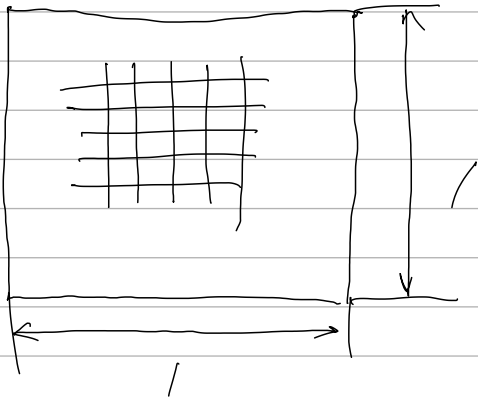
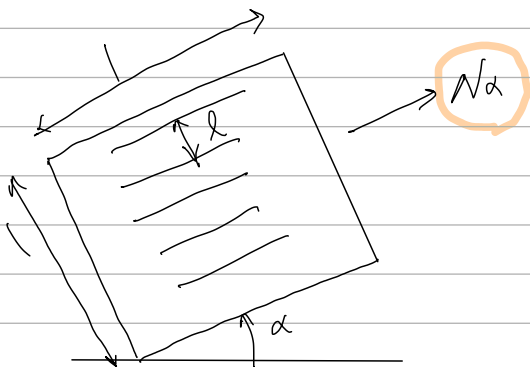


Chapter 2. Space frames

2.3. Continuum models for single layer space frames



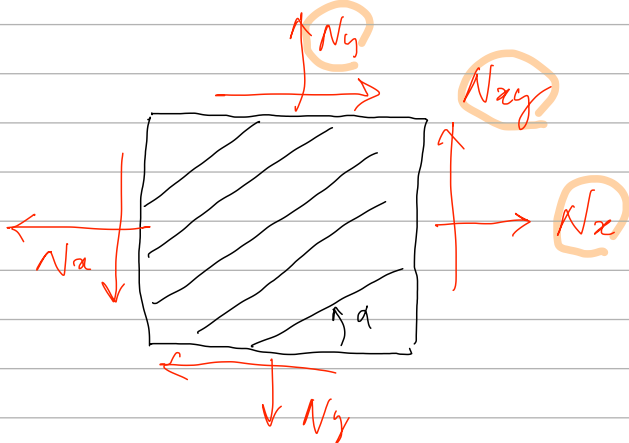
bars: A, E



N_α : sum of the bar axial forces in unit width of space frame

$$\frac{N_\alpha \cdot l}{A} = E \cdot \epsilon_\alpha$$

ϵ_α : normal strain



$$\begin{Bmatrix} N_x \\ N_y \\ N_{xy} \end{Bmatrix} = \begin{Bmatrix} \cos^2 \alpha \\ \sin^2 \alpha \\ \sin \alpha \cos \alpha \end{Bmatrix} N_\alpha$$

Global coordinate
Local coordinate

c.f. Transformation matrices

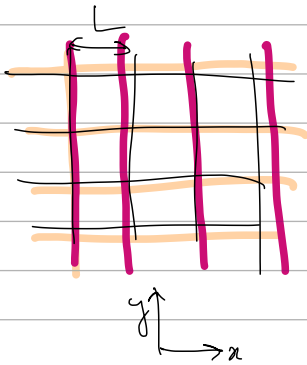
$$\begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix} = \begin{bmatrix} c^2 & s^2 & -2sc \\ s^2 & c^2 & 2sc \\ sc & -sc & c^2 - s^2 \end{bmatrix} \begin{Bmatrix} \sigma_1 \\ \sigma_2 \\ \tau_{12} \end{Bmatrix}$$

$$\begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{Bmatrix} = \begin{bmatrix} c^2 & s^2 & -sc \\ s^2 & c^2 & sc \\ 2sc & -2sc & c^2 - s^2 \end{bmatrix} \begin{Bmatrix} \epsilon_1 \\ \epsilon_2 \\ \gamma_{12} \end{Bmatrix}$$

$$\begin{aligned}
 \begin{Bmatrix} N_x \\ N_y \\ N_{xy} \end{Bmatrix} &= \begin{Bmatrix} \cos^2 \alpha \\ \sin^2 \alpha \\ \sin \alpha \cos \alpha \end{Bmatrix} [N_\alpha] \\
 &= \begin{Bmatrix} \cos^2 \alpha \\ \sin^2 \alpha \\ \sin \alpha \cos \alpha \end{Bmatrix} \frac{AE}{l} \begin{bmatrix} \cos^2 \alpha & \sin^2 \alpha & \sin \alpha \cos \alpha \\ \cos^2 \alpha & \sin^2 \alpha & \sin \alpha \cos \alpha \\ \sin \alpha \cos \alpha & \sin \alpha \cos \alpha & \sin^2 \alpha \cos^2 \alpha \end{bmatrix} \begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{Bmatrix} \\
 &= \frac{AE}{l} \begin{bmatrix} \cos^4 \alpha & \sin^2 \alpha \cos^2 \alpha & \sin \alpha \cos^3 \alpha \\ \sin^2 \alpha \cos^2 \alpha & \sin^4 \alpha & \sin^3 \alpha \cos \alpha \\ \sin \alpha \cos^3 \alpha & \sin^3 \alpha \cos \alpha & \sin^2 \alpha \cos^2 \alpha \end{bmatrix} \begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{Bmatrix}
 \end{aligned}$$

