

One-Dimensional Separated Flow Model

- ❖ Mass and momentum equations for each phase
 - ✓ Two mass and two momentum equations
- ❖ Energy equation
 - ✓ One equation
 - One of the phases can be assumed to be saturated
 - Liquid in bulk boiling/vapor in condensation
 - ✓ Two equations
 - Conditions involving a subcooled liquid and superheated vapor
- ❖ Assumptions
 - ✓ A stratified or annular flow pattern

One-Dimensional Separated Flow Model

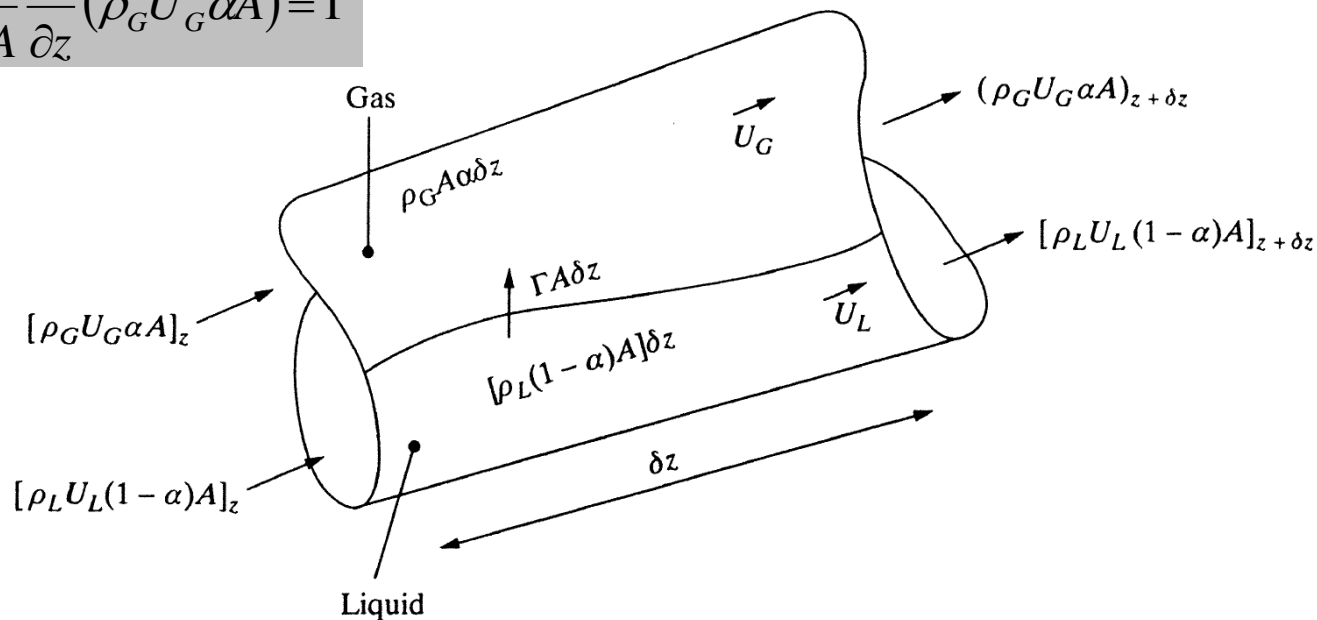
❖ Mass conservation equation

- ✓ Γ : phase change rate per unit volume
 - Positive: evaporation, negative: condensation

- ✓ For gas phase,

$$\frac{\partial \rho_G A \alpha \delta z}{\partial t} = [\rho_G U_G \alpha A]_z - [\rho_G U_G \alpha A]_{z+\delta z} + \Gamma A \delta z$$

$$\frac{\partial \rho_G \alpha}{\partial t} + \frac{1}{A} \frac{\partial}{\partial z} (\rho_G U_G \alpha A) = \Gamma$$



