

Chap 8. Second-order Plastic Hinge Analysis

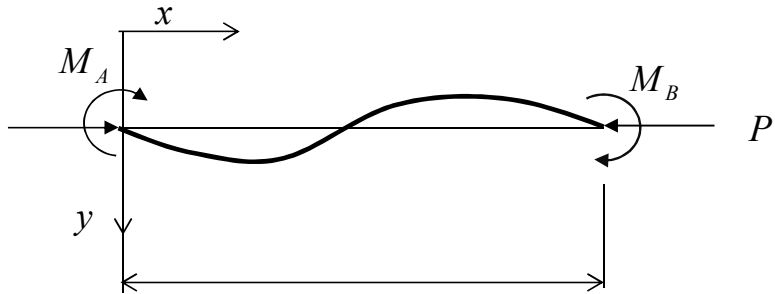
8.1 INTRODUCTION

- Plastic Zone and Plastic Hinge
- Fiber Analysis
- Residual stresses, geometric imperfections, material strain hardening
- Zero length of plastic hinge
- Equilibrium based on deformed structural geometry

8.2 Modeling of Elastic Beam-Column Element

- 8.2.1 Assumptions
 - initially straight and prismatic
 - compact section
 - sufficiently braced
 - Small member deformation but large rigid body displacement
 - Axial shortening due to member curvature bending is neglected

8.2.2 Stability Functions



$$EIy'' + Py = \frac{M_A + M_B}{L}x - M_A$$

$$\rho = L\sqrt{\frac{P}{EI}}$$

$$y = A \sin \rho x + B \cos \rho x + \frac{M_A + M_B}{EIL\rho^2}x - \frac{M_A}{EI\rho^2}$$

Since $y(0) = y(L) = 0$

$$A = -\frac{1}{EI\rho^2 \sin \rho L} (M_A \cos \rho L + M_B)$$

$$B = \frac{M_A}{EI\rho^2}$$

$$y = -\frac{1}{EI\rho^2} \left[\frac{\cos \rho L}{\sin \rho L} \sin \rho x - \cos \rho x - \frac{x}{L} + 1 \right] M_A$$

$$-\frac{1}{EI\rho^2} \left[\frac{1}{\sin \rho L} \sin \rho x - \frac{x}{L} \right] M_B$$

$$\frac{dy}{dx} = -\frac{1}{EI\rho} \left[\frac{\cos \rho L}{\sin \rho L} \cos \rho x + \sin \rho x - \frac{1}{\rho L} \right] M_A$$

$$-\frac{1}{EI\rho} \left[\frac{1}{\sin \rho L} \cos \rho x - \frac{1}{\rho L} \right] M_B$$

$$\begin{cases} \theta_A = \frac{dy}{dx}(0) \\ \theta_B = \frac{dy}{dx}(L) \end{cases}$$

$$P = \frac{EA}{L}e$$

$$\begin{Bmatrix} M_A \\ M_B \\ P \end{Bmatrix} = \frac{EI}{L} \begin{bmatrix} S_1 & S_2 & 0 \\ S_2 & S_1 & 0 \\ 0 & 0 & A/I \end{bmatrix} \begin{Bmatrix} \theta_A \\ \theta_B \\ e \end{Bmatrix}$$

$$S_1 = \begin{cases} \frac{\rho \sin \rho - \rho^2 \cos \rho}{2 - 2 \cos \rho - \rho \sin \rho} & \text{for } P < 0 \\ \frac{\rho^2 \cosh \rho - \rho \sinh \rho}{2 - 2 \cosh \rho} & \text{for } P > 0 \end{cases}$$

$$S_2 = \begin{cases} \frac{\rho^2 - \rho \sin \rho}{2 - 2 \cos \rho - \rho \sin \rho} & \text{for } P < 0 \\ \frac{\rho \sinh \rho - \rho^2}{2 - 2 \cosh \rho + \rho \sinh \rho} & \text{for } P > 0 \end{cases}$$

