Advanced Thermodynamics (M2794.007900)

Chapter 5

Consequences of the First Law

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A.1 Partial Derivatives

Consider a function of three variables, f(x, y, z) = 0

Since only two variables are independent, we can write

$$x = x(y, z), y = y(x, z)$$

Then
$$dx = (\frac{\partial x}{\partial y})_z dy + (\frac{\partial x}{\partial z})_y dz$$
, and
 $dy = (\frac{\partial y}{\partial x})_z dx + (\frac{\partial y}{\partial z})_x dz$

We obtain,

$$dx = \left(\frac{\partial x}{\partial y}\right)_z \left\{ \left(\frac{\partial y}{\partial x}\right)_z dx + \left(\frac{\partial y}{\partial z}\right)_x dz \right\} + \left(\frac{\partial x}{\partial z}\right)_y dz$$



A.1 Partial Derivatives

$$dx = \left(\frac{\partial x}{\partial y}\right)_{z} \left\{ \left(\frac{\partial y}{\partial x}\right)_{z} dx + \left(\frac{\partial y}{\partial z}\right)_{x} dz \right\} + \left(\frac{\partial x}{\partial z}\right)_{y} dz$$
$$= \left(\frac{\partial x}{\partial y}\right)_{z} \left(\frac{\partial y}{\partial x}\right)_{z} dx + \left[\left(\frac{\partial x}{\partial y}\right)_{z} \left(\frac{\partial y}{\partial z}\right)_{x} + \left(\frac{\partial x}{\partial z}\right)_{y} \right] dz$$

If dz = 0 and $dx \neq 0$,

$$\left(\frac{\partial x}{\partial y}\right)_z = \frac{1}{\left(\frac{\partial y}{\partial x}\right)_z}$$
. This expression is known as the *reciprocal relation*.

If dx = 0 and $dz \neq 0$,

$$\left(\frac{\partial x}{\partial y}\right)_{z} \left(\frac{\partial y}{\partial z}\right)_{x} = -\left(\frac{\partial x}{\partial z}\right)_{y}$$



A.1 Partial Derivatives

$$\begin{pmatrix} \frac{\partial x}{\partial y} \\ \frac{\partial z}{\partial z} \end{pmatrix}_{x} = - \begin{pmatrix} \frac{\partial x}{\partial z} \\ \frac{\partial z}{\partial z} \end{pmatrix}_{y}$$
(previous slide)
$$\begin{pmatrix} \frac{\partial x}{\partial z} \\ \frac{\partial z}{\partial z} \end{pmatrix}_{y} = \frac{1}{\begin{pmatrix} \frac{\partial z}{\partial x} \\ \frac{\partial z}{\partial y} \end{pmatrix}_{y}}$$
(using reciprocal relation)

Substituting these equations yield,

$$\left(\frac{\partial x}{\partial y}\right)_{z}\left(\frac{\partial y}{\partial z}\right)_{x}\left(\frac{\partial z}{\partial x}\right)_{y} = -1.$$
 The **cyclical rule**, or **cyclical relation**.



Consider a function u of three variables x, y, z can be written as a function of only two variables and those two variables are independent.

$$u = u(x, y)$$

Alternatively,

$$x = x(u, y)$$

Then

$$\mathrm{d}x = \left(\frac{\partial x}{\partial u}\right)_{\mathcal{Y}} du + \left(\frac{\partial x}{\partial y}\right)_{\mathcal{U}} dy.$$

If we divide the equation by dz while holding u constant,

$$\left(\frac{\partial x}{\partial z}\right)_{u} = \left(\frac{\partial x}{\partial y}\right)_{u} \left(\frac{\partial y}{\partial z}\right)_{u}$$
. The **chain rule of differentiation**.



In general, u = u(T, v)

Using the cyclical and reciprocal relations,

$$\left(\frac{\partial T}{\partial v}\right)_{u} = -\frac{\left(\frac{\partial u}{\partial v}\right)_{T}}{\left(\frac{\partial u}{\partial T}\right)_{v}}$$

For a reversible process, $c_v = (\frac{\partial u}{\partial T})_v$

$$\div \ (\frac{\partial u}{\partial v})_T = -c_v \ (\frac{\partial T}{\partial v})_u$$

Then how can we keep u constant during the expansion?



