

## Chapter 12. Deflection of beams

Loading

$M$

↓  
stresses

$$\sigma = E\varepsilon = -\frac{M}{I}y$$

↓  
strains

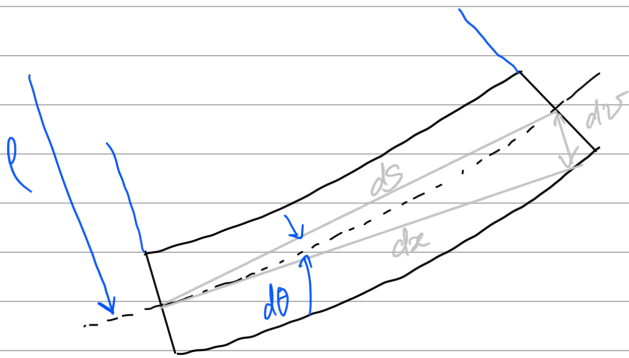
$$\varepsilon = -ky = -\frac{M}{EI}y$$

↓  
deformation

$$k = \frac{M}{EI}$$

Q. How is beam deflection related to  $k$  (or  $1/p$ )

$v$ : deflection of the centroid axis due to bending

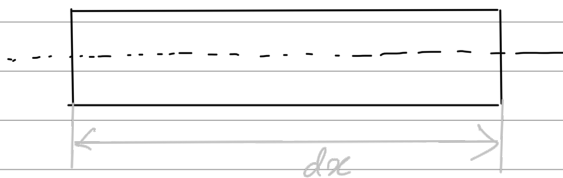


$$① \quad p \, d\theta = ds$$

$$② \quad \tan d\theta = \frac{dv}{dx}$$

$$\rightarrow d\theta = \arctan \frac{dv}{dx}$$

$$③ \quad ds^2 = dx^2 + dv^2$$



$$①, \quad d\theta, \quad ds, \quad dx, \quad dv$$

From ②

$$\begin{aligned} \frac{d\theta}{dx} &= \frac{d}{dx} \left( \arctan \frac{dv}{dx} \right) \\ &= \frac{1}{1 + (dv/dx)^2} \cdot \frac{d}{dx} \left( \frac{dv}{dx} \right) \end{aligned}$$

$$\left( \frac{d}{dx} (\arctan x) = \frac{1}{1+x^2} \right)$$

From ①

$$d\theta = \frac{1}{p} ds$$

$$\rightarrow \frac{1}{p} \frac{ds}{dx} = \frac{d^2v/dx^2}{1 + (dv/dx)^2}$$

$$\text{From ③} \quad \frac{ds^2}{dx^2} = 1 + \frac{dv^2}{dx^2} \quad \rightarrow \quad \frac{ds}{dx} = \sqrt{1 + (dv/dx)^2}$$

$$\therefore k = \frac{1}{\rho} = \frac{d^2v/dx^2}{[1 + (dv/dx)^2]^{3/2}}$$

2f  $dv/dx \approx 0$

$$k = \frac{1}{\rho} = \frac{d^2v/dx^2}{1} = \frac{M}{EI}$$

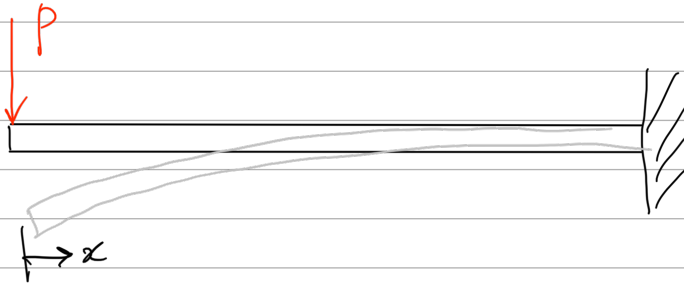
$$\therefore EI \frac{d^2v}{dx^2} = M(x)$$

$$EI \frac{d^3v}{dx^3} = V(x) \quad \left( \frac{dM}{dx} = V \right)$$

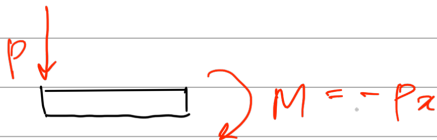
$$EI \frac{d^4v}{dx^4} = w(x) \quad \left( \frac{dV}{dx} = w \right)$$

$EI$  flexural rigidity

Example. 12.2.



F.B.D.



