

Last lecture

Bernoulli Equation as a special case of energy equation.

Pitot → Total or stagnation p ; static pressure; dynamic pressure
Venturi

Ch. 4: Differential Relations.

Mass conservation.

1. Test #1 Oct 14 11-12:15 301-204 & 301-306
(Crupto Ch. 3)

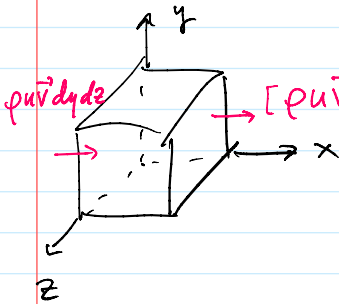
2. Final Exam Dec 9 11-13:30? Tentative date
Dec 14 11-13:30?

Today: Differential form of momentum conservation

$$\Sigma d\vec{F} = \underbrace{\frac{d}{dt} \int_{c.v.} \rho \vec{V} dVol}_{\frac{\partial}{\partial t} (\rho \vec{V}) dx dy dz} + \int_{c.s.} \rho \vec{V} (\vec{V} \cdot \vec{n}) dA$$

stationary
fixed c.v.

$$\vec{V} = u \vec{e}_x + v \vec{e}_y + w \vec{e}_z$$



in x: $[\rho u v + \frac{\partial}{\partial x} (\rho u v) dx] dy dz - \rho u v dy dz = \frac{\partial}{\partial x} (\rho u v) dx dy dz$

y: $\frac{\partial}{\partial y} (\rho v \vec{V}) dy dx dz$

z: $\frac{\partial}{\partial z} (\rho w \vec{V}) dz dy dx$

$$\Sigma d\vec{F} = dx dy dz \left[\frac{\partial}{\partial t} (\rho \vec{V}) + \frac{\partial}{\partial x} (\rho u \vec{V}) + \frac{\partial}{\partial y} (\rho v \vec{V}) + \frac{\partial}{\partial z} (\rho w \vec{V}) \right]$$

$$\Sigma d\vec{F} = dx dy dz \left[\underbrace{\vec{V} \left[\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{V}) \right]}_{=0} + \rho \left(\frac{\partial \vec{V}}{\partial t} + u \frac{\partial \vec{V}}{\partial x} + v \frac{\partial \vec{V}}{\partial y} + w \frac{\partial \vec{V}}{\partial z} \right) \right]$$

Total derivative of \vec{V}
Substantial " " \vec{V}
Material " " \vec{V}

rate of change in \vec{V}
following a particle

$$\frac{D\vec{V}}{Dt} = \frac{d\vec{V}}{dt}$$

$$\Sigma d\vec{F} = \rho \frac{D\vec{V}}{Dt} dx dy dz$$

$$\frac{\Sigma d\vec{F}}{dx dy dz} = \rho \frac{D\vec{V}}{Dt}$$

forces in differential form per unit volume.

body: $\rho \vec{g}$

σ_{ij} stress tensor

Surface: $-\nabla p + \left(\frac{d\vec{F}}{d\text{vol}}\right)_{\text{visc.}}$

surface normal vector in i dir.

$$-\frac{\partial p}{\partial x} \vec{i} + \left(\frac{\partial \tau_{xx}}{\partial x} + \frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z}\right) \vec{i}$$

shear τ acting in j direction.

$$-\frac{\partial p}{\partial y} \vec{j} + \left(\frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \tau_{yy}}{\partial y} + \frac{\partial \tau_{zy}}{\partial z}\right) \vec{j}$$

$$-\frac{\partial p}{\partial z} \vec{k} + \left(\frac{\partial \tau_{xz}}{\partial x} + \frac{\partial \tau_{yz}}{\partial y} + \frac{\partial \tau_{zz}}{\partial z}\right) \vec{k}$$

Newtonian fluid $\tau = \mu \cdot \text{strain rate} \rightarrow \mu \cdot \text{velocity gradient}$

For incompressible ($\rho = \text{constant}$) flows.

