

# 458.308 Process Control & Design

## Lecture 2: Dynamic Modeling

### - Theoretical (or Fundamental) Models

$$\dot{x} = f(x, u, p)$$

$$y = g(x, u, p)$$

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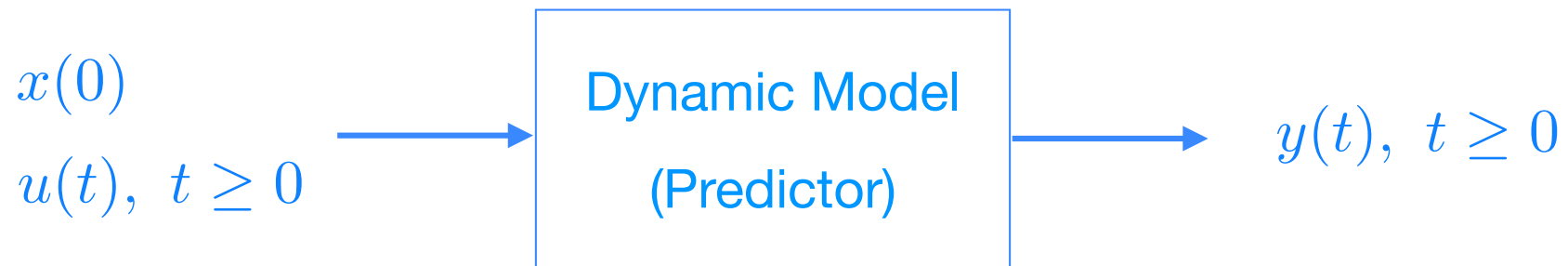
# Objectives

- Write balance equations using **instantaneous** methods
- Incorporate appropriate **constitutive relationships** into the equations
- Determine the state, input, and output variables and the parameters
- Degrees of freedom analysis

# Dynamic Modeling

Express the process's \_\_\_\_\_

- Non steady-state initial condition, parameter changes, disturbances, etc.
- Often in the form of mathematical equation (time differential equation)



# Why Dynamic Modeling?

Process dynamics must be understood well (often at a quantitative level) in order to design and operate the process effectively (e.g., design an effective control system)

## Usage:

- Design (esp. batch processes, cyclic continuous processes that are inherently dynamic)
- Operability assessment (stability of intended operating condition, sensitivity to disturbances)
- Operator training
- Process optimization
- Design of startup / shutdown / transition procedure
- Design of control system

# Modeling Approaches

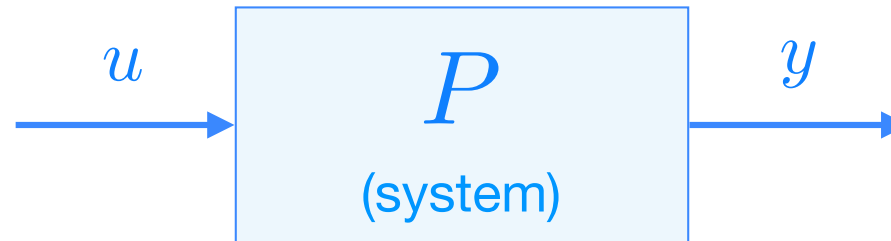
## Fundamental Modeling

- Theoretical/Mechanistic modeling
- **Physico-chemical** understanding + Conservation principle
- Difficult to develop: need detailed process knowledge
- Usually complex: a large set of ODEs or PDEs
- Fundamentally correct - can be used for **exploratory** purposes
- Used for simulation (operator training), optimization, and transition control

## Empirical Modeling

- Explain the **observed** response, cause-effect, pattern, etc.
- Easier to develop: need experimental data
- Usually kept simple: a small set of **linear** ODEs
- Lacks fundamental correctness: may not be useful in applications that require extrapolation beyond the conditions under which data were collected
- Used for controller design

# Linear System



System can be seen as a mapping between input  $u$  and output  $y$

Linear system should satisfy **principle of superposition**

This also implies \_\_\_\_\_

## Principle of superposition

$$y = P(u), \quad u \in \mathcal{D}, \quad y \in \mathcal{R}$$

$$\text{Consider } u_1, u_2 \in \mathcal{D} \longrightarrow y_1 = P(u_1), \quad y_2 = P(u_2)$$

