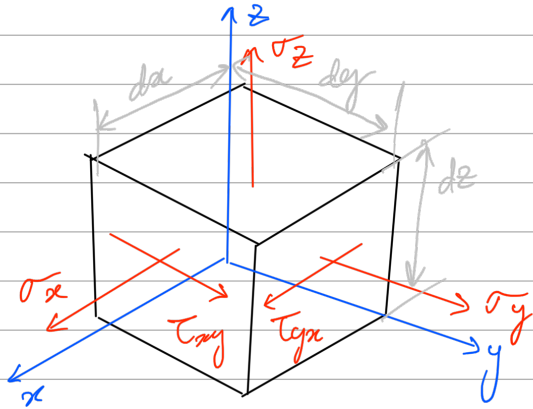


10.6. Material property relationships
(stress-strain relationship
or constitutive equation)



$$\epsilon_x = \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} - \nu \frac{\sigma_z}{E}$$

$$\epsilon_y = -\nu \frac{\sigma_x}{E} + \frac{\sigma_y}{E} - \nu \frac{\sigma_z}{E}$$

$$\epsilon_z = -\nu \frac{\sigma_x}{E} - \nu \frac{\sigma_y}{E} + \frac{\sigma_z}{E}$$

$$\gamma_{xy} = \frac{\tau_{xy}}{G}$$

$$\gamma_{yz} = \frac{\tau_{yz}}{G}$$

$$\gamma_{xz} = \frac{\tau_{xz}}{G}$$

$$\begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \epsilon_z \\ \gamma_{xy} \\ \gamma_{yz} \\ \gamma_{xz} \end{Bmatrix} = \begin{bmatrix} 1/E & -\nu/E & -\nu/E \\ -\nu/E & 1/E & -\nu/E \\ -\nu/E & -\nu/E & 1/E \\ & & & 1/G \\ & & & & 1/G \\ & & & & & 1/G \end{bmatrix} \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \sigma_z \\ \tau_{xy} \\ \tau_{yz} \\ \tau_{xz} \end{Bmatrix}$$

↳ compliance matrix

For homogeneous, isotropic material,

$$G = \frac{E}{2(1+\nu)}$$

→ prove in HW

Example 10.9. $E = 200 \text{ GPa}$, $\nu = 0.3$

strain gage \rightarrow strain state \rightarrow principal strain \rightarrow strain state?
 $(\epsilon_x, \epsilon_y, \epsilon_z)$ $(\epsilon_x, \epsilon_y, \gamma_{xy})$ (ϵ_1, ϵ_2)

$$\begin{aligned} \epsilon_1 &= \frac{\sigma_1}{E} - \nu \frac{\sigma_2}{E} & \sigma_1 &= \frac{E}{1-\nu^2} (\epsilon_1 + \nu \epsilon_2) \\ \epsilon_2 &= \frac{\sigma_2}{E} - \nu \frac{\sigma_1}{E} & \sigma_2 &= \frac{E}{1-\nu^2} (\epsilon_2 + \nu \epsilon_1) \end{aligned}$$

Using $\epsilon_1 = 272 \times 10^{-6}$, $\epsilon_2 = 33.9 \times 10^{-6}$
 $\sigma_1 = 62.0 \text{ MPa}$
 $\sigma_2 = 25.4 \text{ MPa}$

c.f. stiffness matrix

$$\begin{Bmatrix} \sigma_x \\ \sigma_y \\ \sigma_z \\ \tau_{xy} \\ \tau_{yz} \\ \tau_{xz} \end{Bmatrix} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & \nu & & & \\ \nu & 1-\nu & \nu & & & \\ \nu & \nu & 1-\nu & & & \\ & & & 1-2\nu & 0 & 0 \\ & & & 0 & 1-2\nu & 0 \\ & & & 0 & 0 & 1-2\nu \end{bmatrix} \begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \epsilon_z \\ \gamma_{xy} \\ \gamma_{yz} \\ \gamma_{xz} \end{Bmatrix}$$

strain gage \rightarrow strain state \rightarrow stress state \rightarrow principal stress
 $(\epsilon_x, \epsilon_y, \epsilon_z)$ $(\epsilon_x, \epsilon_y, \gamma_{xy})$ $(\sigma_x, \sigma_y, \tau_{xy})$ principal stress

