n t} \left\{ \frac{\zeta}{\sqrt{1-\zeta^2}} x(0) \sin \omega_d t + x(0) \cos \omega_d t \right\} = \frac{x(0)}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \cos \left(\omega_d t - \tan^{-1} \frac{\zeta}{\sqrt{1-\zeta^2}} \right)$$

$$\frac{x_1}{x_n} = \frac{e^{-\zeta\omega_n t_1}}{e^{-\zeta\omega_n (t_1 + (n-1)T)}} =$$



Experimental Determination of Damping Ratio

Logarithmic decrement

$$\ln \frac{x_1}{x_2} = \zeta \omega_n T = \zeta \omega_n \cdot \frac{2\pi}{\omega_d} =$$

$$\ln \frac{x_1}{x_n} = (n-1)\zeta \omega_n T$$

$$\Rightarrow \zeta = \frac{\frac{1}{n-1} \left(\ln \frac{x_1}{x_n} \right)}{\sqrt{4\pi^2 + \left\{ \frac{1}{n-1} \left(\ln \frac{x_1}{x_n} \right) \right\}^2}}$$



Estimate of Response Time

$$x(t) = \frac{x(0)}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \cos\left(\omega_d t - \tan^{-1} \frac{\zeta}{\sqrt{1-\zeta^2}}\right)$$

$$\omega_d = \omega_n \sqrt{1-\zeta^2}, \quad t = \frac{1}{\zeta\omega_n}, \quad \omega_d t = \frac{\sqrt{1-\zeta^2}}{\zeta}$$

$$\tan^{-1} \frac{\zeta}{\sqrt{1-\zeta^2}} = \frac{\pi}{2} - \eta$$