$$EI \frac{d^2v}{dx^2} = -Px$$

$$EI \frac{dv}{dx} = -\frac{Px^2}{2} + C_1$$

B.C.  $\left. \frac{dv}{dx} \right|_{x=L} = -\frac{PL^2}{2} + C_1 = 0$

$$EI \frac{dv}{dx} = \frac{P}{2} (L^2 - x^2)$$

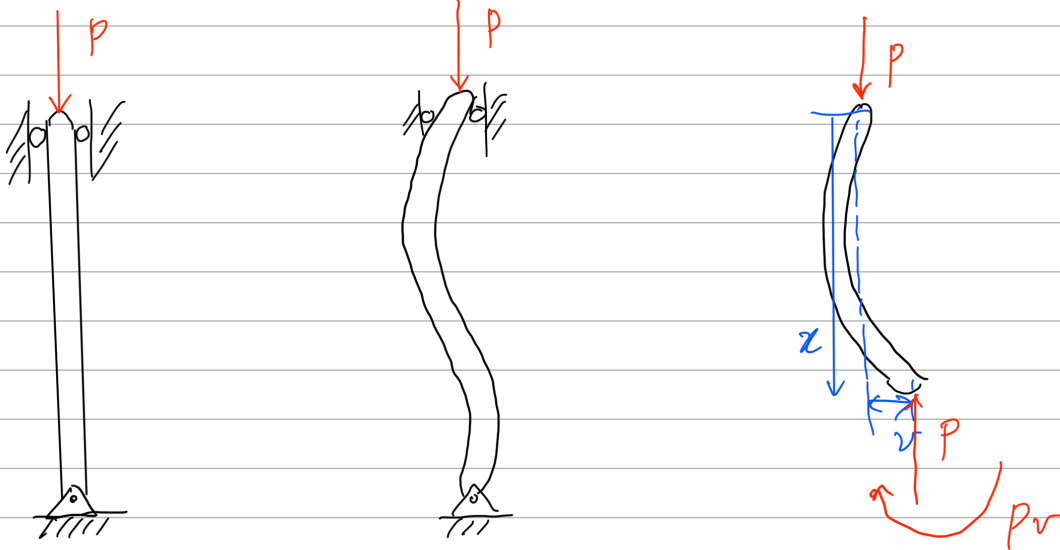
$$EI v = \frac{P}{2} \left( L^2x - \frac{x^3}{3} \right) + C_2$$

B.C.  $v|_{x=L} = \frac{P}{2} \cdot \frac{2}{3} L^3 + C_2 = 0$

## Chapter 13. Buckling of Columns

- Material failure  
(e.g., Tresca, von Mises criteria)
- Structural failure  
: buckling (sudden collapse due to the loss of stability)

### Euler buckling



$$EI \frac{d^2v}{dx^2} = -Pv$$

$$\frac{d^2v}{dx^2} + \frac{P}{EI} v = 0$$

$$v = C_1 \sin \sqrt{\frac{P}{EI}} x + C_2 \cos \sqrt{\frac{P}{EI}} x$$

Boundary condition

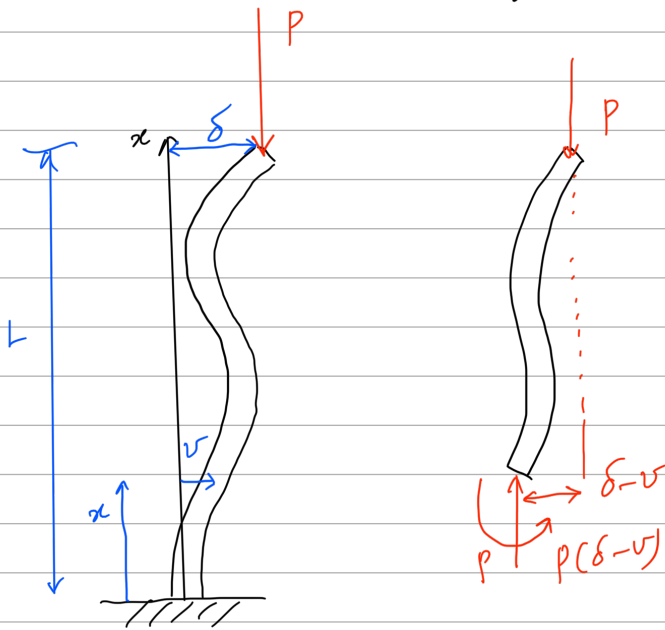
$$v(x=0) = C_2 = 0$$

$$v(x=L) = C_1 \sin \sqrt{\frac{P}{EI}} \cdot L = 0$$

$$\therefore \sqrt{\frac{P}{EI}} \cdot L = n\pi \rightarrow P = \frac{n^2 \pi^2 EI}{L^2}$$

$$P_{cr} = \frac{\pi^2 EI}{L^2} \quad : \text{ Euler load}$$

### 13.3. Columns having various types of supports



$$EI \frac{d^2 v}{dx^2} = P(\delta - v)$$

$$\frac{d^2 v}{dx^2} + \frac{P}{EI} v = \frac{P}{EI} \delta$$

$$\therefore v = C_1 \sin \sqrt{\frac{P}{EI}} x + C_2 \cos \sqrt{\frac{P}{EI}} x + \delta$$

$$\text{B.C. } v(x=0) = C_2 + \delta = 0 \quad \rightarrow \quad C_2 = -\delta$$

$$\left. \frac{dv}{dx} \right|_{x=0} = C_1 \sqrt{\frac{P}{EI}} = 0 \quad \rightarrow \quad C_1 = 0$$

$$v = \delta \left[ 1 - \cos \sqrt{\frac{P}{EI}} x \right]$$

$$v(x=L) = \delta \left[ 1 - \cos \sqrt{\frac{P}{EI}} L \right] = \delta$$

$$\therefore \sqrt{\frac{P}{EI}} L = \frac{n\pi}{2}, \quad n = 1, 3, 5, \dots$$

$$P_{cr} = \frac{\pi^2 EI}{4L^2}$$