4 equations: mass
x-mom
y-mom
z-mom

4 unknowns: p, u, v, w

Thermodynamics (1-D) mass + energy eqns

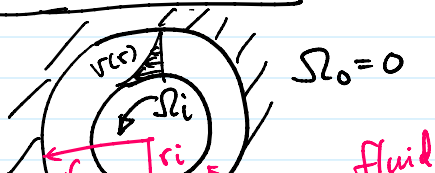
Fluid mechanics (1-D) incompressible \rightarrow mass + mom eqn.

compressible \rightarrow mass + mom + energy + state

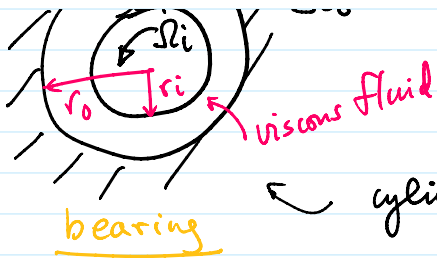
Some examples of solutions. \rightarrow 1. Incompressible

2. Viscous

#1 Couette Flow.



e.g. washing machine
pump
rotating shaft

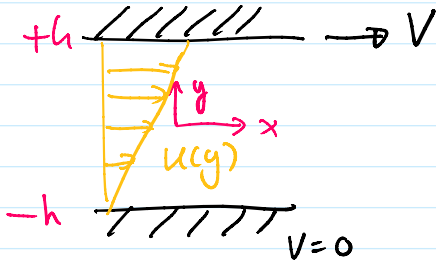


cylindrical coordinates. (r, θ, z)

$$\rightarrow v_{\theta}(r) = C_1 r + \frac{C_2}{r}$$

↓ simplify.

2-D in Cartesian coordinate.



viscous flow between fixed & moving plates.

1. Incompressible
2. Viscous
3. steady $(\frac{\partial}{\partial t} = 0)$
4. neglect z
5. $v = w = 0$
6. $\frac{\partial p}{\partial x} = 0$
7. $\frac{\partial}{\partial z} = 0$

mass: if incompressible then $\nabla \cdot \vec{v} = 0$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \rightarrow u(y)$$

x-mom: $\rho \left(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right) = - \frac{\partial p}{\partial x} + \rho g_x + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$

$$\mu \frac{d^2 u}{dy^2} = 0$$

$u(y) = C_1 y + C_2$ linear velocity distribution

$C_1? C_2?$

Boundary conditions: at $y = +h, u(+h) = V$
 $y = -h, u(-h) = 0$

$$C_1 = \frac{V}{2h}; \quad C_2 = \frac{V}{2}$$

$\tau?$ $\tau = \mu \frac{du}{dy}$

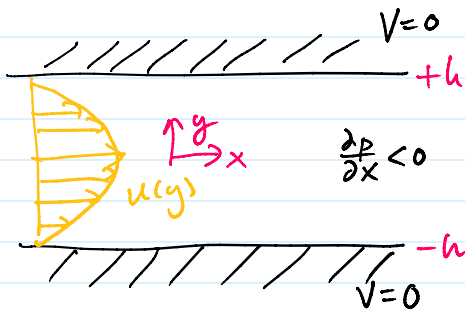
At the interface between a viscous fluid and a solid wall, NO-SLIP BOUNDARY CONDITION exists.