$\epsilon_x = 60 \times 10^{-6}$, $\epsilon_y = 246 \times 10^{-6}$, $\gamma_{xy} = -449 \times 10^{-6}$

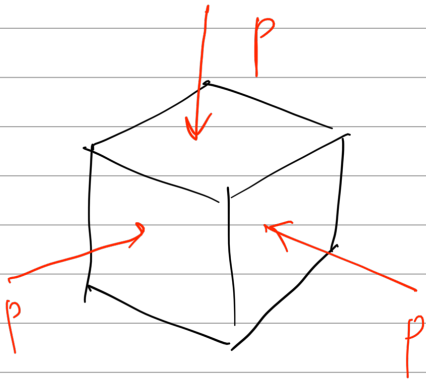
$$\begin{cases} \sigma_x = \frac{E}{1-\nu^2} (\epsilon_x + \nu \epsilon_y) \\ \sigma_y = \frac{E}{1-\nu^2} (\epsilon_y + \nu \epsilon_x) \\ \tau_{xy} = \frac{E}{1+\nu} \gamma_{xy} \end{cases} \rightarrow \begin{cases} \sigma_x = 29.4 \text{ MPa} \\ \sigma_y = 58.0 \text{ MPa} \\ \tau_{xy} = -11.46 \text{ MPa} \end{cases}$$

\rightarrow Mohr's circle $\begin{cases} \sigma_1 = 62 \text{ MPa} \\ \sigma_2 = 25.4 \text{ MPa} \end{cases}$

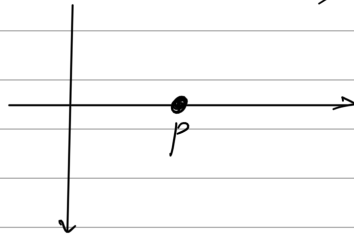
Dilatation

$$\begin{aligned} \delta V &= (1 + \epsilon_x) dx \cdot (1 + \epsilon_y) dy \cdot (1 + \epsilon_z) dz - dx dy dz \\ &\approx (\epsilon_x + \epsilon_y + \epsilon_z) dx dy dz \end{aligned}$$

dilatation $e = \frac{\delta V}{V} = \left(\frac{1-2\nu}{E} \right) \cdot (\sigma_x + \sigma_y + \sigma_z)$



Hydrostatic pressure
(no shear stress)



$$\sigma_x = \sigma_y = \sigma_z = -p$$

$$\frac{p}{e} = -\frac{E}{3(1-2\nu)} = -k \quad ; \quad \text{bulk modulus}$$

if $\nu \approx 1/3$, $k \approx E$

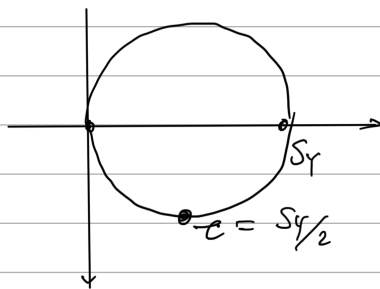
$\nu \approx 0.5$, $k \rightarrow \infty$ (upper ceiling of Poisson's ratio)

10.7. Theories of Failure

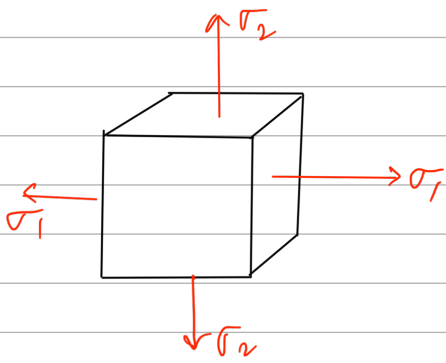
Maximum shear stress theory
(Tresca yield criterion)

: the maximum shear stresses in a multi-axial stress state reaches the value of the maximum shear stress in uni-axial tension
(For ductile materials)