One-Dimensional Separated Flow Model

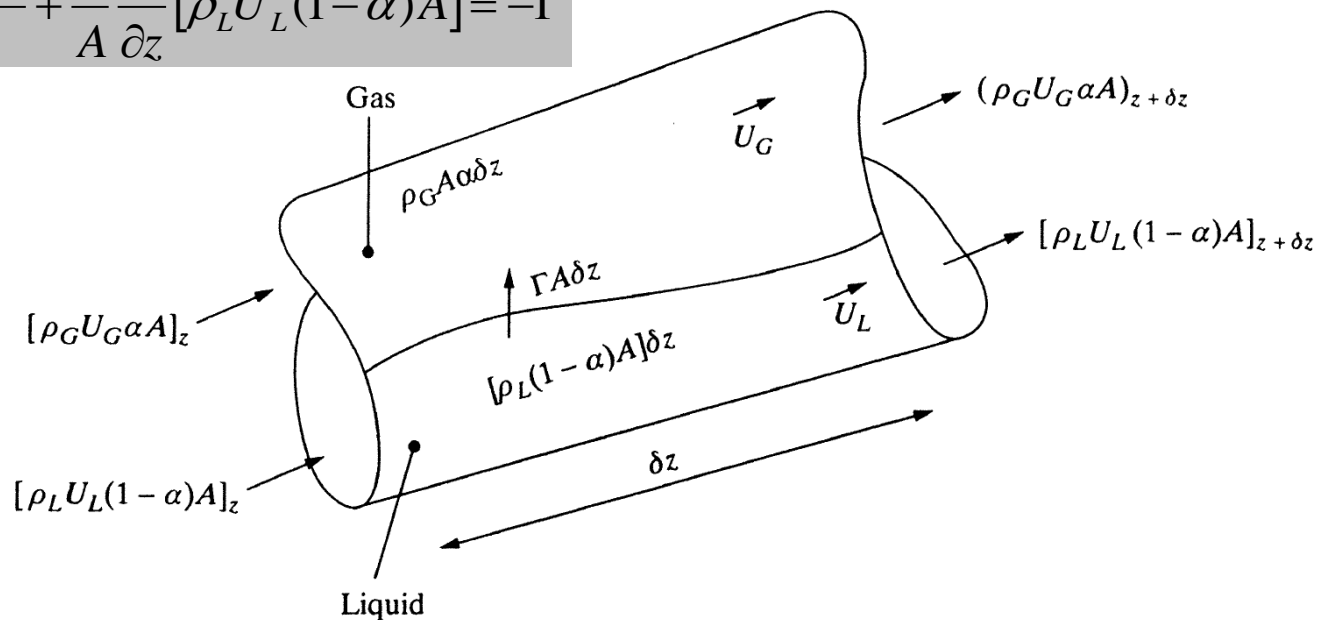
❖ Mass conservation equation

- ✓ Γ : phase change rate per unit volume
 - Positive: evaporation, negative: condensation

- ✓ For liquid phase,

$$\frac{\partial \rho_L A(1-\alpha) \delta z}{\partial t} = [\rho_L U_L (1-\alpha) A]_z - [\rho_L U_L (1-\alpha) A]_{z+\delta z} - \Gamma A \delta z$$

$$\frac{\partial \rho_L (1-\alpha)}{\partial t} + \frac{1}{A} \frac{\partial}{\partial z} [\rho_L U_L (1-\alpha) A] = -\Gamma$$



One-Dimensional Separated Flow Model

❖ Mass conservation equation

- ✓ Γ : phase change rate per unit volume
 - Positive: evaporation, negative: condensation

$$\Gamma = m_I'' a_I'' = \frac{[kg] [m^2]}{[m^2 \cdot s] [m^3]} = \frac{[kg]}{[m^3 \cdot s]} = m_I'' \frac{p_I \delta z}{A \delta z} = \frac{m_I'' p_I}{A}$$

