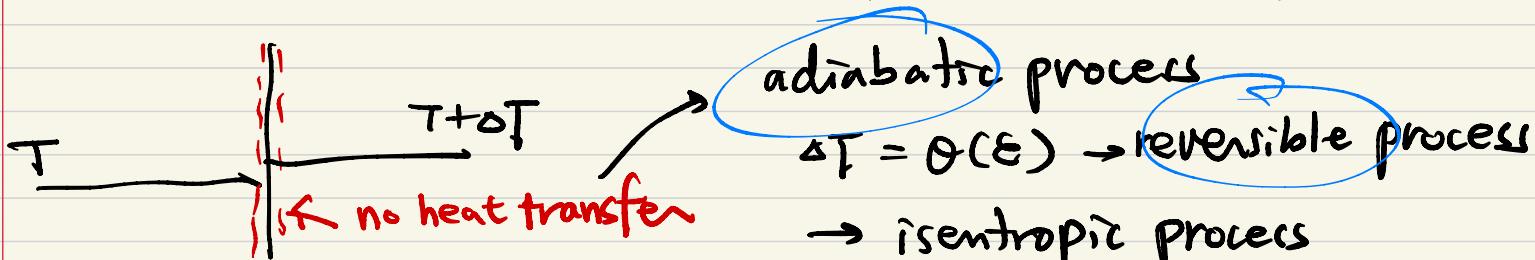


$$c^2 = \frac{\partial P}{\partial \rho} \left( 1 + \frac{\partial P}{\rho} \right)$$

$$P = P(\rho, T)$$

$$\frac{\partial P}{\partial \rho} \rightarrow 0 \quad \left[ \begin{array}{l} \rightarrow \\ \frac{\partial P}{\partial \rho} \rightarrow 0 \end{array} \right] \quad a^2 = \frac{\partial P}{\partial \rho} \Big|_{?} : \text{Speed of sound}$$

Newton (1686) obtained 'a' using the assumption of isothermal process.  $\rightarrow$  20% error. wrong!



$$\Delta S = 0: \quad P/\rho^k = \text{const} \rightarrow \ln P - k \ln \rho = \text{const}$$

$$\frac{dP}{P} - k \frac{d\rho}{\rho} = 0$$

$$\rightarrow \frac{\partial P}{\partial S} = k \frac{P}{\rho} = k \rho T = a^2 \quad \therefore a = \sqrt{kRT}$$

④ 1 atm,  $15^\circ\text{C}$ , air  $c = 340 \text{ m/s}$

water       $1490 \text{ m/s}$       ) bulk modulus  
steel       $5060 \text{ m/s}$

$$\text{water } Ma = 0.3 \rightarrow V = 0.3 \times 1500 = 450 \text{ m/s} \times$$

↳ incomp.

### 9.3 Adiabatic and Isentropic steady flow

$$Pr = \frac{\nu}{\alpha} < 1 \quad \text{air } Pr = 0.7 (0.71)$$

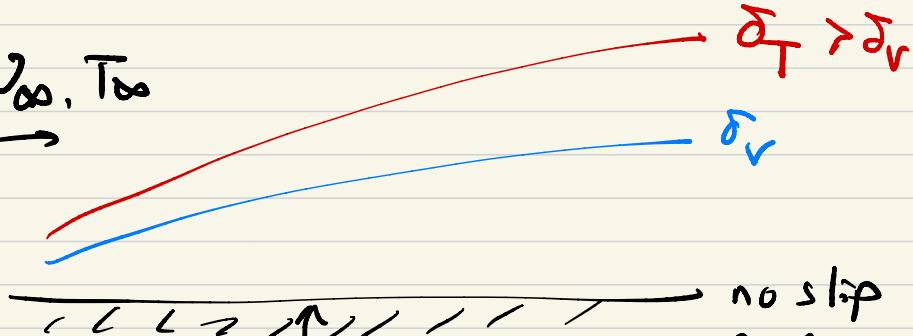
$$\rightarrow V < \alpha$$

$$\rho \frac{dV}{dx} = \dots + \cancel{\mu T^2 V}$$

$$\rho C_p \frac{\partial T}{\partial x} = \dots + \cancel{\alpha V^2 T}$$

$$V_\infty, T_\infty$$

→



$$\gamma'' = 0$$

$$T_{\text{fluid}} = T_{\text{solid}}$$

energy conservation  $B = E$ ,  $\beta = \frac{dE}{dm} = e$

$$e = \hat{u} + \frac{1}{2}v^2 + gz + e_{\text{internal}} + e_{\text{kinetic}} + e_{\text{potential}} + \dots$$