For any scalar values of  $a$  and  $b$ :

$$P(au_1 + bu_2) = aP(u_1) + bP(u_2) = ay_1 + by_2$$

# Determine if the systems given below are linear or not

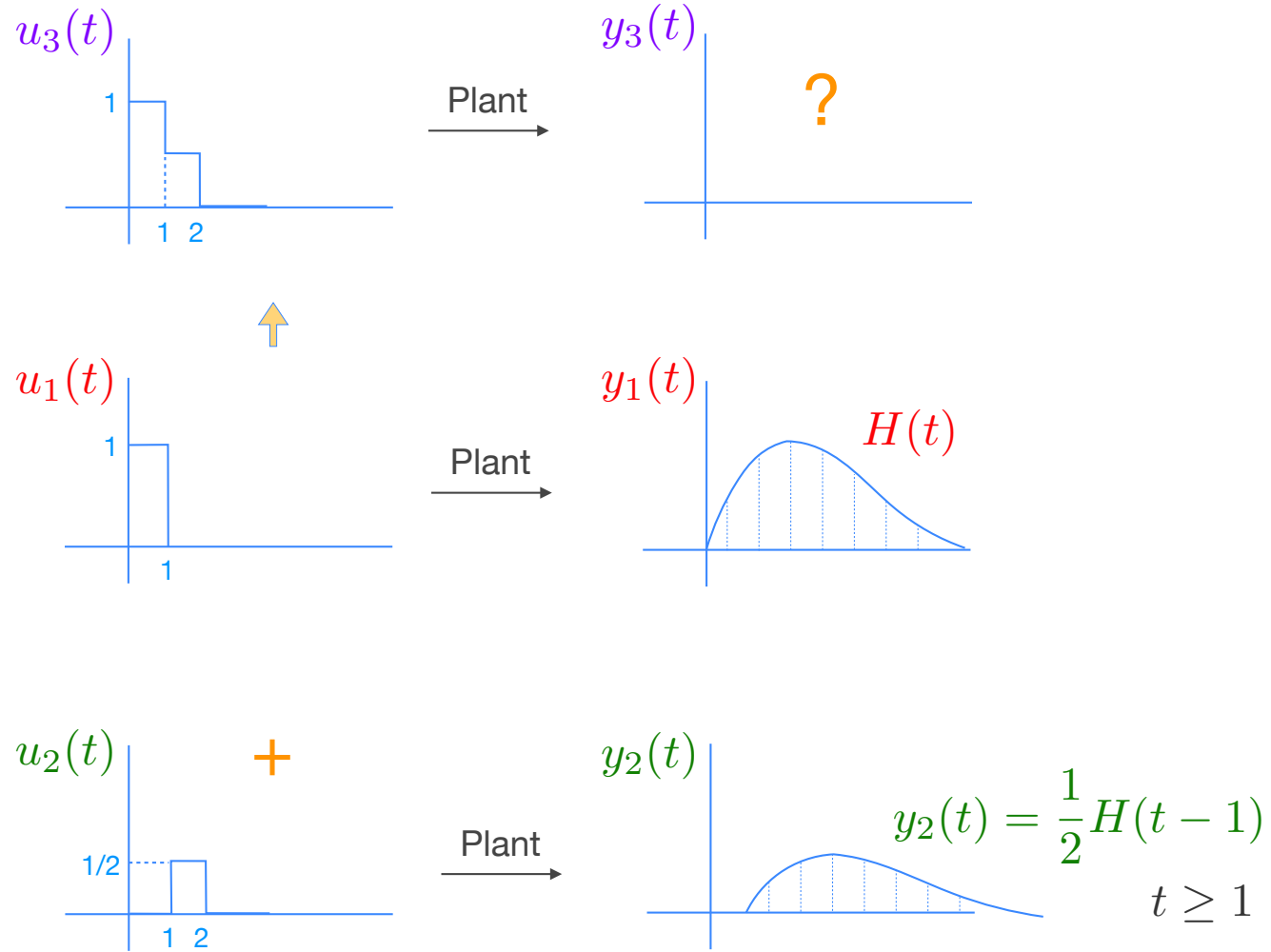
1  $y = 3u$

2  $y = 3u + 2$

3 Can you make (System 2) a linear system by defining a **deviation variable**?

Note: People often call affine system linear system

# Advantage of Superposition Principle





# Linear vs. Nonlinear Dynamics

- Linear model represents significant simplification
  - Knowledge of the output response to one particular input change can be **extrapolated** to predict the output responses to many different (sometimes any) input changes.
  - Gives **approximate** behavior (locally valid around some particular operating point) for most nonlinear processes. In this case, variables must be expressed in terms of **deviations from the steady-state values at the chosen operating point**
- Nonlinear Dynamic System: Any system that is **NOT** linear

# How can you tell if a differential equation is linear or not?



If all the terms appear in the equation are **linear functions of variables**, it is linear. Otherwise it is not.