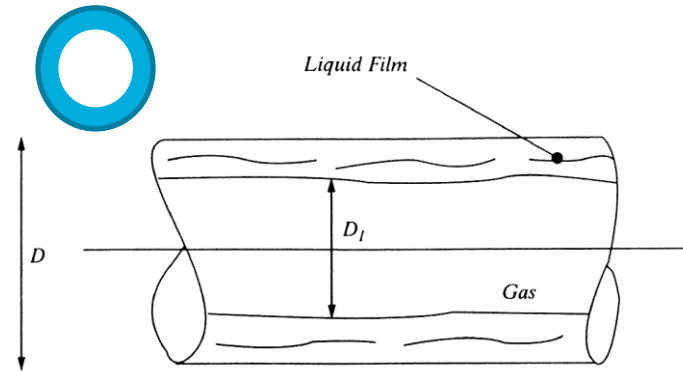
- a_I'' : interfacial area concentration (IAC), interfacial surface area per unit volume
- p_I : interfacial perimeter, interfacial surface area per unit channel length

✓ Interfacial perimeter

- For ideal annular flow in a pipe

$$D_I / D = \sqrt{\alpha} \quad ; \quad p_I = \pi D \sqrt{\alpha} \quad ; \quad \Gamma = \frac{4m_I'' \sqrt{\alpha}}{D}$$

$$p_I = \pi D_I$$



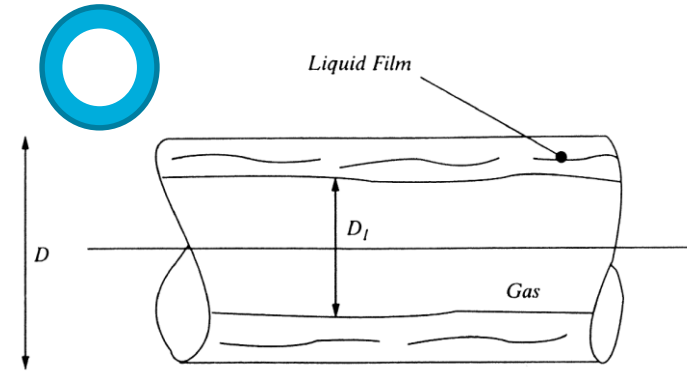
$$\alpha = \frac{A_g}{A} = \frac{\pi D_I^2 / 4}{\pi D^2 / 4}$$

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❖ Mass conservation equation

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$$\Gamma = m_I'' a_I'' = \frac{[kg] [m^2]}{[m^2 \cdot s] [m^3]} = \frac{[kg]}{[m^3 \cdot s]} = m_I'' \frac{p_I \delta z}{A \delta z} = \frac{m_I'' p_I}{A}$$



- a_I'' : interfacial area concentration (IAC), interfacial surface area per unit volume
- p_I : interfacial perimeter, interfacial surface area per unit channel length

✓ Interfacial perimeter

- For ideal bubbly flow

$$p_I = \frac{N_B \pi d_B^2 V}{\delta z} = N_B \pi d_B^2 A$$

Bubble number density (#/m³)

Number of bubbles in a unit volume

$$N_B \left(\frac{\pi d_B^3}{6} \right) = \frac{\#}{V} \left(\frac{\pi d_B^3}{6} \right) = \frac{V_{bubble}}{V} = \alpha$$

$$N_B = \frac{\#}{V}$$

$$N_B = \frac{\alpha}{\left(\frac{\pi d_B^3}{6} \right)}$$

One-Dimensional Separated Flow Model

❖ Interfacial area concentration (IAC)

