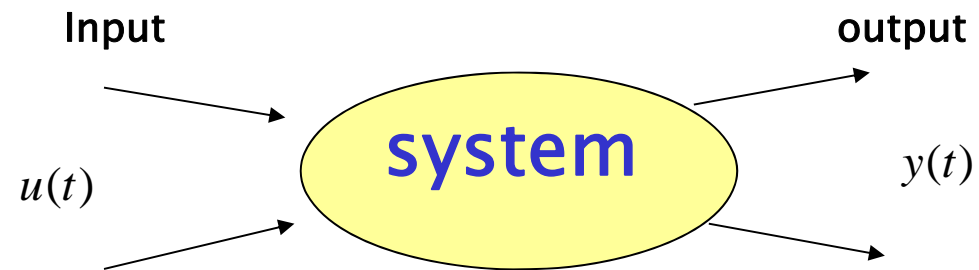


Lecture 12

Control Systems

System, Input and Output



Output depends on inputs !

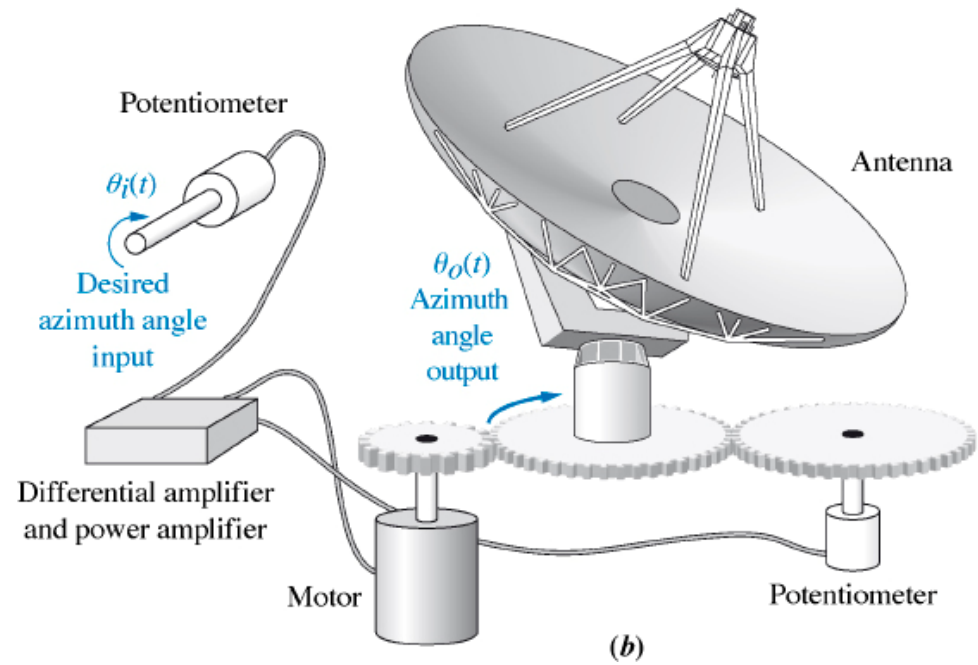
Control systems

Open loop control

Closed loop control

Closed loop transfer function

Antenna azimuth position control system



a DC Servomotor System

servo-motor system

R_a : armature resistance, Ω

i_a : armature current, A

i_f : field current, A

e_a : applied armature voltage, V

e_b : back emf, V

θ_1 : angular displacement of the motor shaft, rad

θ_2 : angular displacement of the load shaft, rad

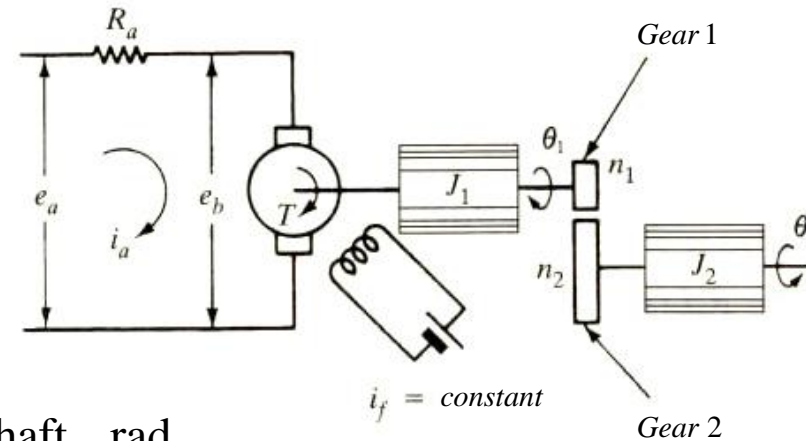
