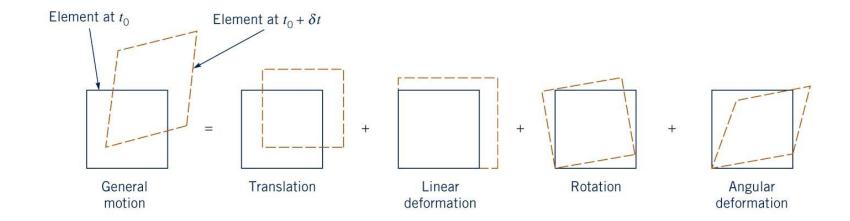
# Lecture 8

# Stress-Strain Relation (1)







#### Lecture 8 Stress-Strain Relation (1)

#### **Contents**

8.1 General Stress-Strain System

#### **Objectives**

- Understand tensor systems of stress and strain
- Study difference between displacement and deformation





Newton's 2<sup>nd</sup> law of motion → Navier-Stokes Equation

$$\Sigma \vec{F} = m\vec{a} = \frac{d}{dt} \left( m\vec{V} \right)$$

Body force: gravitational force

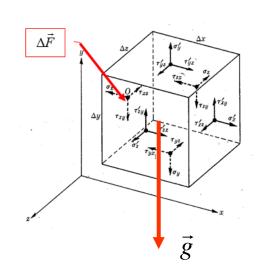
Surface force: normal force, shear force

$$\frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z} + \rho g_x = \rho a_x$$

$$\frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_{y}}{\partial y} + \frac{\partial \tau_{zy}}{\partial x} + \rho g_{y} = \rho a_{y}$$

$$\frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_{y}}{\partial y} + \frac{\partial \tau_{zy}}{\partial x} + \rho g_{y} = \rho a_{y}$$

$$\frac{\partial \tau_{xz}}{\partial x} + \frac{\partial \tau_{yz}}{\partial y} + \frac{\partial \sigma_{z}}{\partial z} + \rho g_{z} = \rho a_{z}$$



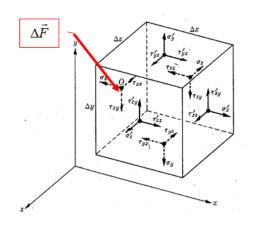




Parallelepiped, cube → infinitesimal C.V.

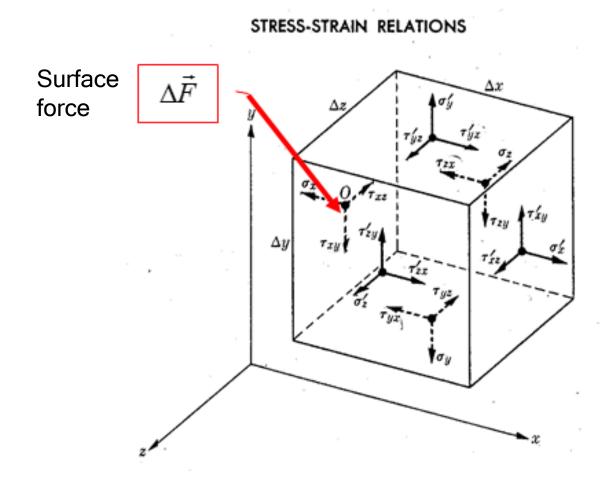
#### 8.1.1 Surface Stress

Surface stresses: 
$$\begin{cases} \text{normal stress - } & \sigma_{xx} \\ \text{shear stress - } & \tau_{xy} \end{cases}$$













$$\sigma_{xx} = \sigma_{x} = \lim_{\Delta A_{x} \to 0} \frac{\Delta F_{x}}{\Delta A_{x}}$$
$$(\Delta A_{x} = \Delta y \Delta z)$$

$$\sigma_{xx} = \sigma_{x} = \lim_{\Delta A_{x} \to 0} \frac{\Delta F_{x}}{\Delta A_{x}} \qquad \tau_{xy} = \lim_{\Delta A_{x} \to 0} \frac{\Delta F_{y}}{\Delta A_{x}} \qquad \tau_{xz} = \lim_{\Delta A_{x} \to 0} \frac{\Delta F_{z}}{\Delta A_{x}}$$

$$(\Delta A_{x} = \Delta y \Delta z)$$

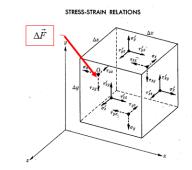
$$\tau_{yx} = \lim_{\Delta A_y \to 0} \frac{\Delta F_x}{\Delta A_y}$$
$$(\Delta A_y = \Delta x \Delta z)$$