 $du = \delta q - \delta w \Rightarrow \text{free expansion}$ = 0 = 0 (adiabatic) (no work)



thermal insulation

$$T_1 = T_0 + \int_{v_0}^{v_1} (\frac{\partial T}{\partial v})_u dv, \quad \eta \equiv (\frac{\partial T}{\partial v})_u$$
: Joule's coefficient

From Joule's experimental result,

$$|\eta| = \left| \left(\frac{\partial T}{\partial v} \right)_u \right| < 0.001 \text{ K kilomole m}^{-3}$$



For a Van der Waals gas, (Problem 5-3)

$$\eta = -\frac{a}{v^2 c_v}$$

For an ideal gas,

by using the equation du = Tds - Pdv,

$$\left(\frac{\partial u}{\partial v}\right)_T = T \left(\frac{\partial s}{\partial v}\right)_T - P = T \left(\frac{\partial P}{\partial T}\right)_v - P = T \left[\frac{\partial}{\partial T} \left(\frac{RT}{v}\right)\right]_v - P$$
$$= \frac{RT}{v} - P = 0$$
Then $u = u(T)$



For a real gas,

by using the equation
$$dq = \left(\frac{\partial u}{\partial T}\right)_v dT + \left[\left(\frac{\partial u}{\partial v}\right)_T + P\right] dv$$
 divide by the

temperature T,

$$\frac{dq}{T} = \frac{1}{T} \left(\frac{\partial u}{\partial T} \right)_{v} dT + \frac{1}{T} \left[\left(\frac{\partial u}{\partial v} \right)_{T} + P \right] dv$$
$$\frac{\partial}{\partial v} \left(\frac{1}{T} \frac{\partial u}{\partial T} \right) = \frac{\partial}{\partial T} \left[\frac{1}{T} \left(\frac{\partial u}{\partial v} + P \right) \right]$$
$$\frac{1}{T} \frac{\partial^{2} u}{\partial v \partial T} = -\frac{1}{T^{2}} \left(\frac{\partial u}{\partial v} + P \right) + \frac{1}{T} \frac{\partial^{2} u}{\partial v \partial T} + \frac{1}{T} \frac{\partial P}{\partial T}$$
$$\left(\frac{\partial u}{\partial v} \right)_{T} = T \left(\frac{\partial P}{\partial T} \right)_{v} - P$$



Since the process takes place in an insulated cylinder,



 $\delta q = 0$

Specific work done in forcing the gas through the plug, $w_1 = \int_{v_1}^0 P_1 dv = -P_1 v_1$

Specific work done by the gas in the expansion, $w_2 = \int_0^{v_2} P_2 dv = P_2 v_2$



The total work, $w = w_1 + w_2 = P_2 v_2 - P_1 v_1 = u_1 - u_2$

Thus, a throttling process occurs at constant enthalpy.



Joule-Thomson coefficient



$$\mu_{JT} \equiv (\frac{\partial T}{\partial P})_h$$

the point where $\mu_{IT} = 0$ is called inversion point.

from h = h(T, P), $dh = \left(\frac{\partial h}{\partial T}\right)_P dT + \left(\frac{\partial h}{\partial P}\right)_T dP$ $\mu_{JT} \equiv \left(\frac{\partial T}{\partial P}\right)_h = \left[\frac{T_2 - T_1}{P_2 - P_1}\right]_h$ $T_2 = T_1 - \mu(P_2 - P_1)$ The gas is cooling when the μ is positive

and heating when the μ is negative



Joule-Thomson coefficient

$$\mu_{JT} = \left(\frac{\partial T}{\partial P}\right)_h = -\left(\frac{\partial T}{\partial h}\right)_P \left(\frac{\partial h}{\partial P}\right)_T = \frac{1}{C_P} \left[T\left(\frac{\partial v}{\partial T}\right)_P - v\right]$$

For a real gas,

$$Tds = dh - vdP$$
$$T\left(\frac{\partial s}{\partial P}\right)_{T} = \left(\frac{\partial h}{\partial P}\right)_{T} - v$$
$$\left(\frac{\partial s}{\partial P}\right)_{T} = \frac{\partial(s,T)}{\partial(P,T)} = \frac{\partial(v,P)}{\partial(P,T)} = -\left(\frac{\partial v}{\partial T}\right)_{P}$$
$$\left(\frac{\partial h}{\partial P}\right)_{T} = -T\left(\frac{\partial v}{\partial T}\right)_{P} + v$$



Joule-Thomson coefficient

$$\mu_{JT} = \left(\frac{\partial T}{\partial P}\right)_h = -\left(\frac{\partial T}{\partial h}\right)_P \left(\frac{\partial h}{\partial P}\right)_T = \frac{1}{C_P} \left[T\left(\frac{\partial v}{\partial T}\right)_P - v\right]$$

For an ideal gas, $\mu_{JT} = 0$,

$$\left(\frac{\partial h}{\partial P}\right)_T = 0 \text{ and } h = h(T)$$

For a Van der Waals gas,
$$P = \frac{RT}{v-b} - \frac{a}{v^2}$$

$$\mu_{JT} = \frac{1}{c_P} \left[\frac{\frac{2a}{RT} \left(1 - \frac{b}{v}\right)^2 - b}{1 - \frac{2a}{vRT} \left(1 - \frac{b}{v}\right)^2} \right]$$
If $\mu_{JT} = 0$, $T_i = \frac{2a}{bR} \left(1 - \frac{b}{v}\right)^2$



Carnot cycle



Fig. The concept of a heat engine.