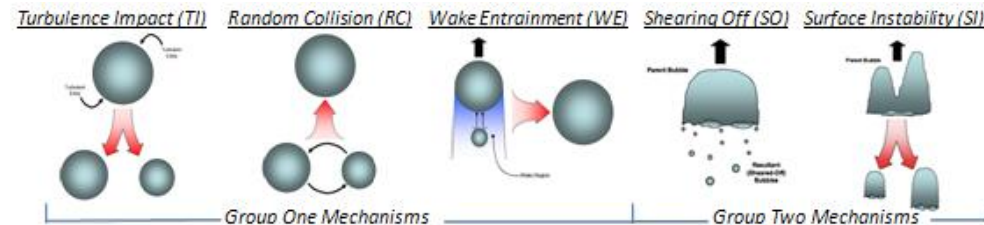
✓ a''_I : interfacial surface area per unit volume

$$\Gamma = m''_I a''_I$$

- In two-fluid model
 - Mass transfer, momentum transfer, energy transfer
 $\propto IAC$
 - IAC varies depending on flow regime
 - Large uncertainty in developing flows

❖ Interfacial area transport equation

- ✓ Transport equation of IAC
- ✓ Dynamic flow regime map?
- ✓ TRACE, CATHARE, SPACE
- ✓ Can replace the conventional flow regime map?



One-Dimensional Separated Flow Model

❖ Momentum conservation equation

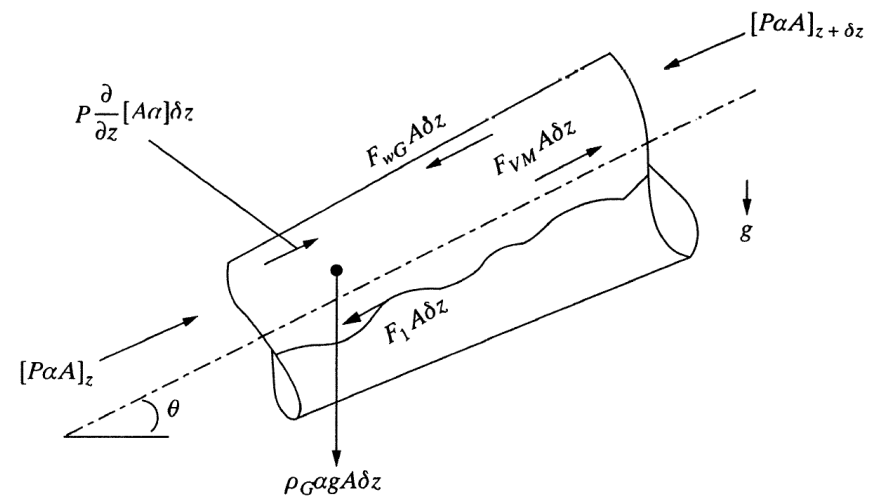
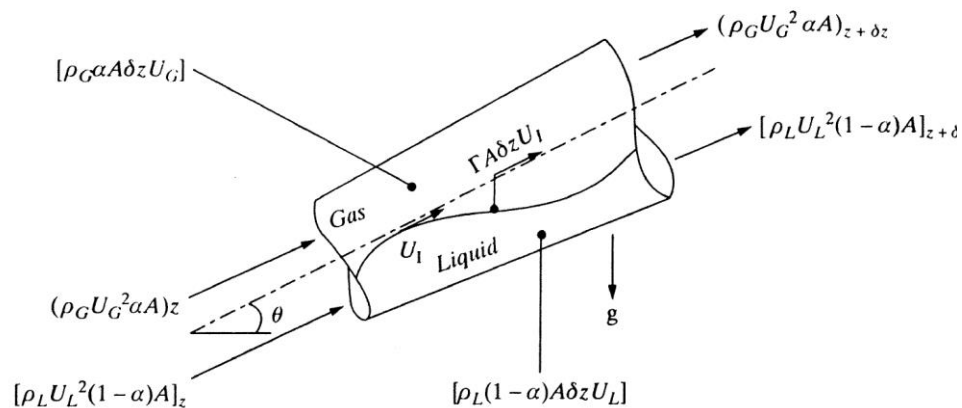
- ✓ U_I : interfacial velocity, the axial velocity at the interphase
- ✓ For gas phase