T : torque developed by the motor, N-m

J_1 : equivalent moment of inertia of the motor, kg-m^2

J_2 : equivalent moment of inertia of the load, kg-m^2

The torque of motor : $T = K i_a$

For a constant flux, the induced voltage : $e_b = K_b \frac{d\theta}{dt}$ K_b : back emf constant



DC Servomotor System

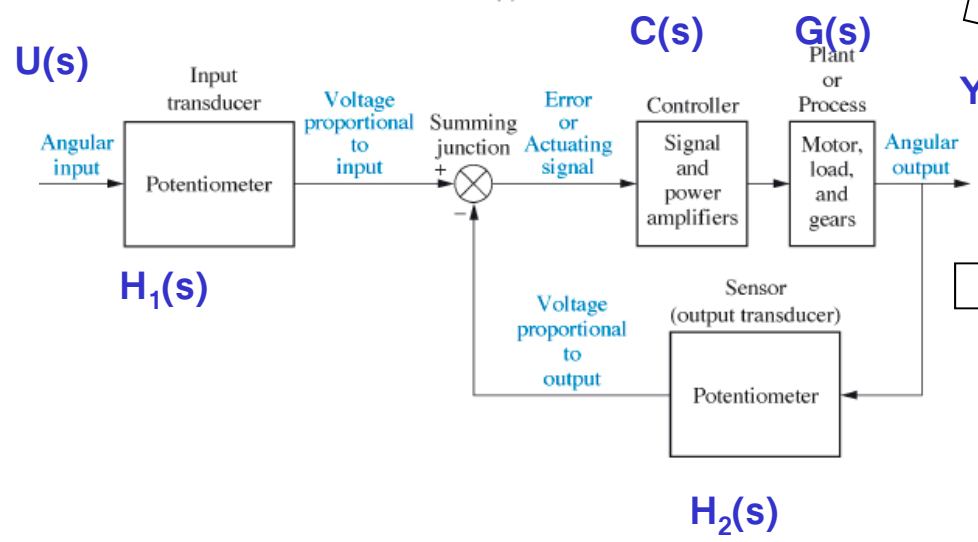
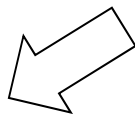
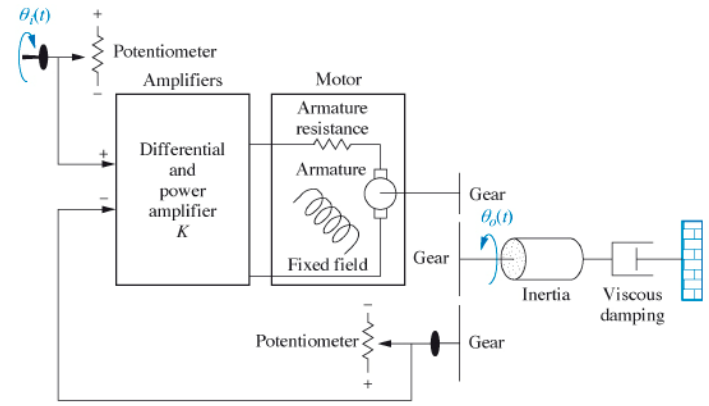
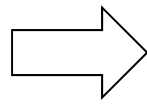
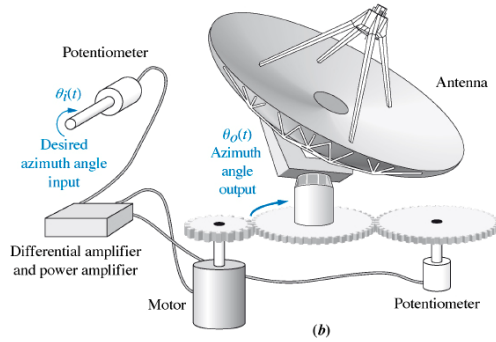
Armature circuit D.E : $R_a i_a + e_b = e_a$ Inertia and friction : $J_{1eq} = J_1 + \left(\frac{n_1}{n_2}\right)^2 J_2$