Linear

$$\frac{dy}{dt} + 3y = 2u(t)$$

$$a_n \frac{d^n y}{dt^n} + a_{n-1} \frac{d^{n-1} y}{dt^{n-1}} + \cdots + a_1 \frac{dy}{dt} = bu(t)$$

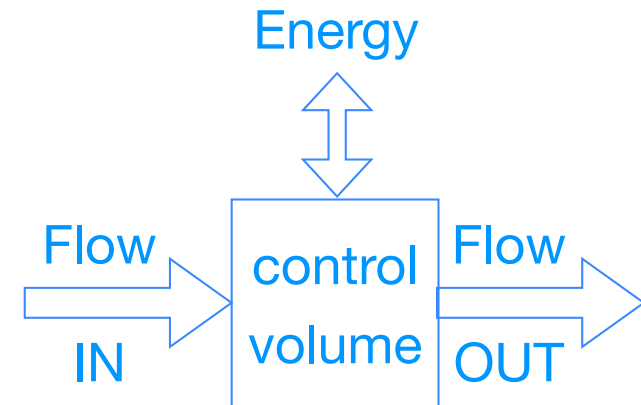
Nonlinear

$$\frac{dy}{dt} + y \sin y = 2u(t)$$

$$\frac{d^2 y}{dt^2} + 3 \frac{dy}{dt} + 2e^{-y} = u(t)$$

$$\frac{dy}{dt} y + \frac{d^2 y}{dt^2} + y^3 = 3u(t)$$

# Modeling Principle



## 1. Total mass balance

$$\left[ \begin{array}{l} \text{Rate of} \\ \text{mass accumulation} \\ \text{within CV} \end{array} \right] = \left[ \begin{array}{l} \text{Rate of} \\ \text{mass in} \\ \text{from surroundings} \end{array} \right] - \left[ \begin{array}{l} \text{Rate of} \\ \text{mass out} \\ \text{to surroundings} \end{array} \right]$$

## 2. Component mass (molar) balance

$$\left[ \begin{array}{l} \text{Rate of component} \\ \text{mass accumulation} \\ \text{within CV} \end{array} \right] = \left[ \begin{array}{l} \text{Rate of component} \\ \text{mass in} \\ \text{from surroundings} \end{array} \right] - \left[ \begin{array}{l} \text{Rate of component} \\ \text{mass out} \\ \text{to surroundings} \end{array} \right] + \left[ \begin{array}{l} \text{Rate of component} \\ \text{mass creation} \\ \text{within CV} \end{array} \right]$$

# Modeling Principle

## 3. Total energy balance

$$\left[ \begin{array}{l} \text{Rate of} \\ \text{energy accumulation} \\ \text{within CV} \end{array} \right] = \left[ \begin{array}{l} \text{Rate of} \\ \text{energy in} \\ \text{from surroundings} \end{array} \right] - \left[ \begin{array}{l} \text{Rate of} \\ \text{energy out} \\ \text{to surroundings} \end{array} \right]$$

For flow systems,

$$\left[ \begin{array}{l} \text{Rate of} \\ \text{enthalpy accumulation} \\ \text{within CV} \end{array} \right] = \left[ \begin{array}{l} \text{Rate of} \\ \text{enthalpy in} \\ \text{by material flow} \end{array} \right] - \left[ \begin{array}{l} \text{Rate of} \\ \text{enthalpy out} \\ \text{by material flow} \end{array} \right] + \left[ \begin{array}{l} \text{Rate of total} \\ \text{heat addition} \\ \text{from surroundings} \end{array} \right]$$

