

Boundary Layer Flows (2)







Contents

- 17.1 Equation for 2-D Boundary Layers
- 17.2 Laminar Boundary Layers

Objectives

- Derive the equation for 2-D boundary layers and integral equation
- Study the solution for laminar boundary layer flows





- Boundary Layer Theory
- Prandtl (1904) suggested the boundary layer concept in which viscous effects are important only in the boundary layer regions, and outside boundary layer, the fluid acts as if it were inviscid.
- A necessary condition for this structure of the flow is that the <u>Reynolds</u> <u>number be large</u>.







17.1.1 Boundary-layer thickness definitions

- In the boundary layer, the fluid change its velocity from the upstream value of U to zero on the solid surface (no slip).
- The velocity profile is given as

u = u(x, y) (17.1)

- In actuality, there is no sharp edge to the boundary layer, that is, $u \rightarrow U$.

 \rightarrow very intermittent









(1) Boundary-layer thickness, δ

- Define δ as the distance to the point where the velocity is within 1% of the free-stream velocity, U







(2) Mass displacement thickness, $\delta^*(\delta_l)$

 $= \rho Q = \rho U A = \rho U \delta^* \times 1$

- \sim is the thickness of an <u>imaginary</u> layer of fluid of velocity *U*.
- \sim is the thickness of mass flux rate (flowrate) equal to the amount of defect

$$\rho U \delta^* = \frac{\rho \int_0^h (U - u) dy}{mass \ defect} \qquad h \ge \delta$$

$$\delta^* = \int_0^h (1 - \frac{u}{U}) dy \qquad (17.3)$$

[Re] mass flux = mass/time

a





b

(3) Momentum thickness, $\theta(\delta_2)$

- Velocity retardation within δ causes a <u>reduction in the rate of momentum</u> <u>flux</u>.

 $\rightarrow \theta$ is the thickness of an imaginary layer of fluid of velocity U for which the <u>momentum flux rate</u> equals the reduction caused by the velocity profile.

$$\rho \theta U^2 = \rho \int_0^h (U-u)u dy = \rho \int_0^h (Uu-u^2) dy$$

$$\theta = \int_{0}^{h} \frac{u}{U} (1 - \frac{u}{U}) dy$$
 (17.4)





[Re] momentum in θ = mass × velocity = $\rho \theta U \times U = \rho \theta U^2$ momentum in shaded area = $\int [\rho(U-u) \times u] dy$

 $\delta > \delta^* > \theta$

(4) Energy thickness, δ_3

$$\frac{1}{2}\rho U^{3}\delta_{3} = \frac{1}{2}\int_{0}^{h}\rho u(U^{2} - u^{2})dy$$
$$\delta_{3} = \int_{0}^{h}\frac{u}{U}(1 - \frac{u^{2}}{U^{2}})dy$$

(17.5)





- Two-dimensional boundary layer equations by Prandtl
- \rightarrow simplification of the N-S Eq. using <u>order-of-magnitude arguments</u>
- Start with <u>2D dimensionless N-S eq</u>. for incompressible fluid with negligible gravitational effects





Within thin and small curvature boundary layer

$$u \gg v, \qquad x \gg y$$
$$\frac{\partial u}{\partial y} \gg \frac{\partial u}{\partial x}$$

$$\frac{\partial p}{\partial y} \text{ is small} \sim may \text{ be neglected}$$

dimensionless boundary-layer thickness $\delta^{\!\circ}$

$$\delta^{\circ} = \frac{\delta}{L} \to \delta^{\circ} \ll 1$$

 \therefore scale for decreasing order

$$\frac{1}{\delta^{\circ^2}} > \frac{1}{\delta^{\circ}} > 1 > \delta^{\circ} > \delta^{\circ^2}$$





