## **Chapter 6 Equations of Continuity and Motion**

### **Session 6-1 Continuity equation**







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#### Objectives

- Derive 3D equations of continuity and motion
- Derive Navier-Stokes equation for Newtonian fluid
- Study solutions for simplified cases of laminar flow
- Derive Bernoulli equation for irrotational motion and frictionless flow
- Study solutions for vortex motions





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• Derivation of 3-D Eq.

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conservation of mass \rightarrow continuity eq.
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conservation of momentum  $\rightarrow$  eq. of motion  $\rightarrow$  Navier-Strokes eq.

Consider <u>infinitesimal</u> control volume ( $\Delta x \Delta y \Delta z$ )  $\rightarrow$  point form [Cf] Finite control volume – arbitrary CV  $\rightarrow$  integral form

Apply principle of conservation of matter to the CV

 $\rightarrow$  sum of net flux = time rate change of mass inside C.V.

1) mass flux per unit time (mass flow)

$$=\frac{mass}{time} = \rho \frac{vol}{time} = \rho Q = \rho u \Delta A$$



## **6.1 Continuity Equation**







- net flux through face perpendicular to x-axis
  - = flux in -flux out

$$=\rho u \Delta y \Delta z - \left(\rho u + \frac{\partial(\rho u)}{\partial x} \Delta x\right) \Delta y \Delta z = -\frac{\partial(\rho u)}{\partial x} \Delta x \Delta y \Delta z$$

• net flux through face perpendicular to *y*-axis



• net flux through face perpendicular to z-axis

$$= -\frac{\partial(\rho w)}{\partial z} \Delta x \Delta y \Delta z$$



(A)



2) time rate change of mass inside C.V.

$$= \frac{\partial}{\partial t} (\rho \Delta x \Delta y \Delta z)$$

Thus, equating (A) and (B) gives

$$\frac{\partial}{\partial t}(\rho\Delta x\Delta y\Delta z) = -\frac{\partial(\rho u)}{\partial x}\Delta x\Delta y\Delta z - \frac{\partial(\rho v)}{\partial y}\Delta x\Delta y\Delta z - \frac{\partial(\rho w)}{\partial z}\Delta x\Delta y\Delta z$$

$$LHS = \frac{\partial}{\partial t} (\rho \Delta x \Delta y \Delta z) = \rho \frac{\partial}{\partial t} (\Delta x \Delta y \Delta z) + \Delta x \Delta y \Delta z \frac{\partial \rho}{\partial t}$$





(B)

Since C.V. is fixed 
$$\rightarrow \frac{\partial (\Delta x \Delta y \Delta z)}{\partial t} = 0$$
  
 $\therefore LHS = \Delta x \Delta y \Delta z \frac{\partial \rho}{\partial t}$ 

Cancelling terms makes

$$\frac{\partial \rho}{\partial t} + \frac{\partial (\rho u)}{\partial x} + \frac{\partial (\rho v)}{\partial y} + \frac{\partial (\rho w)}{\partial z} = 0$$
$$\frac{\partial \rho}{\partial t} + \nabla \cdot \rho \vec{q} = 0$$

(6.1)

→ Continuity Eq. for compressible fluid in unsteady flow (point form)





The 2<sup>nd</sup> term of Eq. (6.1) can be expressed as



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Substituting (i) into Eq (6.1) yields





[Re] Total derivative (total rate of density change)

$$\frac{d\rho}{dt} = \frac{\partial\rho}{\partial t} + \frac{\partial\rho}{\partial x}\frac{dx}{dt} + \frac{\partial\rho}{\partial y}\frac{dy}{dt} + \frac{\partial\rho}{\partial z}\frac{dz}{dt}$$
$$= \frac{\partial\rho}{\partial t} + u\frac{\partial\rho}{\partial x} + v\frac{\partial\rho}{\partial y} + w\frac{\partial\rho}{\partial z}$$

1) For steady-state conditions

$$\rightarrow \frac{\partial \rho}{\partial t} = 0$$





# 6.1 Continuity Equation

#### Then (6.1) becomes

$$\frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho v)}{\partial y} + \frac{\partial(\rho w)}{\partial z} = \nabla \cdot (\rho \vec{q}) = 0$$
(6.3)