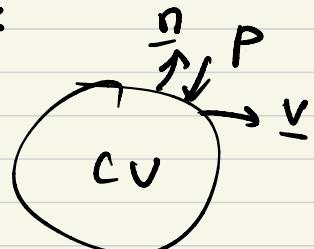
$$\Rightarrow \frac{dQ}{dt} - \frac{dW}{dt} = \frac{dE}{dt} = \frac{d}{dt} \int_{CV} e \rho dV + \int_{CS} e \rho (\underline{v} \cdot \underline{n}) dA$$

$$\begin{aligned}\dot{W} &= \frac{dW}{dt} = \dot{W}_{\text{shaft}} + \dot{W}_{\text{press}} + \dot{W}_{\text{viscous stress}} \\ &= \dot{W}_s + \dot{W}_p + \dot{W}_v\end{aligned}$$

$$\dot{W}_p = \int_{CS} p (\underline{v} \cdot \underline{n}) dA$$

$$\dot{W}_v = - \int_{CS} \underline{\sigma} \cdot \underline{v} dA \quad \underline{\sigma} : \text{stress vector on } dA$$

$$\begin{aligned}\Rightarrow \dot{Q} - \dot{W}_s - \dot{W}_v &= \frac{\partial}{\partial t} \int_{CV} e \rho dV + \int_{CS} \left( ce + \frac{P}{S} \right) \rho (\underline{v} \cdot \underline{n}) dA \\ &= \hat{u} + \frac{1}{2}v^2 + gz + \frac{P}{\rho} = h + \frac{1}{2}v^2 + gz\end{aligned}$$



1D & Steady:  $i \rightarrow i$

$$\dot{Q} - \dot{W}_C - \dot{W}_V = -m(h_i + \frac{1}{2}V_i^2 + qz_i) + m(h_2 + \frac{1}{2}V_2^2 + qz_2)$$

$$\rightarrow h_i + \frac{1}{2}V_i^2 + qz_i = h_2 + \frac{1}{2}V_2^2 + qz_2 - g + \dot{W}_S/m + \dot{W}/m$$

$\stackrel{\text{"Q/m}}{\longrightarrow}$

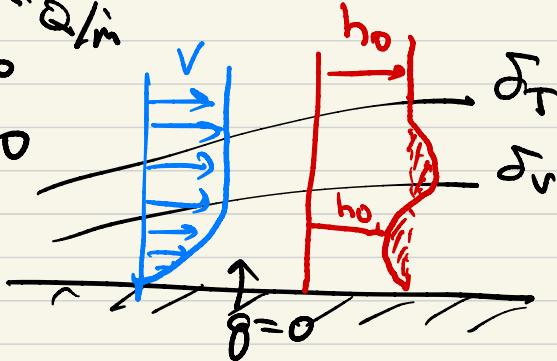
inside the bdry layer:  $g \neq 0, \dot{W}_V \neq 0$

outside " " :  $g = 0, \dot{W}_V = 0$

$$h_i + \frac{1}{2}V_i^2 + qz_i = h_2 + \frac{1}{2}V_2^2 + qz_2$$

(neglect  $q(z_2 - z_i) \approx 0$ )

$$\rightarrow h_i + \frac{1}{2}V_i^2 = h_2 + \frac{1}{2}V_2^2 = \text{const} \quad \boxed{h_0 = h + \frac{1}{2}V^2}$$