Laplace transforms of these equations :

$$K_b s \Theta(s) = E_b(s), \quad (L_a s + R_a) I_a(s) + E_b(s) = E_a(s), \quad (J s^2 + b s) \Theta(s) = T(s) = K I_a(s)$$

$$\begin{aligned} T.F = \frac{\Theta(s)}{E_a(s)} &= \frac{K}{s(R_a J s + R_a b + K K_b)} = \frac{\frac{K}{R_a J}}{s \left(s + \frac{R_a b + K K_b}{R_a J} \right)} \\ &= \frac{K_m}{s(T_m s + 1)} \end{aligned}$$

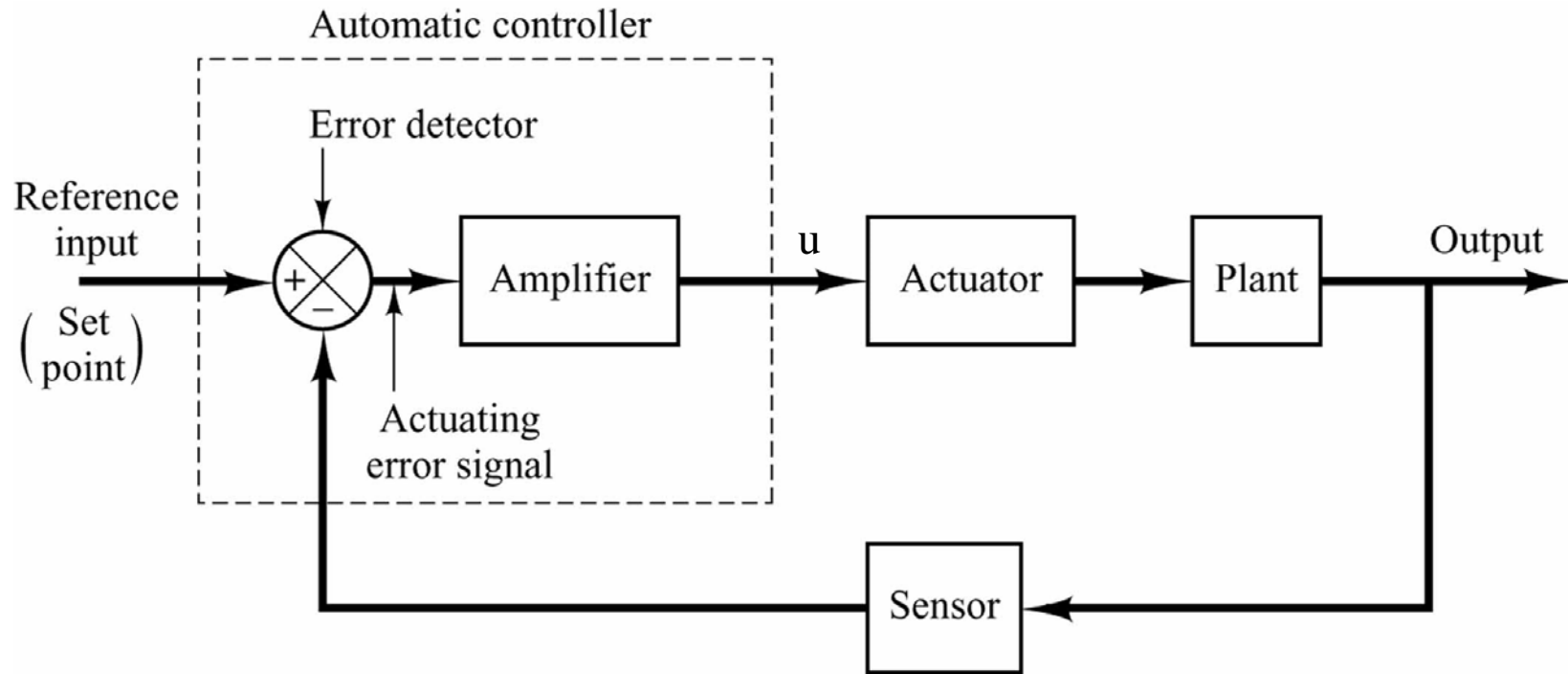
System → Schematic → Block Diagram → Transfer Functions