$$\tau_{zx} = \lim_{\Delta A_z \to 0} \frac{\Delta F_x}{\Delta A_z}$$
$$(\Delta A_z = \Delta x \Delta y)$$

$$\sigma_{yy} = \sigma_{y} = \lim_{\Delta A_{y} \to 0} \frac{\Delta F_{y}}{\Delta A_{y}} \quad \tau_{yz} = \lim_{\Delta A_{y} \to 0} \frac{\Delta F_{z}}{\Delta A_{y}}$$

$$\tau_{zy} = \lim_{\Delta A_z \to 0} \frac{\Delta F_y}{\Delta A_z}$$
 $\sigma_{zz} = \sigma_z = \lim_{\Delta A_z \to 0} \frac{\Delta F_z}{\Delta A_z}$ 

Same surface but different direction



Same direction but different surface

where  $\Delta F_x$ ,  $\Delta F_v$ ,  $\Delta F_z$ = component of force vector  $\Delta F_z$ 





subscripts

 $\sigma_{x}$ : subscript indicates the <u>direction of stress</u>

 $\tau_{xy}$ : 1st - direction of the normal to the face on which  $\tau$  acts

2nd - direction in which τ acts

- general stress system: <u>stress tensor</u>
- ~ 9 scalar components (Cauchy's formula)

$$egin{pmatrix} oldsymbol{\sigma}_{xx} & oldsymbol{ au}_{xy} & oldsymbol{ au}_{xz} \ oldsymbol{ au}_{yx} & oldsymbol{\sigma}_{yy} & oldsymbol{ au}_{yz} \ oldsymbol{ au}_{zx} & oldsymbol{ au}_{zy} & oldsymbol{\sigma}_{zz} \end{pmatrix}$$





#### [Re] Tensor

~ an ordered array of entities which is invariant under coordinate transformation; includes scalars & vectors ~  $3^n$ 

Oth order - 1 component, scalar (mass, length, pressure, energy)

1st order - 3 components, vector (velocity, force, acceleration)

2nd order - 9 components (stress, rate of strain, turbulent diffusion)





At three other surfaces,

$$\sigma_{x}' = \sigma_{x} + \frac{\partial \sigma_{x}}{\partial x} \Delta x$$

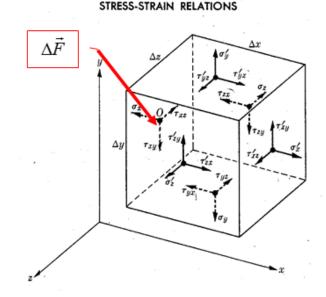
$$\sigma_{y}' = \sigma_{y} + \frac{\partial \sigma_{y}}{\partial y} \Delta y$$

$$\sigma_{z}' = \sigma_{z} + \frac{\partial \sigma_{z}}{\partial z} \Delta z$$

$$\tau_{xy}' = \tau_{xy} + \frac{\partial \tau_{xy}}{\partial x} \Delta x$$

$$\tau_{yx}' = \tau_{yx} + \frac{\partial \tau_{yx}}{\partial y} \Delta y$$

$$\tau_{zx}' = \tau_{zx} + \frac{\partial \tau_{zx}}{\partial z} \Delta z$$







- ♦ Shear stress is symmetric. → moment is zero
- → Shear stress pairs with subscripts differing in order are equal.

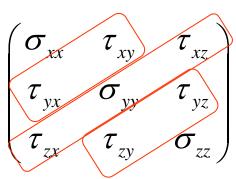
$$\rightarrow au_{xy} = au_{yx}$$

#### [Proof]

In <u>static equilibrium</u>, sum of all moments and sum of all forces equal zero for the element.

First, apply Newton's 2nd law

$$\sum F = m \frac{du}{dt}$$







Then, consider torque (angular momentum), T

$$\sum T = \frac{d}{dt}(rmu) = \frac{d}{dt}(r^2m\omega) = \frac{d}{dt}(I\omega) = I\frac{d\omega}{dt}$$

where 
$$l = moment of inertia = r^2 m$$
 $r = radius of gyration$ 
 $\frac{d\omega}{dt} = angular acceleration$ 