2) For incompressible fluid (whether or not flow is steady)

$$\rightarrow \frac{d\rho}{dt} = 0 \tag{6.4}$$

Then (6.2) becomes

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = \nabla \cdot \vec{q} = 0$$

(6.5)





[Re] Continuity equation in polar (cylindrical) coordinates

- u, r radial
- $v, \theta$  azimuthal

w, z - axial

For compressible fluid of unsteady flow



$$\frac{\partial \rho}{\partial t} + \frac{1}{r} \frac{\partial (\rho u r)}{\partial r} + \frac{1}{r} \frac{\partial (\rho v)}{\partial \theta} + \frac{\partial (\rho w)}{\partial z} = 0$$

### For incompressible fluid

$$\frac{1}{r}\frac{\partial(ur)}{\partial r} + \frac{1}{r}\frac{\partial v}{\partial \theta} + \frac{\partial w}{\partial z} = 0$$





For incompressible fluid and flow of axial symmetry

$$\frac{\partial \rho}{\partial t} = 0, \quad \frac{\partial \rho}{\partial r} = \frac{\partial \rho}{\partial \theta} = \frac{\partial \rho}{\partial z} = 0, \qquad \frac{\partial (\rho v)}{\partial \theta} = 0$$

$$\therefore \frac{1}{r} \frac{\partial(ur)}{\partial r} + \frac{\partial w}{\partial z} = 0 \quad \Rightarrow 2\text{-D boundary layer flow}$$

Example: submerged jet







# 6.2 Stream Function in 2-D, Incompressible Flows

2-D incompressible continuity eq. is

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

In 2D flowfield, define stream function 
$$\psi(x, y)$$
 as

$$u = -\frac{\partial \psi}{\partial y}$$
(6.8)  
$$v = \frac{\partial \psi}{\partial x}$$
(6.9)

• We can have a simplified equation to determine only one unknown function.



(6.7)



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# 6.2 Stream Function in 2-D, Incompressible Flows

### Then LHS of Eq. (6.7) becomes

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = \frac{\partial}{\partial x} \left( -\frac{\partial \psi}{\partial y} \right) + \frac{\partial}{\partial y} \left( \frac{\partial \psi}{\partial x} \right) = -\frac{\partial^2 \psi}{\partial x \partial y} + \frac{\partial^2 \psi}{\partial x \partial y} = 0$$
(6.10)

### $\rightarrow$ Thus, continuity equation is satisfied.

1) Apply stream function to the equation for <u>a stream line</u> in 2-D flow Eq. (2.10): vdx - udy = 0 (6.11)

Substitute (6.8) into (6.11)

$$\frac{\partial \psi}{\partial x} dx + \frac{\partial \psi}{\partial y} dy = d\psi = 0$$
(6.12)





# 6.2 Stream Function in 2-D, Incompressible Flows

 $\psi$  = constant

(6.13)

 $\rightarrow$  The stream function is constant <u>along a streamline</u>.

2) Apply stream function to the law of conservation of mass

$$-qdn = -udy + vdx \tag{6.14}$$

Substitute (6.8) into (6.14)

$$-qdn = \frac{\partial \psi}{\partial y}dy + \frac{\partial \psi}{\partial x}dx = d\psi \qquad (6.15)$$







→ Change in  $\psi$  ( $d\psi$ ) between adjacent streamlines is equal to the <u>volume</u> rate of flow per unit width.







#### 6.3.1 Rotation and vorticity

Assume the <u>rate of rotation</u> of fluid element  $\Delta x$  and  $\Delta y$  about *z*-axis is positive when it rotates <u>counterclockwise</u>.