$$Y(s) = \frac{C(s)G(s)H_1(s)}{1 + C(s)G(s)H_2(s)} U(s)$$

Antenna azimuth position control system

Control Systems



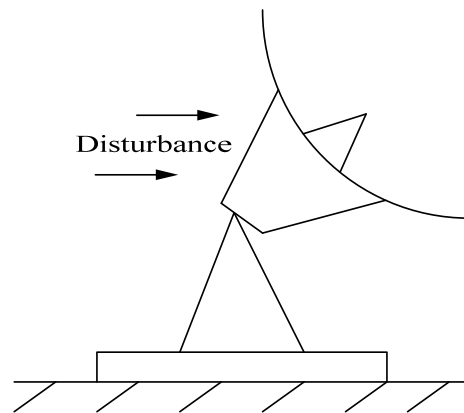
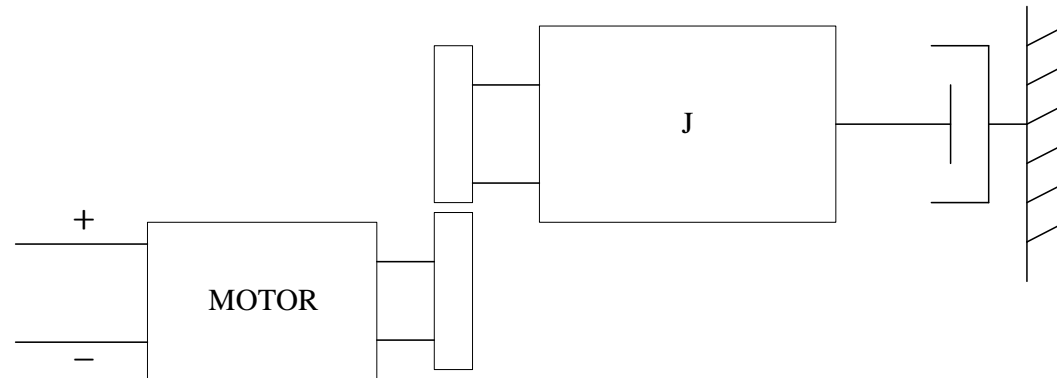
$$u=f(r,c)$$

PID controllers

Proportional Control

$$U = k_p e$$

k_p : The proportional Gain



Antenna Position Control System

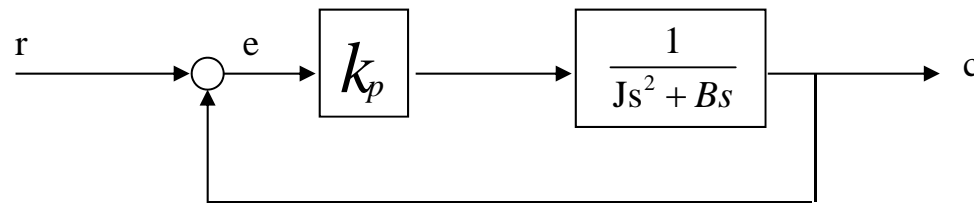
controllers

Proportional Control

=

$$U = k_p e$$

k_p : The proportional Gain



$$\frac{c(s)}{r(s)} = \frac{\frac{K_p}{Js^2 + Bs}}{1 + \frac{K_p}{Js^2 + Bs}} = \frac{K_p}{Js^2 + Bs + K_p}$$

