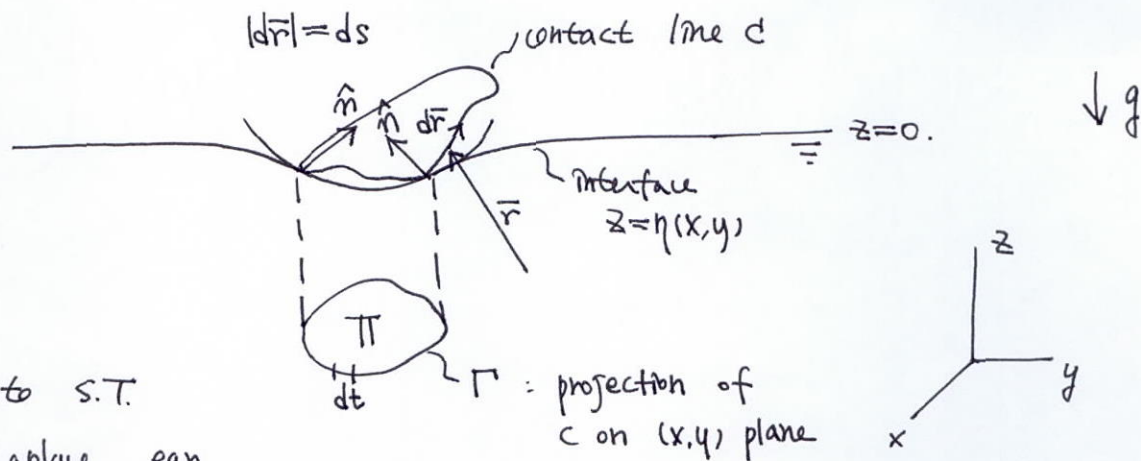


Force on a partly submerged body



* Force due to S.T.

Young-Laplace eqn.

$$p_i - p_o = -\sigma \kappa$$

$$p_i = p_o + \rho g \eta$$

$$\rho g \eta(x, y) = -\sigma \nabla \cdot \hat{n} \quad \dots (1)$$

curvature $\kappa = -\nabla \cdot \hat{n}$

\hat{n} : unit normal vector to the interface pointing out of the liquid.

$$\begin{aligned} \hat{n}(x, y) &= n_1 \hat{i} + n_2 \hat{j} + n_3 \hat{k} \\ &= \frac{1}{\sqrt{1 + \eta_x^2 + \eta_y^2}} (-\eta_x \hat{i} - \eta_y \hat{j} + \hat{k}) \end{aligned}$$

$$\begin{aligned} \nabla \cdot \hat{n} &= \left(\frac{\partial}{\partial x} \hat{i} + \frac{\partial}{\partial y} \hat{j} + \frac{\partial}{\partial z} \hat{k} \right) \cdot \hat{n} \\ &= \frac{\partial n_1}{\partial x} + \frac{\partial n_2}{\partial y} \end{aligned}$$

Integrating (1) over (x, y) plane outside Γ ,

$$-\int_{R^2/\Gamma} \sigma \nabla \cdot \hat{n} dA = \int_{R^2/\Gamma} \rho g \eta dA$$

$$\text{LHS} = \sigma \int_{\Gamma} \hat{\nu} \cdot \hat{n} \, dt \quad \text{by divergence theorem}$$

$$\begin{aligned} \text{RHS} &= \rho g \cdot (\text{volume between the interface } z=\eta \text{ and the surface } z=0) \\ &= W_M \quad (\text{weight of the liquid displaced by the meniscus}) \end{aligned}$$

We will show that $\text{LHS} = \hat{k} \cdot \bar{F}_{st}$: vertical component of surface tension force

• surface tension force on $ds \perp d\vec{r}, \perp \hat{n}$

$$d\bar{F}_{st} = \sigma \, d\vec{r} \times \hat{n}$$

$$\bar{F}_{st} = \sigma \int_c \frac{d\vec{r}}{ds} \times \hat{n} \, ds$$

$$= \sigma \int_c \dot{\vec{r}}(s) \times \hat{n} \, ds. \quad \dot{\vec{r}} = \frac{d\vec{r}}{ds}$$

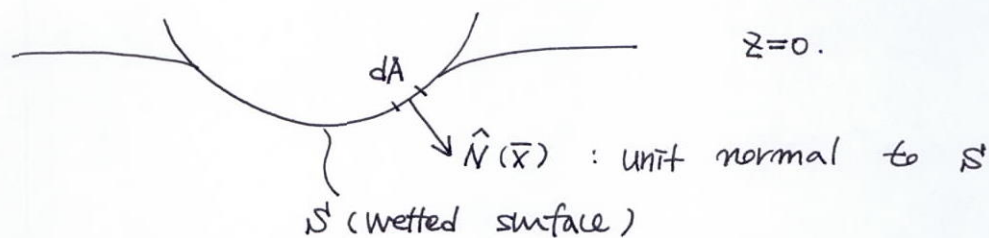
$$\begin{aligned} \hat{k} \cdot \bar{F}_{st} &= \sigma \int_c \hat{k} \cdot \dot{\vec{r}} \times \hat{n} \, ds = \sigma \int_c \hat{n} \cdot \underbrace{\hat{k} \times \dot{\vec{r}}}_{\substack{\text{vector} \\ \text{identity}}} \, ds \\ &= \hat{k} \times d\vec{r} \\ &= \hat{\nu} \, dt \end{aligned}$$



$\hat{\nu}$: inward normal vector of Γ

$$\therefore \hat{k} \cdot \bar{F}_{st} = \sigma \int_{\Gamma} \hat{n} \cdot \hat{\nu} \, dt = W_M.$$

* Force due to hydrostatic pressure



$$\bar{F}_p = \int_S \rho g z \hat{N}(\bar{x}) dA$$

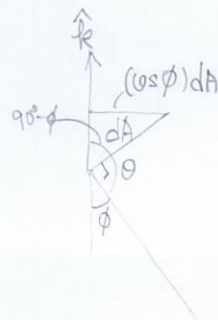
vertical component of \bar{F}_p

$$\hat{k} \cdot \bar{F}_p = \rho g \int_S z \underbrace{\hat{k} \cdot \hat{N}(\bar{x})}_{= \pm dx dy} dA$$

for depressed interface $\hat{N}(\bar{x}) \searrow$: $\hat{k} \cdot \hat{N} dA = -dx dy$

$$\hat{k} \cdot \bar{F}_p = -\rho g \int_{\Pi} z(x,y) dx dy$$

= W_v (weight of liquid in the vertical cylinder through d , which intersects the horizontal plane in the curve Γ)



$$\hat{k} \cdot \hat{N} dA = -\cos \phi dA$$

* Total vertical force on the body

$$\hat{k} \cdot (\bar{F}_{se} + \bar{F}_p) = W_u + W_v$$

: weight of liquid that would fill the region bounded below by the interface and the wetted surface of the body, and above by the undisturbed interface $z=0$.