$h_0$  varies inside the thermal bdry layer,

but its average value is the same because of  
energy conservation

$\uparrow$  stagnation enthalpy

For perfect gas,  $h = c_p T \rightarrow c_p T_0 = c_p T + \frac{1}{2} V^2$

$$V_{\max} = \sqrt{2c_p T_0} = \left( 2 \frac{kR}{kT_0} T_0 \right)^{\frac{1}{2}}$$

$$c_p T_0 = c_p T + \frac{1}{2} V^2 \rightarrow \frac{T_0}{T} = 1 + \frac{V^2}{2c_p T} = 1 + \frac{(k-1)V^2}{2a^2}$$

$$\left( c_p T = \frac{kR}{k-1} T = \frac{a^2}{k-1} \right)$$

$$\rightarrow \boxed{\frac{T_0}{T} = 1 + \frac{k-1}{2} M_a^2} \quad M_a = V/a$$

adibatic

$$\boxed{\frac{a_0}{a} = \left( \frac{T_0}{T} \right)^{\frac{1}{2}} = \left( 1 + \frac{k-1}{2} M_a^2 \right)^{\frac{1}{2}}} \quad \text{adibatic}$$

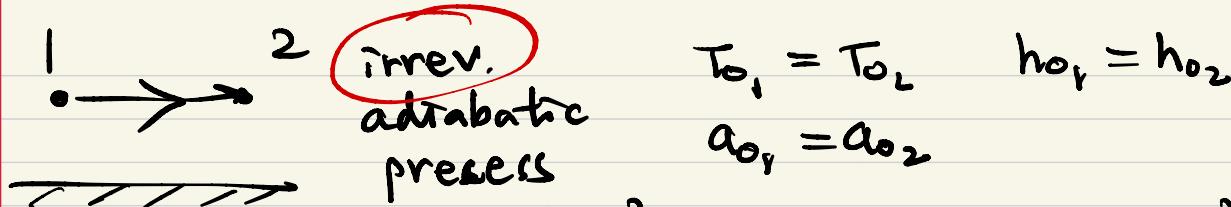
If isentropic process

$P_0$ : stag. press

$\rho_0$ : density

$$\frac{P_0}{P} = \left( \frac{T_0}{T} \right)^{\frac{k}{k-1}} = \left( 1 + \frac{k-1}{2} M_a^2 \right)^{\frac{k}{k-1}}$$

$$\frac{\rho_0}{\rho} = \left( \frac{T_0}{T} \right)^{\frac{1}{k-1}} = \left( 1 + \frac{k-1}{2} M_a^2 \right)^{\frac{1}{k-1}}$$



$$P_1 + \frac{1}{2} \rho_1 V_1^2 = P_{0_1} \neq P_{0_2} = P_2 + \frac{1}{2} \rho_2 V_2^2$$

$$P_{0_1} \neq P_{0_2}$$

isentropic process :  $T_{0_1} = T_{0_2}$ ,  $a_{0_1} = a_{0_2}$ ,  $h_{0_1} = h_{0_2}$ ,  $P_{0_1} = P_{0_2}$

$$P_{0_1} = P_{0_2}$$

(adiabatic)  $h + \frac{1}{2} V^2 = h_0 = \text{const}$

$\rightarrow dh + VdV = 0$

$Tds = dh - \frac{1}{\rho} dP = 0 \rightarrow dh = \frac{1}{\rho} dP$

↑  
isentropic

$\frac{dp}{\rho} + VdV = 0$   
 Bernoulli eq.  
 isentropic process

- $(a_0, T_0, P_0, \rho_0)$ : stagnation values
- $(a^*, T^*, p^*, \rho^*)$ : critical values  $\text{at } Ma=1$   
sonic