controllers

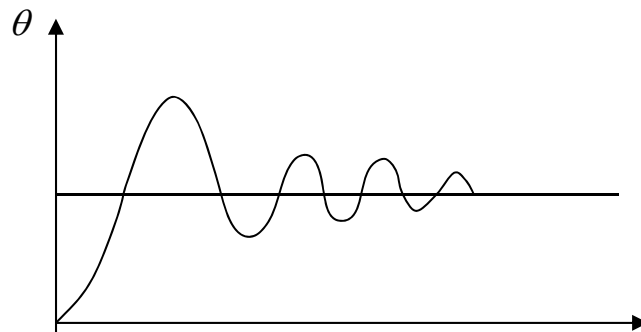
Proportional Control

=

- Step input response $r(s) = \frac{1}{s}$

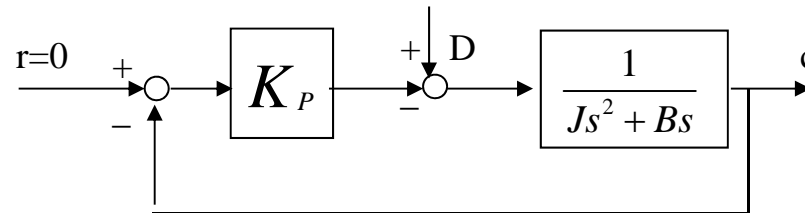
- steady state response $c(t) = \lim_{s \rightarrow 0} sC(s)$

$$= \lim_{s \rightarrow 0} s \frac{K_p}{Js^2 + Bs + K_p} \frac{1}{s} = 1$$



Proportional Control

- Response to torque disturbance



$$C(s) = \frac{K_p}{Js^2 + Bs + K_p} r(s) + \frac{1}{Js^2 + Bs + K_p} D(s)$$

Assume that $D(s) = \frac{1}{s}$ steady state response

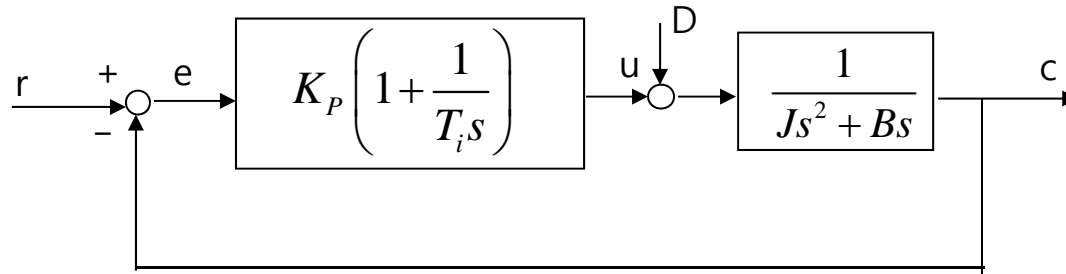
$$c(t) = \lim_{s \rightarrow \infty} s \cdot \frac{1}{Js^2 + Bs + K_p} \cdot \frac{1}{s} = \frac{1}{K_p} : \text{steady-state error}$$

- Large K_p → small steady-state error
- large motor power is needed
- oscillations
- large w_n $\left(w_n = \sqrt{\frac{K_p}{J}} \right)$

→ small damping ratio $\zeta = \frac{B}{2\sqrt{JK_p}}$

Proportional-Integral Control (PI control)

- Response to torque disturbance



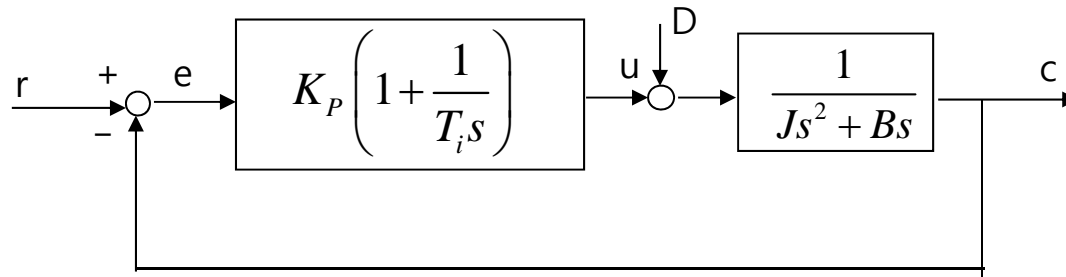
$$\frac{U(s)}{E(s)} = K_P \left(1 + \frac{1}{T_i s} \right)$$

$$u(t) = K_p e(t) + \frac{K_P}{T_i} \int_0^t e(t) dt$$

$$\frac{C(s)}{R(s)} = \frac{K_P s + \frac{1}{T_i} K_P}{J s^3 + B s^2 + K_P s + \frac{K_P}{T_i}}$$