## 4. Total momentum balance

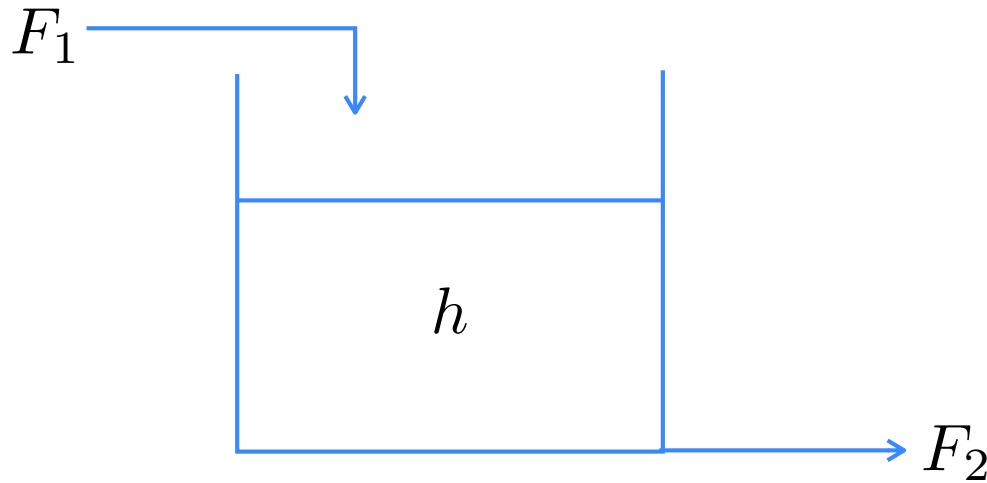
$$\left[ \begin{array}{l} \text{Rate of} \\ \text{momentum (mv) accumulation} \\ \text{within CV} \end{array} \right] = \left[ \begin{array}{l} \text{Net force} \\ \text{working} \\ \text{on CV} \end{array} \right]$$

# Modeling Procedure

1. Draw the control volume (Inside C.V., all intrinsic properties should be assumed same)
2. Identify which balances are necessary
  - Changing volume, height: Total Mass Balance
  - Changing concentration: Component mass or molar balance
  - Changing temperature: Energy balance
3. Write down mathematical expression for each term
4. Put them together into equation
5. Incorporate constitutive equations if needed
6. Use the chain rule, etc. to simplify

# Examples of Mass Balance

## 1. Surge Tank



Given

$\rho$  : liquid density

$A$  : cross-sectional area of vessel

$F_1, F_2$  : volumetric flow rate (vol./time)

Total M.B.

Total mass inside CV:  $m = \rho V$

Rate of mass accumulation inside CV:  $\frac{d(\rho V)}{dt}$

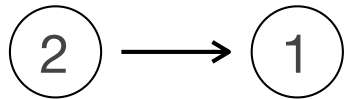
Rate of mass into CV by flow:  $\rho F_1$

Rate of mass out of CV by flow:  $\rho F_2$

$$\frac{d(\rho V)}{dt} = \rho F_1 - \rho F_2 \quad \text{--- (1)}$$

# Constitutive Equation?

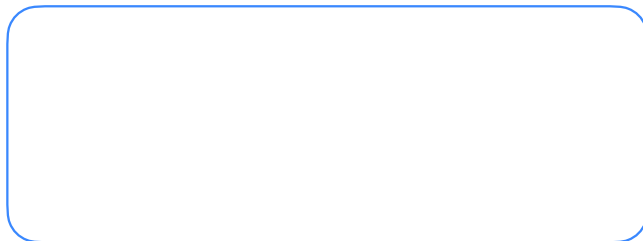
$$V = Ah \quad \text{-----} \quad (2)$$



$$\frac{d(\rho Ah)}{dt} = \rho F_1 - \rho F_2$$

If  $\rho$  and  $A$  are constants,

$$\rho A \frac{dh}{dt} = \rho F_1 - \rho F_2$$

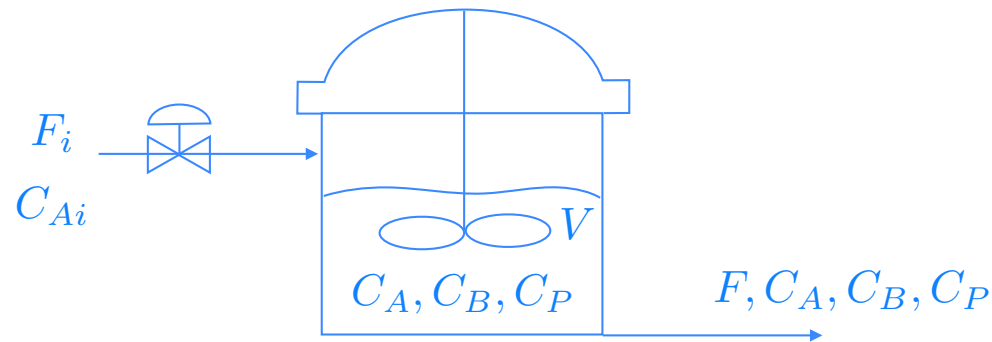


state variable:  $h$   
input variables:  $F_1, F_2$   
output variable:  $h$   
parameter:  $A$

## 2. Isothermal Chemical Reactor

Ethylene oxide (A) is reacted with water (B) in a CSTR to form ethylene glycol (P):  $A + B \rightarrow P$ . Assume the temperature is maintained at a constant value and that the water is in large excess. Also assume that  $\rho_i = \rho$

Develop a model to find the concentration for each species as a function of time.