↳ Homogenized stiffness matrix

e.g.,



Horizontal : $\alpha = 0, l = L$

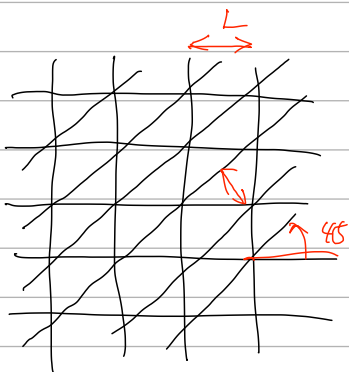
$$\frac{AE}{L} \begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

Vertical : $\alpha = 90, l = L$

$$\frac{AE}{L} \begin{bmatrix} 0 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$\rightarrow \begin{Bmatrix} N_x \\ N_y \\ N_{xy} \end{Bmatrix} = \frac{AE}{L} \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{Bmatrix} \quad (*)$$

Rank 2 : lack of shearing stress
(exists zero-stiffness deformation mode)



Add diagonal stiffness term

$$\alpha = 45^\circ \quad l = \frac{\sqrt{2}L}{2}$$

Diagonal

$$\frac{AE}{\sqrt{2}L} \begin{bmatrix} 1/4 & 1/4 & 1/4 \\ 1/4 & 1/4 & 1/4 \\ 1/4 & 1/4 & 1/4 \end{bmatrix}$$

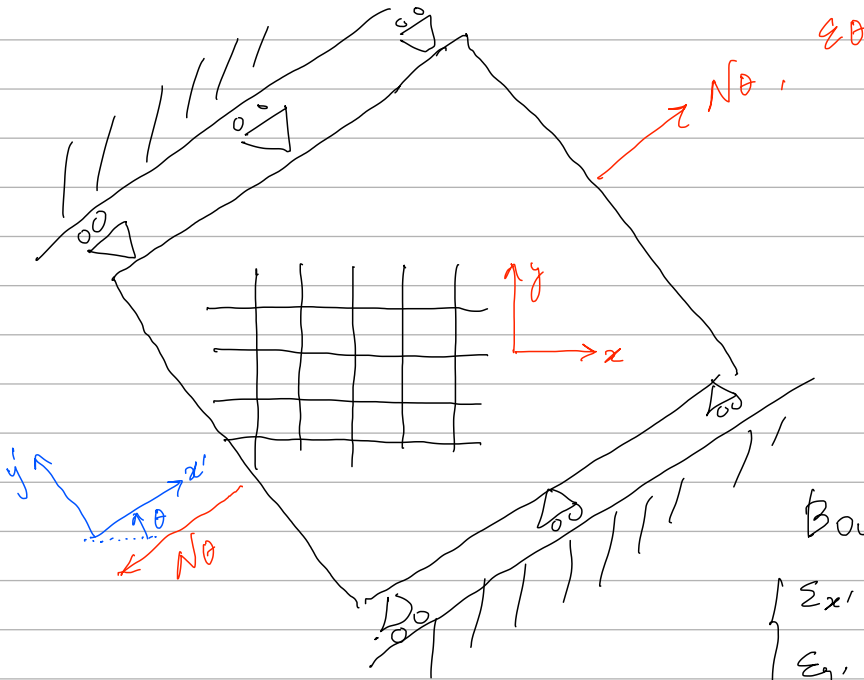
Summing up with (*)

$$\frac{AE}{L} \begin{bmatrix} 4a & a & a \\ a & 4a & a \\ a & a & a \end{bmatrix}$$

$$a = \frac{1}{2\sqrt{2}}$$

Rank 3 (full rank)

2.3.2. Stiffness in direction θ



Stiffness of the space frame in θ -direction

$$E_{\theta}^* = \frac{N_{\theta}}{\epsilon_{\theta}}$$

Boundary condition

$$\begin{pmatrix} \epsilon_{x'} \\ \epsilon_{y'} \\ \gamma_{xy} \end{pmatrix} = \begin{pmatrix} \epsilon_{\theta} \\ 0 \\ 0 \end{pmatrix}$$

→ plane strain condition

From strain transformation matrix

$$\begin{pmatrix} \epsilon_{x'} \\ \epsilon_{y'} \\ \gamma_{xy} \end{pmatrix} = \begin{bmatrix} c^2 & s^2 & -2sc \\ s^2 & c^2 & 2sc \\ 2sc & -2sc & c^2 - s^2 \end{bmatrix} \begin{pmatrix} \epsilon_{\theta} \\ 0 \\ 0 \end{pmatrix}$$