vary at irreversible adiabatic process

critical velocity  $V^* = a^* = \sqrt{kRT^*} = \sqrt{\frac{2k}{k+1} RT_0}$

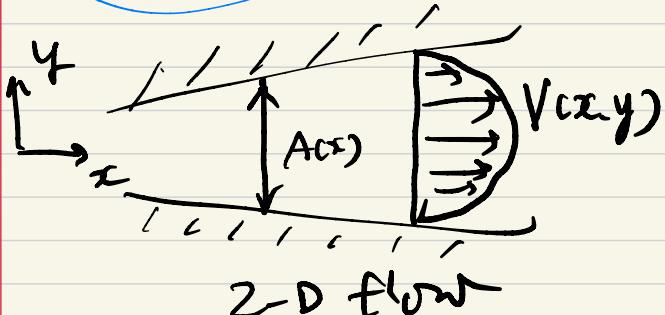
$(Ma=1)$

$$\frac{a^*}{a_0} = \left(\frac{2}{k+1}\right)^{\frac{1}{2}}, \quad \frac{P^*}{P_0} = \left(\frac{2}{k+1}\right)^{\frac{1}{k+1}}, \quad \frac{\rho^*}{\rho_0} = \left(\frac{2}{k+1}\right)^{\frac{k}{k+1}}, \quad \frac{T^*}{T_0} = \frac{2}{k+1}$$

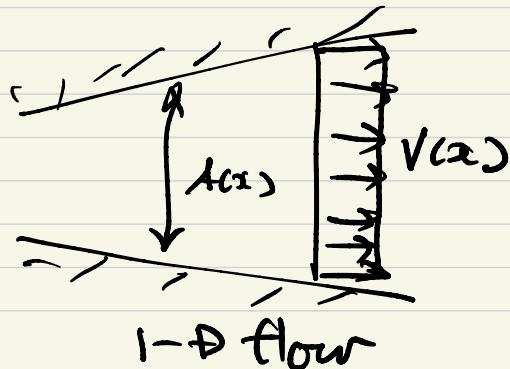
only ft of  $k$ .

9.4

Isentropic flow w/ area change



approx.



$$\begin{aligned} P_{01} &\neq P_{02} \\ P_{01} &\neq P_{02} \\ T_{01} &= T_{02} \end{aligned}$$

$$\text{cont: } p(x) V(x) A(x) = \dot{m} = \text{const}$$

$$dp \cdot VA + p dV \cdot A + pV \cdot dA = 0$$

$$\rightarrow \frac{dp}{p} + \frac{dV}{V} + \frac{dA}{A} = 0$$

$$\text{Bernoulli eq: } \frac{dp}{p} + vdv = 0$$

$$\text{speed of sound: } dp = a^2 df$$

$$\boxed{\frac{dv}{V} = \frac{da}{A} \cdot \frac{1}{M_a^2 - 1} = -\frac{dp}{pv^2}}$$