Order of magnitude

$x^{\circ} \sim O(1)$	$x^{\circ} = \frac{x}{L}$
$y^{\circ} \sim O(\delta^{\circ})$	$y^\circ = \frac{y}{L}$
$u^{\circ} \sim O(1)$ $v^{\circ} \sim O(\delta^{\circ})$	$u^{\circ} = \frac{u}{V_0}$
$\frac{\partial u^{\circ}}{\partial x^{\circ}} \sim O(1)$	$v^{\circ} = \frac{v}{V_0}$
$\frac{\partial v^{\circ}}{\partial y^{\circ}} \sim O(1) \leftarrow continuity\left(\frac{\partial v^{\circ}}{\partial y^{\circ}} = -\frac{\partial u^{\circ}}{\partial x^{\circ}}\right)$	$p^{\circ} = \frac{p}{\rho V_0^2}$
$\frac{\partial u^{\circ}}{\partial y^{\circ}} \sim O\left(\frac{1}{\delta^{\circ}}\right)$	
$\frac{\partial v^{\circ}}{\partial x^{\circ}} \sim O\left(\delta^{\circ}\right)$	





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$$\frac{\partial^2 u^{\circ}}{\partial (x^{\circ})^2} = \frac{\partial}{\partial x^{\circ}} \left(\frac{\partial u^{\circ}}{\partial x^{\circ}} \right) \sim O(1)$$

$$\frac{\partial^2 v^{\circ}}{\partial (y^{\circ})^2} = \frac{\partial}{\partial y^{\circ}} \left(\frac{\partial v^{\circ}}{\partial y^{\circ}} \right) \sim O(\frac{1}{\delta^{\circ}})$$

$$\frac{\partial u^{\circ}}{\partial t^{\circ}} = \frac{\partial u^{\circ}}{\partial x^{\circ}} \frac{\partial x^{\circ}}{\partial t^{\circ}} = u^{\circ} \frac{\partial u^{\circ}}{\partial x^{\circ}} \sim O(1)$$

$$\frac{\partial v^{\circ}}{\partial t^{\circ}} = \frac{\partial v^{\circ}}{\partial x^{\circ}} \frac{\partial x^{\circ}}{\partial t^{\circ}} = u^{\circ} \frac{\partial v^{\circ}}{\partial x^{\circ}} \sim O(\delta^{\circ})$$

$$\operatorname{Re} = \frac{\rho u x}{\mu} \sim O(\frac{1}{\delta^{\circ 2}})$$





$$x:\frac{\partial u^{\circ}}{\partial t^{\circ}} + u^{\circ}\frac{\partial u^{\circ}}{\partial x^{\circ}} + v^{\circ}\frac{\partial u^{\circ}}{\partial y^{\circ}} = -\frac{\partial p^{\circ}}{\partial x^{\circ}} + \frac{1}{\operatorname{Re}}\left(\frac{\partial^{2}u^{\circ}}{\partial x^{\circ^{2}}} + \frac{\partial^{2}u^{\circ}}{\partial y^{\circ^{2}}}\right)$$
(17.6)
$$1 \quad 1 \times 1 \quad \delta^{\circ} \times 1/\delta^{\circ} \qquad \delta^{\circ^{2}}(1+1/\delta^{\circ^{2}}) \to 1$$

$$y:\frac{\partial v^{\circ}}{\partial t^{\circ}} + u^{\circ}\frac{\partial v^{\circ}}{\partial x^{\circ}} + v^{\circ}\frac{\partial v^{\circ}}{\partial y^{\circ}} = -\frac{\partial p^{\circ}}{\partial y^{\circ}} + \frac{1}{\operatorname{Re}}\left(\frac{\partial^{2} v^{\circ}}{\partial x^{\circ^{2}}} + \frac{\partial^{2} v^{\circ}}{\partial y^{\circ^{2}}}\right)$$
$$\delta^{\circ} \quad 1 \times \delta^{\circ} \quad \delta^{\circ} \times 1 \qquad \delta^{\circ^{2}}(\delta^{\circ} + 1/\delta^{\circ}) \rightarrow \delta^{\circ}$$

Continuity:
$$\frac{\partial u^{\circ}}{\partial x^{\circ}} + \frac{\partial v^{\circ}}{\partial y^{\circ}} = 0$$

1 1





Therefore, eliminate all terms of order less than unity in Eq. (17.6) and revert to dimensional terms

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \frac{\mu}{\rho} \frac{\partial^2 u}{\partial y^2}$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$
(17.7)

 \rightarrow Prandtl's 2-D boundary-layer equation

BC: 1)
$$y = 0$$
; $u = 0, v = 0$

2)
$$y = \infty$$
; $u = U(x)$

(17.8)

Unknowns: *u*, *v*, *p*; Eqs. = $2 \rightarrow$ needs <u>assumptions for *p*</u>





17.1.3 Integral momentum equation for 2-D boundary layers

- Prandtl's equation can be integrated to obtain the <u>relation between boundary</u> <u>shear stress and velocity distribution</u> for steady motion of incompressible fluid.