- Uni-axial tension

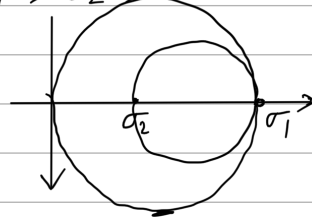


- Multi-axial stress state



(Note: principal stresses and axes)

(1) $\sigma_1 > 0, \sigma_2 > 0$ (both tension)
 if $\sigma_1 > \sigma_2 > 0$



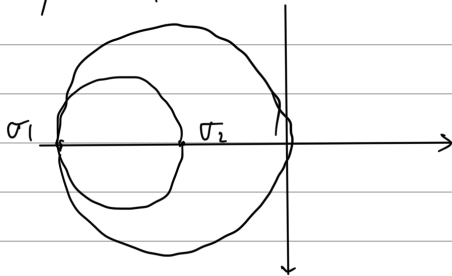
$$\frac{\sigma_1}{2} = \frac{\sigma_Y}{2}$$

if $\sigma_2 > \sigma_1 > 0$

$$\frac{\sigma_2}{2} = \frac{\sigma_Y}{2}$$

(2) $\sigma_1 < 0, \sigma_2 < 0$ (both compression)

if $\sigma_1 < \sigma_2 < 0$



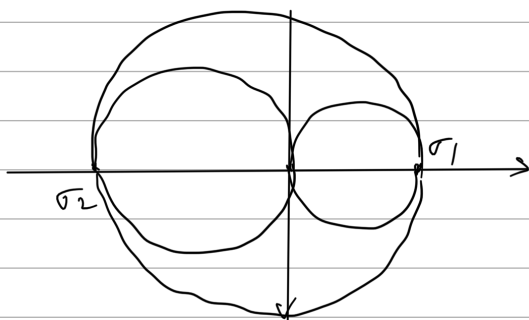
$$-\frac{\sigma_1}{2} = \frac{\sigma_Y}{2}$$

if $\sigma_2 < \sigma_1 < 0$

$$-\frac{\sigma_2}{2} = \frac{\sigma_Y}{2}$$

(3) σ_1 and σ_2 have different sign

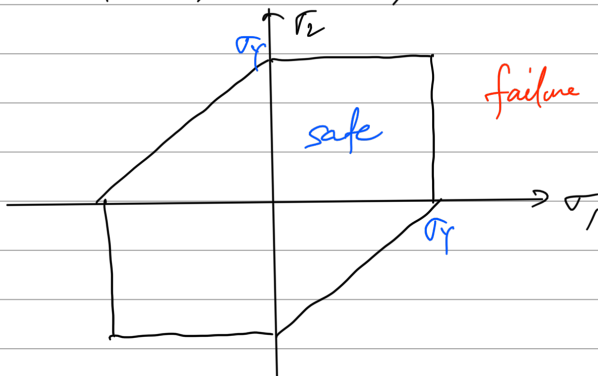
$\sigma_1 > 0, \sigma_2 < 0$



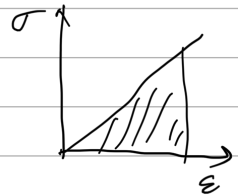
$$\frac{\sigma_1 - \sigma_2}{2} = \frac{\sigma_Y}{2}$$

$\sigma_1 < 0, \sigma_2 > 0$

$$\frac{\sigma_2 - \sigma_1}{2} = \frac{\sigma_Y}{2}$$



Maximum distortion energy theory (von Mises criterion)



• Strain energy density

$$\begin{aligned}u &= \frac{1}{2} \sigma_1 \epsilon_1 + \frac{1}{2} \sigma_2 \epsilon_2 + \frac{1}{2} \sigma_3 \epsilon_3 \\&= \frac{1}{2} \sigma_1 \left[\frac{\sigma_1}{E} - \frac{\nu}{E} (\sigma_2 + \sigma_3) \right] + \frac{1}{2} \sigma_2 \left[\frac{\sigma_2}{E} - \frac{\nu}{E} (\sigma_1 + \sigma_3) \right] \\&\quad + \frac{1}{2} \sigma_3 \left[\frac{\sigma_3}{E} - \frac{\nu}{E} (\sigma_1 + \sigma_2) \right] \\&= \frac{1}{2E} \left[\sigma_1^2 + \sigma_2^2 + \sigma_3^2 - 2\nu (\sigma_1 \sigma_2 + \sigma_2 \sigma_3 + \sigma_1 \sigma_3) \right]\end{aligned}$$