Thus,

$$\sum T = mr^2 \frac{d\omega}{dt} \tag{A}$$



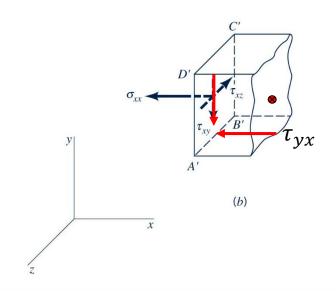


Now, take a moment about a centroid axis in the z-direction

$$LHS = \sum T = \left(\Delta y \Delta z \tau_{xy}\right) \frac{\Delta x}{2} - \left(\tau_{yx} \Delta x \Delta z\right) \frac{\Delta y}{2} = \frac{\Delta x \Delta y \Delta z}{2} \left(\tau_{xy} - \tau_{yx}\right)$$

$$RHS = \rho dvolr^{2} \frac{d\omega}{dt} = \Delta x \Delta y \Delta z \rho r^{2} \frac{d\omega}{dt}$$

$$\therefore \left(\tau_{xy} - \tau_{yx}\right) \Delta x \Delta y \Delta z = 2\Delta x \Delta y \Delta z \rho r^2 \frac{d\omega}{dt}$$







After canceling terms, this gives

$$\tau_{xy} - \tau_{yx} = 2\rho r^2 \frac{d\omega}{dt}$$

$$\lim_{\Delta x, \Delta y, \Delta z \to 0} r^2 \to 0$$

$$\tau_{xy} - \tau_{yx} = 0$$

$$\therefore \quad \tau_{xy} = \tau_{yx}$$

Components of Lame: 6 components 
$$\begin{bmatrix} \sigma_{xx} & \tau_{xy} & \tau_{xz} \\ \tau_{xy} & \sigma_{yy} & \tau_{yz} \\ \tau_{xz} & \tau_{yz} & \sigma_{zz} \end{bmatrix}$$





#### 8.1.2 Motion and deformation

Motion and deformation of fluid element

Motion

translation (displacement) - 단순이동(변위)

rotation - 회전

Deformation

linear (normal) deformation - 선형변형(수축 및 팽창)

angular (shear) deformation - 각변형(전단변형)



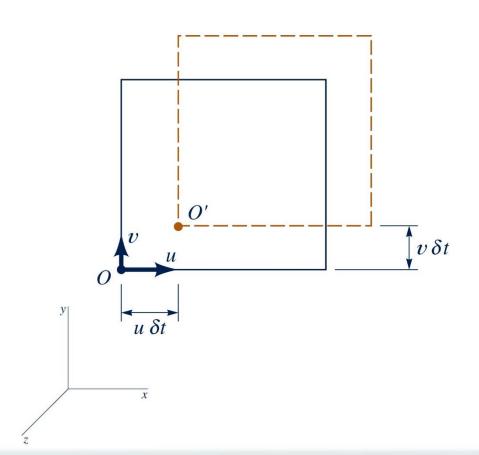


(1) Motion: <u>no change in shape</u>

1) Translation:  $\xi$ ,  $\eta$ 

$$\xi = u \, dt, \ u = \frac{d\xi}{dt}$$

$$\eta = v \, dt, \ v = \frac{d\eta}{dt}$$





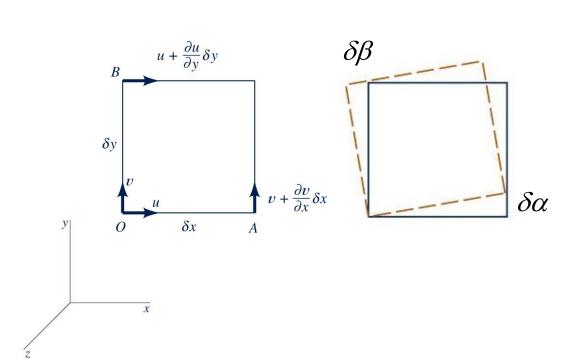


#### 2) Rotation ← Shear flow

$$\tan \delta \alpha = \frac{\frac{\partial v}{\partial x} \delta x \delta t}{\delta x} = \frac{\partial v}{\partial x} \delta t$$

$$\tan \delta \beta = \frac{-\frac{\partial u}{\partial y} \delta y \delta t}{\delta y} = -\frac{\partial u}{\partial y} \delta t$$

$$\omega_z = \frac{1}{2} \left( \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right)$$







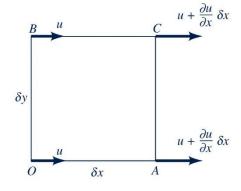
- (2) Deformation: change in shape
  - 1) Linear deformation <u>normal strain (변형율)</u>