$$\frac{\partial \rho_G U_G \alpha A \delta z}{\partial t} = (\rho_G U_G^2 \alpha A)_z - (\rho_G U_G^2 \alpha A)_{z+\delta z} + (P \alpha A)_z - (P \alpha A)_{z+\delta z} + \Gamma U_I$$

$$- A \delta z F_{wG} + P \frac{\partial}{\partial z} [A \alpha] \delta z - A \delta z \rho_G \alpha g \sin \theta - F_I A \delta z + F_{VM} A \delta z$$

$$\frac{\partial \rho_G U_G \alpha}{\partial t} + \frac{1}{A} \frac{\partial}{\partial z} (\rho_G U_G^2 \alpha A) - \Gamma U_I = -\alpha \frac{\partial P}{\partial z} - \rho_G \alpha g \sin \theta - F_{wG} - F_I + F_{VM}$$

$$F_I = f(U_G - U_L)$$



One-Dimensional Separated Flow Model

❖ Momentum conservation equation

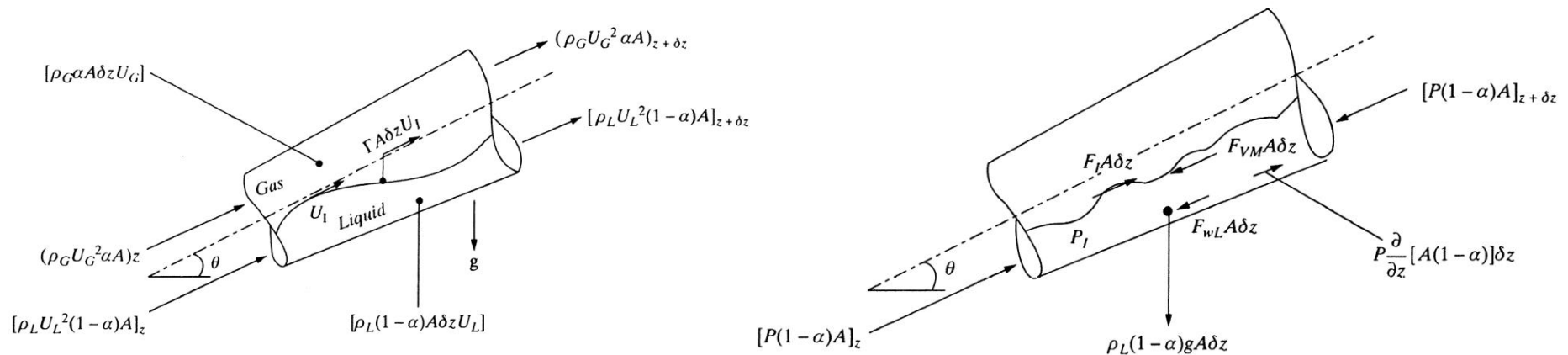
- ✓ U_I : interfacial velocity, the axial velocity at the interphase
- ✓ For liquid phase

$$F_I = f(U_G - U_L)$$

$$\frac{\partial \rho_L U_L (1-\alpha) A \delta z}{\partial t} = [\rho_L U_L^2 (1-\alpha) A]_z - [\rho_L U_L^2 (1-\alpha) A]_{z+\delta z} + [P(1-\alpha) A]_z - [P(1-\alpha) A]_{z+\delta z} - \Gamma U_I$$

$$- A \delta z F_{wL} + P \frac{\partial}{\partial z} [A(1-\alpha)] \delta z - A \delta z \rho_L (1-\alpha) g \sin \theta + F_I A \delta z - F_{VM} A \delta z$$

$$\frac{\partial \rho_L U_L (1-\alpha)}{\partial t} + \frac{1}{A} \frac{\partial}{\partial z} [\rho_L U_L^2 (1-\alpha) A] + \Gamma U_I = -(1-\alpha) \frac{\partial P}{\partial z} - \rho_L (1-\alpha) g \sin \theta - F_{wL} + F_I - F_{VM}$$