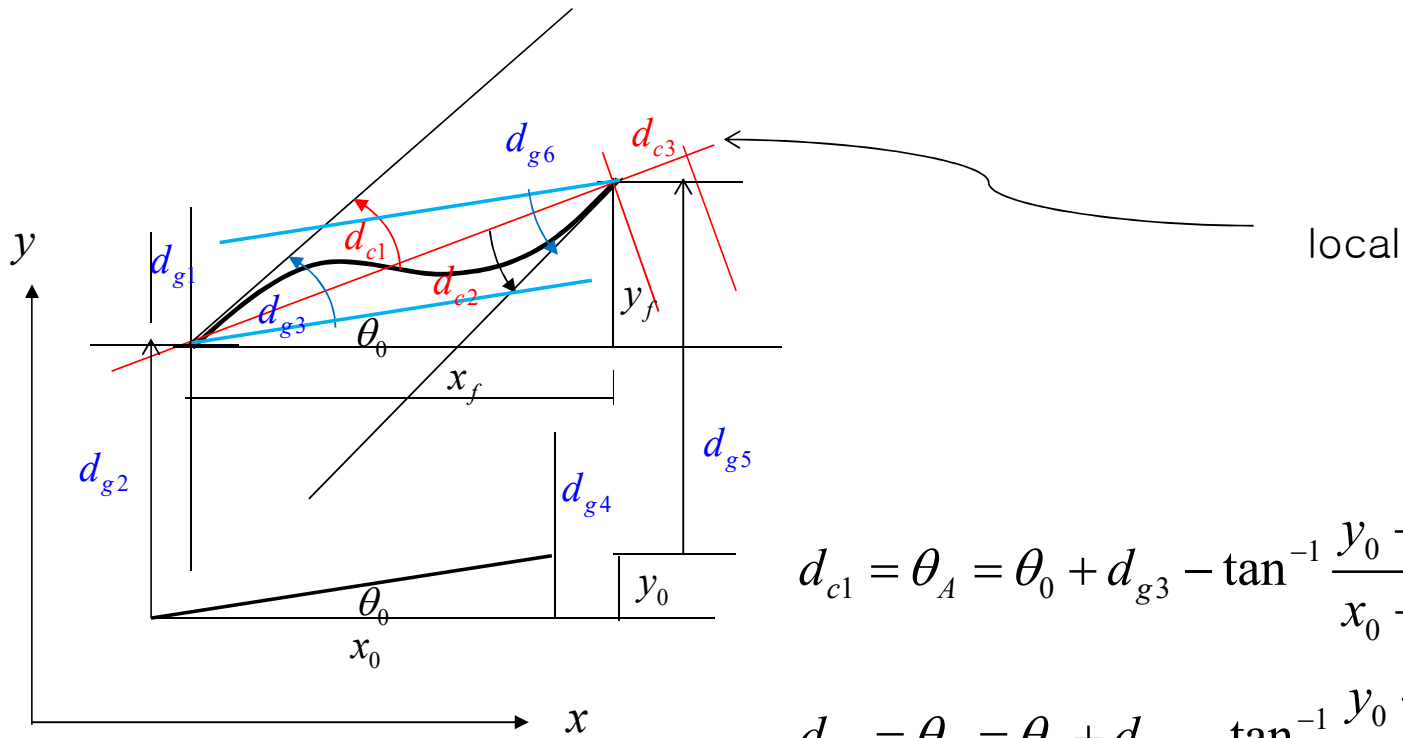
8.2.3 Tangent Stiffness Relationship

Element force-displacement equation

$$\begin{Bmatrix} \dot{M}_A \\ \dot{M}_B \\ \dot{P} \end{Bmatrix} = \frac{EI}{L} \begin{bmatrix} S_1 & S_2 & 0 \\ S_2 & S_1 & 0 \\ 0 & 0 & A/I \end{bmatrix} \begin{Bmatrix} \dot{\theta}_A \\ \dot{\theta}_B \\ \dot{e} \end{Bmatrix}$$

Vector notation

$$\mathbf{\dot{f}}_c = \mathbf{k}_c \mathbf{\dot{d}}_c$$



$$d_{c1} = \theta_A = \theta_0 + d_{g3} - \tan^{-1} \frac{y_0 + d_{g5} - d_{g2}}{x_0 + d_{g4} - d_{g1}}$$

$$d_{c2} = \theta_B = \theta_0 + d_{g6} - \tan^{-1} \frac{y_0 + d_{g5} - d_{g2}}{x_0 + d_{g4} - d_{g1}}$$

$$d_{c3} = \frac{(2x_0 + d_{g4} - d_{g1})(d_{g4} - d_{g1}) + (2y_0 + d_{g5} - d_{g2})(d_{g5} - d_{g2})}{L_f + L}$$