Assumptions:

constant density:
$$d \rho = 0$$

steady motion: $\frac{\partial(\cdot)}{\partial t} = 0$
pressure gradient = 0: $\frac{\partial p}{\partial x} = 0$
BC's: $@ y = h; \tau = 0, u = U$
 $@ y = 0; \tau = \tau_0, u = 0$





- Prandtl's 2-D boundary-layer equations become as follows:



Integrate Eq. (A) w.r.t. y

$$\int_{y=0}^{y=h\geq\delta} \left(\frac{u}{\partial x} + v \frac{\partial u}{\partial y} \right) dy = \frac{\mu}{\rho} \int_{y=0}^{y=h} \frac{\partial^2 u}{\partial y^2} dy \qquad (C)$$



$$(3) = \mu \int_{0}^{h} \frac{\partial^{2} u}{\partial y^{2}} dy = \int_{0}^{h} \frac{\partial}{\partial y} \left(\mu \frac{\partial u}{\partial y} \right) dy = \int_{0}^{h} \frac{\partial \tau}{\partial y} dy = [\tau]_{0}^{h}$$
$$= \tau \Big|_{y=h} - \tau \Big|_{y=0} = 0 - \tau_{0} = -\tau_{0}$$
$$(2) = \int_{0}^{h} v \frac{\partial u}{\partial y} dy = \int_{0}^{h} \frac{\partial u v}{\partial y} dy + \int_{0}^{h} \frac{\partial v}{\partial y} dy$$
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$$(4) = \int_{0}^{h} \frac{\partial v}{\partial y} dy$$
$$(5)$$
[Re] Integration by parts: $\int v u dy = uv - \int uv dy$

$$(4) = \int_0^h \frac{\partial uv}{\partial y} dy = [uv]_0^h = Uv_h - 0 = Uv$$





(D)

Continuity Eq.:
$$\frac{\partial v}{\partial y} = -\frac{\partial u}{\partial x}$$
 (i)
 $\rightarrow v = -\int_{0}^{h} \frac{\partial u}{\partial x} dy$ (ii)

Substitute (i) into (5)

$$(5) = \int_0^h u \left(-\frac{\partial u}{\partial x} \right) dy = -\int_0^h u \frac{\partial u}{\partial x} dy$$
(iii)

Substitute (ii) into 4

$$(4) = Uv = -U \int_0^h \frac{\partial u}{\partial x} dy$$





Eq. (D) becomes

$$\int_{0}^{h} v \frac{\partial u}{\partial y} dy = -U \int_{0}^{h} \frac{\partial u}{\partial x} dy + \int_{0}^{h} u \frac{\partial u}{\partial x} dy$$

Then, (C) becomes

$$\int_{0}^{h} u \frac{\partial u}{\partial x} dy - U \int_{0}^{h} \frac{\partial u}{\partial x} dy + \int_{0}^{h} u \frac{\partial u}{\partial x} dy = -\frac{\tau_{0}}{\rho}$$
(F)

For steady motion with and U = const., (F) becomes

$$\frac{\tau_0}{\rho} = U \int_0^h \frac{\partial u}{\partial x} dy - 2 \int_0^h u \frac{\partial u}{\partial x} dy = \int_0^h \frac{\partial U u}{\partial x} dy - \int_0^h \frac{\partial u^2}{\partial x} dy$$
$$= \int_0^h \frac{\partial}{\partial x} [u(U-u)] dy = \frac{\partial}{\partial x} \int_0^h u(U-u) dy = \frac{\partial}{\partial x} (\theta U^2)$$
$$\theta U^2$$





(E)

where θ = momentum thickness

$$\frac{\tau_0}{\rho} = \frac{\partial}{\partial x} (U^2 \theta) = U^2 \frac{\partial \theta}{\partial x}$$
$$\frac{\partial \theta}{\partial x} = \frac{\tau_0}{\rho U^2} = \left(\frac{u^*}{U}\right)^2$$