### Total Mass Balance

$$\frac{d(\rho V)}{dt} = \rho_i F_i - \rho F \quad \longrightarrow \quad \boxed{\frac{dV}{dt} = F_i - F} \quad \textcircled{1} \quad \text{note: } \frac{dV}{dt} \neq 0$$



Component Mass Balances  $C_A, C_B, C_P$   $\left[ \frac{\text{moles}}{\text{vol}} \right]$

$$A : \frac{d(V C_A M_A)}{dt} = F_i C_{Ai} M_A - F C_A M_A + r_A \cdot V \cdot M_A$$

2

$M_A$  : molecular weight [mass/mol]

Similarly,

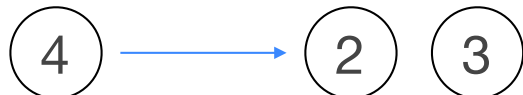
$$P : \frac{d(V C_p)}{dt} = 0 - F C_P + r_p V$$

3

Constitutive equations:

$$r_A = -k C_A, \quad r_p = k C_A$$

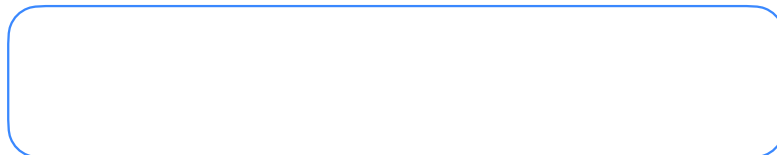
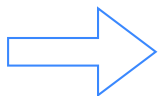
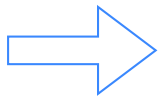
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$$\frac{d(VC_A)}{dt} = F_i C_{Ai} - FC_A - VkC_A \quad (2')$$

$$\frac{d(VC_P)}{dt} = -FC_P + VkC_A \quad (3')$$

$$\begin{aligned} (2') \quad \frac{d(VC_A)}{dt} &= V \frac{dC_A}{dt} + C_A \frac{dV}{dt} \\ &= V \frac{dC_A}{dt} + C_A (F_i - F) && \text{LHS} \\ &= F_i C_{Ai} - FC_A - VkC_A && \text{RHS} \end{aligned}$$



(2'')

Similarly, (3') becomes

$$\frac{dC_P}{dt} = -\frac{F_i}{V}C_P + kC_A \quad (3'')$$

State Variables

$$x = \begin{bmatrix} V \\ C_A \\ C_P \end{bmatrix}$$

Input Variables

$$u = \begin{bmatrix} F_i \\ F \\ C_{Ai} \end{bmatrix}$$

Parameters

$$p = [k]$$

# Energy Balance

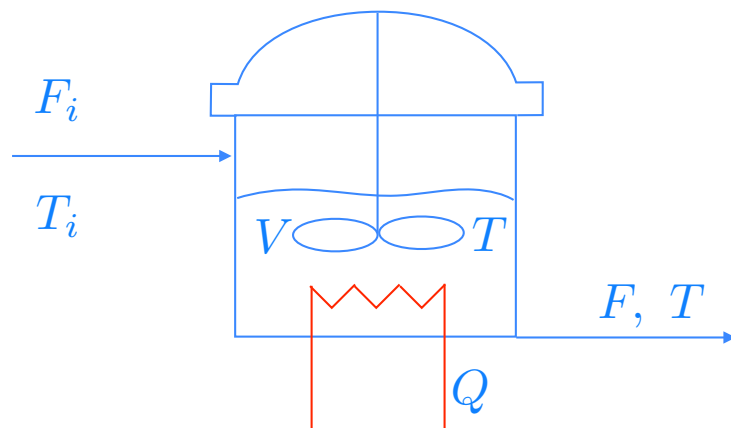
## Preliminary

$$TE \text{ (total energy)} = U \text{ (internal)} + KE \text{ (kinetic)} + PE \text{ (potential)}$$

negligible compared to thermal energy

$$\text{For flow systems: } H = U + pV$$

## Stirred Tank Heater



### Assume

$F_i, T_i$  can vary

Tank is insulated

$Q$  (rate of heat added per time) can vary

Develop a model to find the tank liquid temperature as a function of time

## Material Balance

$$\frac{d(V\rho)}{dt} = F_i\rho_i - F\rho$$

## Energy Balance

$$\frac{dT E}{dt} = F_i\rho_i T \hat{E}_i - F\rho T \hat{E} + Q + \underbrace{W_T}_{\text{work done on the system}}$$