In (a), work is done on the system and is converted to heat.

In (b), heat is extracted from a reservoir and is converted to mechanical work.

This configuration is not possible.



Carnot cycle



Fig. The concept of a heat engine.

Can the work done by the system be equal to the heat in?

The second law of thermodynamics states *unequivocally* that it is impossible to

construct a perfect heat engine.

Thus case (b) must be modified as case (c)



Clausius statement

It is impossible to construct a device that operates in a cycle and whose sole effect is to transfer heat from a cooler body to a hotter body.

Kelvin-Planck statement

It is impossible to construct a device that operates in a cycle and produces no other effect than the performance of work and the exchange of heat with a single reservoir.



Carnot cycle



Fig. P-V and T-S diagrams of Carnot cycle.

- 1-2: isothermal expansion
- 2-3: adiabatic expansion
- 3-4: isothermal compression
- 4 1 : adiabatic compression



The efficiency of the engine,

$$\eta = \frac{W}{Q_H} = \frac{|W|}{|Q_H|} = \frac{\text{output}}{\text{input}}$$

Applying the first law to the system,

$$\Delta U = Q_H + Q_L - W = |Q_H| - |Q_L| - |W| \qquad Q_L < 0$$

Since the system is in a cyclical process, $\Delta U = 0$. Then,

$$W = Q_H + Q_L$$
 or $W = |Q_H| - |Q_L|$

Substituting the equations,

$$\eta = \frac{Q_L + Q_H}{Q_H} = 1 + \frac{Q_L}{Q_H} = 1 - \frac{|Q_L|}{|Q_H|}$$



For an ideal gas, Pv = RT, u = u(T),

For isothermal process, $Q_H = W_{12} = n\overline{R} T_b \ln \frac{V_2}{V_1} > 0$

$$-Q_L = W_{34} = n\bar{R} T_a \ln \frac{V_4}{V_3} < 0$$

For adiabatic process, $PV^{\gamma} = constant$, $T_b V_2^{\gamma-1} = T_a V_3^{\gamma-1}$ $T_b V_1^{\gamma-1} = T_a V_4^{\gamma-1}$ $\Rightarrow \frac{V_2}{V_1} = \frac{V_3}{V_4}$ $(s_2 - s_1 = c_v \ln \frac{T_2}{T_1} + R \ln \frac{v_2}{v_1})$ $= c_p \ln \frac{T_2}{T_1} - R \ln \frac{P_2}{P_1} = 0$ $c_v = \frac{1}{\kappa - 1} R, \quad c_p = \frac{\kappa}{\kappa - 1} R$ $\frac{P_2}{P_1} = \left(\frac{v_1}{v_2}\right)^{\kappa}$

The efficiency of the Carnot cycle, $\eta = 1 - \frac{Q_L}{Q_H} = 1 - \frac{T_a}{T_b}$



Carnot engine has the maximum efficiency for any engine that one might design.

- 1. Carnot engine operates between two reservoirs and that it is reversible.
- If a working substance other than an ideal gas is used, the shape of curves in the P-V diagram will be different.
- 3. The efficiency would be 100 percent if we were able to obtain a low temperature reservoir at absolute zero. \rightarrow However this is forbidden by the third law.



Carnot refrigerator

 T_b Reverse process of Carnot engine Q_H Coefficient of performance(COP) Q_H W $COP \equiv -\frac{Q_L}{W} = \frac{|Q_L|}{|W|} = \frac{|Q_L|}{|Q_2| - |Q_L|} = \frac{T_1}{T_2 - T_1}$ T_a

We introduce a minus sign in order to make the COP a positive quantity. The heat Q_L is extracted from the low temperature reservoir and W is the work done

on the system. Q_L is positive (heat flow *into* the system) and W is negative (work done *on* the system)





- 1. The refrigerant is a substance chosen to be a saturated liquid at the pressure and temperature of condenser.
- 2. The liquid undergoes a throttling process in which it is cooled and is partially vaporized.





Fig. Schematic diagram of a typical refrigerator

- 3. The vaporization is completed in the evaporator: the heat is absorbed by the refrigerant from the low temperature reservoir (the interior refrigerator space).
- 4. The low pressure vapor is then adiabatically compressed and isobarically cooled until it becomes a liquid again.