• Hydrostatic pressure ($\sigma_{avg} = \frac{1}{3}(\sigma_1 + \sigma_2 + \sigma_3)$) does not contribute to the failure, but stress deviator (i.e., $\sigma_1 - \sigma_{avg}$, $\sigma_2 - \sigma_{avg}$, $\sigma_3 - \sigma_{avg}$) does.

• Distortional energy U_d

$$\begin{aligned}U_d &= \frac{1}{2E} \left[(\sigma_1 - \sigma_{avg})^2 + (\sigma_2 - \sigma_{avg})^2 + (\sigma_3 - \sigma_{avg})^2 \right. \\&\quad \left. - 2\nu \left[(\sigma_1 - \sigma_{avg})(\sigma_2 - \sigma_{avg}) + \dots \right] \right] \\&= \frac{1+\nu}{6E} \left[(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_1 - \sigma_3)^2 \right]\end{aligned}$$

From uni-axial tension test

$$(U_d)_Y = \frac{1+\nu}{6E} \cdot 2\sigma_Y^2$$

$$\therefore \frac{1}{2} \left[(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_1 - \sigma_3)^2 \right] = \sigma_Y^2$$

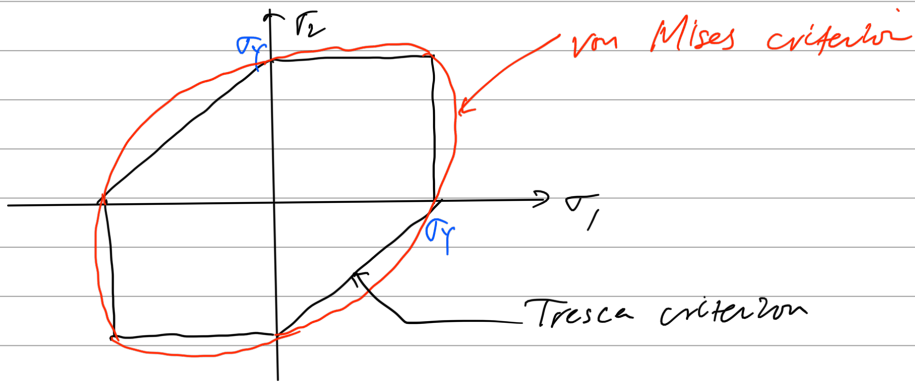
$$\sqrt{\frac{1}{2} \left[(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_1 - \sigma_3)^2 \right]} = \sigma_{vM}$$

: von Mises stress

Under plane stress condition

$$\frac{1}{2} \left(\sigma_1^2 - 2\sigma_1\sigma_2 + \sigma_2^2 + \sigma_2^2 + \sigma_1^2 \right) = \sigma_Y^2$$

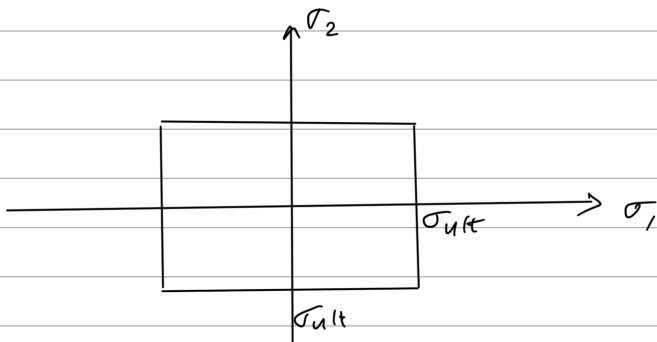
$$\rightarrow \sigma_1^2 - \sigma_1\sigma_2 + \sigma_2^2 = \sigma_Y^2$$



Maximum normal stress theory
(for brittle materials)

$$|\sigma_1| = \sigma_{ult}$$

$$|\sigma_2| = \sigma_{ult}$$



Mohr - Coulomb failure criterion