We neglect the kinetic and potential energy:

$$\frac{dU}{dt} = F_i\rho_i \hat{U}_i - F\rho \hat{U} + Q + W_T$$

$$W_T = W_s + F_i p_i - F p$$

$$\frac{dU}{dt} = F_i\rho_i \left( \hat{U}_i + \frac{p_i}{\rho_i} \right) - F\rho \left( \hat{U}_i + \frac{p}{\rho} \right) + Q + W_s$$

neglect the mean pressure change (liquid)

$$\frac{dH}{dt} - \frac{dpV}{dt} = F_i\rho_i \hat{H}_i - F\rho \hat{H} + Q + W_s$$

$$\frac{dpV}{dt} = \cancel{V \frac{dp}{dt}} + p \frac{dV}{dt}$$

constant volume

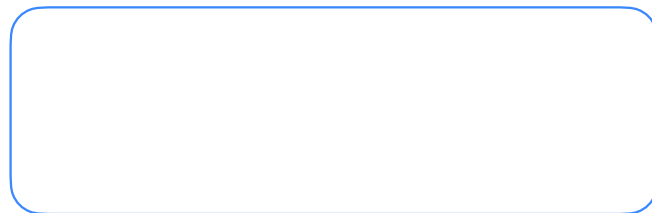
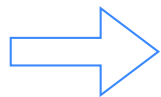
**Assume** that the heat capacity is constant (hence, no phase change)

$$\frac{d[V\rho c_p(T - T_{ref})]}{dt} = F_i\rho_i c_p(T_i - T_{ref}) + Q - F\rho c_p(T - T_{ref}) + W_s$$

**Assume** constant density and volume (hence,  $F_i = F$ )

$$\frac{d[V\rho c_p(T - T_{ref})]}{dt} = F\rho c_p [(T_i - T_{ref}) - (T - T_{ref})] + Q + W_s$$

Note



state:  $T$

inputs:  $F, T_i, Q$

parameters:  $V, \rho, c_p$

# Linear? Nonlinear? Examples We Worked on

Surge Tank

$$\frac{dh}{dt} = \frac{F_1 - F_2}{A} \quad \longleftrightarrow \quad \frac{dy}{dt} = \frac{u_1 - u_2}{A}$$

CSTR (constant hold-up)

$$\frac{dC_A}{dt} = \frac{F_i}{V}(C_{Ai} - C_A) - kC_A$$
$$\longleftrightarrow \quad \frac{dy}{dt} = \frac{u_1}{V}(u_2 - y) - ky$$

# Degrees of Freedom Analysis

DoF: Number of variables that can be specified **independently**

$$\mathbf{N}_F = \mathbf{N}_V - \mathbf{N}_E$$

$\mathbf{N}_F$  : Degree of freedom (# of independent variables)

$\mathbf{N}_V$  : Total number of variables

$\mathbf{N}_E$  : Number of equations (# of dependent variables)

Another perspective: # of equations needed?

$$\mathbf{N}_E = \mathbf{N}_V - \mathbf{N}_F$$



# DoF: Heated Tank Example

$$\frac{dT}{dt} = \frac{F}{V}(T_i - T) + \frac{Q}{V\rho c_p}$$

state:  $T$

inputs:  $F, T_i, Q$

parameters:  $V, \rho, c_p$

# Lumped Parameter System

- Spatial dependence of variables is ignored
  - Well-mixed system
  - Systems with insignificant temperature or concentration gradient
  - Variables are functions of time only, not spatial position
  - Ordinary differential equation model
- Examples
  - Mixer, CSTR
  - Tray of distillation column
  - Steel ball for which heat conduction within much faster than heat transfer to the surrounding

# Distributed Parameter System

- Variables have spatial dependence
  - Instead of  $y(t)$ , you have  $y(t, z)$ ,  $y(t, r, z)$ , etc.
- Type of equation
  - PDEs
- Examples
  - Counter-current heat exchanger
  - Plug-flow reactor or packed-tube reactor
  - Heat conduction through a plate
  - Almost all systems show some spatial variations (e.g., due to imperfect mixing) but many systems can be treated as lumped parameter system