Boundary condition

exists.

$$\vec{V}_{\text{solid}} = \vec{V}_{\text{fluid}} \text{ at the interface}$$

#2 Poiseuille flow
($\frac{\partial p}{\partial x} \neq 0$)



1. Incompressible
2. Viscous
3. steady
4. neglect \vec{g}
5. $v = w = 0$
6. $\frac{\partial}{\partial z} = 0$

mass: $\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \rightarrow u(y)$

X-mom: $0 = -\frac{\partial p}{\partial x} + \rho g_x + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$

$$0 = -\frac{\partial p}{\partial x} + \mu \frac{d^2 u}{dy^2}$$

$$u(y) = \frac{1}{\mu} \frac{dp}{dx} \frac{y^2}{2} + C_1 y + C_2$$

B.C. @ $y = \pm h$, $u(+h) = u(-h) = 0$

$$C_1 = 0$$

$$C_2 = -\frac{dp}{dx} \frac{h^2}{2\mu}$$

$$u(y) = -\frac{dp}{dx} \frac{h^2}{2\mu} \left(1 - \frac{y^2}{h^2} \right)$$

u_{max} at $y=0 \rightarrow u_{\text{max}} = \left(-\frac{dp}{dx} \right) \frac{h^2}{2\mu} > 0$ because $\frac{dp}{dx} < 0$

τ at the wall and/or in the fluid

$$\tau_{\text{wall}} = \mu \left(\frac{du}{dy} \right)_{y=\pm h}$$

$$\tau_{\text{wall}} = -\frac{2\mu U_{\text{max}}}{h} \quad \text{at } y = +h$$

$$+\frac{2\mu U_{\text{max}}}{h} \quad \text{at } y = -h.$$

Last lecture

Momentum eqn for Newtonian fluid in differential form

Navier-Stokes eqn.

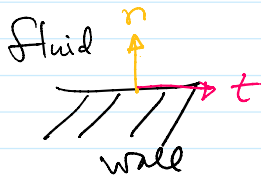
Viscous solutions

- Couette
- Poiseuille

Boundary conditions.

1. fluid & wall
interface

viscous flow



$$\underline{\vec{V}_{\text{fluid}} = \vec{V}_{\text{wall}}} \quad (\text{except in rarefied flow (space)})$$

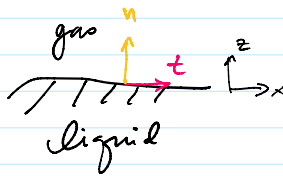
$$\underline{V_{n \text{ fluid}} = V_{n \text{ wall}}} \quad (\text{impermeable})$$

$$\underline{V_{t \text{ fluid}} = V_{t \text{ wall}}} \quad (\text{viscous}); \quad V_{t \text{ fluid}} \neq V_{t \text{ wall}} \quad (\text{inviscid})$$

(rarefied)

extremely low density

2. gas & liquid
interface



$$1. V_{n \text{ gas}} = V_{n \text{ liquid}}$$

$$2. \tau_{\text{gas}} = \tau_{\text{liquid}}$$

$$\tau_{zy}$$

$$\tau_{zx}$$

$$3. P_{\text{liquid}} \approx P_{\text{gas}} \quad (\text{neglecting surface tension curvature})$$

Functions.

1. Stream function ψ

Assume 1. steady 2. incompressible 3. 2-D flow

continuity equation

$$\underline{\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0}$$

$$\vec{V} = u\vec{i} + v\vec{j}$$

introduce a stream function ψ (definition: $u = \frac{\partial \psi}{\partial y}$ & $v = -\frac{\partial \psi}{\partial x}$)

continuity equation is automatically satisfied

solve the momentum equation for ψ .

Continuity equation is automatically satisfied
 solve the momentum equation for ψ .

Relationship between streamline and stream function ψ ?

tangent to velocity

$$\left. \frac{dy}{dx} \right| = \frac{v}{u}$$

$$\frac{dx}{u} = \frac{dy}{v} = \frac{dz}{w}$$

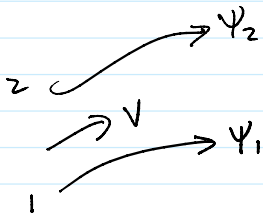
$$u = \frac{\partial \psi}{\partial y} \quad \& \quad v = -\frac{\partial \psi}{\partial x}$$

$$u dy - v dx = 0$$

$$\frac{\partial \psi}{\partial y} dy + \frac{\partial \psi}{\partial x} dx = d\psi = 0$$