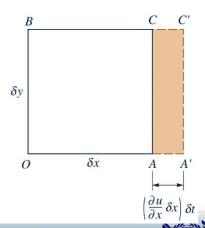
$$\varepsilon = \frac{\text{change in length}}{\text{original length}}$$

Non-dimensional

$$\varepsilon_{x} = \frac{\left(u + \frac{\partial u}{\partial x} dx\right) dt - u dt}{dx} = \frac{\partial u}{\partial x} dt = \frac{\partial \xi}{\partial x}$$

$$\varepsilon_{y} = \frac{\partial \eta}{\partial y}$$





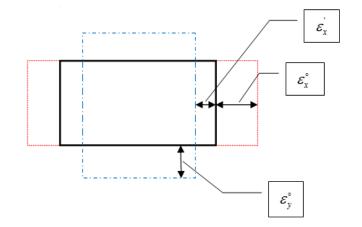


- For compressible fluid, changes in temperature or pressure cause change in volume.
- For <u>incompressible</u> fluid, if length in 2-D increases, then length in another 1-D decreases in order to make <u>total volume unchanged</u>.

$$\varepsilon_{x}^{0} = \frac{\partial \xi}{\partial x} \qquad \varepsilon_{y}^{0} = \frac{\partial \eta}{\partial y}$$

 $\mathcal{E}_{x}$ ' = elongation in the x-dir. due to  $\sigma_{y}$ 

 $\varepsilon_x$ " = elongation in the x - dir. due to  $\sigma_z$ 



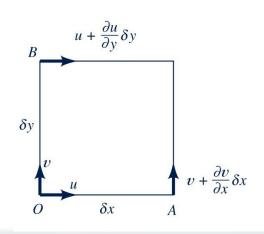


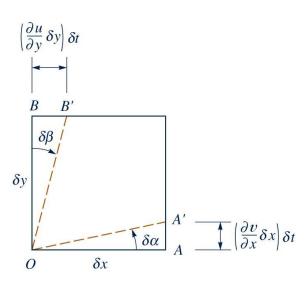


2) Angular deformation-shear strain

$$\gamma_{xy} = \frac{\partial \eta}{\partial x} + \frac{\partial \xi}{\partial y}$$











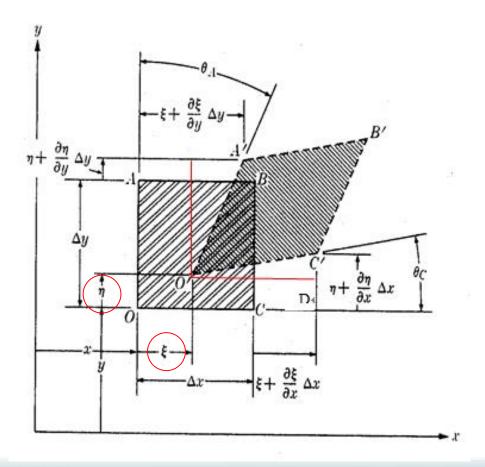
#### 8.1.3 Strain components

○ Strain  $\gamma$  normal strain:  $\varepsilon$  ← linear deformation shear strain:  $\gamma$  ← angular deformation





#### Consider a small element *OABC*







i) Displacement (translation):  $\xi$ ,  $\eta$ ,  $\zeta$ 

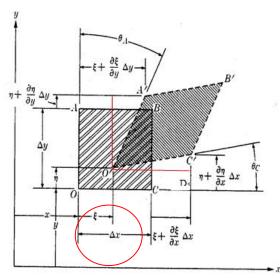
$$O(x, y, z) \rightarrow O'(x + \xi, y + \eta, z + \zeta)$$

$$C(x + \Delta x, y + \Delta y, z + \Delta z) \rightarrow$$

$$C'\left(x+\Delta x+\xi+\frac{\partial\xi}{\partial x}\Delta x, y+\eta+\frac{\partial\eta}{\partial x}\Delta x, z+\zeta+\frac{\partial\zeta}{\partial x}\Delta x\right)$$