One-Dimensional Separated Flow Model

❖ Momentum conservation equation

✓ Interfacial momentum transfer terms

$$\frac{\partial \rho_G U_G \alpha}{\partial t} + \frac{1}{A} \frac{\partial}{\partial z} (\rho_G U_G^2 \alpha A) - \Gamma U_I = -\alpha \frac{\partial P}{\partial z} - \rho_G \alpha g \sin \theta - F_{wG} - F_I + F_{VM}$$

$$\frac{\partial \rho_L U_L (1-\alpha)}{\partial t} + \frac{1}{A} \frac{\partial}{\partial z} [\rho_L U_L^2 (1-\alpha) A] + \Gamma U_I = -(1-\alpha) \frac{\partial P}{\partial z} - \rho_L (1-\alpha) g \sin \theta - F_{wL} + F_I - F_{VM}$$

One-Dimensional Separated Flow Model

❖ Momentum conservation equation

✓ Closure relations

- Interface velocity

$$U_I = \frac{1}{2}(U_L + U_G) \quad \text{or} \quad U_I = U_{donor} \quad U_{donor} = \begin{cases} U_L & \text{for evaporation} \\ U_G & \text{for condensation} \end{cases}$$

- Interfacial friction force (for stratified or annular flow)

$$F_I = \tau_I a_I'' \quad \tau_I = f_I \frac{1}{2} \rho_G |U_G - U_L| (U_G - U_L) \quad f_I = 0.005[1 + 75(1 - \alpha)]$$

- Interfacial drag force (for bubbly flow)

$$F_I = F_D N \quad N = \alpha / (\pi d_B^3 / 6) \quad F_D = C_D \rho_L \frac{\pi d_B^2}{4} \frac{1}{2} |U_G - U_L| (U_G - U_L)$$

Bubble number density (#/m³)

$$N(\pi d_B^3 / 6) = \frac{V_{bubble}}{V} = \alpha \quad N = \frac{\alpha}{(\pi d_B^3 / 6)}$$

One-Dimensional Separated Flow Model

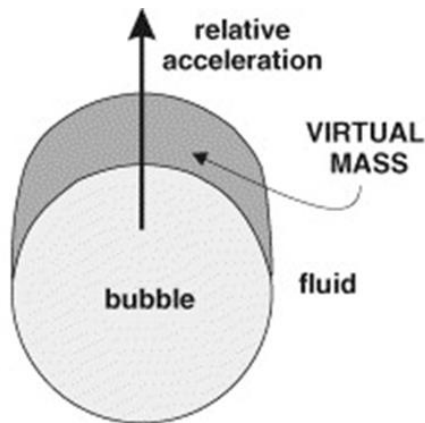
❖ Momentum conservation equation

✓ Closure relations

- Interfacial drag/friction force: caused by velocity difference
- Virtual mass (added mass) force term: caused by acceleration difference

$$F_{VM} = -C_{VM} \left[\frac{\partial U_G}{\partial t} + U_G \frac{\partial U_G}{\partial z} - \frac{\partial U_L}{\partial t} - U_L \frac{\partial U_L}{\partial z} \right]$$

$$C_{VM} = C' \alpha (1 - \alpha) \bar{\rho} \quad \bar{\rho} = \alpha \rho_G + (1 - \alpha) \rho_L \quad C' \approx 1 \quad \text{Watanabe et al. (1990)}$$



An accelerating or decelerating body must move some volume of surrounding fluid as it moves through it. Added mass is a common issue because the object and surrounding fluid cannot occupy the same physical space simultaneously.