No steady-state error for reference input

Proportional–Integral Control (PI control)



$$\frac{C(s)}{D(s)} = \frac{\frac{1}{Js^2 + Bs}}{1 + K_P \left(1 + \frac{1}{T_i s}\right) \frac{1}{Js^2 + Bs}}$$

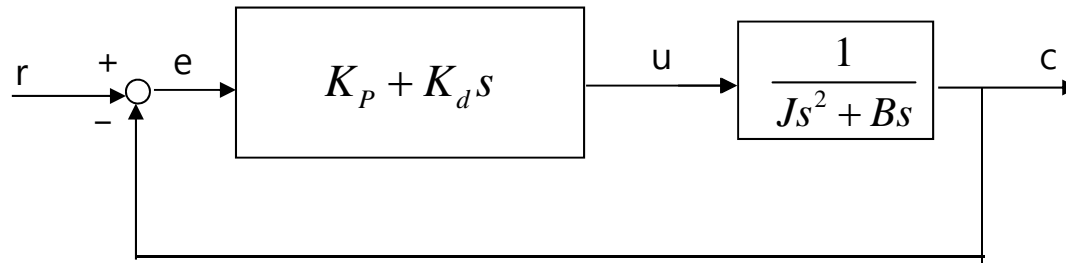
$$= \frac{s}{Js^3 + Bs^2 + K_P s + \frac{K_P}{T_i}}$$

$$D(s) = \frac{1}{s}$$

$$C(t) = \lim_{s \rightarrow 0} s \frac{s}{Js^3 + Bs^2 + K_P s + \frac{K_P}{T_i}} \frac{1}{s} = 0$$

No steady-state error for step disturbance

Proportional-Derivative Control (PD control)



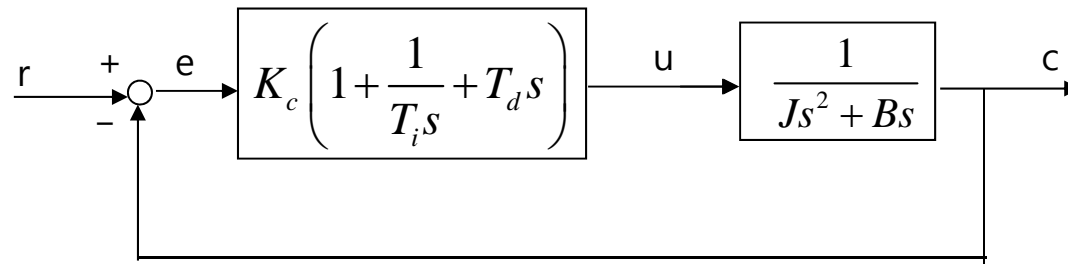
$$u(t) = K_P e(t) + K_d \frac{d}{dt} e(t)$$

$$\frac{C(s)}{R(s)} = \frac{K_P + K_d s}{J s^2 + (B + K_d) s + K_P}$$

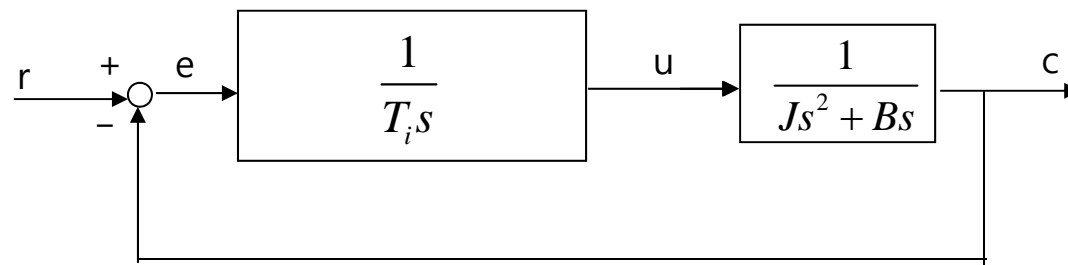
$$\zeta = \frac{B + K_d}{2\sqrt{JK_P}} : \text{increased effective damping}$$

Classifications of controllers

Proportional-Integral-Derivative Control (PID control)