- <u>time rate of rotation</u> of  $\Delta x$ -face about *z*-axis

$$=\frac{1}{\Delta t}\frac{\left[\left\{v + \left(\frac{\partial v}{\partial x}\right)\Delta x\right\} - v\right]\Delta t}{\Delta x} = \frac{\partial v}{\partial x}$$







- time rate of rotation of  $\Delta y$ -face about z - axis

$$= \frac{1}{\Delta t} \frac{\left[ u + \left( \frac{\partial u}{\partial y} \Delta y \right) - u \right] \Delta t}{\Delta y} = -\frac{\partial u}{\partial y}$$

net rate of rotation = average of rotations of  $\Delta x$ -and  $\Delta y$ -face

$$\omega_z = \frac{1}{2} \left( \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right)$$





Doing the same way for x-axis, and y-axis

$$\omega_{x} = \frac{1}{2} \left( \frac{\partial w}{\partial y} - \frac{\partial v}{\partial z} \right)$$
$$\omega_{y} = \frac{1}{2} \left( \frac{\partial u}{\partial z} - \frac{\partial w}{\partial x} \right)$$

(6.16a)

1) Rotation

$$\vec{\omega} = \frac{1}{2} \left( \frac{\partial w}{\partial y} - \frac{\partial v}{\partial z} \right) \vec{i} + \frac{1}{2} \left( \frac{\partial u}{\partial z} - \frac{\partial w}{\partial x} \right) \vec{j} + \frac{1}{2} \left( \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) \vec{k}$$
$$= \frac{1}{2} \left( \nabla \times \vec{q} \right) = \frac{1}{2} curl \vec{q}$$
(6.16b)





Magnitude:

$$\left|\vec{\omega}\right| = \sqrt{\omega_x^2 + \omega_y^2 + \omega_z^2}$$

a) Ideal fluid  $\rightarrow$  irrotational flow

$$\nabla \times \vec{q} = 0$$
  

$$\omega_x = \omega_y = \omega_z = 0$$
  

$$\frac{\partial w}{\partial y} = \frac{\partial v}{\partial z}, \quad \frac{\partial u}{\partial z} = \frac{\partial w}{\partial x}, \quad \frac{\partial v}{\partial x} = \frac{\partial u}{\partial y}$$

b) Viscous fluid  $\rightarrow$  rotational flow

 $\nabla \times \vec{q} \neq 0$ 







2) Vorticity

$$\vec{\zeta} = curl \ \vec{q} = \nabla \times \vec{q} = 2\vec{\omega}$$

Rotation in cylindrical coordinates

$$\omega_r = \frac{1}{2} \left( \frac{1}{r} \frac{\partial v_z}{\partial \theta} - \frac{\partial v_\theta}{\partial z} \right)$$

$$\omega_{\theta} = \frac{1}{2} \left( \frac{\partial v_r}{\partial z} - \frac{\partial v_z}{\partial r} \right)$$

$$\omega_{z} = \frac{1}{2} \left( -\frac{1}{r} \frac{\partial v_{r}}{\partial \theta} + \frac{v_{\theta}}{r} + \frac{\partial v_{\theta}}{\partial r} \right)$$

















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6.3.2 Circulation

 $\Gamma$  = <u>line integral of the tangential velocity component</u> about any closed contour *S* 

$$\Gamma = \oint \vec{q} \cdot d\vec{s} \tag{6.19}$$







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– take line integral from A to B, C, D, A ~ infinitesimal CV

$$d\Gamma \cong \left[ u - \frac{\partial u}{\partial y} \frac{dy}{2} \right] dx + \left[ v + \frac{\partial v}{\partial x} \frac{dx}{2} \right] dy - \left[ u + \frac{\partial u}{\partial y} \frac{dy}{2} \right] dx - \left[ v - \frac{\partial v}{\partial x} \frac{dx}{2} \right] dy$$
$$= \left( \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) dx dy$$
$$d\Gamma \cong \left( \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) dx dy$$

$$\Gamma = \iint_{A} \left( \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) dA = \iint_{A} 2\omega_{z} dA = \iint_{A} \left( \nabla \times \vec{q} \right)_{z} dA$$
(6.20)





For irrotational flow,

circulation  $\Gamma = 0$  (if there is no singularity vorticity source).

[Re] Fluid motion and deformation of fluid element

translation

rotation

Motion

y y z dt

Deformation { linear deformation angular deformation