Global and local displacement Relationship

$$\begin{Bmatrix} \dot{\theta}_A \\ \dot{\theta}_B \\ \dot{e} \end{Bmatrix} = \begin{bmatrix} -\sin \theta / L & \cos \theta / L & 1 & \sin \theta / L & -\cos \theta / L & 0 \\ -\sin \theta / L & \cos \theta / L & 0 & \sin \theta / L & -\cos \theta / L & 1 \\ -\cos \theta & -\sin \theta & 0 & \cos \theta & \sin \theta & 0 \end{bmatrix} \begin{Bmatrix} \dot{d}_{g1} \\ \dot{d}_{g2} \\ \dot{d}_{g3} \\ \dot{d}_{g4} \\ \dot{d}_{g5} \\ \dot{d}_{g6} \end{Bmatrix}$$

$$\dot{\mathbf{d}}_c = \mathbf{T}_{cg} \dot{\mathbf{d}}_g$$

Relationship between global and local displacement

$$\begin{Bmatrix} f_{g1} \\ f_{g2} \\ f_{g3} \\ f_{g4} \\ f_{g5} \\ f_{g6} \end{Bmatrix} = \begin{bmatrix} -\sin \theta / L & -\sin \theta / L & -\cos \theta \\ \cos \theta / L & \cos \theta / L & -\sin \theta \\ 1 & 0 & 0 \\ \sin \theta / L & \sin \theta / L & \cos \theta \\ -\cos \theta / L & -\cos \theta / L & \sin \theta \\ 0 & 1 & 0 \end{bmatrix} \begin{Bmatrix} M_A \\ M_B \\ P \end{Bmatrix}$$

$$\mathbf{f}_g = \mathbf{T}_{cg}^T \mathbf{f}_c$$

Taking derivative

$$\dot{\mathbf{f}}_g = \mathbf{T}_{cg}^T \dot{\mathbf{f}}_c + \dot{\mathbf{T}}_{cg}^T \mathbf{f}_c$$

$$\dot{\mathbf{f}}_c = \mathbf{k}_c \dot{\mathbf{d}}_c$$

$$\dot{\mathbf{f}}_g = \mathbf{T}_{cg}^T \dot{\mathbf{f}}_c + \dot{\mathbf{T}}_{cg}^T \mathbf{f}_c$$

$$\dot{\mathbf{f}}_g = \mathbf{T}_{cg}^T \mathbf{k}_c \mathbf{T}_{cg} \dot{\mathbf{d}}_g + \dot{\mathbf{T}}_{cg}^T \mathbf{f}_c$$

$$\dot{\mathbf{T}}_{cg}^T = \left[\frac{\partial \dot{\mathbf{T}}_{cg}^T}{\partial d_{gk}} \right] \dot{\mathbf{d}}_{gk}, k = 1, 2, \dots, 6.$$

$$\dot{\mathbf{T}}_{cg}^T = \left[\frac{\partial^2 \mathbf{d}_{ci}}{\partial d_{gj} \partial d_{gk}} \right]^T \dot{\mathbf{d}}_{gk}, i = 1, 2, \dots, 6. \quad j = 1, 2, \dots, 6 \quad k = 1, 2, \dots, 6$$

$$\dot{\mathbf{T}}_{cg}^T = \left[\frac{\partial^2 \mathbf{d}_{c1}}{\partial d_{gj} \partial d_{gk}} \mid \frac{\partial^2 \mathbf{d}_{c2}}{\partial d_{gj} \partial d_{gk}} \mid \frac{\partial^2 \mathbf{d}_{c3}}{\partial d_{gj} \partial d_{gk}} \right]^T \dot{\mathbf{d}}_{gk} = [\mathbf{T}_1 \mid \mathbf{T}_2 \mid \mathbf{T}_3] \dot{\mathbf{d}}_{gk}$$

$$\mathbf{T}_1 = \mathbf{T}_2 = \frac{1}{L^2} \begin{bmatrix} -2sc & c^2 - s^2 & 0 & 2sc & -(c^2 - s^2) & 0 \\ & 2sc & 0 & -(c^2 - s^2) & -2sc & 0 \\ & & 0 & 0 & 0 & 0 \\ & & & -2sc & c^2 - s^2 & 0 \\ & \text{Sym} & & & 2sc & 0 \\ & & & & & 0 \end{bmatrix}$$

$$\mathbf{T}_3 = \frac{1}{L^2} \begin{bmatrix} s^2 & -sc & 0 & -s^2 & sc & 0 \\ & c^2 & 0 & sc & -c^2 & 0 \\ & & 0 & 0 & 0 & 0 \\ & & & s^2 & -sc & 0 \\ & \text{Sym} & & & c^2 & 0 \\ & & & & & 0 \end{bmatrix}$$

$$\dot{\mathbf{f}}_g = \mathbf{T}_{cg}^T \mathbf{k}_c \mathbf{T}_{cg} \dot{\mathbf{d}}_g + \dot{\mathbf{T}}_{cg}^T \mathbf{f}_c$$

$$\dot{\mathbf{f}}_g = \left(\mathbf{T}_{cg}^T \mathbf{k}_c \mathbf{T}_{cg} + \mathbf{T}_1 M_A + \mathbf{T}_2 M_B + \mathbf{T}_3 P \right) \dot{\mathbf{d}}_g$$