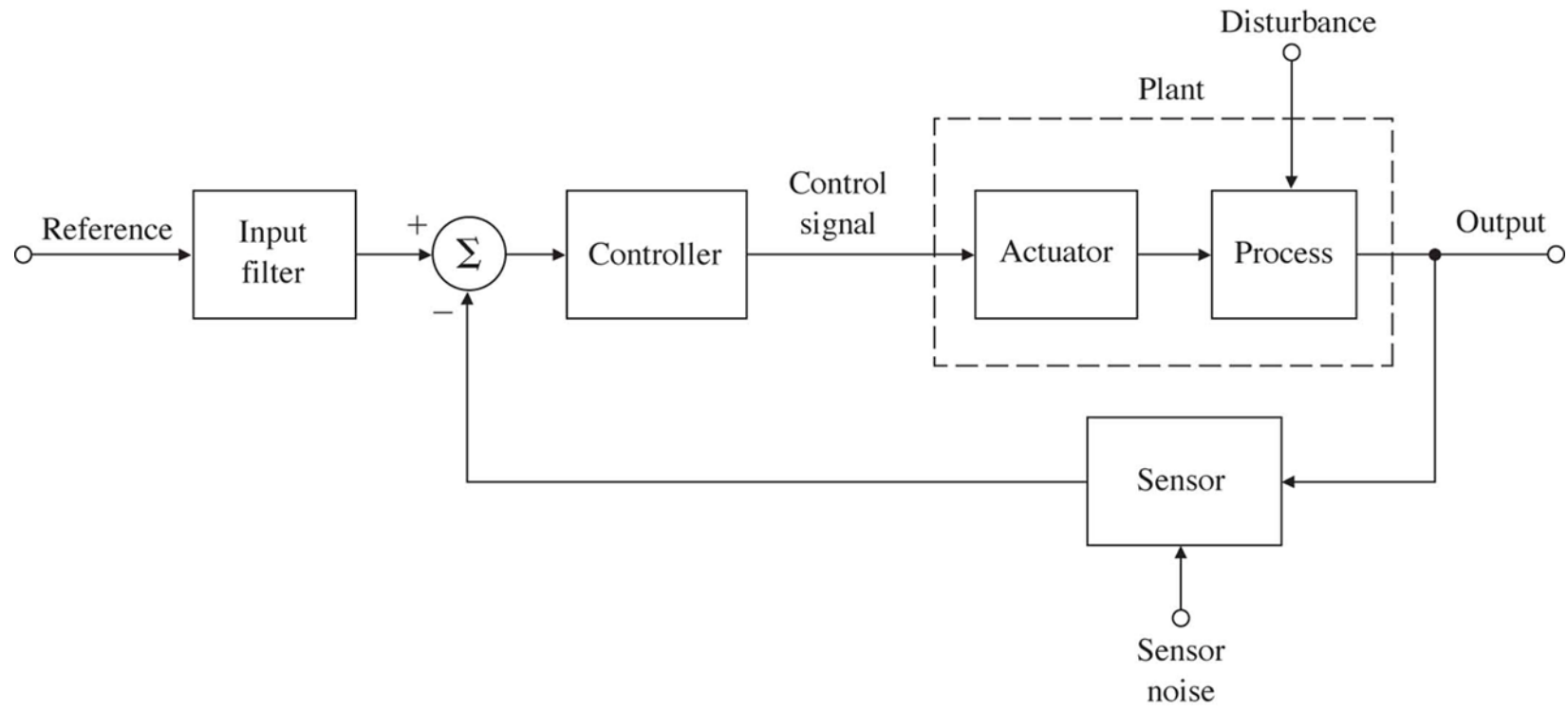


I control



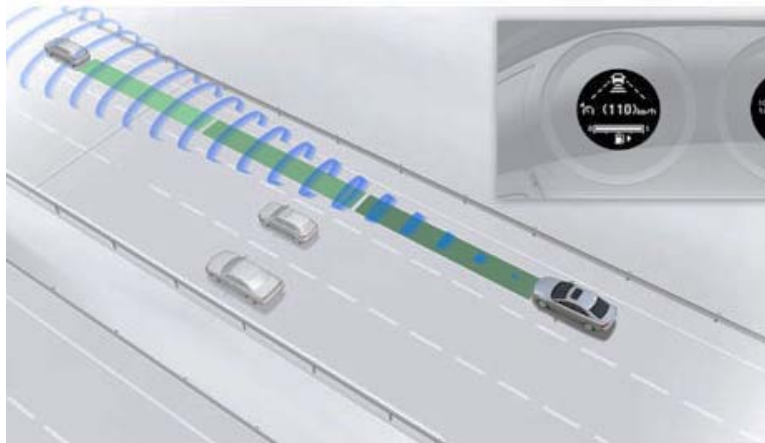
$$\frac{C(s)}{R(s)} = \frac{\frac{1}{T_i s} \frac{1}{Js^2 + Bs}}{1 + \frac{1}{T_i s} \frac{1}{Js^2 + Bs}} = \frac{\frac{1}{T_i}}{Js^3 + Bs^2 + \frac{1}{T_i}}$$

Component block diagram of an elementary feedback control



Vehicle control systems

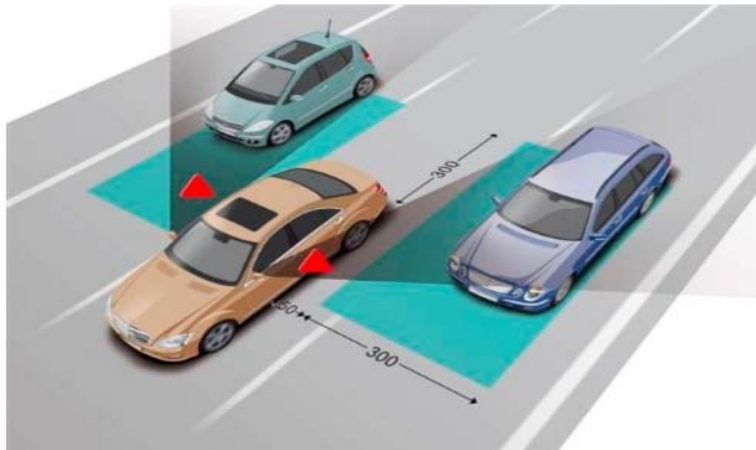
Driver Assistance Systems



Smart Cruise Control System



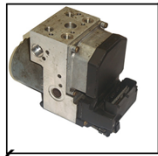
Lane Keeping Assist System



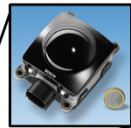
Blind Spot Detection System

Hyundai Motor Company, GENESIS, in 2008