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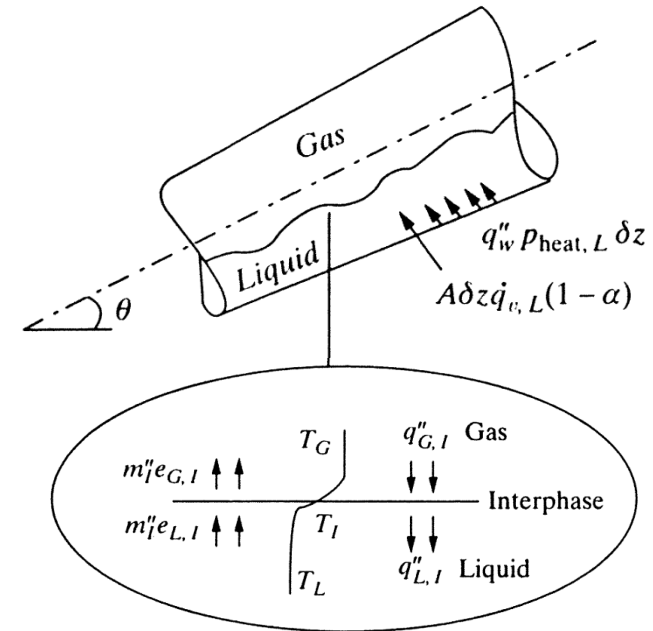
❖ Energy conservation equation

- ✓ T_I : Temperature at the interphase
- ✓ q''_{LI} , q''_{GI} : heat fluxes between liquid and gas and the interphase
- ✓ For gas phase,

$$\frac{\partial}{\partial t} \left[\rho_G \alpha \left(e_G - \frac{P}{\rho_G} \right) A \delta z \right] = (\rho_G e_G U_G \alpha A)_z - (\rho_G e_G U_G \alpha A)_{z+\delta z}$$

$$+ \Gamma e_{GI} A \delta z - P \frac{\partial \alpha}{\partial t} A \delta z$$

$$+ p_{heat,G} \dot{q}_w'' \delta z - p_I \dot{q}_{GI}'' \delta z + \dot{q}_{v,G} \alpha A \delta z - (F_I - F_{VM}) U_I A \delta z$$



$$\frac{\partial}{\partial t} \left[\rho_G \alpha \left(e_G - \frac{P}{\rho_G} \right) \right] + \frac{1}{A} \frac{\partial}{\partial z} [\rho_G e_G U_G \alpha A] - \Gamma e_{GI} + P \frac{\partial \alpha}{\partial t}$$

$$- \frac{p_{heat,G} \dot{q}_w''}{A} + \frac{p_I \dot{q}_{GI}''}{A} - \dot{q}_{v,G} \alpha + (F_I - F_{VM}) U_I = 0$$

One-Dimensional Separated Flow Model

❖ Energy conservation equation

✓ For gas phase,

$$\frac{\partial}{\partial t} \left[\rho_G \alpha \left(e_G - \frac{P}{\rho_G} \right) \right] + \frac{1}{A} \frac{\partial}{\partial z} [\rho_G e_G U_G \alpha A] - \Gamma e_{GI} + P \frac{\partial \alpha}{\partial t} - \frac{p_{heat,G} \dot{q}_w''}{A} + \frac{p_I \dot{q}_{GI}''}{A} - \dot{q}_{v,G} \alpha + (F_I - F_{VM}) U_I = 0$$

✓ For liquid phase,

$$\frac{\partial}{\partial t} \left[\rho_L (1 - \alpha) \left(e_L - \frac{P}{\rho_L} \right) \right] + \frac{1}{A} \frac{\partial}{\partial z} [\rho_L e_L U_L (1 - \alpha) A] + \Gamma e_{LI} - P \frac{\partial \alpha}{\partial t} - \frac{p_{heat,L} \dot{q}_w''}{A} - \frac{p_I \dot{q}_{LI}''}{A} - \dot{q}_{v,L} (1 - \alpha) - (F_I - F_{VM}) U_I = 0$$