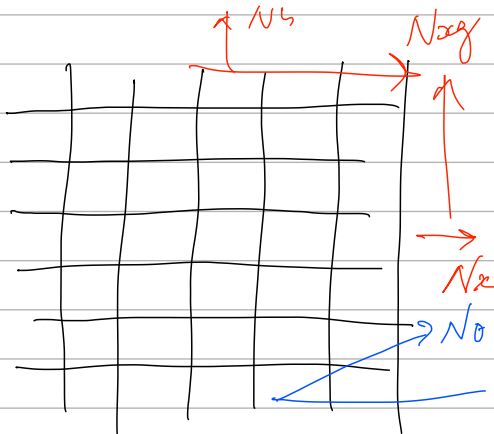
$$= \begin{pmatrix} c^2 \\ s^2 \\ 2sc \end{pmatrix} \epsilon_{\theta}$$

Likewise.

$$\begin{Bmatrix} N_\theta \\ \vdots \end{Bmatrix} = \begin{bmatrix} c^2 & s^2 & 2sc \\ s^2 & c^2 & -2sc \\ -sc & sc & c^2 - s^2 \end{bmatrix} \begin{Bmatrix} N_x \\ N_y \\ N_{xy} \end{Bmatrix}$$

$$\underline{N_\theta = N_x \cos^2 \theta + N_y \sin^2 \theta + 2 N_{xy} \sin \theta \cos \theta}$$

2.4.1. Square lattice



Recall

$$\begin{Bmatrix} N_x \\ N_y \\ N_{xy} \end{Bmatrix} = \frac{AE}{L} \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{Bmatrix}$$

$$N_\theta = \frac{AE}{L} \underline{\epsilon_x} \cdot \cos^2 \theta + \frac{AE}{L} \underline{\epsilon_y} \cdot \sin^2 \theta$$

$$= \frac{AE}{L} \cdot \cos^4 \theta \epsilon_0 + \frac{AE}{L} \sin^4 \theta \cdot \epsilon_0$$

$$= \frac{AE}{L} (\cos^4 \theta + \sin^4 \theta) \cdot \epsilon_0$$

$$\underline{E_\theta^*} = \frac{N_\theta}{\epsilon_0} = \underline{\frac{AE}{L} (\cos^4 \theta + \sin^4 \theta)}$$

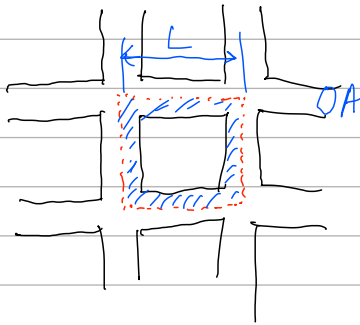
2.4. In-plane efficiency of single-layer space frames

: Stiffness is compared to that of a homogeneous, continuous plate of arbitrary thickness t and made from the same material as the bars of our space frame

$$\underline{M_\theta} = \frac{E_\theta^* / \rho^*}{E_0 / \rho} = \frac{E_\theta^*}{\alpha E_0}$$

$$\rho^* = \frac{\text{mass of bars in repeating unit}}{\text{Volume of repeating unit}}$$

$$\alpha = \frac{\rho^*}{\rho} ; \text{ relative density} \\ \text{ (i.e., volume fraction)}$$



$$\alpha = \frac{2A \cdot L}{L^2 \cdot t}$$

Recall B-5-L. Plate under plane stress

$$\begin{pmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{pmatrix} = \begin{bmatrix} 1/E & -\nu/E & 0 \\ -\nu/E & 1/E & 0 \\ 0 & 0 & 1/G \end{bmatrix} \begin{pmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{pmatrix}$$

$$\begin{pmatrix} N_x \\ N_y \\ N_{xy} \end{pmatrix} = \begin{pmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{pmatrix} \cdot t = \frac{Et}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix} \begin{pmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{pmatrix}$$

$$\text{if } \nu = 1/3 \quad = \frac{9}{8} Et \begin{bmatrix} 1 & 1/3 & 0 \\ 1/3 & 1 & 0 \\ 0 & 0 & 1/3 \end{bmatrix} \begin{pmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{pmatrix}$$

Under $\epsilon_y = \gamma_{xy} = 0$

$$N_x = \frac{9}{8} Et \epsilon_x$$

$$\therefore E\theta = \frac{N_x}{s_x} = \frac{9}{8} Et$$

$$M_\theta = \frac{E\theta}{\alpha E\theta} = \frac{\cancel{4Et} (\cos^2\theta + \sin^2\theta)}{\frac{9}{8} Et} \cdot \frac{\cancel{L^2}}{2A}$$

$$\left(\alpha = \frac{2A \cdot L}{L^2 \cdot t} \right) = \frac{4}{9} (\cos^2\theta + \sin^2\theta)$$