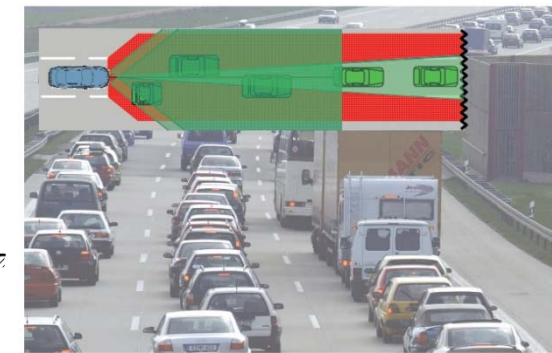
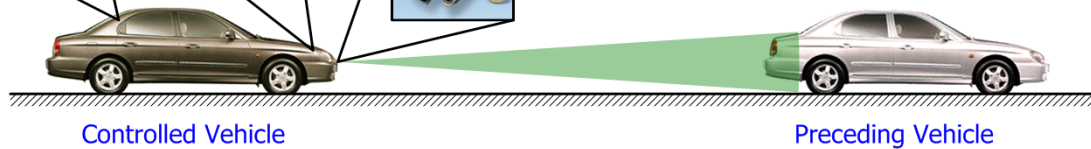
Acceleration Control
(Engine Control)



Deceleration
(ABS,ESC,ESC+)



Lidar / Radar
(Detecting Preceding Vehicle)



Vehicle Control systems

Cruise Control

Automated Emergency Braking Systems

System Control (next semester)

Controller design

- feedback control systems (closed-loop control systems)
 - Root Locus method: pole placement
 - Frequency Response method : lead/lag compensators
 - analysis and design Using MATLAB
 - control system simulation using MATLAB/SIMULINK
-
- PID Control
 - state space method

End of Introduction to Control Systems