- ii) Deformation: due to system of external forces  $OABC \rightarrow O'A'B'C'$
- 1) Normal strain,  $\varepsilon$

$$\varepsilon = \frac{\text{change in length}}{\text{original length}}$$





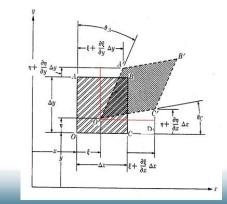


$$\varepsilon_{x} = \lim_{\Delta x \to 0} \frac{O'C' - OC}{OC} = \lim_{\Delta x \to 0} \frac{\left\{ \left( x + \Delta x + \xi + \frac{\partial \xi}{\partial x} \Delta x \right) - \left( x + \xi \right) \right\} - \Delta x}{\Delta x} = \frac{\partial \xi}{\partial x}$$
(8.1a)

$$\varepsilon_{y} = \lim_{\Delta y \to 0} \frac{O'A' - OA}{OA} = \lim_{\Delta y \to 0} \frac{\left\{ \left( y + \Delta y + \eta + \frac{\partial \eta}{\partial y} \Delta y \right) - \left( y + \eta \right) \right\} - \Delta y}{\Delta y} = \frac{\partial \eta}{\partial y}$$
 (8.1b)

$$\varepsilon_z = \frac{\partial \zeta}{\partial z} \tag{8.1c}$$

 $\sim$   $\varepsilon$  is positive when <u>element elongates</u> under deformation







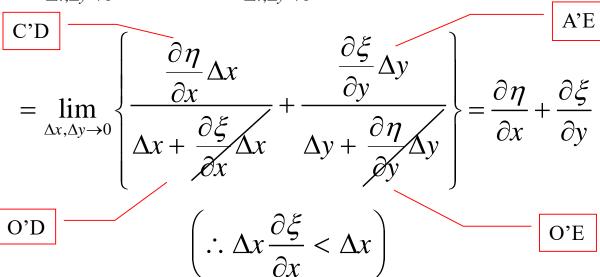
#### 2) Shear strain, $\gamma$

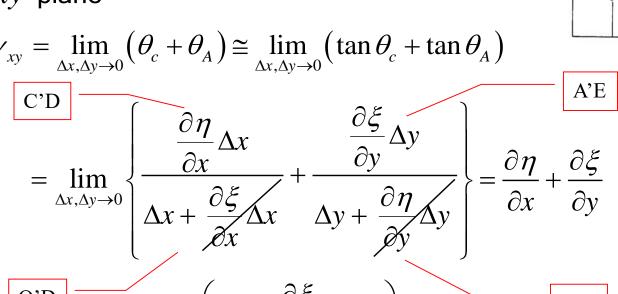
<u>change in angle</u> between <u>two originally</u>

perpendicular elements

For xy-plane

$$\gamma_{xy} = \lim_{\Delta x, \Delta y \to 0} (\theta_c + \theta_A) \cong \lim_{\Delta x, \Delta y \to 0} (\tan \theta_c + \tan \theta_A)$$









$$\gamma_{xy} = \frac{\partial \eta}{\partial x} + \frac{\partial \xi}{\partial y}$$

$$\gamma_{yz} = \frac{\partial \zeta}{\partial y} + \frac{\partial \eta}{\partial z}$$

$$\gamma_{zx} = \frac{\partial \xi}{\partial z} + \frac{\partial \zeta}{\partial x}$$

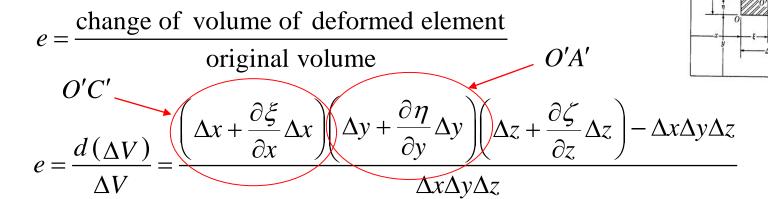
(2) displacement vector  $\vec{\delta}$ 

$$\vec{\delta} = \xi \vec{i} + \eta \vec{j} + \zeta \vec{k}$$

(8.3)



#### (3) Volume dilation (dilatation)



$$\cong \frac{\partial \xi}{\partial x} + \frac{\partial \eta}{\partial y} + \frac{\partial \zeta}{\partial z} = \varepsilon_x + \varepsilon_y + \varepsilon_z \tag{8.4}$$

$$e = \varepsilon_x + \varepsilon_y + \varepsilon_z \tag{8.5}$$

$$e = \frac{\partial \xi}{\partial x} + \frac{\partial \eta}{\partial y} + \frac{\partial \zeta}{\partial z} = \nabla \cdot \vec{\delta} \quad \text{---- divergence}$$
 (8.6)





#### **Problems**

#### [Homework Assignment-Special work]

Due: 1 week from today

1. Fabricate your own "Stress Cube" using paper box.