Introduce surface (frictional) resistance coefficient c_f

$$\tau_0 = \frac{\rho}{2} c_f U^2$$

$$c_f = \frac{v_0}{\frac{\rho}{2}U^2}$$

Combine (17.9) with (17.10)

$$c_f = 2\frac{\partial\theta}{\partial x} \qquad (17.11)$$





(17.9)

Integral momentum equation for <u>unsteady motion</u>

→ unsteady motion:
$$\frac{\partial()}{\partial t} \neq 0$$

→ pressure gradient, $\frac{\partial p}{\partial x} \neq 0$

First, simplify Eq. (17.7) for external flow where viscous influence is negligible. $\partial U = \partial U = \partial U = \partial U = 1 \partial p = u \partial^2 U$

$$\frac{\partial U}{\partial t} + U \frac{\partial U}{\partial x} + v \frac{\partial U}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \frac{\mu}{\rho} \frac{\partial^2 U}{\partial y^2}$$
$$\rho \frac{\partial U}{\partial t} + \rho U \frac{\partial U}{\partial x} = -\frac{\partial p}{\partial x}$$





(A)

Substitute (A) into (17.7)

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = \frac{\partial U}{\partial t} + U \frac{\partial U}{\partial x} + \frac{\mu}{\rho} \frac{\partial^2 u}{\partial y^2}$$
$$-\frac{1}{\rho} \frac{\partial p}{\partial x}$$

$$\int_{0}^{h} \frac{\mu}{\rho} \frac{\partial^{2} u}{\partial y^{2}} dy = \int_{0}^{h} \left\{ \frac{\partial u}{\partial t} - \frac{\partial U}{\partial t} + \frac{u}{\partial x} \frac{\partial u}{\partial x} - U \frac{\partial U}{\partial x} + \frac{v}{\partial y} \right\} dy$$
(B)

$$(1) \qquad (2) \qquad (3) \qquad (4)$$

$$(1): \int_0^h \frac{\mu}{\rho} \frac{\partial^2 u}{\partial y^2} \, dy = -\frac{\tau_0}{\rho}$$

$$(2): \int_0^h \left(\frac{\partial u}{\partial t} - \frac{\partial U}{\partial t}\right) dy = \int_0^h \frac{\partial}{\partial t} (u - U) dy = \frac{\partial}{\partial t} \int_0^h (u - U) dy = -\frac{\partial}{\partial t} U \delta^{*} dy$$

 $-U\delta^*$



Integrate



$$(3) = \int_{0}^{h} \left(u \frac{\partial u}{\partial x} - u \frac{\partial U}{\partial x} \right) dy + \int \left(u \frac{\partial U}{\partial x} - U \frac{\partial U}{\partial x} \right) dy$$

$$(3)-1 = \int_{0}^{h} \left\{ u \frac{\partial}{\partial x} (u - U) \right\} dy$$

$$(3)-2 = \int_{0}^{h} \left\{ (u - U) \frac{\partial U}{\partial x} \right\} dy = \frac{\partial U}{\partial x} \int_{0}^{h} (u - U) dy = \frac{\partial U}{\partial x} (-U\delta^{*})$$

$$(4) = \int_{0}^{h} v \frac{\partial u}{\partial y} dy = -U \int_{0}^{h} \frac{\partial u}{\partial x} dy + \int u \frac{\partial u}{\partial x} dy = \int_{0}^{h} (u - U) \frac{\partial u}{\partial x} dy$$

$$Eq.(E)$$





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Combine ③-1 and ④

$$\int_{0}^{h} u \frac{\partial}{\partial x} (u - U) dy + \int_{0}^{h} (u - U) \frac{\partial u}{\partial x} dy = \int_{0}^{h} \left[u \frac{\partial}{\partial x} (u - U) + (u - U) \frac{\partial u}{\partial x} \right] dy$$

$$= \int_{0}^{h} \frac{\partial}{\partial x} \{ u(u - U) \} dy = \frac{\partial}{\partial x} \int_{0}^{h} u(u - U) dy = \frac{\partial}{\partial x} (-\theta U^{2})$$
Substituting all these into (B) yields

$$-\frac{\tau_{0}}{\rho} = -\frac{\partial}{\partial t}(U\delta^{*}) - U\frac{\partial U}{\partial x}\delta^{*} - \frac{\partial}{\partial x}(\theta U^{2})$$
$$\frac{\tau_{0}}{\rho} = \frac{\partial}{\partial x}(U^{2}\theta) + U\frac{\partial U}{\partial x}\delta^{*} + \frac{\partial}{\partial t}(U\delta^{*})$$
(17.12)