One-Dimensional Separated Flow Model

❖ Energy conservation equation

✓ Closure relations

- T_I : Temperature at the interphase

$$T_I = T_{sat}(P)$$

- Energy balance at the interphase

$$\dot{m}_I'' \left(h_L + \frac{1}{2} U_{LI}^2 \right)_I - q_{LI}'' = \dot{m}_I'' \left(h_G + \frac{1}{2} U_{GI}^2 \right)_I - q_{GI}''$$

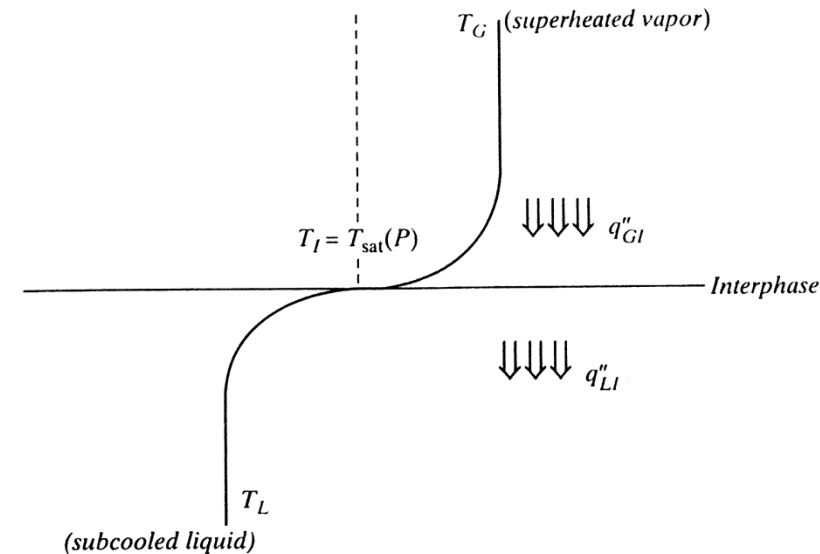
- Interphase sensible heat transfer

$$q_{GI}'' = H_{GI} (T_G - T_I) \quad q_{LI}'' = H_{LI} (T_I - T_L)$$

H_{kl} Interfacial heat transfer coefficient for k-phase

- Then, the phase changer rate reduces to

$$\dot{m}_I'' = \frac{H_{GI} (T_G - T_I) - H_{LI} (T_I - T_L)}{h_{fg}}$$



One-Dimensional Separated Flow Model

❖ Summary

- ✓ Unknowns: U_L , U_G , h_L , h_G , P , and α
- ✓ Six conservation equations \Rightarrow six equation model (two-fluid model)
 - Nine equation model \Rightarrow two-phase three-field model

$$\begin{aligned} \frac{\partial \rho_G \alpha}{\partial t} + \frac{1}{A} \frac{\partial}{\partial z} (\rho_G U_G \alpha A) &= \Gamma && \text{Three equations for gas} \\ \frac{\partial \rho_G U_G \alpha}{\partial t} + \frac{1}{A} \frac{\partial}{\partial z} (\rho_G U_G^2 \alpha A) - \Gamma U_I &= -\alpha \frac{\partial P}{\partial z} - \rho_G \alpha g \sin \theta - F_{wG} - F_I + F_{VM} \\ \frac{\partial}{\partial t} \left[\rho_G \alpha \left(e_G - \frac{P}{\rho_G} \right) \right] + \frac{1}{A} \frac{\partial}{\partial z} [\rho_G e_G U_G \alpha A] - \Gamma e_{GI} &+ P \frac{\partial \alpha}{\partial t} \\ - \frac{P_{heat,G} \dot{q}_w''}{A} + \frac{P_I \dot{q}_{GI}''}{A} - \dot{q}_{v,G} \alpha + (F_I - F_{VM}) U_I &= 0 \end{aligned}$$