isentropic process

### • Subsonic ( $M_a < 1$ )

$$\xrightarrow{\quad \nearrow \quad} \quad dA > 0 \quad \rightarrow$$

Subsonic diffuser

$$dv < 0$$

$$dp > 0$$

$$df > 0$$

$$\xrightarrow{\quad \searrow \quad} \quad df < 0 \quad \rightarrow$$

Subsonic nozzle

$$dv > 0$$

$$dp < 0$$

$$df < 0$$

### • Supersonic ( $M_a > 1$ )

$$\xrightarrow{\quad \nearrow \quad} \quad dA > 0 \quad \rightarrow$$

Supersonic nozzle

$$dv > 0$$

$$dp < 0$$

$$df < 0$$

$$\xrightarrow{\quad \searrow \quad} \quad dA < 0 \quad \rightarrow$$

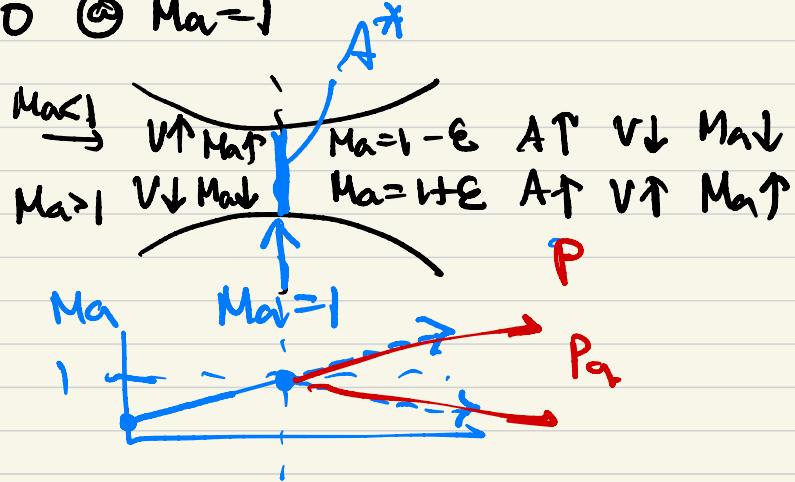
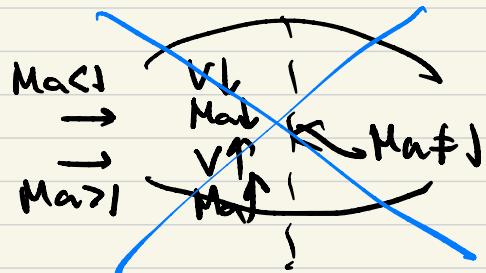
Supersonic diffuser

$$dv < 0$$

$$dp > 0$$

$$df > 0$$

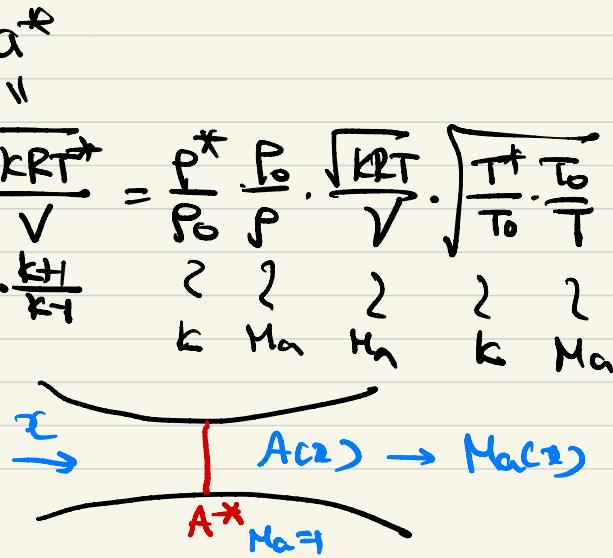
when  $Ma_1 = 1$ ?  $\rightarrow dA = 0$  @  $Ma_1 = 1$



- Perfect gas relations

$$\rho V A = \rho^* V^* A^*$$

$$\begin{aligned} \rightarrow \frac{A}{A^*} &= \frac{\rho^* V^*}{\rho V} = \frac{\rho^*}{\rho_0} \cdot \frac{P_0}{P} \cdot \frac{\sqrt{kRT^*}}{V} = \frac{\rho^*}{\rho_0} \frac{P_0}{P} \cdot \frac{\sqrt{kRT}}{V} \cdot \sqrt{\frac{T^*}{T_0} \cdot \frac{T_0}{T}} \\ &= \frac{1}{Ma_1} \left[ \frac{1 + \frac{1}{2} c_{b1} Ma_1^2}{\frac{1}{2} (k+1)} \right]^{1/2} \cdot \frac{\sqrt{k+1}}{k-1} \end{aligned}$$



For air ( $k=1.4$ )

$$\frac{A}{A_x} = \frac{1}{Ma_1} \cdot \frac{(1+0.2M_a)^2}{1.728}^{3/2}$$

$$\frac{T_0}{T} = 1 + 0.2 Ma^2$$

$$\frac{P_0}{P} = (1 + 0.2 M_a^2)^{2.5}$$

$$\frac{P_0}{P} = (1 + 0.2 M_a^2)^{3.5}$$

Given  
 $T_0$   
 $P_0$   
 $\gamma_0$

$A_x$

isentropic process

$\frac{P_0}{P_a} = f(M_a) \rightarrow M_{a,x} = \star$

$T_{a,x}$   $P_{a,x}$

$A_x$  ( $Ma=1$ )