$\hookrightarrow \psi = \text{constant on a streamline}$

$$\left. \frac{dy}{dx} \right|_{\psi = \text{const.}} = \frac{v}{u}$$



$$\psi_2 - \psi_1 = \int_1^2 d\psi = \int_1^2 (\vec{V} \cdot \vec{n}) dA = Q_{1-2} \quad \text{volume flow rate between streamlines 1 \& 2.}$$

$\nabla \times \vec{V} \rightarrow \nabla \times \vec{V} = 0$ irrotational flow (no rigid body motion)
 $\nabla \times \vec{V} \neq 0$ rotational flows (rigid body motion)

Vorticity

$$\hookrightarrow \nabla \times \vec{V} = \begin{vmatrix} \vec{i} & \vec{j} & \vec{k} \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ u & v & w \end{vmatrix} \rightarrow \text{in 2-D flow} \rightarrow \nabla \times \vec{V} = -\nabla^2 \psi \vec{k}$$

Laplace Equation $\nabla^2 \psi = 0$ if irrotational flow

Poisson Equation $\nabla^2 \psi = \text{constant}$ if rotational flow

Vorticity $\vec{\Omega}$ & Irrotationality:

$$\vec{\Omega} = \nabla \times \vec{V}$$

$\hookrightarrow \nabla \times \vec{V} = 0$ zero vorticity in irrotational flow

Angular velocity = $\frac{1}{2} \vec{\Omega}$

for 2-D flow in xy plane

$$\text{angular velocity} = \frac{1}{2} \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right)$$

2. Potential function ϕ

if flow is irrotational, then potential function ϕ exists.

$$\nabla \times \vec{V} = 0$$

$$\vec{V} = \nabla \phi$$

$$\phi \rightarrow u = \frac{\partial \phi}{\partial x}, v = \frac{\partial \phi}{\partial y}, w = \frac{\partial \phi}{\partial z}$$

for 1. steady 2. incompressible 3. 2-D 4. irrotational flow

ψ

ϕ

$$u = \frac{\partial \psi}{\partial y} = \frac{\partial \phi}{\partial x}$$

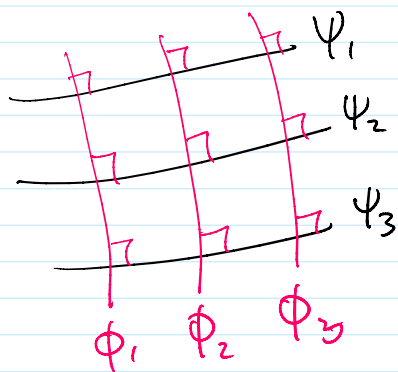
$$v = -\frac{\partial \psi}{\partial x} = \frac{\partial \phi}{\partial y}$$

on a constant ϕ line $\rightarrow d\phi = 0 = \frac{\partial \phi}{\partial x} dx + \frac{\partial \phi}{\partial y} dy$

$$d\phi = 0 = u dx + v dy$$

$$\frac{dy}{dx} \Big|_{\phi = \text{constant}} = -\frac{u}{v} \quad \text{vs.}$$

$$\frac{dy}{dx} \Big|_{\psi = \text{constant}} = \frac{v}{u}$$



if incompressible flow

$$\nabla \cdot \vec{V} = 0$$

if incompressible & irrotational flow

$$\nabla \cdot (\nabla \phi) = \nabla^2 \phi = 0$$

if incompressible & 2-D

\downarrow
 ψ

\downarrow
 $\nabla^2 \psi = 0$ irrotational

$\nabla^2 \psi = c$ rotational

$$\nabla^2 \psi = c \quad \nabla^2 \psi = \text{max}$$

LaPlace Eqn & Poisson Eqn \rightarrow linear differential equations

\rightarrow superposition possible.

if a is a solution & b is a solution, so is $a+b$.