 \rightarrow Karman's integral momentum equation





17.2.1 The Blasius solution for laminar flow

- Blasius (1908) solved the Prandtl's boundary layer equations for steady laminar flow.
- Assume the pressure gradient $\frac{\partial p}{\partial x}$ is zero.
- Then, Eq. (17.7) will reduce to

$$u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} = \frac{\mu}{\rho}\frac{\partial^2 u}{\partial y^2}$$
(17.13a)

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$
 (17.13b)



2.

2.



- The boundary conditions are

At the wall

y = 0: u = 0, v = 0 (17.14a)

Beyond the boundary layer $y = \infty$: u = U

(17.14b)

 Blasius obtained the solution by assuming <u>similar velocity profiles</u> along the plate at every x.

$$\frac{u}{U} = F\left(\frac{y}{\delta}\right) = F(\eta) \tag{17.15}$$

where $F \eta$ is the same for all x, and $\eta = \frac{y}{x / \operatorname{Re}_{x}^{1/2}}$.





- Introducing a stream function makes Eq. (17-13a) into the ordinary differential equation

$$\psi = -\sqrt{\nu x U} f(\eta) \tag{17.16a}$$

$$u = -\partial \psi / \partial y = Uf'$$
(17.16b)

$$v = \partial \psi / \partial x = \frac{1}{2} \sqrt{\nu U / x} \eta f' - f$$
 (17.16c)

$$ff'' + f''' = 0$$
 (17.17)

- Blasius obtained the solution of Eq. (17.17) in the form of a power series expanded about $\eta = 0$.





Laminar Flow along a Flat Plate (the Blasius Solution)

$\eta = y(U/\nu x)^{1/2}$	$f'(\eta) = u/U$	η	$f'(\pmb{\eta})$
0	0	3.6	0.9233
0.4	0.1328	4.0	0.9555
0.8	0.2647	4.4	0.9759
1.2	0.3938	4.8	0.9878
1.6	0.5168	5.0	0.9916
2.0	0.6298	5.2	0.9943
2.4	0.7290	5.6	0.9975
2.8	0.8115	6.0	0.9990
3.2	0.8761	∞	1.0000



 $f'(\eta) = \frac{u}{U}$





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FIG. 10-4. Velocity profiles in a laminar boundary layer on a flat plate [2].





From the solution, it is found that

$$\frac{u}{U} = 0.992$$
 when $\eta = 5 = \frac{y}{x / \text{Re}_x^{1/2}}$

Thus, we have the thickness of the boundary layer as

$$y = \delta_{lam} = \frac{5x}{\text{Re}_x^{1/2}}$$
 (17.18)

Then, based on definition of the mass and momentum thicknesses, we have

$$\delta_{lam}^{*} = \frac{1.73x}{\text{Re}_{x}^{1/2}}$$
(17.19)
$$\theta_{lam} = \frac{0.664x}{\text{Re}_{x}^{1/2}}$$
(17.20)





The local wall shear stress is given by the value of the velocity gradient at the wall

$$\tau_{0} = \mu \left(\frac{\partial u}{\partial y}\right)_{y=0} = 0.332 \mu R e_{x}^{1/2} \frac{U}{x}$$
(17.21)

Thus, the local wall shear stress coefficient is given as



FIG. 10-5. Local shear stress coefficients for laminar boundary layer on flat plate [3].





Drag for one side of a plate of width b is computed to be

$$D = b \int_{0}^{l} \tau_{0}(x) dx = \frac{1.328}{\operatorname{Re}_{l}^{1/2}} \rho \frac{U^{2}}{2} bl$$
 (17.24)

Using the relation

$$D = C_f \frac{\rho}{2} U^2 bl \tag{17.25}$$

We get the coefficient of surface resistance as

$$C_f = \frac{1.328}{Re_i^{1/2}} \tag{17.26}$$