$$\dot{\mathbf{f}}_g = \mathbf{k}_g \dot{\mathbf{d}}_g$$

8.3 Modeling of Truss Elements

$$\dot{\mathbf{f}}_g = \left(\mathbf{T}_{cg}^T \mathbf{k}_c \mathbf{T}_{cg} + \cancel{\mathbf{T}_1 M_A} + \cancel{\mathbf{T}_2 M_B} + \mathbf{T}_3 P \right) \dot{\mathbf{d}}_g$$

$$\dot{\mathbf{f}}_g = \left(\mathbf{T}_{cg}^T \mathbf{k}_c \mathbf{T}_{cg} + \mathbf{T}_3 P \right) \dot{\mathbf{d}}_g$$

$$\dot{\mathbf{f}}_g = \begin{Bmatrix} \dot{f}_{g1} \\ \dot{f}_{g2} \\ \dot{f}_{g3} \\ \dot{f}_{g4} \end{Bmatrix} \quad \dot{\mathbf{d}}_g = \begin{Bmatrix} \dot{d}_{g1} \\ \dot{d}_{g2} \\ \dot{d}_{g3} \\ \dot{d}_{g4} \end{Bmatrix}$$

$$\mathbf{T}_{cg} = [-\cos \theta \quad -\sin \theta \quad \cos \theta \quad \sin \theta]^T$$

$$\mathbf{T}_3 = \begin{bmatrix} \sin^2 \theta & -\sin \theta \cos \theta & -\sin^2 \theta & \sin \theta \cos \theta \\ & \cos^2 \theta & \sin \theta \cos \theta & \cos^2 \theta \\ & & \sin^2 \theta & -\sin \theta \cos \theta \\ \text{sym} & & & \cos^2 \theta \end{bmatrix}$$

8.4.2 Modifies Tangent Stiffness Relationship

$$\begin{Bmatrix} \Delta \dot{M}_{pcA} \\ \dot{M}_B \\ \dot{P} \end{Bmatrix} = \frac{EI}{L} \begin{bmatrix} S_1 & S_2 & 0 \\ S_2 & S_1 & 0 \\ 0 & 0 & A/I \end{bmatrix} \begin{Bmatrix} \dot{\theta}_A \\ \dot{\theta}_B \\ \dot{e} \end{Bmatrix}$$

$$\dot{\theta}_A = \frac{L \Delta M_{pcA}}{EIS_1} - \frac{S_2}{S_1} \dot{\theta}_B$$

$$\begin{Bmatrix} \dot{M}_A \\ \dot{M}_B \\ \dot{P} \end{Bmatrix} = \frac{EI}{L} \begin{bmatrix} 0 & 0 & 0 \\ 0 & (S_1 - S_2^2)/S_1 & 0 \\ 0 & 0 & A/I \end{bmatrix} \begin{Bmatrix} \dot{\theta}_A \\ \dot{\theta}_B \\ \dot{e} \end{Bmatrix} + \begin{Bmatrix} 1 \\ S_2/S_1 \\ 0 \end{Bmatrix} \Delta M_{pcA}$$

$$\begin{Bmatrix} \dot{M}_A \\ \dot{M}_B \\ \dot{P} \end{Bmatrix} = \frac{EI}{L} \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & A/I \end{bmatrix} \begin{Bmatrix} \dot{\theta}_A \\ \dot{\theta}_B \\ \dot{e} \end{Bmatrix} + \begin{Bmatrix} \Delta M_{pcA} \\ \Delta M_{pcB} \\ 0 \end{Bmatrix}$$

$$\dot{\mathbf{f}}_c = \mathbf{k}_{ch} \dot{\mathbf{d}}_c + \dot{\mathbf{f}}_{cp}$$

$$\dot{\mathbf{f}}_g = \left(\mathbf{T}_{cg}^T \mathbf{k}_{ch} \mathbf{T}_{cg} + \mathbf{T}_1 M_A + \mathbf{T}_2 M_B + \mathbf{T}_3 P \right) \dot{\mathbf{d}}_g + \mathbf{T}_{cg}^T \dot{\mathbf{f}}_{cp}$$

$$\dot{\mathbf{f}}_g = \mathbf{k}_{gh} \dot{\mathbf{d}}_g + \dot{\mathbf{f}}_{gp}$$

$$\mathbf{k}_{gh} = \mathbf{T}_{cg}^T \mathbf{k}_{ch} \mathbf{T}_{cg} + \mathbf{T}_1 M_A + \mathbf{T}_2 M_B + \mathbf{T}_3 P$$

$$\dot{\mathbf{f}}_{gp} = \mathbf{T}_{cg}^T \dot{\mathbf{f}}_{cp}$$